

NONLINEAR SOIL-STRUCTURE INTERACTION OF SKEW HIGHWAY BRIDGES

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## ABSTRACT

This report is one in a series to result from the study, "An Investigation of the Effectiveness of Existing Bridge Design Methodology in Providing Adequate Structural Resistance to Seismic Disturbances", sponsored by the U. S. Department of Transportation. Federal Highway Administration. Descriptions are given of the analytical investigations of the seismic response of skew highway bridges where soil-structure interaction effects are important.

Four different mathematical model elements are incorporated into the three dimensional computer program which possesses the capability of performing linear or nonlinear time-history dynamic response analysis. Solid finite element modelling is used for the backfill soils and the abutment walls. The bridge deck, pier columns and pier caps are modelled using prismatic beam elements. A frictional element is used to model the discontinuous behavior at the interfaces of the backfill soils and abutments. Boundary elements provide foundation flexibility at the base of columns supported on either piles or spread footings. In the nonlinear mathematical model the effects of separation, impact and slippage at the interfaces between the abutment walls and the backfill soils are taken into consideration.

Computational efficiency is achieved through the use of mathematical techniques including matrix reduction procedures, iteration procedures and variable time steps.

A number of analytical solutions are carried out considering a skewed three-span bridge with backfill soils. Different mathematical models are used to study the parameters which may influence the seismic response of the bridge.

Finally, conclusions are deduced from the analytical results.



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## I. INTRODUCTION

Presented in this report is a study of the behavior of short, skew highway bridges interacting with their surrounding soils during strong motion earthquakes. The first part of the study defines a three-dimensional, nonlinear mathematical model for the complete bridge-soil system while the second part develops the associated computer program for carrying out time-history dynamic response analysis.

The mathematical model consists of (1) linear, elastic, three-dimensional solid finite elements representing backfill soils and abutment walls, (2) linear, elastic prismatic beam elements representing the bridge deck, pier columns, and pier caps, (3) nonlinear friction elements representing the discontinuous behavior of separation, impact, and slippage at the interfaces between backfills and abutment walls, and (4) discrete translational and rotational linear springs representing foundation flexibilities at the bases of supporting columns.

In developing the computer program for time-history dynamic response analysis, considerable effort has been spent in achieving computational efficiency. Special programming techniques including the use of matrix reduction procedures, iteration procedures, and variable time steps were used. The matrix reduction procedures reduce the number of coupled equations involved by constraining certain degrees of freedom without decreasing the number of nodal points in the system. The iteration procedures used in the nonlinear analysis express the stiffness matrix in the incremental equilibrium equation in terms of constant initial values placed on the left hand side of the equation and time dependent values associated with the nonlinearities which

are placed on the right hand side of the equation to form an effective force vector. Total equilibrium is enforced at each time step in the numerical integration and variable time steps are used so that the "overshoot" errors never exceed a prescribed limit.

The stiffness properties of the four basic types of elements used in the mathematical model are described in Chapter II and the numerical techniques used in the dynamic response analyses are presented in Chapter III. Some numerical results are given in Chapter IV and certain conclusions are deduced in Chapter V. Finally, the computer program for carrying out time-history response analyses is listed in Appendix A.

## II. BASIC ELEMENT STIFFNESSES

### A. SOLID FINITE ELEMENT

The three-dimensional linear finite element used to model the abutment walls and backfill soils is the eight-node isoparametric hexahedron shown in Fig. 1 which contains incompatible deformation modes [27, 29]. The local and global coordinates of the element are related through a set of interpolation functions, namely,

$$\begin{aligned} x &= \sum_{i=1}^8 h_i x_i \\ y &= \sum_{i=1}^8 h_i y_i \\ z &= \sum_{i=1}^8 h_i z_i \end{aligned} \quad (1)$$

where  $x_i$ ,  $y_i$ , and  $z_i$  are the global coordinates of nodal point  $i$  and

$$\begin{aligned} h_1 &= 1/8 (1 + \eta)(1 - \xi)(1 - \zeta) \\ h_2 &= 1/8 (1 + \eta)(1 + \xi)(1 - \zeta) \\ h_3 &= 1/8 (1 - \eta)(1 + \xi)(1 - \zeta) \\ h_4 &= 1/8 (1 - \eta)(1 - \xi)(1 - \zeta) \\ h_5 &= 1/8 (1 + \eta)(1 - \xi)(1 + \zeta) \\ h_6 &= 1/8 (1 + \eta)(1 + \xi)(1 + \zeta) \\ h_7 &= 1/8 (1 - \eta)(1 + \xi)(1 + \zeta) \\ h_8 &= 1/8 (1 - \eta)(1 - \xi)(1 + \zeta) \end{aligned} \quad (2)$$

are the interpolation functions. The displacements within the element  $u_x$ ,  $u_y$ , and  $u_z$  are related to the nodal displacements  $u_{xi}$ ,  $u_{yi}$ , and  $u_{zi}$  ( $i = 1, 2, \dots, 8$ ) through the equations

$$\begin{aligned} u_x &= \sum_{i=1}^8 h_i u_{xi} + h_9 \alpha_{x1} + h_{10} \alpha_{x2} + h_{11} \alpha_{x3} \\ u_y &= \sum_{i=1}^8 h_i u_{yi} + h_9 \alpha_{y1} + h_{10} \alpha_{y2} + h_{11} \alpha_{y3} \\ u_z &= \sum_{i=1}^8 h_i u_{zi} + h_9 \alpha_{z1} + h_{10} \alpha_{z2} + h_{11} \alpha_{z3} \end{aligned} \quad (3)$$

where

$$\begin{aligned} h_9 &= (1 - \eta^2) \\ h_{10} &= (1 - \xi^2) \\ h_{11} &= (1 - \zeta^2) \end{aligned} \quad (4)$$

The degrees of freedom corresponding to the last three terms in Eq. (3) are eliminated at the element level by static condensation; thus, the final dimensions of the element stiffness matrix are  $24 \times 24$ . Linear elastic isotropic material properties are specified for each element; however, these properties can vary from element to element.

#### B. FRICTIONAL ELEMENT

The frictional element representing separation, impact, and slippage at the interfaces of abutment walls and backfill soils uses relative displacements as the independent degrees-of-freedom to avoid numerical difficulties [11, 26]. Figure 2 shows the nodal displacements of the frictional element along with the corresponding

displacements for its top-half element and its bottom-half element.

The equations relating the displacements at nodal point k are

$$\begin{aligned} u_{xk}^T &= u_{xk}^B + \Delta u_{xk} \\ u_{yk}^T &= u_{yk}^B + \Delta u_{yk} \\ u_{zk}^T &= u_{zk}^B + \Delta u_{zk} \end{aligned} \quad (5)$$

where superscripts T and B refer to the top- and bottom-half elements, respectively. Similar equations exist for nodal points i, j, and l.

The degrees of freedom of the top-half element are obtained from the frictional element nodal displacements  $\{\Delta u\}$  and the upper nodal displacements of the bottom-half element  $\{u_u^B\}$  through a transformation matrix [A] as given by

$$\{u^T\} = \begin{Bmatrix} u_u^T \\ u_l^T \end{Bmatrix} = \begin{bmatrix} I & 0 & 0 \\ 0 & I & I \end{bmatrix} \begin{Bmatrix} \Delta u \\ u_u^B \end{Bmatrix} \quad (6)$$

or

$$\{u^T\} = [A]\{u\} \quad (7)$$

where  $I$  is a  $12 \times 12$  unit matrix and  $0$  is a  $12 \times 12$  null matrix. Subscripts u and l refer to the upper 4 nodes and the lower 4 nodes of the element, respectively.

The procedure used in forming the stiffness matrix of the frictional element can be summarized as follows:

- (1) Form the  $24 \times 24$  top-half and bottom-half stiffness matrices in global coordinates by the standard procedure used for a solid element.
- (2) Retain the bottom-half  $24 \times 24$  element stiffness matrix but transform the top-half  $24 \times 24$  element stiffness matrix into a  $36 \times 36$  matrix using the relation

$$\begin{matrix} [K] \\ 36 \times 36 \end{matrix} = \begin{matrix} [A]^T \\ 36 \times 24 \end{matrix} \begin{matrix} [k] \\ 24 \times 24 \end{matrix} \begin{matrix} [A] \\ 24 \times 36 \end{matrix} \quad (8)$$

- (3) Form the  $12 \times 12$  frictional element stiffness matrix in local coordinates and then transform to the global coordinates.

To form the frictional element stiffness matrix in local coordinates, the relative normal and tangential displacements  $\Delta u_n$ ,  $\Delta u_s$ , and  $\Delta u_t$  as shown in Fig. 3 are assumed to vary linearly within the element, i.e.

$$\begin{aligned} \Delta u_n &= \sum_{i=1}^4 h_i \Delta u_{ni} \\ \Delta u_s &= \sum_{i=1}^4 h_i \Delta u_{si} \\ \Delta u_t &= \sum_{i=1}^4 h_i \Delta u_{ti} \end{aligned} \quad (9)$$

where

$$\begin{aligned} h_1 &= 1/4 (1 - \eta)(1 - \xi) \\ h_2 &= 1/4 (1 + \eta)(1 - \xi) \\ h_3 &= 1/4 (1 + \eta)(1 + \xi) \\ h_4 &= 1/4 (1 - \eta)(1 + \xi) \end{aligned} \quad (10)$$

Assuming all strains are constant throughout the thickness  $W$  and neglecting the in-plane normal strains, the only remaining effective strain components are the normal strain  $\varepsilon_{nn}$  and the two in-plane shear strains  $\varepsilon_{ns}$  and  $\varepsilon_{nt}$  as given by

$$\begin{aligned}\varepsilon_{nn} &= \frac{1}{W} \Delta u_n \\ \varepsilon_{ns} &= \frac{1}{W} \Delta u_s \\ \varepsilon_{nt} &= \frac{1}{W} \Delta u_t\end{aligned}\quad (11)$$

Making use of Eqs. (9)-(11), the strain-displacement relations for the element become

$$\left\{ \begin{array}{l} \varepsilon_{nn} \\ \varepsilon_{ns} \\ \varepsilon_{nt} \end{array} \right\} = \frac{W}{4} \left[ \begin{array}{cccccccccc} h_1 & 0 & 0 & h_2 & 0 & 0 & h_3 & 0 & 0 & h_4 & 0 & 0 \\ 0 & h_1 & 0 & 0 & h_2 & 0 & 0 & h_3 & 0 & 0 & h_4 & 0 \\ 0 & 0 & h_1 & 0 & 0 & h_2 & 0 & 0 & h_3 & 0 & 0 & h_4 \end{array} \right] \left\{ \begin{array}{l} \Delta u_{n1} \\ \Delta u_{s1} \\ \Delta u_{t1} \\ \Delta u_{n2} \\ \Delta u_{s2} \\ \Delta u_{t2} \\ \Delta u_{n3} \\ \Delta u_{s3} \\ \Delta u_{t3} \\ \Delta u_{n4} \\ \Delta u_{s4} \\ \Delta u_{t4} \end{array} \right\} \quad (12)$$

or

$$\{\varepsilon\} = [B] \{\Delta u\} \quad (13)$$

The corresponding stress-strain relations are given by

$$\begin{Bmatrix} \sigma_{nn} \\ \sigma_{ns} \\ \sigma_{nt} \end{Bmatrix} = \begin{bmatrix} C_n & 0 & 0 \\ 0 & C_s & 0 \\ 0 & 0 & C_s \end{bmatrix} \begin{Bmatrix} \varepsilon_{nn} \\ \varepsilon_{ns} \\ \varepsilon_{nt} \end{Bmatrix} \quad (14)$$

or

$$\{\sigma\} = [C] \{\varepsilon\} \quad (15)$$

where  $C_n$  and  $C_s$  are the normal and shear stiffnesses, respectively.

The  $12 \times 12$  stiffness matrix for the frictional element in local coordinates can now be evaluated using standard techniques [29], i.e.

$$\underline{K}_{nst} = \int_{Vol} \underline{B}^T \underline{C} \underline{B} dv \quad (16)$$

Transformation to the XYZ global coordinates is accomplished using the following relation through the coordinate transformation matrix T

$$\underline{K} = \underline{T}^T \underline{K}_{nst} \underline{T} \quad (17)$$

The nonlinear tangential stress-strain relation is assumed to be the elastic-perfectly plastic relation obtained from the Mohr-Coulomb yield criterion shown in Fig. 4, i.e.

$$C_s = G \text{ (shear modulus of frictional element)} \quad (18)$$

when

$$\tau < c + \sigma_{nn} \tan \phi \text{ (elastic)} \quad (19)$$

and

$$c_s = 0 \quad (20)$$

when

$$\tau = c + \sigma_{nn} \tan \phi \text{ (plastic)} \quad (21)$$

in which  $c$  and  $\phi$  are the cohesion and angle of friction, respectively, and  $\tau$  is either  $\sigma_{ns}$  or  $\sigma_{nt}$ .

The nonlinear normal stress-strain relation for the element is assumed to be bilinear as shown in Fig. 5 with  $C_n$  being assigned a very large value when contact is present at the interface of abutment and soil and a zero value when separation occurs.

### C. BEAM ELEMENT

The prismatic beam element used to represent the bridge deck, pier columns, and pier caps, was assumed to be linear elastic. The deformations considered in the element were those caused by torsion, bending about the two principal axes of the cross-section, axial force, and the two-components of transverse shear. Thus, the  $12 \times 12$  stiffness matrix for the element is of the standard form [21]

$$\begin{bmatrix}
 \frac{EA}{\ell} & & & & & \\
 0 & \frac{12EI_z}{\ell^3(1+\phi_y)} & & & & \\
 0 & 0 & \frac{12EI_y}{\ell^3(1+\phi_z)} & & & \\
 0 & 0 & 0 & \frac{GJ}{\ell} & & \text{Symmetric} \\
 0 & 0 & \frac{-6EI_y}{\ell^2(1+\phi_z)} & 0 & \frac{(4+\phi_z)EI_y}{\ell(1+\phi_z)} & \\
 0 & \frac{6EI_z}{\ell^2(1+\phi_y)} & 0 & 0 & 0 & \frac{(4+\phi_y)EI_z}{\ell(1+\phi_y)} \\
 \end{bmatrix}_{12 \times 12} = \boxed{\begin{array}{cccccc}
 -\frac{EA}{\ell} & 0 & 0 & 0 & 0 & \frac{AE}{\ell} \\
 0 & \frac{-12EI_z}{\ell^3(1+\phi_y)} & 0 & 0 & 0 & \frac{-6EI_z}{\ell^2(1+\phi_y)} & \frac{12EI_z}{\ell^3(1+\phi_y)} \\
 0 & 0 & \frac{-12EI_y}{\ell^3(1+\phi_z)} & 0 & \frac{6EI_y}{\ell^2(1+\phi_z)} & 0 & \frac{12EI_y}{\ell^3(1+\phi_z)} \\
 0 & 0 & 0 & \frac{-GJ}{\ell} & 0 & 0 & 0 & \frac{GJ}{\ell} \\
 0 & 0 & \frac{-6EI_y}{\ell^2(1+\phi_z)} & 0 & \frac{(2-\phi_z)EI_y}{\ell(1+\phi_z)} & 0 & \frac{6EI_y}{\ell^2(1+\phi_z)} & 0 & \frac{(4+\phi_z)EI_y}{\ell(1+\phi_z)} \\
 0 & \frac{6EI_z}{\ell^2(1+\phi_y)} & 0 & 0 & 0 & \frac{(2-\phi_y)EI_z}{\ell(1+\phi_y)} & 0 & \frac{-6EI_z}{\ell^2(1+\phi_y)} & 0 & \frac{(4+\phi_y)EI_z}{\ell(1+\phi_y)}
 \end{array}} \quad (22)$$

where  $\phi_y$  and  $\phi_z$  are shear deformation parameters given by

$$\phi_y = \frac{12EI_z}{G A_{sy} \frac{\ell^2}{\ell^2}} = 24(1+\nu) \frac{A}{A_{sy}} \left( \frac{\gamma_z}{\ell} \right)^2 \quad (23)$$

and

$$\phi_z = \frac{12EI_y}{G A_{sz} \frac{\ell^2}{\ell^2}} = 24(1+\nu) \frac{A}{A_{sz}} \left( \frac{\gamma_y}{\ell} \right)^2 \quad (24)$$

If  $(\gamma_z/\ell)$  and  $(\gamma_y/\ell)$ , representing ratios of radius of gyration to element length, are small compared with unity as in the case of a slender member, both  $\phi_y$  and  $\phi_z$  can be taken equal to zero in Eq. (22).

#### D. BOUNDARY ELEMENT

The boundary element is used for modelling foundation flexibility at the base of columns supported on either piles or mat footings and soil flexibility at both horizontal and vertical boundaries of the backfill models, when necessary. The element consists of 3 translational and 3 rotational degrees of freedom as shown in Fig. 6. The individual stiffness in each degree of freedom can be approximated using either numerical or closed form solutions [4, 10, 20]. The  $6 \times 6$  element stiffness matrix has diagonal terms only as given by

$$[k]_{6 \times 6} = \begin{bmatrix} k_x & & & & & \\ & k_y & & & & \\ & & k_z & & & \\ & & & k_\alpha & & \\ & & & & k_\beta & \\ & & & & & k_\gamma \end{bmatrix} \quad (25)$$

### III. SOLUTION TECHNIQUES

This chapter discusses the formulation and solution of the dynamic equilibrium equations of motion for the complete soil-structure system. Included are discussions of the Guyan matrix reduction procedure, the step-by-step integration and iteration procedures used in solving the equations, the variable time step procedure for controlling overshooting errors, and finally the prescribed earthquake ground motions used in the study.

#### A. EQUATIONS OF MOTION

The dynamic force equilibrium equations of motion associated with the nodes of the complete soil-structure system can be expressed in the form [7]

$$\underline{F}^I + \underline{F}^D + \underline{F}^S = \underline{R} \quad (26)$$

where  $\underline{F}^I$  is the inertia force vector,  $\underline{F}^D$  is the damping force vector,  $\underline{F}^S$  is the internal resisting force vector caused by deformations in the system, and  $\underline{R}$  is the external load vector. For linear elastic systems, the internal resisting force vector can be expressed in terms of the nodal displacement vector  $\underline{u}$  through the relation

$$\underline{F}^S = \underline{K} \underline{u} \quad (27)$$

where  $\underline{K}$  is the stiffness matrix of the structure. Likewise, the inertia and damping force vectors can be expressed in the form

$$\underline{F}^I = \underline{M} \ddot{\underline{u}} \quad (28)$$

$$\underline{F}^D = \underline{C} \dot{\underline{u}} \quad (29)$$

where  $\underline{M}$  and  $\underline{C}$  are the mass and damping matrices, respectively. For rigid base earthquake excitation, the external force vector has the form

$$\underline{R} = -\ddot{\underline{u}}_g \underline{M} \begin{cases} 1 \\ 0 \\ 0 \\ \vdots \\ \cdot \\ \vdots \\ 1 \\ 0 \\ 0 \end{cases} + \underline{g}_x \begin{cases} 0 \\ 1 \\ 0 \\ \vdots \\ \cdot \\ \vdots \\ 0 \\ 1 \\ 0 \end{cases} + \underline{g}_y \begin{cases} 0 \\ 0 \\ 1 \\ \vdots \\ \cdot \\ \vdots \\ 0 \\ 0 \\ 1 \end{cases} + \underline{g}_z \begin{cases} 0 \\ 0 \\ 0 \\ \vdots \\ \cdot \\ \vdots \\ 0 \\ 0 \\ 1 \end{cases} \quad (30)$$

where  $\ddot{\underline{u}}_g$  is the prescribed one-dimensional ground acceleration and  $\underline{g}_x$ ,  $\underline{g}_y$ , and  $\underline{g}_z$  are its direction cosines with respect to the x, y, and z axes, respectively.

For nonlinear systems, it is convenient to write the dynamic force equilibrium equations of motion in the incremental form [25]

$$\left( \underline{F}_t^I + \Delta \underline{F}_{-t}^I \right) + \left( \underline{F}_t^D + \Delta \underline{F}_{-t}^D \right) + \left( \underline{F}_t^S + \Delta \underline{F}_{-t}^S \right) = \underline{R}_{t+\Delta t} \quad (31)$$

where subscripts  $t$  and  $t+\Delta t$  represent times at the beginning and end of a time increment of duration  $\Delta t$ , respectively. The incremental force vectors over the interval  $\Delta t$  become

$$\Delta \underline{F}_{-t}^I = \underline{M}_t \Delta \ddot{\underline{u}}_{-t} \quad (32)$$

$$\Delta \underline{F}_{-t}^D = \underline{C}_t \Delta \dot{\underline{u}}_{-t} \quad (33)$$

$$\Delta \underline{F}_t^S = \underline{K}_t \Delta \underline{u}_t; \quad (34)$$

thus, Eq. (31) can be written in the form

$$\underline{M} \ddot{\underline{u}}_t + \underline{C}_t \dot{\underline{u}}_t + \underline{K}_t \underline{u}_t = \underline{R}_{t+\Delta t} - \left( \underline{F}_t^I + \underline{F}_t^D + \underline{F}_t^S \right) \quad (35)$$

This equation can be solved by standard numerical procedures for  $\dot{\underline{u}}_t$ .

Note that the subscript  $t$  associated with matrices  $\underline{M}_t$ ,  $\underline{C}_t$ , and  $\underline{K}_t$  indicate that these physical properties vary with the time-dependent response; thus, the incremental form given by Eq. (35) is approximate unless complete equilibrium is achieved at the end of each time increment  $\Delta t$ . Normally, complete equilibrium is not achieved in which case the residual force vector

$$\Delta \underline{P}_{t+\Delta t} = \underline{R}_{t+\Delta t} - \left( \underline{F}_{t+\Delta t}^I + \underline{F}_{t+\Delta t}^D + \underline{F}_{t+\Delta t}^S \right) \quad (36)$$

indicates the errors involved. To correct for these errors, the residual force vector can be evaluated at the end of each time increment  $\Delta t$  and then be added to the right hand side of Eq. (35) before proceeding on with the numerical iteration.

In the nonlinear analysis of this investigation, matrices  $\underline{M}_t$  and  $\underline{C}_t$  are considered constant in time, i.e.  $\underline{M}_t = \underline{M}$  and  $\underline{C}_t = \underline{C}$ ; however, matrix  $\underline{K}_t$  must be retained in its time dependent form requiring retriangularization at each time step [7]. It is convenient to express  $\underline{K}_t$  in terms of its initial tangent value  $\underline{K}$  and its incremental change  $\Delta \underline{K}_t$ . The force vector associated with the nonlinear incremental changes can be placed on the right hand side of the equilibrium equation and be treated as an effective load vector; thus, Eq. (35) can be written in the form

$$\underline{M} \ddot{\underline{u}}_t + \underline{C} \dot{\underline{u}}_t + (\underline{K} - \Delta \underline{K}_t) \underline{u}_t = \underline{R}_{t+\Delta t} - \left( \underline{F}_t^I + \underline{F}_t^D + \underline{F}_t^S \right) \quad (37)$$

or

$$\underline{M} \Delta \ddot{\underline{u}}_t + \underline{C} \Delta \dot{\underline{u}}_t + \underline{K} \Delta \underline{u}_t = \underline{R}_{t+\Delta t} - \left( \underline{F}_t^I + \underline{F}_t^D + \underline{F}_t^S \right) + \Delta \underline{K}_t \Delta \underline{u}_t \quad (38)$$

### B. MASS MATRIX

In the present investigation, all masses are assumed concentrated at the nodal points which leads to a diagonal mass matrix of the form

$$\underline{M} = \text{diag} < M_1 \ M_2 \ \dots \ M_n > \quad (39)$$

where  $M_i$  is the mass associated with the  $i^{\text{th}}$  degree of freedom and  $n$  is the total number of degrees of freedom in the system. No rotational moments of inertia are assigned to the lumped masses; therefore, the  $M_i$ 's associated with rotational degrees of freedom equal zero.

### C. STIFFNESS MATRIX

As pointed out previously, the complete stiffness matrix  $\underline{K}_t$  at a particular time  $t$  can be expressed as the sum of the initial tangent stiffness matrix  $\underline{K}$  and the incremental stiffness matrix  $\Delta \underline{K}_t$ , i.e.

$$\underline{K}_t = \underline{K} - \Delta \underline{K}_t \quad (40)$$

The individual element stiffnesses for each time interval are obtained by the procedures described in Chapter II. The initial tangent stiffness matrix  $\underline{K}$  is assembled by the standard direct stiffness method [6]; however, the incremental stiffness matrix  $\Delta \underline{K}_t$  must be treated differently. Since in the numerical examples carried out, the frictional element was the only nonlinear element in the system, the incremental stiffness matrix  $\Delta \underline{K}_t$  for the entire soil-structure system contained relatively few nonzero elements. Thus, the effort involved

in computing the term  $\Delta K_{-t} \Delta u_{-t}$  in Eq. (38) is relatively small even though an iteration is involved for each time step  $\Delta t$ . This iteration starts by using the initial values  $\Delta u_{-t-\Delta t}$  for  $\Delta u_{-t}$  which are then changed through successive iterations towards their correct values  $\Delta u_{-t}$ . These iterative multiplications are carried out at the element level to reduce computational effort [11], i.e. one makes use of the relation

$$\Delta K_{-t} \Delta u = \sum_{m=1}^N \Delta K_{-t}^m \Delta u \quad (41)$$

in which  $\Delta K_{-t}^m$  is the incremental stiffness associated with the  $m^{th}$  frictional element and  $N$  is the total number of frictional elements.

In this equation, subscript  $t$  has been dropped from  $\Delta u$  to reflect the changing values associated with the iteration process.

#### D. DAMPING MATRIX

Various methods have been used by investigators in evaluating the viscous damping matrix corresponding to matrix  $C$  in Eq. (38) [13]. Wilson and Penzien describe two methods for evaluating this matrix [28]. The first method relates the modal damping ratios to the coefficients in the Caughey series form for  $C$  [3]. If only the first two terms in this series are used, Rayleigh damping results, i.e. the damping matrix is a linear combination of the mass and stiffness matrices. The second method of Wilson and Penzien is a direct approach whereby the damping matrix is expressed in terms of a series of matrices each controlling damping in only one normal mode. Clough describes another type of damping called "structural damping" which yields a damping force vector proportional to displacements but in

phase with the velocities [7]. The above three types of damping will now be described in more detail.

1. Rayleigh Damping - Rayleigh damping is given by the first two terms of the Caughey series, i.e.

$$\underline{C} = \alpha \underline{M} + \beta \underline{K} \quad (42)$$

where  $\alpha$  and  $\beta$  are scalar quantities having units consistent with the other units involved in this equation. By properly selecting these scalar values, the damping ratios in two normal modes can be controlled. It can be shown that these quantities are related to the damping ratios for modes  $i$  and  $j$  through the equations

$$\alpha = \frac{2 \omega_i \omega_j (\xi_j \omega_i - \xi_i \omega_j)}{(\omega_i^2 - \omega_j^2)} \quad (43)$$

$$\beta = \frac{2 (\xi_i \omega_i - \xi_j \omega_j)}{(\omega_i^2 - \omega_j^2)} \quad (44)$$

Further, it can be shown that if  $\alpha$  and  $\beta$  satisfy these equations, the damping ratio in any other normal mode, say mode  $n$ , is given by

$$\xi_n = \frac{\alpha + \beta \omega_n^2}{2 \omega_n} \quad (45)$$

In the present investigation, the numerical values of  $\alpha$  and  $\beta$  were determined by specifying the damping ratios in the two most dominant modes of the initial elastic system. These values were then held constant throughout the time history of response including those periods of time when the system responded inelastically.

2. Direct Damping - By the direct method of Wilson and Penzien, the damping matrix controlling mode  $r$  only is given by

$$\underline{C}_r = \beta_r \underline{\theta}_r \underline{\theta}_r^T \quad (46)$$

where the  $\underline{\theta}_r$  represents the mass normalized mode shape matrix given by

$$\underline{\theta}_r = \underline{M} \underline{\phi}_r \quad (47)$$

where  $\underline{\phi}_r$  is the  $r^{\text{th}}$  mode shape vector. The scalar quantity  $\beta_r$  is obtained using the relation

$$\beta_r = \frac{2 \xi_r \omega_r}{M_r} \quad (48)$$

where  $\xi_r$  is the damping ratio of the  $r^{\text{th}}$  mode,  $\omega_r$  is the frequency of the  $r^{\text{th}}$  mode, and  $M_r$  is the generalized mass of the  $r^{\text{th}}$  mode, i.e.

$$M_r = \underline{\phi}_r^T \underline{M} \underline{\phi}_r \quad (49)$$

The total damping matrix is then given by a summation over all  $N$  modes; thus,

$$\underline{C} = \sum_{r=1}^N \underline{C}_r \quad (50)$$

(3) Structural Damping - For structural damping, the damping force vector is proportional to the elastic force vector  $\underline{F}_t^S$  as given by

$$\underline{F}_t^D = b \underline{F}_t^S \quad (51)$$

where  $b$  is the proportionality factor and where the sign of each

component in the vector is the same as the sign of the corresponding velocity component (the triangular "hat" symbol above vector  $\underline{F}_t^S$  denotes this procedure in selecting the sign of each component). If damping is variable throughout the system, the proportionality factor  $b$  is replaced by a diagonal intensity matrix  $\hat{B}$ ; thus,

$$\underline{F}_t^D = \underline{B} \hat{\underline{F}}_t^S \quad (52)$$

$$\Delta\underline{F}_t^D = \underline{B} \hat{\Delta\underline{F}}_t^S = \underline{B} (\underline{K} - \Delta\underline{K}_t) \Delta\hat{\underline{u}}_t \quad (53)$$

where  $\Delta\hat{\underline{u}}_t$  is the vector  $\Delta\underline{u}_t$  with signs corresponding to the signs in the vector  $\hat{\underline{u}}_t$ .

#### E. GUYAN MATRIX REDUCTION

1. Linear Systems - To reduce computational effort the Guyan Matrix Reduction technique has been used effectively for linear systems [12, 14, 16, 17, 22]. By this technique, the independent degrees of freedom  $\underline{u}$  in the system are separated into two sets  $\underline{u}_o$  and  $\underline{u}_a$  such that  $\underline{u}_o$  can be eliminated before solving a reduced set of equations of motion. Consider first the static equilibrium equation

$$\underline{K} \underline{u} = \underline{R} \quad (54)$$

which may be written in the partitioned form

$$\begin{bmatrix} \underline{K}_{aa} & \underline{K}_{ao} \\ \underline{K}_{ao}^T & \underline{K}_{oo} \end{bmatrix} \begin{Bmatrix} \underline{u}_a \\ \underline{u}_o \end{Bmatrix} = \begin{Bmatrix} \underline{R}_a \\ \underline{R}_o \end{Bmatrix} \quad (55)$$

Solving for  $\underline{u}_o$  in the second of Eqs. (55) and substituting back into the first gives the reduced static equilibrium equation

$$\underline{\underline{K}}_{aa}^R \underline{u}_a = \underline{\underline{R}}_a^R \quad (56)$$

where the reduced stiffness matrix  $\underline{\underline{K}}_{aa}^R$  and the reduced force vector are given by

$$\underline{\underline{K}}_{aa}^R = \underline{\underline{K}}_{aa} + \underline{\underline{K}}_{ao} \underline{G}_o \quad (57)$$

and

$$\underline{\underline{R}}_a^R = \underline{\underline{R}}_a + \underline{G}_o^T \underline{\underline{R}}_o \quad (58)$$

respectively, and where the transformation stiffness matrix  $\underline{G}_o$  is obtained by solving the equation

$$\underline{\underline{K}}_{oo} \underline{G}_o = - \underline{\underline{K}}_{ao}^T \quad (59)$$

Having solved for  $\underline{u}_a$  in Eq. (56),  $\underline{u}_o$  can then be obtained using the relation

$$\underline{u}_o = \underline{\underline{K}}_{oo}^{-1} \underline{\underline{R}}_o + \underline{G}_o \underline{u}_a \quad (60)$$

In the present analysis, the response quantities of main interest are the displacements of the column nodal points. Therefore, when using the Guyan Matrix Reduction method, it is advantageous to let the vector  $\underline{u}_o$  contain the displacements of the nodal points within the backfills and the bridge deck and to place the displacements of the remaining nodal points within the columns in the vector  $\underline{u}_a$ .

In carrying out a linear dynamic analysis, the above matrix reduction procedure can also be used to reduce the inertia term  $\underline{\underline{M}} \ddot{\underline{u}}$  to the form  $\underline{\underline{M}}_{aa}^R \ddot{\underline{u}}_a$  in which case the reduced mass matrix is given by

$$\underline{\underline{M}}_{aa}^R = \underline{\underline{M}}_{aa} + \underline{\underline{M}}_{ao} \underline{G}_o + \underline{G}_o^T \underline{\underline{M}}_{ao}^T + \underline{G}_o^T \underline{\underline{M}}_{oo} \underline{G}_o \quad (61)$$

When using a lumped mass system, the off-diagonal terms in matrix  $\underline{M}$  are all zero in which case Eq. (61) reduces to

$$\underline{M}_{aa}^R = \underline{M}_{aa} + \underline{G}_o^T \underline{M}_{oo} \underline{G}_o \quad (62)$$

2. Nonlinear Systems - For a general nonlinear system, the Guyan reduction procedure may be very inefficient due to the time dependency of stiffness matrix  $\underline{K}_t$  which could require using the reduction procedure each time step of the numerical integration. However, for the nonlinear system considered herein, the reduction procedure is still very efficient as in the case of the linear system for two reasons. First, the nonlinear system considered has relatively few nonlinear elements which are concentrated at the interfaces of abutments and backfills. Second, the procedure allows a shifting of the time dependent stiffness coefficients to the right hand side of the equation of motion so that the reduction procedure need only be applied once during the entire time of integration.

Following the same reasoning as in the case of the linear system, consider first the quasi-static equilibrium equation

$$\underline{K}_t \underline{u} = \underline{R} \quad (63)$$

which corresponds to Eq. (54) in the linear case. Making use of Eq. (40), this equation can be written in the form

$$[\underline{K} - \Delta \underline{K}_t] \underline{u} = \underline{R} \quad (64)$$

Separating vector  $\underline{u}$  into two parts, namely  $\underline{u}_o$  and  $\underline{u}_a$ , this equation becomes

$$\begin{bmatrix} K_{aa} & K_{ao} \\ K_{ao}^T & K_{oo} \end{bmatrix} \begin{Bmatrix} u_a \\ u_o \end{Bmatrix} - \begin{bmatrix} \Delta K_{-t}^R & 0 \\ 0 & 0 \end{bmatrix} \begin{Bmatrix} u_a \\ u_o \end{Bmatrix} = \begin{Bmatrix} R_a \\ R_o \end{Bmatrix} \quad (65)$$

It is necessary, of course, that vector  $\underline{u}_o$  which is to be eliminated not contain any components having time dependent stiffness coefficients consistent with the form of Eq. (65). Note that the nonlinear (or time dependent) reduced stiffness matrix  $\Delta K_{-t}^R$  is the upper left submatrix in the coefficient matrix of the second term with the other three submatrices being zero matrices.

Solving for  $\underline{u}_o$  in the second of Eqs. (65) and substituting back into the first gives the reduced static equilibrium equation

$$K_{aa}^R u_a = R_a^R + \Delta K_{-t}^R u_a \quad (66)$$

where the reduced stiffness matrix  $K_{aa}^R$  and the reduced force vector  $R_a^R$  are of the same forms given by Eqs. (57) and (58), respectively.

When carrying out a dynamic analysis, the reduction of the inertia term  $M \ddot{\underline{u}}$  to the reduced form  $M_{aa}^R \ddot{\underline{u}}_a$  is identical to that previously discussed for the linear system, i.e. Eqs. (61) and (62) are all applicable to the nonlinear case being considered.

Since the damping matrix is expressed in terms of the mass and stiffness matrices as shown by Eqs. 42, 46, and 53, the reduced mass and stiffness matrices can be used directly in defining the reduced damping matrix.

#### F. STEP-BY-STEP INTEGRATION PROCEDURES

After applying the Guyan reduction procedure as previously described, the resulting incremental reduced equations of motion are

identical in form to those given by matrix Eq. (38), except that all quantities are of the reduced form. These equations can be solved numerically using various procedures [1, 19, 23]. The differences in these procedures relate to the analytical form of the variation in acceleration over the time interval  $\Delta t$ . In the investigation presented herein, two different forms have been used, namely, the constant acceleration method and the Wilson  $\theta$ -method. These forms express the velocity and displacement vectors at time  $t + \Delta t$  in terms of the velocity and displacement vector at time  $t$  and the acceleration vectors at times  $t + \Delta t$  and  $t + \theta\Delta t$  when using the constant acceleration and Wilson  $\theta$ -methods, respectively. Since three forms of damping were used in the investigation, the step-by-step integration procedures will be developed for each case.

1. Constant-Acceleration Method - This form of acceleration over interval  $\Delta t$  leads to the relations

$$\dot{\underline{u}}_{t+\Delta t} = \dot{\underline{u}}_t + \frac{1}{2} \Delta t \ddot{\underline{u}}_t + \frac{1}{2} \Delta t \ddot{\underline{u}}_{t+\Delta t} \quad (67)$$

$$\underline{u}_{t+\Delta t} = \underline{u}_t + \Delta t \dot{\underline{u}}_t + \frac{1}{4} \Delta t^2 \ddot{\underline{u}}_t + \frac{1}{4} \Delta t^2 \ddot{\underline{u}}_{t+\Delta t} \quad (68)$$

Introducing incremental vectors as defined by

$$\Delta \ddot{\underline{u}}_t \equiv \ddot{\underline{u}}_{t+\Delta t} - \ddot{\underline{u}}_t \quad (69)$$

$$\Delta \dot{\underline{u}}_t \equiv \dot{\underline{u}}_{t+\Delta t} - \dot{\underline{u}}_t \quad (70)$$

$$\Delta \underline{u}_t \equiv \underline{u}_{t+\Delta t} - \underline{u}_t \quad (71)$$

and making use of Eqs. (67) and (68), one obtains

$$\Delta \ddot{u}_t = \frac{4}{\Delta t^2} \Delta u_t - \frac{4}{\Delta t} \dot{u}_t - 2 \ddot{u}_t \quad (72)$$

$$\Delta \dot{u}_t = \frac{2}{\Delta t} \Delta u_t - 2 \dot{u}_t \quad (73)$$

Using these relations, Eq. (38) can be written in the form

$$\bar{K} \Delta u_t = p_{t+\Delta t} + \Delta K_t \Delta u_t \quad (74)$$

where  $\bar{K}$  and  $p_{t+\Delta t}$  take on different forms depending upon the type of damping assumed as follows:

(a) Direct Damping

$$\bar{K} = \frac{4}{\Delta t^2} M + \frac{2}{\Delta t} C + K \quad (75)$$

$$\begin{aligned} p_{t+\Delta t} &= R_{t+\Delta t} + M \left( \frac{4}{\Delta t} \dot{u}_t + \ddot{u}_t \right) + C \dot{u}_t \\ &- K u_t + \sum_{i=\Delta t}^{t-\Delta t} \Delta K_i \Delta u_i \end{aligned} \quad (76)$$

(b) Rayleigh Damping

$$\bar{K} = \frac{4}{\Delta t^2} M + \frac{2}{\Delta t} C + K \quad (77)$$

$$\begin{aligned} p_{t+\Delta t} &= R_{t+\Delta t} + M \left( \frac{4}{\Delta t} \dot{u}_t + \ddot{u}_t \right) + C \dot{u}_t \\ &- K u_t + \sum_{i=\Delta t}^{t-\Delta t} \Delta K_i \Delta u_i \end{aligned} \quad (78)$$

$$C = \alpha M + \beta K \quad (79)$$

(c) Structural Damping

$$\bar{K} = \frac{4}{\Delta t^2} M + K \quad (80)$$

$$\begin{aligned}
 p_{t+\Delta t} &= \underline{r}_{t+\Delta t} + \underline{M} \left( \frac{4}{\Delta t} \dot{\underline{u}}_t + \ddot{\underline{u}}_t \right) - \underline{K} \underline{u}_t \\
 &+ \sum_{i=\Delta t}^{t-\Delta t} \Delta \underline{K}_i \Delta \underline{u}_i - \underline{B} \underline{K} \sum_{i=\Delta t}^{t-\Delta t} \Delta \hat{\underline{u}}_i + \sum_{i=\Delta t}^{t-\Delta t} \underline{B} \Delta \underline{K}_i \Delta \hat{\underline{u}}_i \\
 &- \underline{B} \underline{K} \Delta \hat{\underline{u}}_t + \underline{B} \Delta \underline{K}_t \Delta \hat{\underline{u}}_t
 \end{aligned} \tag{81}$$

2. Wilson θ-Method - This form of acceleration which assumes a linear variation over the interval  $\tau = \theta \Delta t$  (where  $\theta \geq 1.0$ ), leads to the relations

$$\dot{\underline{u}}_{t+\tau} = \dot{\underline{u}}_t + \frac{\tau}{2} (\ddot{\underline{u}}_{t+\tau} + \ddot{\underline{u}}_t) \tag{82}$$

$$\underline{u}_{t+\tau} = \underline{u}_t + \tau \dot{\underline{u}}_t + \frac{\tau^2}{6} (\ddot{\underline{u}}_{t+\tau} + 2 \ddot{\underline{u}}_t); \tag{83}$$

thus, one obtains

$$\ddot{\underline{u}}_{t+\tau} = \frac{6}{\tau^2} (\underline{u}_{t+\tau} - \underline{u}_t) - \frac{6}{\tau} \dot{\underline{u}}_t - 2 \ddot{\underline{u}}_t \tag{84}$$

$$\dot{\underline{u}}_{t+\tau} = \frac{3}{\tau} (\underline{u}_{t+\tau} - \underline{u}_t) - 2 \dot{\underline{u}}_t - \frac{\tau}{2} \ddot{\underline{u}}_t \tag{85}$$

Using these relations, Eq. (38) can again be written in the form

$$\underline{K} \Delta \underline{u}_t = \underline{p}_{t+\Delta t} + \Delta \underline{K}_t \Delta \underline{u}_t \tag{86}$$

where  $\underline{K}$  and  $\underline{p}_{t+\Delta t}$  take on different forms as follows:

(a) Direct Damping

$$\underline{K} = \frac{6}{\tau^2} \underline{M} + \frac{3}{\tau} \underline{C} + \underline{K} \tag{87}$$

$$\begin{aligned} \underline{\underline{p}}_{t+\Delta t} &= \underline{\underline{R}}_t + \theta (\underline{\underline{R}}_{t+\Delta t} - \underline{\underline{R}}_t) + \underline{\underline{M}} \left( \frac{6}{\tau^2} \dot{\underline{\underline{u}}}_t + 2 \ddot{\underline{\underline{u}}}_t \right) \\ &+ \underline{\underline{C}} \left( 2 \dot{\underline{\underline{u}}}_t + \frac{\tau}{2} \ddot{\underline{\underline{u}}}_t \right) - \underline{\underline{K}} \underline{\underline{u}}_t + \sum_{i=\Delta t}^{t-\Delta t} \Delta \underline{\underline{K}}_i \Delta \underline{\underline{u}}_i \end{aligned} \quad (88)$$

(b) Rayleigh Damping

$$\underline{\underline{K}} = \frac{6}{\tau^2} \underline{\underline{M}} + \frac{3}{\tau} \underline{\underline{C}} + \underline{\underline{K}} \quad (89)$$

$$\begin{aligned} \underline{\underline{p}}_{t+\Delta t} &= \underline{\underline{R}}_t + \theta (\underline{\underline{R}}_{t+\Delta t} - \underline{\underline{R}}_t) + \underline{\underline{M}} \left( \frac{6}{\tau^2} \dot{\underline{\underline{u}}}_t + 2 \ddot{\underline{\underline{u}}}_t \right) \\ &+ \underline{\underline{C}} \left( 2 \dot{\underline{\underline{u}}}_t + \frac{\tau}{2} \ddot{\underline{\underline{u}}}_t \right) - \underline{\underline{K}} \underline{\underline{u}}_t + \sum_{i=\Delta t}^{t-\Delta t} \Delta \underline{\underline{K}}_i \Delta \underline{\underline{u}}_i \end{aligned} \quad (90)$$

$$\underline{\underline{C}} = \alpha \underline{\underline{M}} + \beta \underline{\underline{K}} \quad (91)$$

(c) Structural Damping

$$\underline{\underline{K}} = \frac{6}{\tau^2} \underline{\underline{M}} + \underline{\underline{K}} \quad (92)$$

$$\begin{aligned} \underline{\underline{p}}_{t+\Delta t} &= \underline{\underline{R}}_t + \theta (\underline{\underline{R}}_{t+\Delta t} - \underline{\underline{R}}_t) + \underline{\underline{M}} \left( \frac{6}{\tau} \dot{\underline{\underline{u}}}_t + 2 \ddot{\underline{\underline{u}}}_t \right) \\ &- \underline{\underline{K}} \underline{\underline{u}}_t + \sum_{i=\Delta t}^{t-\Delta t} \Delta \underline{\underline{K}}_i \Delta \underline{\underline{u}}_i - \underline{\underline{B}} \underline{\underline{K}} \sum_{i=\Delta t}^{t-\Delta t} \Delta \hat{\underline{\underline{u}}}_i \\ &+ \sum_{i=\Delta t}^{t-\Delta t} \underline{\underline{B}} \Delta \underline{\underline{K}}_i \Delta \hat{\underline{\underline{u}}}_i - \underline{\underline{B}} \underline{\underline{K}} \Delta \hat{\underline{\underline{u}}}_t + \underline{\underline{B}} \Delta \underline{\underline{K}}_t \Delta \hat{\underline{\underline{u}}}_t \end{aligned} \quad (93)$$

#### G. ITERATION PROCEDURES

The dynamic equilibrium equations of motion, Eq. (38), can be solved by iteration for the unknown vector  $\Delta \underline{\underline{u}}_t$  which appears on both sides of the equation. Two different solution methods have been employed in the present investigation [2, 5, 9, 11, 18]. To explain

these two procedures, express the incremental vector  $\Delta \underline{u}_t$  as  $\underline{x}^t$  which is to be approached iteratively through  $\underline{x}_n^t$ ,  $n = 1, 2, \dots, N$ . In the first procedure, the total values in  $\underline{x}_n^t$  are determined for all iterative steps. In the second procedure, only the incremental values  $\Delta \underline{x}_n^t$  of  $\underline{x}^t$  are determined by successive iteration until sufficient convergence is reached. Convergence is based on two different criteria. One being the Euclidean norm of the difference in incremental displacement vectors obtained by consecutive iterations, i.e.  $\underline{x}_{n+1}^t - \underline{x}_n^t$ , and the second being the differences in successive values of  $x_i$ . Convergence is judged to be satisfactory when the differences in successive values of  $x_i$  drop to a certain pre-assigned tolerance level.

To explain further the first procedure mentioned above, consider  $\underline{x}_n^t$  which is an approximation of  $\underline{x}^t$ . An improved value of  $\underline{x}_n^t$  is obtained by solving for  $\underline{x}_{n+1}^t$  using the equation

$$\bar{\underline{K}} \underline{x}_{n+1}^t = \underline{p} + \Delta \underline{K} \underline{x}_n^t \quad n = 1, 2, \dots, N \quad (94)$$

To start this iteration,  $\underline{x}_1^t$  is assumed to be the value finally reached for the previous time step, i.e. equal to  $\underline{x}^{t-1}$ . The second procedure makes use of the relations

$$\bar{\underline{K}} \underline{x}_0^t = \underline{p} + \Delta \underline{K} \underline{x}^{t-1} \quad (95)$$

$$\bar{\underline{K}} \Delta \underline{x}_n^t = \Delta \underline{K} \Delta \underline{x}_{n-1}^t \quad n = 1, 2, \dots, N \quad (96)$$

$$\underline{x}_n^t = \sum_{i=1}^n \Delta \underline{x}_i^t + \underline{x}_0^t \quad (97)$$

$$\Delta \underline{x}_0^t = \underline{x}_0^t - \underline{x}^{t-1} \quad (98)$$

#### H. OVERSHOOT TOLERANCE AND VARIABLE TIME STEP

As explained previously in Chapter II and illustrated in Fig. 5, the normal stress-strain relation of the frictional element is a bilinear elastic function which is assigned a very large modulus in the compression region and zero modulus in the tension region. During transition from one region to the other, the regular numerical integration procedure permits an overshoot error to occur as shown in Fig. 7a. Experience shows that this error can accumulate over a number of cycles as shown in Fig. 7b; thus, becoming unacceptably large.

To reduce this error to an acceptable level, a variable time step interval can be used over the last regular interval which passes through the transition, i.e. the interval is changed to  $\Delta t/n$ , whenever it is found necessary to do so. Since the numerous frictional elements experience the transition at different instants of time during response, it is impractical from a computer usage point of view to apply the shorter interval every time a transition occurs. However, it is practical to use the shorter intervals provided they are used only when the overshoot error introduced by the regular interval exceeds a specified tolerance value. Thus, by properly specifying a tolerance value and the transition integration interval  $\Delta t/n$ , the overshoot errors can be controlled and the computer time will remain within practical limits.

The detailed procedure used in this investigation was as follows:

- (1) First, specify overshoot tolerance limits of strain in the tension and compression regions as designated by zones 1 and 2 in Fig. 8.

- (2) When the transition occurs from compression to tension, yielding without stress change is assumed to take place immediately after the strain at the end of a regular interval falls within either zone 1 or zone 2. Upon the return transition from tension to compression, the large modulus is introduced only at the end of a regular interval falling in the same zone in which the preceding compression to tension transition was allowed to take place. Figure 8 illustrates this procedure, once when the transitions in both directions take place in zone 2 and again when the transitions take place in zone 1.
- (3) When the overshoot is so large that the strain either falls outside both zones or falls in an unacceptable zone as described in (2) above, the computation returns to the beginning of the regular time step and proceeds forward again using the smaller time step  $\Delta t/n$ . In the present investigation a value of 10 was used for n and found to always satisfy the acceptable overshoot error. After the integration proceeds over the regular transition interval  $\Delta t$  in n steps, the method returns back to using the regular interval  $\Delta t$ .

#### I. EARTHQUAKE INPUT

In the present investigation, the ground motion was prescribed in accordance with the acceleration time-history shown in Fig. 9. This artificial accelerogram was generated by A. K. Chopra, et al., to simulate the ground motions produced by the San Fernando earthquake at the site of the Olive View Hospital located about six miles southwest of the epicenter [8]. It has a peak acceleration of 0.5g and a uniform phase of high intensity shaking for 8 seconds.

The input motion was assumed to be in the longitudinal direction of the bridge for the present study. The computer program developed in the investigation does however permit multi-directional inputs in arbitrary directions with respect to the bridge axis.

#### IV. NUMERICAL EXAMPLES

A number of analytical solutions have been carried out to demonstrate the methods previously described. The bridge used for this purpose was a skewed structure similar to the North Connector Undercrossing located approximately 800 feet northerly of the Route 5-San Fernando Road Interchange in the city and county of Los Angeles. Three equal spans were assumed for the idealized bridge deck as shown in Fig. 10.

Five different mathematical models (A-E) were selected for this structure as shown in Fig. 11. These models differ only in the type of skew permitted and in the arrangement of abutments and backfills. Model A has no skew and the backfills extend laterally only over the width of the bridge deck. Model B is identical to Model A except the deck is skewed  $37.5^\circ$ . Model C has one abutment and its backfill similar to Model A while the other abutment and its backfill are similar to Model B. The elevation views of Models A, B, and C are identical as shown in Fig. 11d. The backfills in each case extend longitudinally a distance 1.5 times their depth H. All of these three models have identical abutment and columns which are assumed to be fixed at their bases. Model D is identical to Model B except that the backfill behind each abutment extends a distance  $7H$  in the longitudinal direction and a distance  $6H$  beyond the deck in the transverse direction. Each backfill in this case is modelled using 4 equal layers in depth with their finite elements having 3 different widths in the longitudinal direction as shown in Fig. 11f. Model E is identical to Model D except that the abutments and backfills are of

depth  $2H$  and the bases of the columns are provided with linear translational and rotational springs representing foundation flexibility. The backfill soils are modelled using three layers of depth  $H/3$  and one layer of depth  $H$  as shown in Fig. 11g.

Numerical results are presented for Models A-E in the subsequent sections of this chapter. Section A presents the results of linear analyses while Section B presents the results of nonlinear analyses. Computational efficiencies are demonstrated in Section C for one example case.

#### A. LINEAR ANALYSES

1. Duration of Input Acceleration - The computed longitudinal component of displacement at the top of the right bridge columns for the entire 15 seconds of input is shown in Fig. 12. A 3.775 second segment of this displacement time-history from about 5.7 to 9.4 seconds is shown again in Fig. 13a. If instead of using the entire 15 seconds of input only the input in this 3.775 second interval is used, the computed longitudinal component of displacement at the top of the right bridge column has the time-history shown in Fig. 13b. The displacement time histories in Figs 13a and 13b agree very well except in the beginning. This difference is, of course, due to the differences in the initial conditions imposed at the beginning of the 3.775 second segment. The point of this comparison is that the transient response caused by changes in initial conditions lasts only a very short time. Therefore, in the interest of saving of computer costs, it was decided that the methodology and computer program capabilities could be checked adequately using only the 3.775

second duration input. Therefore, all analytical results presented subsequently are computed using this input.

2. Effects of Skew on Bridge Response - The longitudinal displacement time-histories for the top of the right bridge column are shown for Models A, B, and C in Figs. 14a, 14b, and 14c, respectively. The dissimilarities in amplitudes and shapes noted in these wave forms are due to differences in amplitudes and phasing of the backfill forces on the two abutments.

Figures 15a and 15b show the time-histories of the transverse shear component in the left and right columns of Model B. The relatively low values of shear and the similarity in time-histories indicate that the dynamic backfill forces at the two abutments were nearly in-phase resulting in low torsional response of the bridge.

Figures 16a and 16b show the time histories of the transverse shear component in the same two columns for Model C. The relatively large values of shear produced and the dissimilarities noted for the two columns in this case indicate that large torsional response developed due to the presence of skew at only one abutment. The backfill forces at the two abutments had large out-of-phase components.

Figures 17, 18, and 19 show time-histories of backfill force on the left and right abutment walls for Models A, B, and C, respectively. It is noted that the dynamic pressures on both walls for Models A and B are nearly in-phase, i.e. when the pressure is positive on one abutment, it is negative on the other, and vice versa. However, for Model C as shown in Fig. 19, these dynamic pressures on the two abutments differ considerably in amplitude and in their phasing. These results again provide evidence that unequal skews produce large torsional response.

To provide further comparisons of the results for Models A-C, maximum dynamic amplitudes of displacement, shear, and wall force are presented in Table 1. As indicated by the values in rows (1) and (2), the maximum amplitudes of longitudinal displacement and longitudinal shear in the right column are greatly reduced by the presence of skewed abutments. Rows (3) and (4) in this table, give maximum values of lateral shear in the left and right columns, respectively. Row (5) gives the ratio of maximum lateral shear to maximum longitudinal shear produced in the right column. The increase in this ratio with skewness indicates the corresponding increase in torsional response which induces a differential shear force between the two columns as shown at the top of Table 1. Half the difference in the shear forces of these two columns is the shear produced by torsional response. The maximum values of these torsional shears are 0.13 and 6.03 kips for Models B and C, respectively, as shown in row (6). Although the magnitude of maximum torsional shear is negligible for Model B, it is large for Model C. The maximum amplitudes of the dynamic wall force are shown in rows (7) through (10). The ratios of maximum positive pressure on the left abutment to maximum negative pressure on the right abutment and maximum negative pressure on the left abutment to maximum positive pressure on the right abutment for both Models A and B are all equal to 1.0 which indicates the two wall pressures are in-phase with each other. Finally as indicated in row (11), the time history of the resultant of both backfill forces  $p(t)$  acts longitudinally along the axis of symmetry in the case of Model A but acts at angle  $\theta(t)$  to the longitudinal axis in the case of Model B; causing no torsion in each case. However, in the case of Model C, the resultant force  $p(t)$  acting

at an angle  $\theta(t)$  has an eccentricity about the elastic center of the bridge. This is equivalent to its acting through the elastic center but with a torque  $T(t)$  applied as shown in the table.

3. Effects of Foundation Flexibility - To study the effects of foundation flexibility on dynamic response, let us compare the results for Models D and E. The foundation flexibilities at the base of each column of Model E are modelled using 3 translational and 3 rotational springs with their spring constants established using elastic half-space theory. The flexibility at the base of each abutment wall is provided by finite element modelling of the soil below its base as shown in Fig. 11g.

The time-histories of longitudinal displacement at the top of the left column for Models D and E are shown in Figs. 20a and 20b, respectively. Noting the different displacement scales used, these two wave forms differ considerably in form and in their peak amplitudes.

To provide further comparative data, the maximum dynamic amplitudes of displacement and acceleration of the bridge deck, column shear forces, and backfill soil forces are listed in Table 2. Based on the ratios of corresponding responses for Models E and D given in rows (1) through (4) of the last column of this table, it is quite clear that the overall response of Model E having foundation flexibility is considerably greater than that for Model D. The ratios in rows (5) through (8) indicate however that the backfill soil forces for Model E are less than those of Model D. All of these ratios simply indicate that Model E has less constraint provided by its backfills than does Model D; thus, the bridge structural response is higher for Model E.

Rows (9) and (10) in Table 2 show ratios of maximum column shears to maximum total backfill force on one abutment wall. Comparing the magnitudes of these ratios confirms the above statement explaining the reason for higher overall structural response in the case of Model E.

#### B. NONLINEAR ANALYSIS

To study the effects of nonlinearities on seismic response, results obtained by linear and nonlinear analyses for Models A, B, and E in Fig. 10 are compared. Specifically, the effects of impact and separation between abutment wall and backfill soil are investigated and ratios of maximum response obtained by both methods of analysis are compared.

1. Effects of Impact - The most distinctive difference between the results obtained by linear and nonlinear analyses is the high acceleration produced at the point of impact in the nonlinear case. A typical acceleration time-history response for Model A is shown in Fig. 21. The high peaks of acceleration in this wave form are produced at moments of impact. While these acceleration peaks are high near the point of impact, the influence is very localized, i.e. the amplitudes of the peaks produced by impact decay rapidly with distance from the point of impact. Acceleration time-histories at the top of the left column as produced without and with impacts are shown in Figs. 22a and 22b for Model A. While the general features of the two wave forms are essentially the same, localized differences in the form of high frequency noise caused by impact are noted. This feature is better observed in Fig. 23 which shows an expanded-scale view of the first second of time-history shown in Fig. 22b.

2. Effects of Separation - A characteristic feature of allowing separation between wall and backfill soil to occur is that only positive pressure is permitted at the interface. Therefore, the backfill soils at the interfaces of both end abutments can have phase differences in their responses. Figures 24a and 24b show the time-histories of soil force on the left and right abutment walls, respectively, as determined by the nonlinear analysis for skewed Model B. Clearly there are significant out-of-phase components of response between the two abutments. Note that a small overshoot error is present during certain moments of the time-history. As previously pointed out, this overshoot error can be controlled by reducing the integration time-step and by introducing the variable time-step procedure.

The out-of-phase components of soil force on the end abutments produces a torsional response of the bridge structure. This effect is quite apparent when observing the unequal lateral shears produced in the two columns. This comparison can be made in Fig. 25 which shows the transverse shear time-histories for the two columns of Model B. While the frequency content of the two wave forms in this figure are similar, significant differences are present in the amplitudes. The maximum transverse shear produced in the left column is 16.81 kips while the maximum transverse shear in the right column is 18.95 kips. The maximum difference in the two shears is 3.72 kips.

3. Comparison of Amplitudes - For further comparison, maximum amplitudes of response obtained by linear and nonlinear analyses for Models A, B and E are shown in Table 3. The particular responses presented are longitudinal displacement and acceleration at the top of the left column and the shears in both principal directions of the

left column. In Models A and B, principal shears  $V_2$  and  $V_3$  are the lateral and longitudinal shears, respectively, as the column is oriented with one principal axis coinciding with the longitudinal axis of the bridge. In Model E, the column is placed so that one principal axis is oriented  $52.5^\circ$  from the longitudinal bridge axis.

The maximum amplitudes of dynamic response are listed for both linear and nonlinear response and for comparison purposes the ratios of linear to nonlinear response amplitudes are shown. From the results shown in Row 1 of Table 3, it is quite apparent that the displacements of Models A and B produced by nonlinear response are larger than the corresponding displacements produced by linear response. However, the reverse comparison is seen for Model E. From the results in Row 2 it is seen that the accelerations produced by the linear response are larger than the corresponding accelerations produced by nonlinear response. The differences in the amplitudes for both types of response are relatively small however. Row 3 shows a large difference in the transverse lateral shears produced in Model B. This large difference results from the torsional response produced in the nonlinear case. Row 4 shows only small differences in the longitudinal shears produced by the two types of response.

#### C. SEISMIC LOAD TRANSFER TO COLUMNS AND ABUTMENTS

It is of particular importance to know the division of the total longitudinal seismic deck force between the supporting columns and the abutments. To check this behavior characteristic, consider the unskewed Model A which experienced a maximum longitudinal deck acceleration of  $1.09g$  as shown in Table 3. The tributary bridge weight for each column (center to center of spans of deck plus one-half of columns) in this case is 340 kips; thus, the estimated maximum column shear

based on this tributary weight is 371 kips ( $340 \times 1.09 = 371$ ). Since the maximum calculated column shear as shown in Table 3 is only 47.6 kips, it is clear that most of the tributary seismic deck force (87%) is transferred to the foundation through the abutment walls. To further check this transfer characteristic, let us consider the total deck seismic force plus the seismic forces produced in the upper-half portions of both columns. The maximum combined seismic force in this case amounts to 1078 kips ( $989 \times 1.09 = 1078$ ) which occurs at about 2.1 seconds. The algebraic sum of the two abutment wall forces at this same instant of time is 855 kips ( $404 + 451 = 855$ ; see Figs. 17a and 17b). Considering the bridge as a whole, this information indicates that about 79% of the maximum seismic force in the total deck is transferred to the foundation through the interaction of abutment walls with the backfills. Further, calculations show the maximum combined longitudinal shear in the two columns which occurs at the critical time of 2.1 seconds is approximately 94 kips. Therefore about 9% ( $94/1078$ ) of the maximum seismic force is transferred to the foundation through the columns. The remaining 12% of the maximum seismic force is transferred to the foundation by shear in the abutment walls.

Making comparisons as shown above for the other bridge models gives similar results.

#### D. COMPUTATIONAL EFFICIENCY

Computational efficiency in the computer program is achieved through careful program arrangement and the use of three mathematical schemes, namely, the matrix reduction procedure, the iteration method, and variable time steps.

The increased efficiency through program arrangement is achieved using overlay programs which can reduce considerably the required core memory.

The increased efficiency through matrix reduction results from a decrease in the number of simultaneous equations involved and a decrease in bandwidths. If this scheme reduces the number of degrees of freedom from  $N$  to  $N^1$  and the bandwidth from  $m$  to  $m^1$ , the ratio of computational effort required using matrix reduction to the effort required without matrix reduction is  $N^m^2/N^{m^1^2}$ .

The iteration procedure used allows the normal multiple triangularization and backsubstitution of the nonlinear stiffness matrix at each time step to be substituted by only a single triangularization and backsubstitution. If the average number of iterations per time step is "i", then the ratio of computational effort required without using this scheme to the computation effort required using it is  $N^{m^1^2}/N^{m^1}i$  which equals  $m^1/i$ .

The increased efficiency using variable time steps is quite apparent; therefore, no further discussion of this procedure is needed.

To illustrate the savings in computational time which can be achieved by the above mentioned schemes consider Model E which has 402 degrees-of-freedom. Matrix reduction reduces this number to 140 and the bandwidth  $m$  which equals 93 can be reduced to 58. Therefore, the ratio of computational efforts mentioned above, i.e.  $N^m^2/N^{m^1^2}$  becomes  $(402)(93)^2/(140)(58)^2$  or 7.4. Since the average number of iterations per time step in the nonlinear case equals 3, the computational effort ratio  $m^1/i$  becomes  $58/3$  or 19.3. Using the variable time step method, the total number of time steps required to produce a

certain accuracy in this case was 4,040, including 8 subdivisions of 5 each to limit overshooting errors. By the standard equal interval procedure, 20,000 time steps would have been required to limit overshooting errors to the same level. Thus, the ratio of computational efforts as influenced by using (or not using) variable time steps is  $20,000/4,040$  which is approximately 5.

Thus, it is seen that for the above example nonlinear solution, the three above mentioned schemes lead to an overall ratio of computational efforts equal to (7.4)(19.3)(5) which approximately equals 720. Clearly, the methods used greatly increase computational efficiency. Without these special techniques the cost of solutions would be prohibitive. Even using these effective methods, the computer time for a single nonlinear solution was as great as 908 seconds using the CDC 7600 computer.

TABLE 1. EFFECTS OF SKEWNESS-MAXIMUM DYNAMIC AMPLITUDE

		LATERAL SHEAR			
Model		$\theta_1 = \theta_2 = 0$	$\theta_1 = \theta_2 = 37.5^\circ$	$\theta_1 = 37.5^\circ \theta_2 = 0$	$\theta_1 = 37.5^\circ \theta_2 = 0$
Long. Disp. @ Top of Rt. Col.	(1)	0.14	0.07	0.17	
Long. Shear, Rt. Col.	(2)	47.37	23.70	5.64	
Lateral Shear	Lt. Col.	(3)	0.	10.47	7.80
	Rt. Col.	(4)	0.	10.49	6.85
Ratio of Shear (4)/(2)		(5)	0.	0.44	1.22
Tors. Shear $\tau/\lambda =  V_L - V_R /2$		(6)	0.	0.13	6.03
Left Wall Force per ft	Tension	(7)	9.27	6.45	10.20
	Comp.	(8)	9.96	5.30	16.40
Rt. Wall Force per ft	Tension	(9)	9.96	5.30	11.20
	Comp.	(10)	9.27	6.45	12.30
Resulting Model	(11)				

Note: All displacements in inches; All forces in kips.

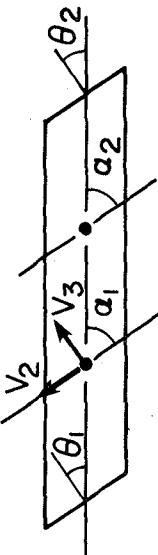
TABLE 2. EFFECTS OF FLEXIBILITY AT BASE

		(a)		(b)	
Maximum Response Values		Model D Fixed Base		Model E Flex. Base	
				Ratio E/D	
Long. Disp. @ Top of Lt. Col.	(1)	0.087		0.178	2.05
Long. Accel. (g)	(2)	0.98		1.36	1.39
Shear @ Lt. Col.	V <sub>2</sub> (3)	31.35		55.70	1.78
	V <sub>3</sub> (4)	42.59		70.60	1.66
Lt. Wall Force per ft	Tension (5)	11.06		9.98	0.91
	Comp. (6)	13.31		8.95	0.67
Rt. Wall Force per ft	Tension (7)	13.31		8.95	0.67
	Comp. (8)	11.06		9.98	0.91
V <sub>2</sub> /Wall Force,	$\frac{(3)}{(5) + (6)}$ (9)	1.29		2.95	2.29
V <sub>3</sub> /Wall Force,	$\frac{(4)}{(5) + (6)}$ (10)	1.75		3.74	2.14

Note: All displacements in inches; All forces in kips.

TABLE 3. LINEAR VS. NONLINEAR - MAXIMUM DYNAMIC RESPONSES

Model	A		B		E	
	$\theta_1 = 0^\circ$	$\theta_2 = 0^\circ$	$\theta_1 = 37.5^\circ$	$\theta_2 = 37.5^\circ$	$\theta_1 = 37.5^\circ$	$\theta_2 = 37.5^\circ$
Type of Analysis Responses	Linear	Non-linear	Ratio Linear Non-linear	Non-linear	Ratio Linear Non-linear	Non-linear
Long. Disp.	(1) 0.14	0.15	0.93	0.071	0.088	0.81
Long. Accel.	(2) 1.09	1.05	1.04	0.80	0.75	1.07
Shear $V_2$	(3)			10.47	16.81	0.62
Shear $V_3$	(4) 47.6	51.5	0.91	29.96	28.63	1.04



Note: All displacements in inches; All accelerations in g's; All shears in kips.

## V. CONCLUSIONS

Based on the studies contained herein for short bridges, conclusions can be deduced as follows:

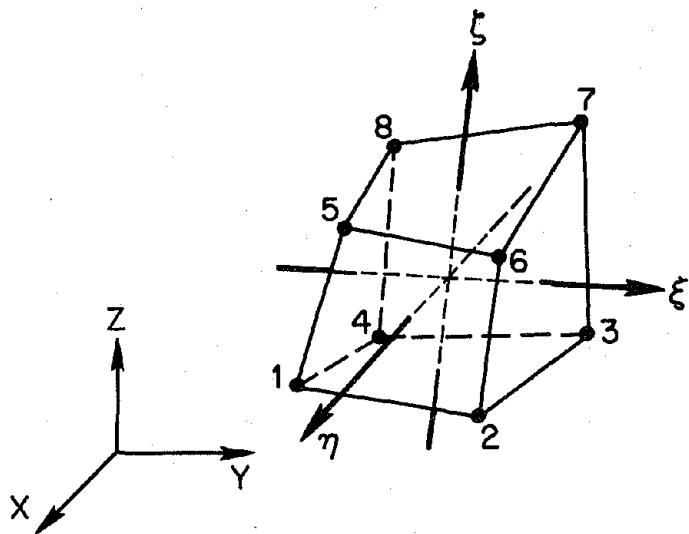
- (1) The total seismic load of the bridge deck is transmitted to the foundation primarily through the abutments with the columns carrying only a small percentage.
- (2) Backfill soil forces on the two end abutments remain essentially in-phase under linear conditions but can develop significant out-of-phase components under nonlinear conditions.
- (3) Skewness of a bridge tends to reduce maximum longitudinal response but it causes coupled lateral response to develop.
- (4) Unequally skewed end abutments can cause both lateral and large torsional responses to develop.
- (5) Foundation flexibilities at the bases of columns and abutments have significant influence on overall bridge response.
- (6) Impacts at the interfaces of abutment walls and the bridge deck cause very large local transient accelerations but they have little effect on the average deck acceleration.
- (7) Separations which occur between abutments and backfill soils cause significant out-of-phase components to develop in the backfill forces.
- (8) The three-dimensional, nonlinear seismic response, including soil-structure interaction, can be treated analytically in a fairly efficient manner.

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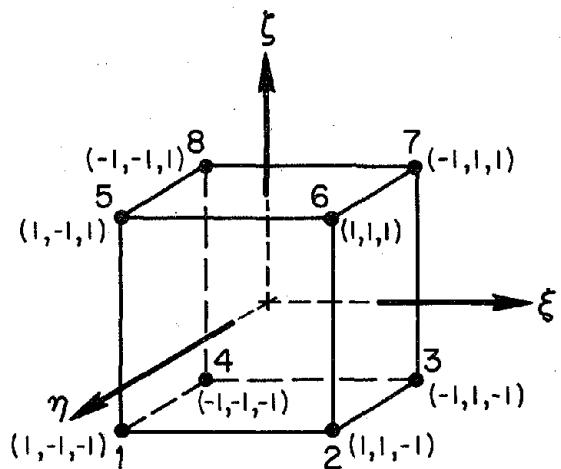
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a) GLOBAL COORDINATES



b) LOCAL COORDINATES

Fig. 1 Three dimensional coordinate systems

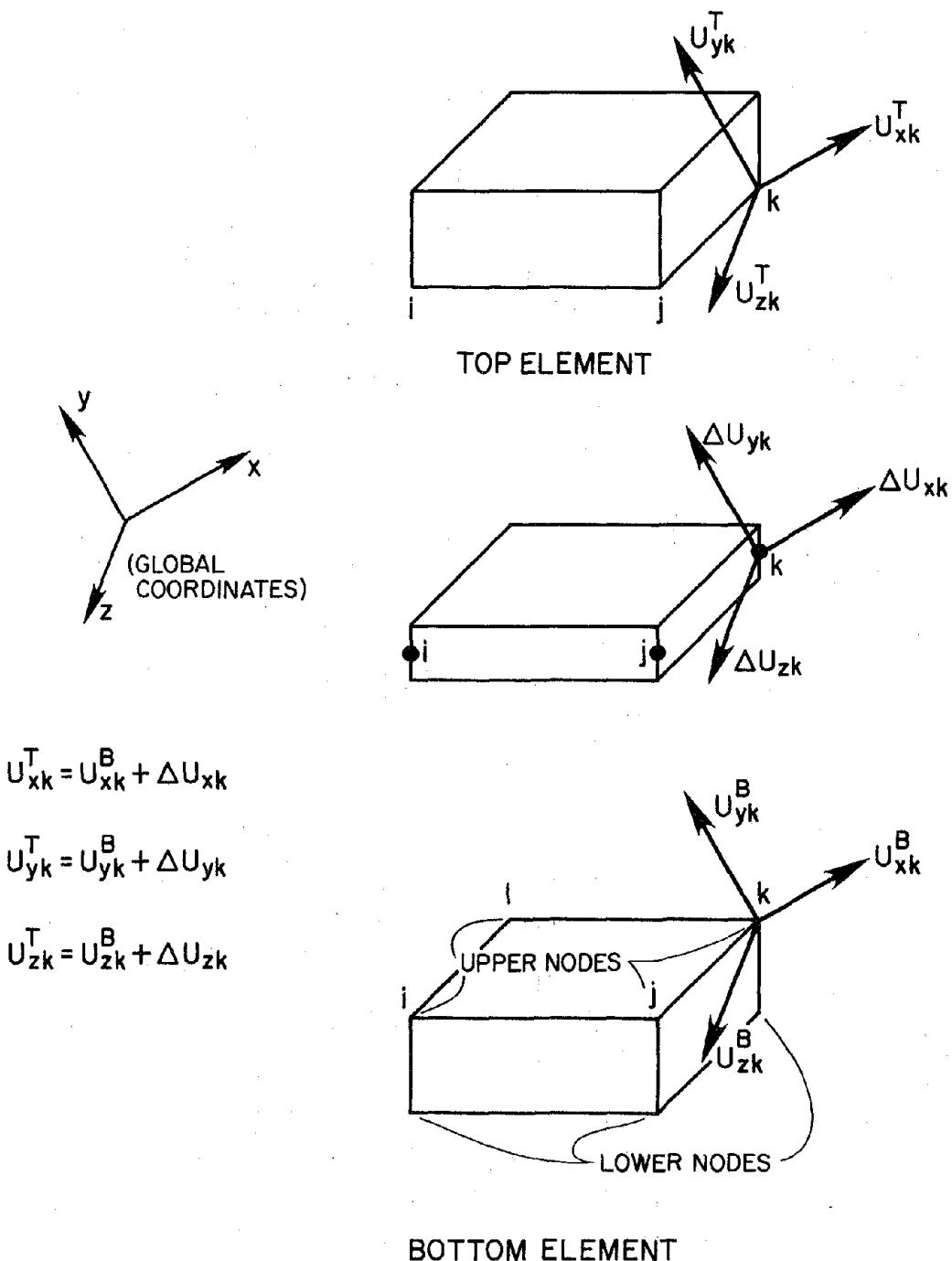


Fig. 2 Frictional element

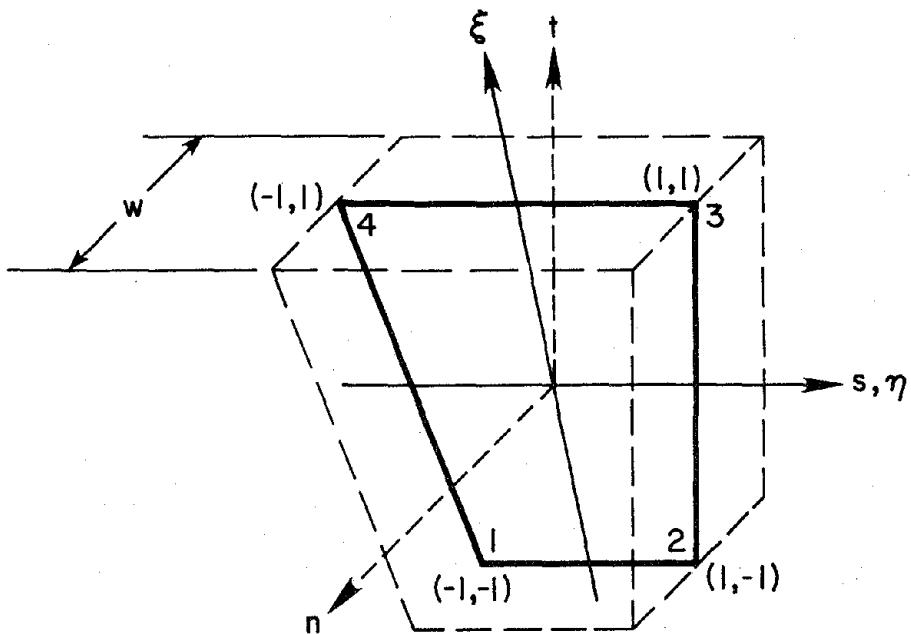


Fig. 3 Frictional element in local coordinates

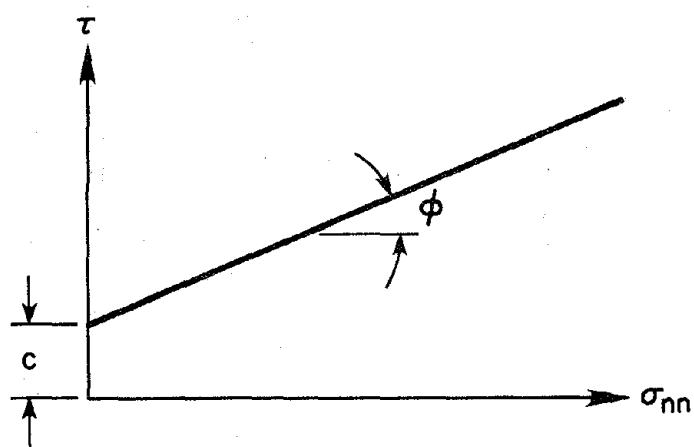


Fig. 4 Mohr-Coulomb yield criterion

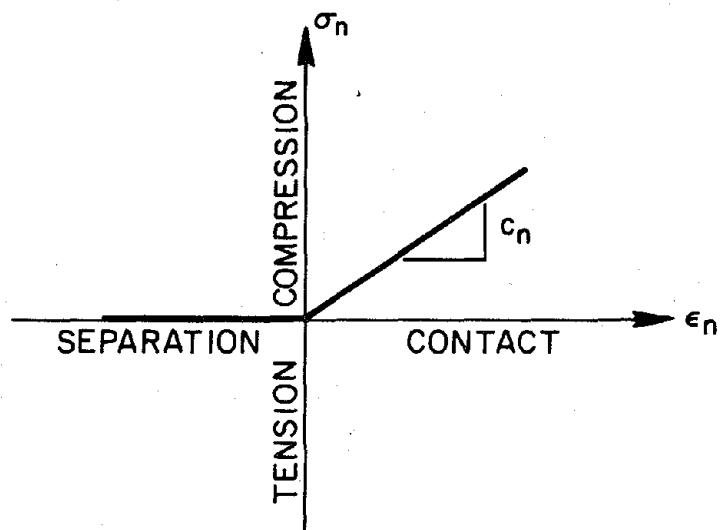


Fig. 5 Normal stress-strain relation for frictional element

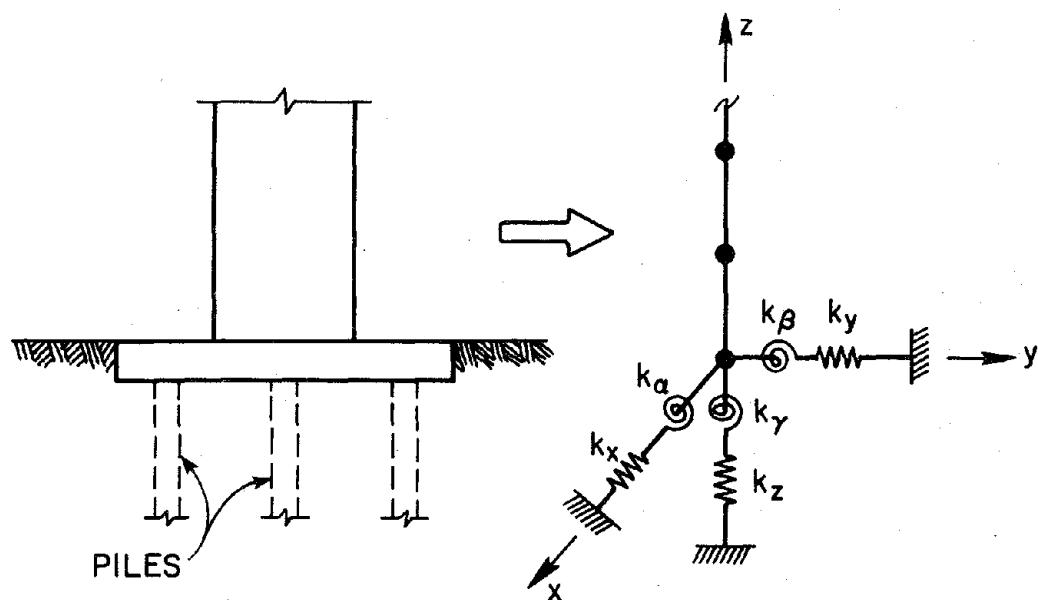
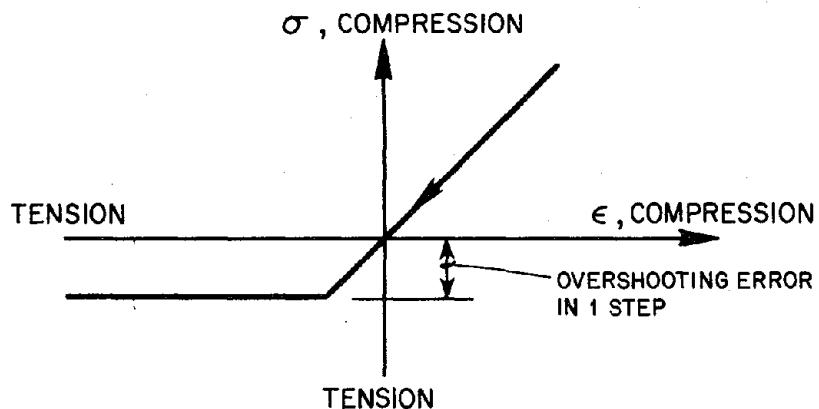
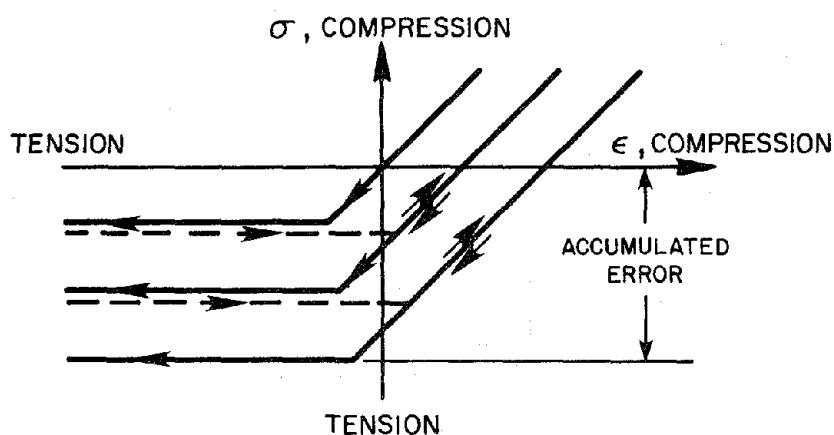


Fig. 6 Column foundation and boundary element



a) OVERSHOOTING IN 1 STEP



b) ACCUMULATIVE OVERSHOOTING ERROR

Fig. 7 Overshooting errors

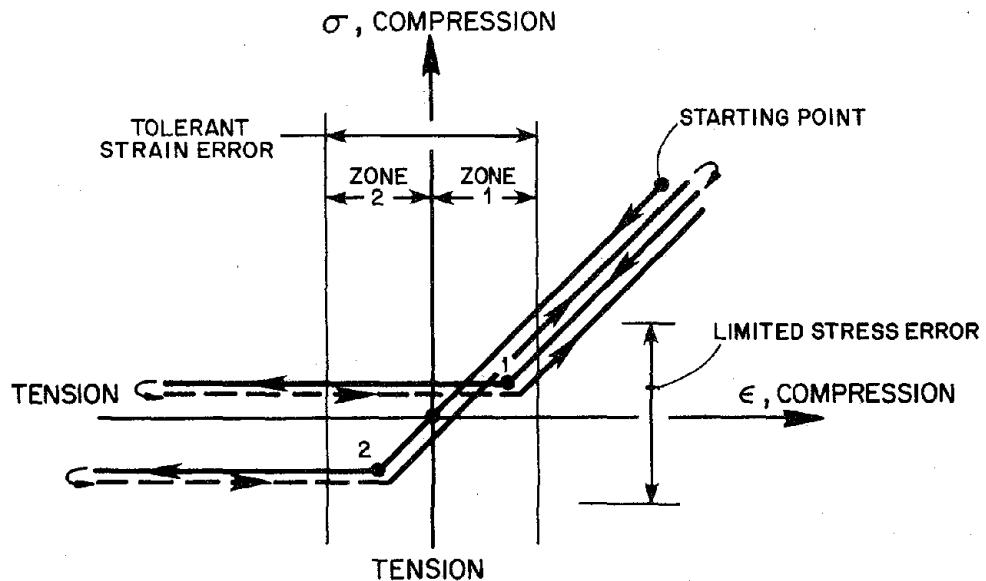


Fig. 8 Limiting overshooting errors

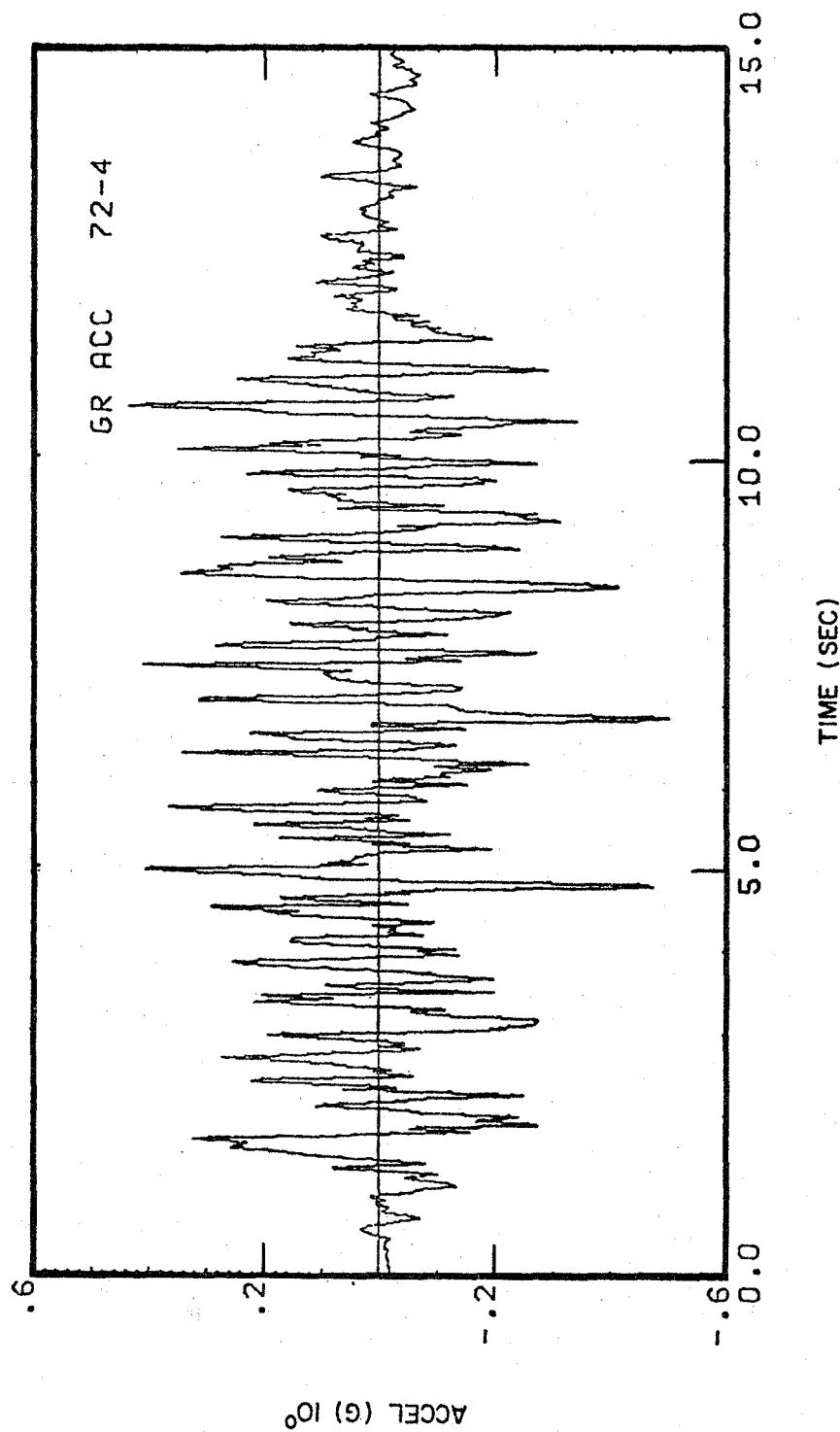


Fig. 9 Simulated ground acceleration record of the San Fernando Earthquake at the Olive View Hospital site

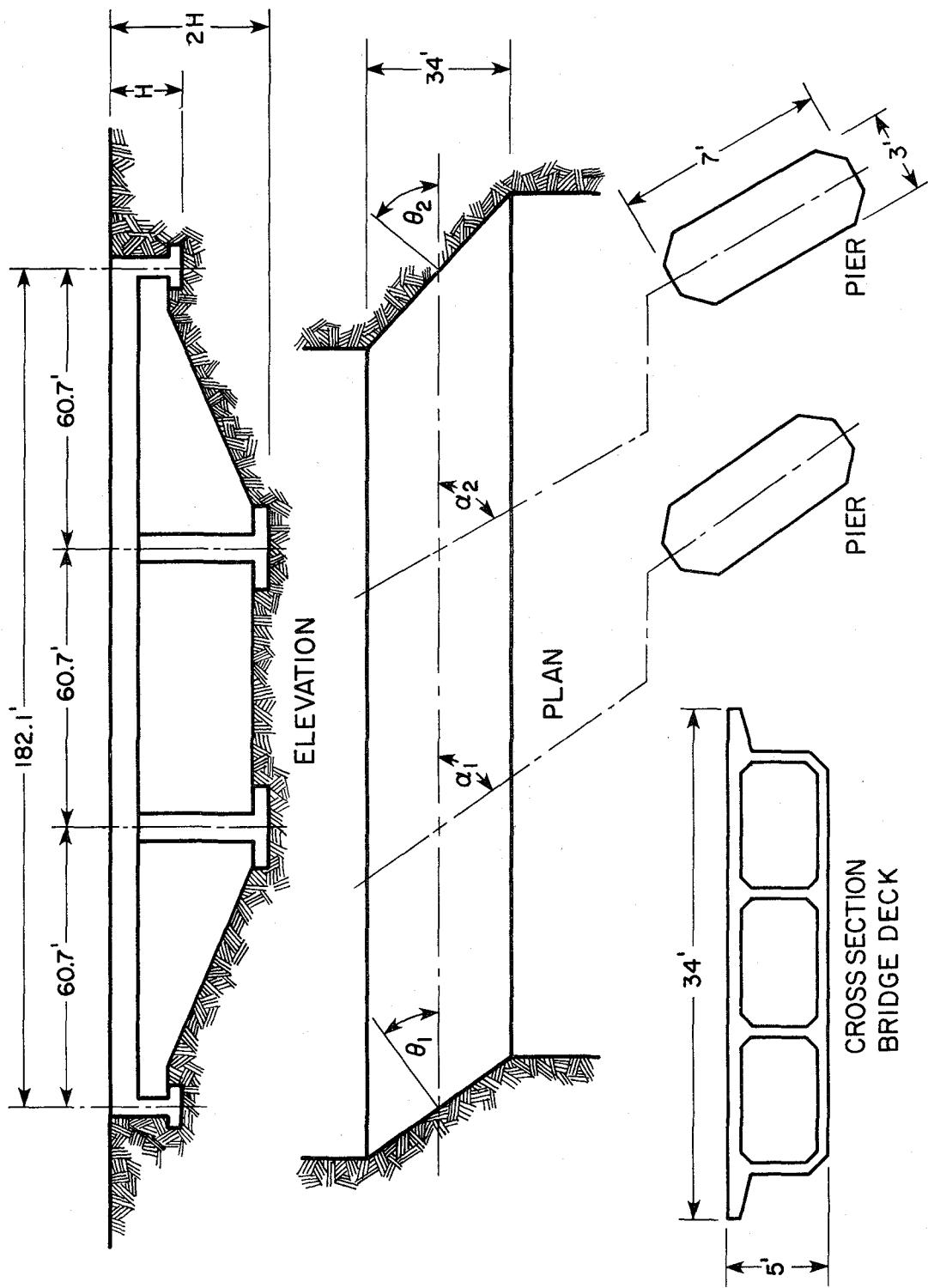


Fig. 10 General plan of model bridge

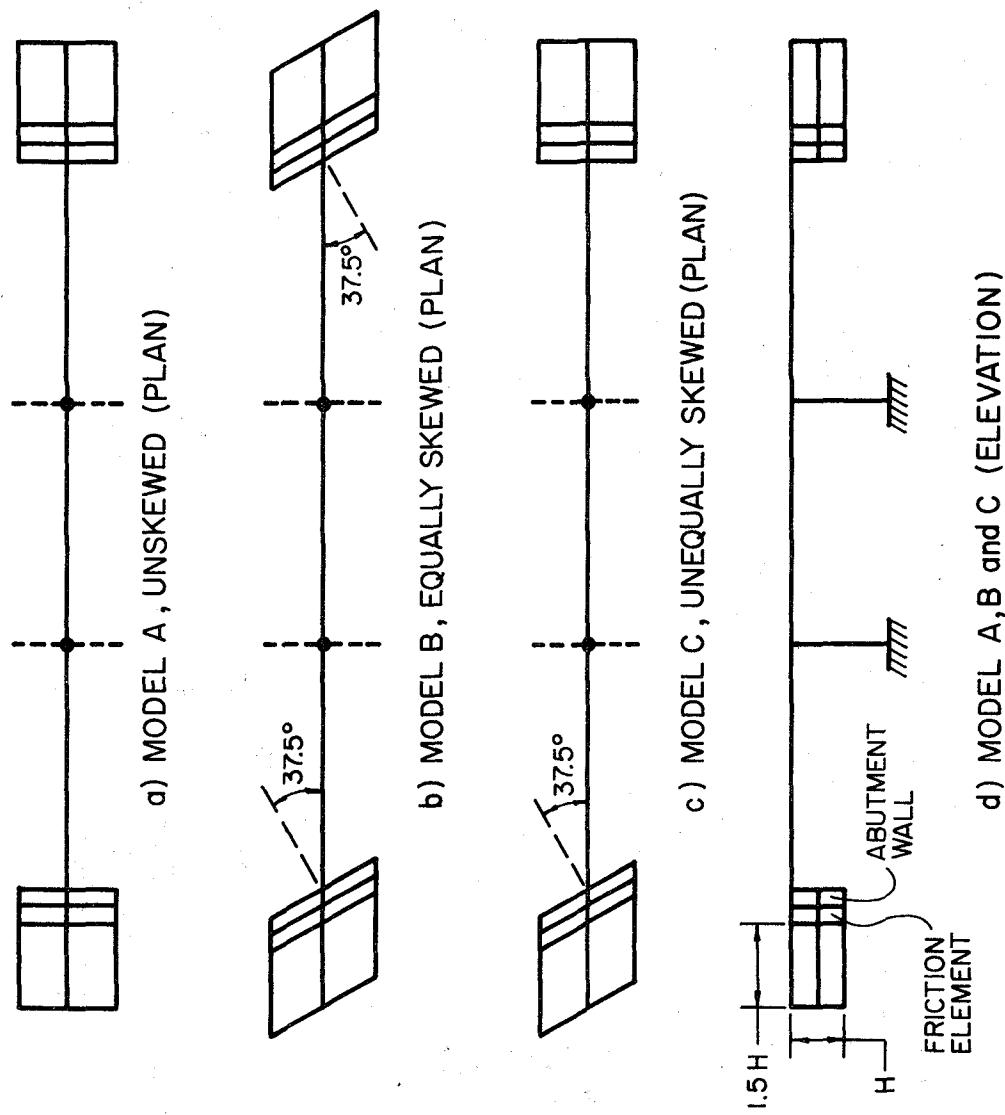


Fig. 11 Mathematical models

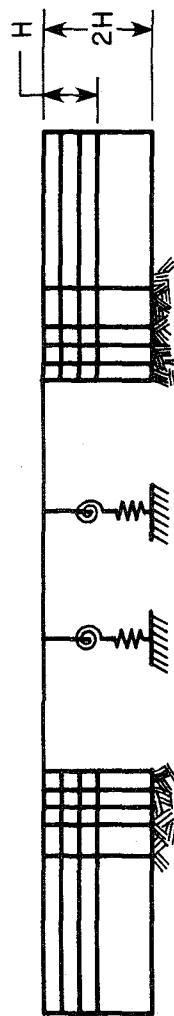
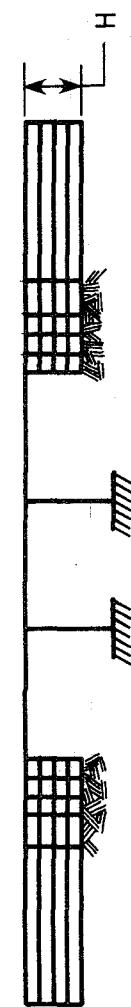
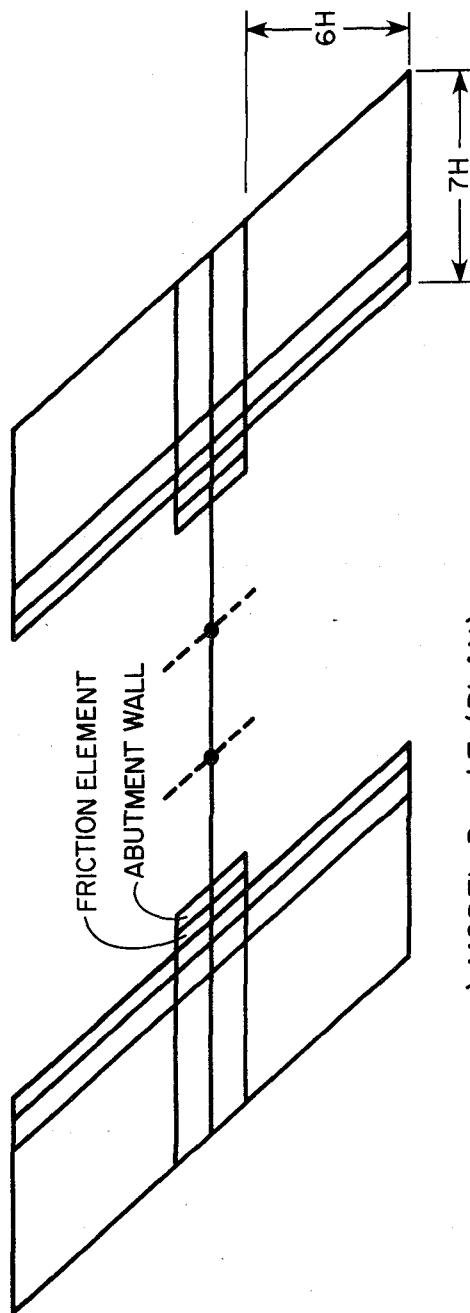


Fig. 11 (cont.) Mathematical models

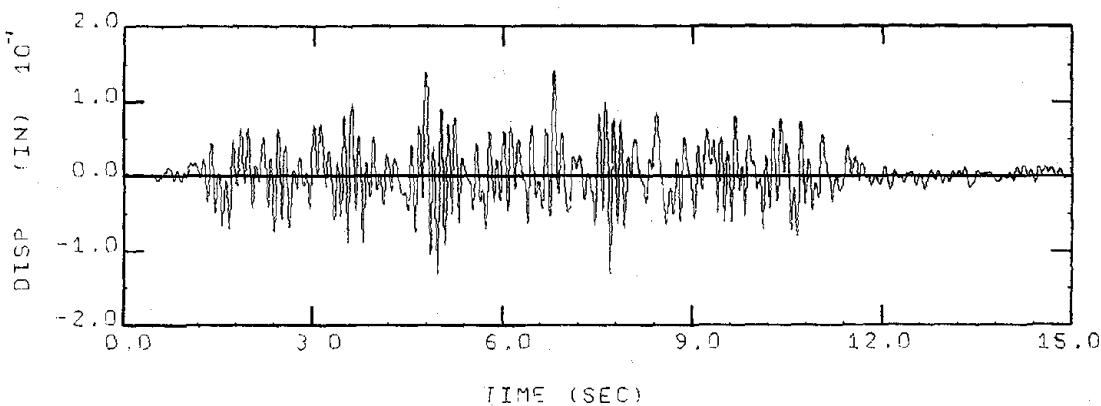


Fig. 12 Longitudinal acceleration at top of right column

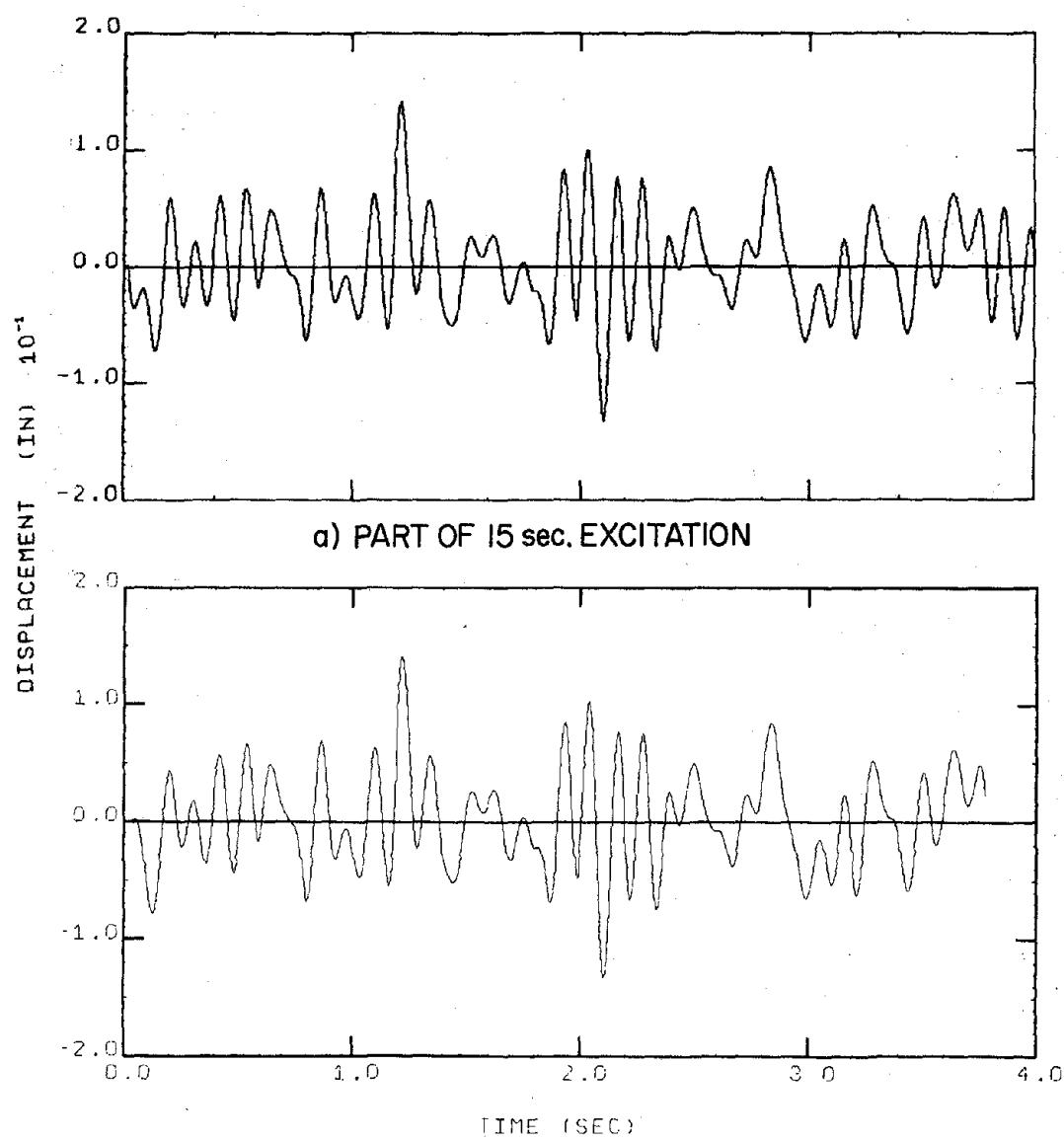


Fig. 13 Longitudinal acceleration at top of right column

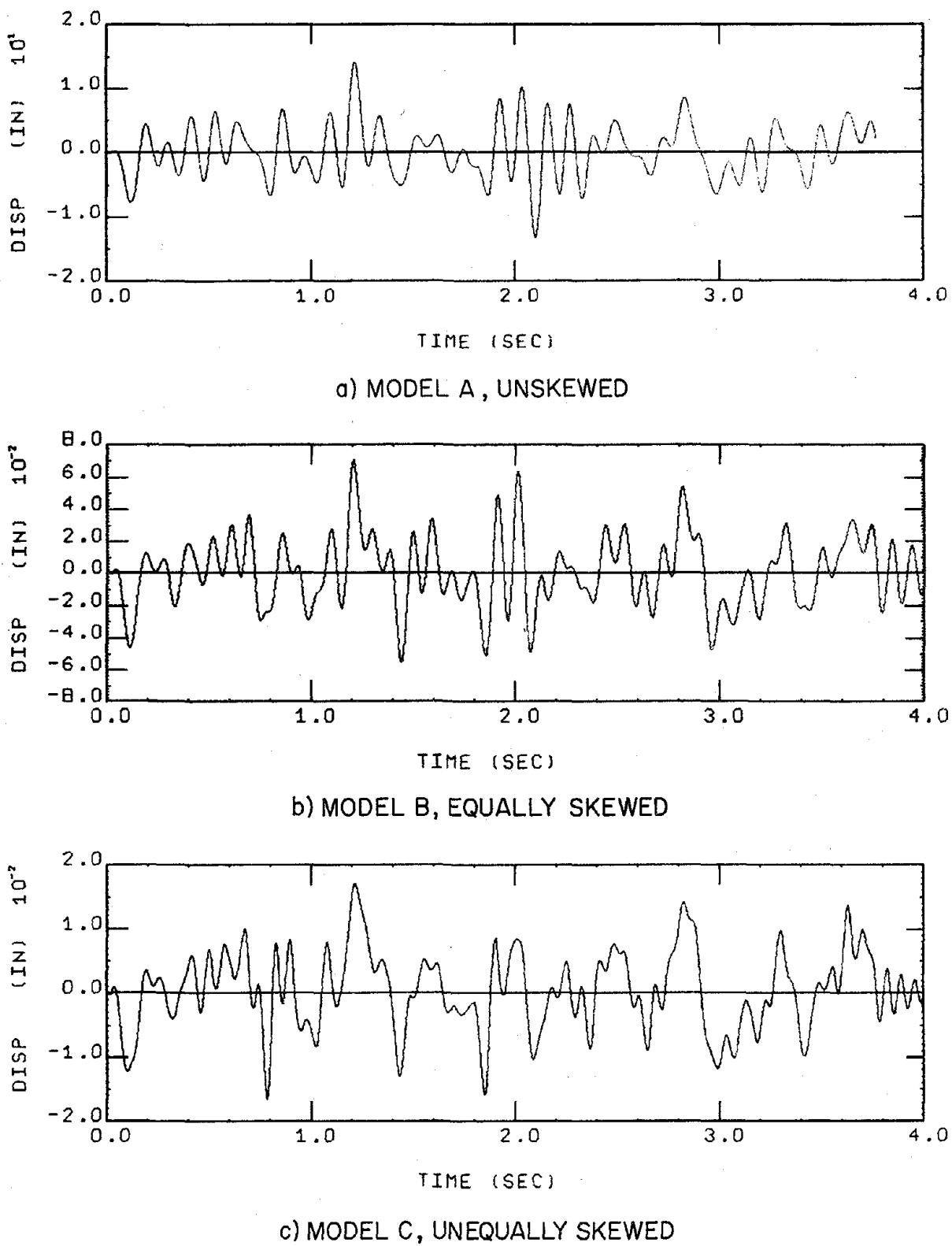


Fig. 14 Longitudinal displacement at top of right column

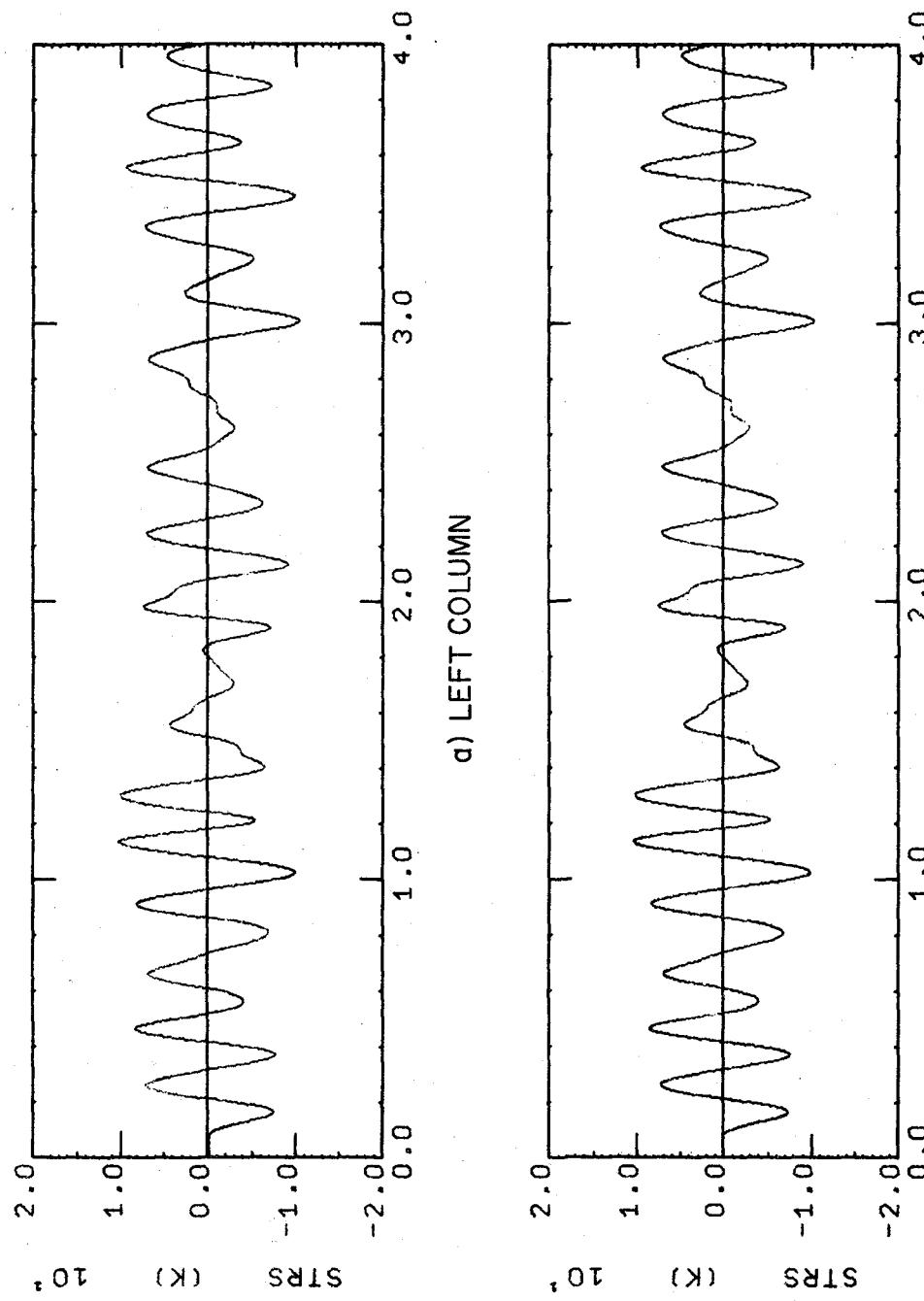


Fig. 15 Lateral shear - equally skewed - Model B

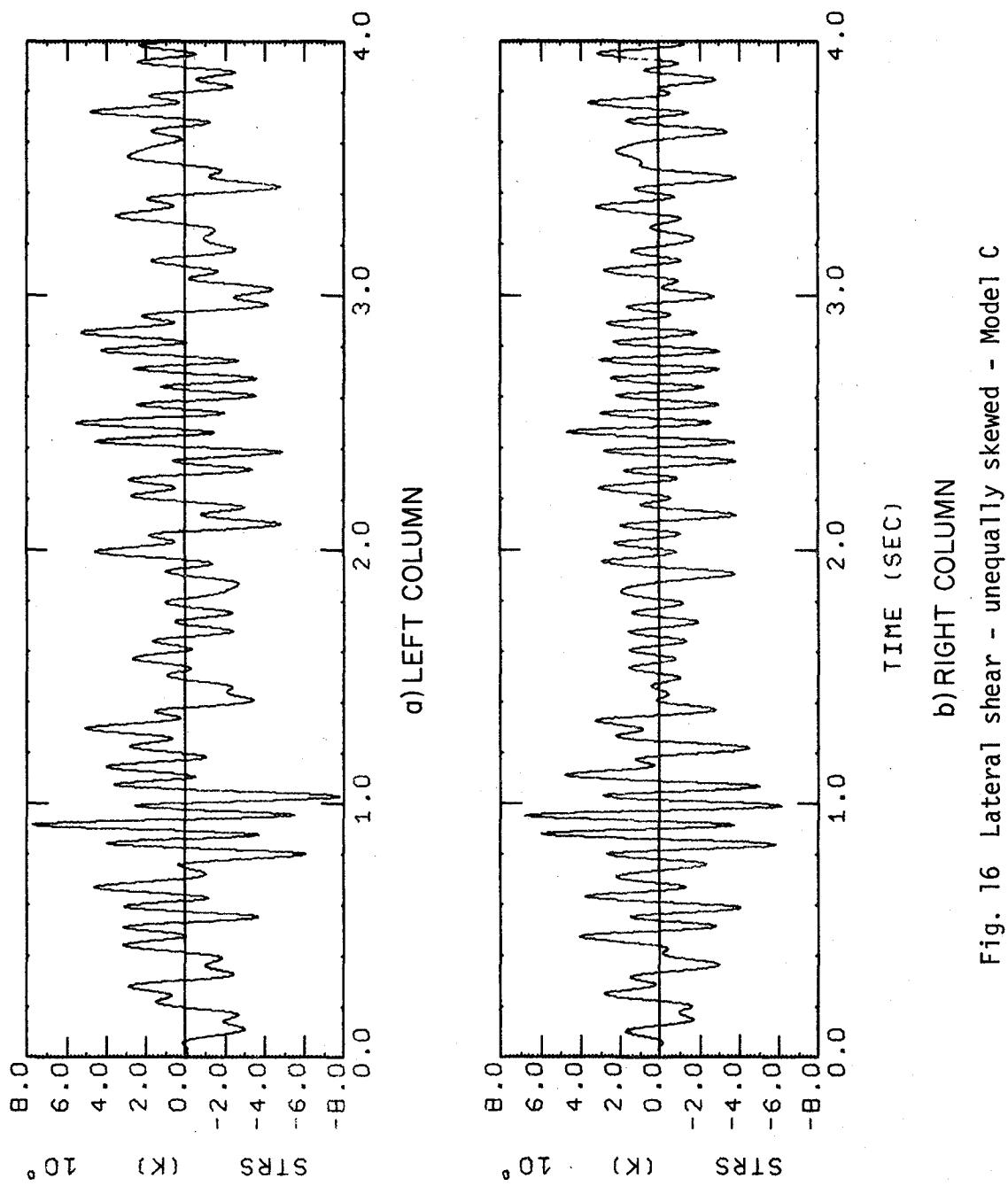


Fig. 16 Lateral shear - unequally skewed - Model C

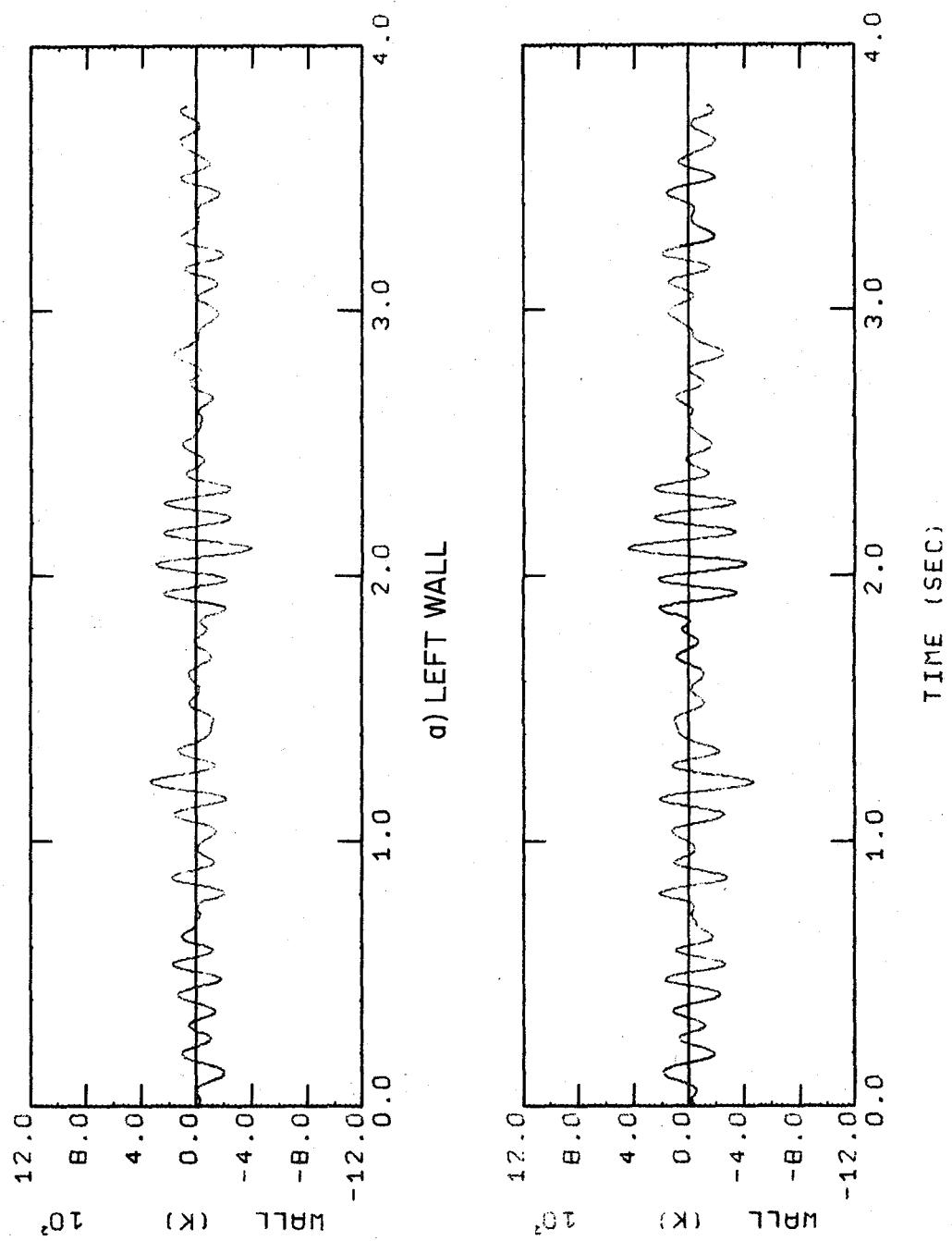


Fig. 17 Wall pressure - unskewed - Model A

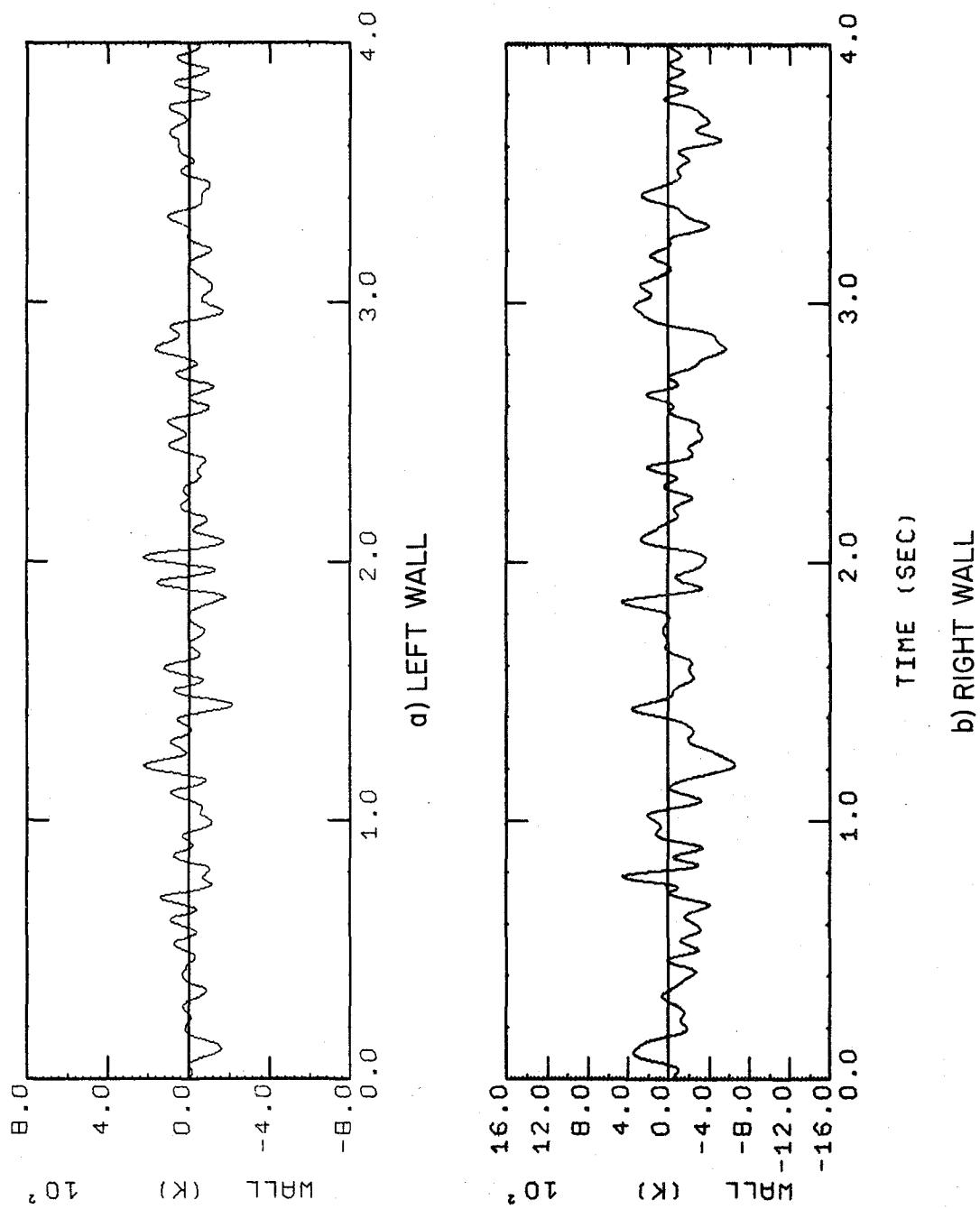


Fig. 18 Wall pressure - equally skewed - Model A

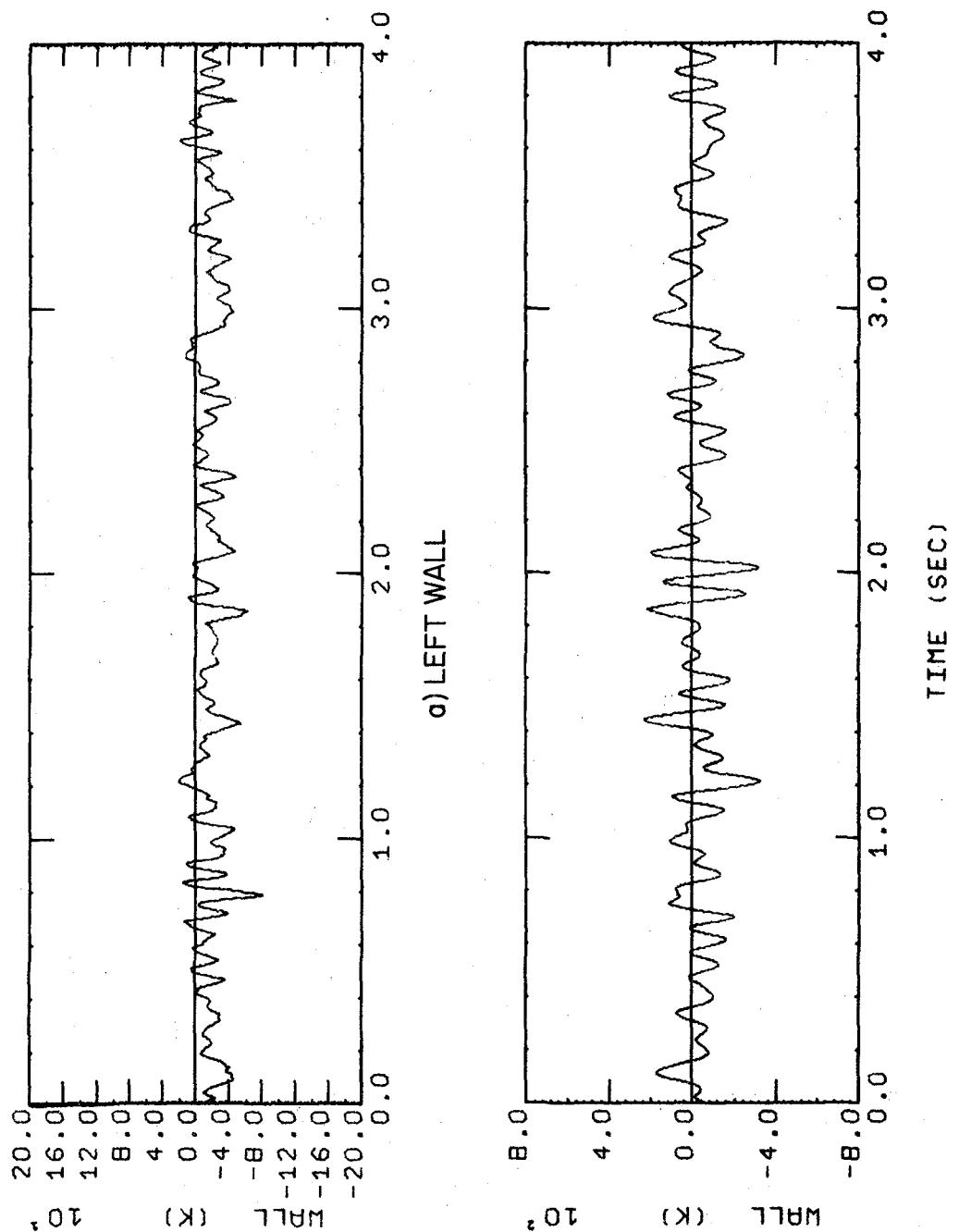


Fig. 19 Wall pressure - unequally skewed - Model C

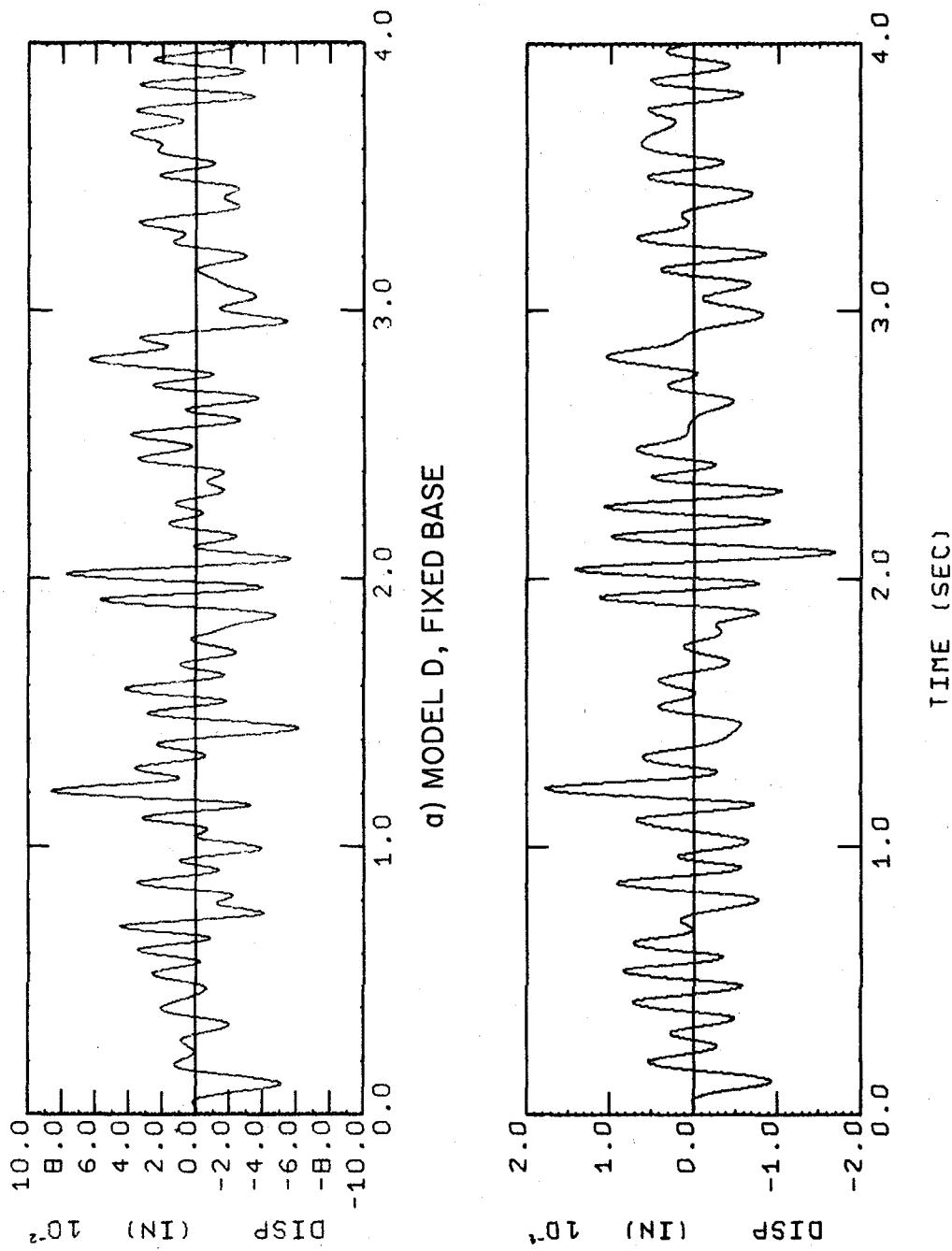


Fig. 20 Longitudinal displacement at top of left column - Model D and E

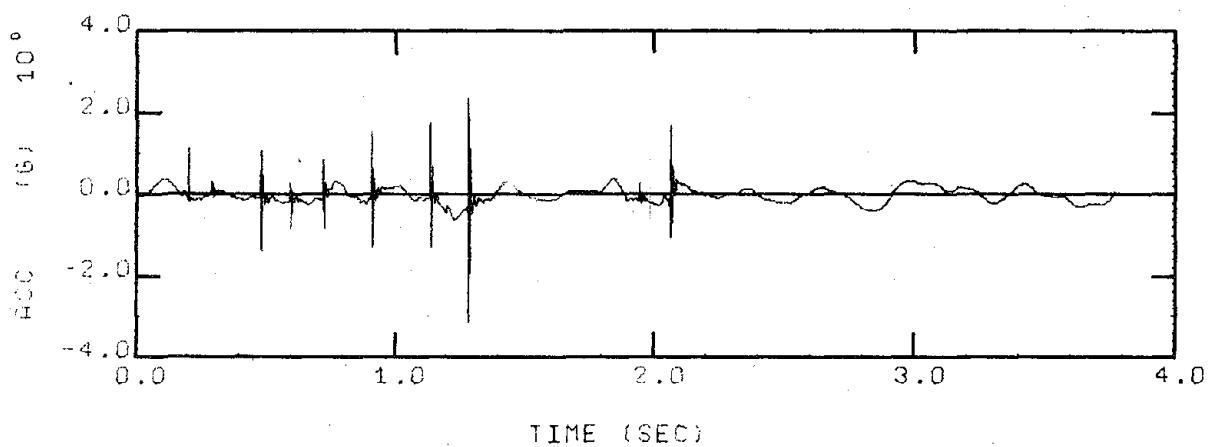


Fig. 21 Acceleration time history at contact point with impact - Model A

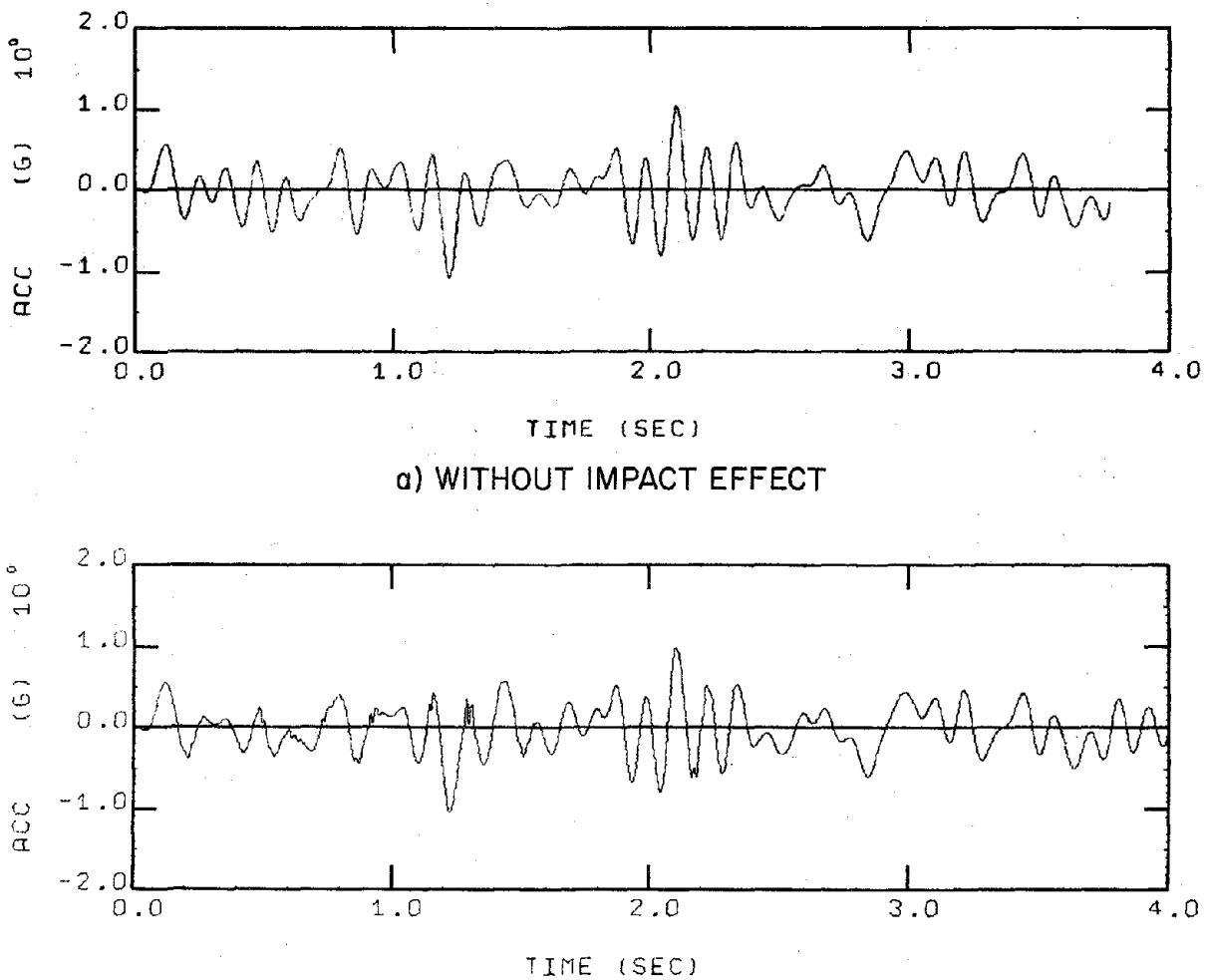


Fig. 22 Comparison of time histories at top of left column without and with impact - Model A

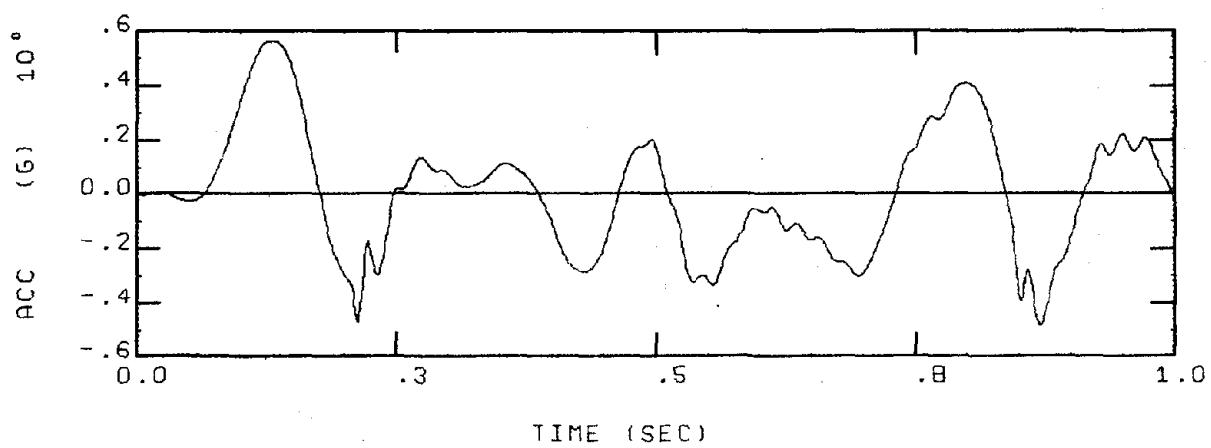


Fig. 23 Expanded scale view of effect of impact on acceleration - Model A

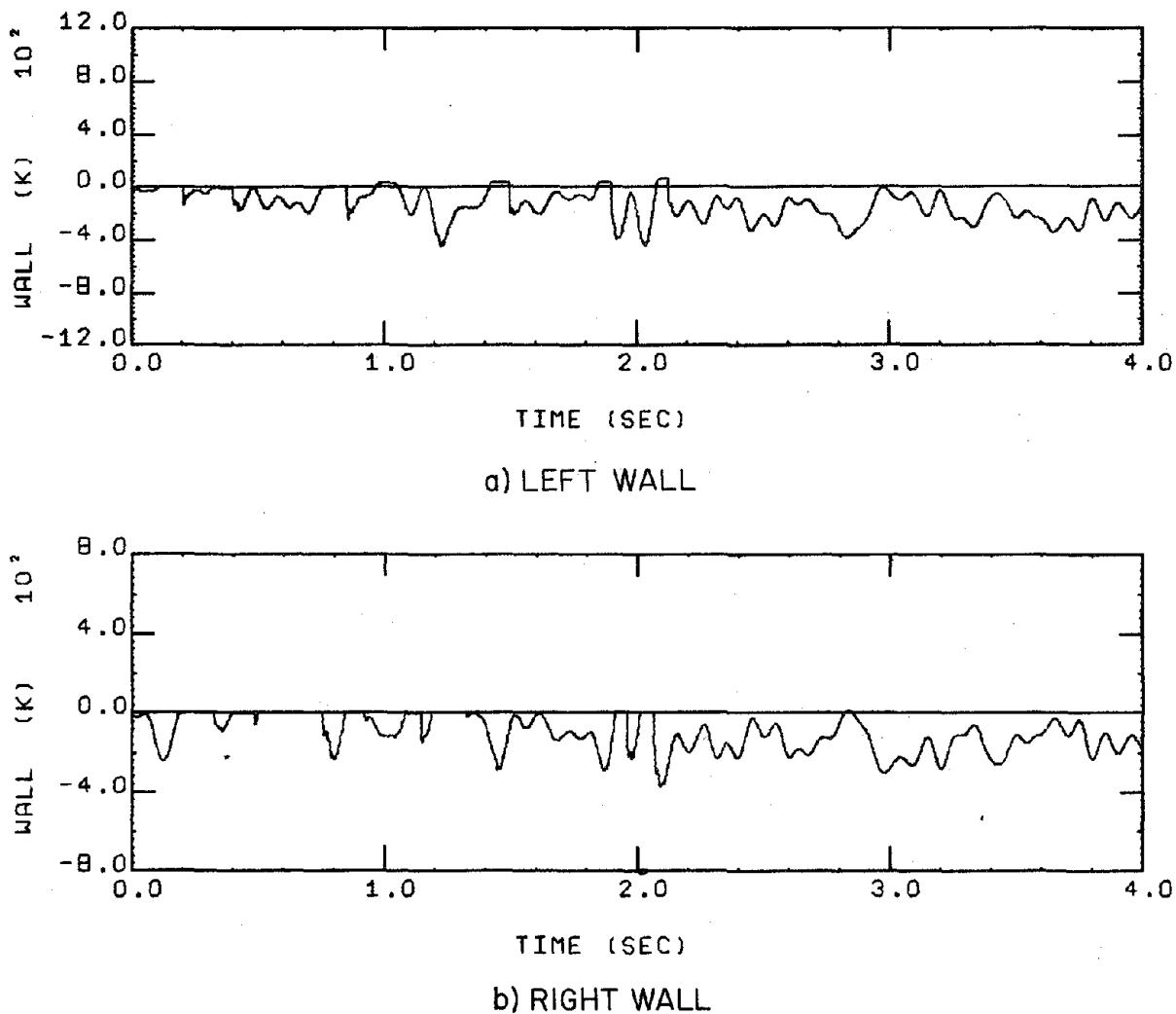


Fig. 24 Non-linear response of wall pressure - Model B

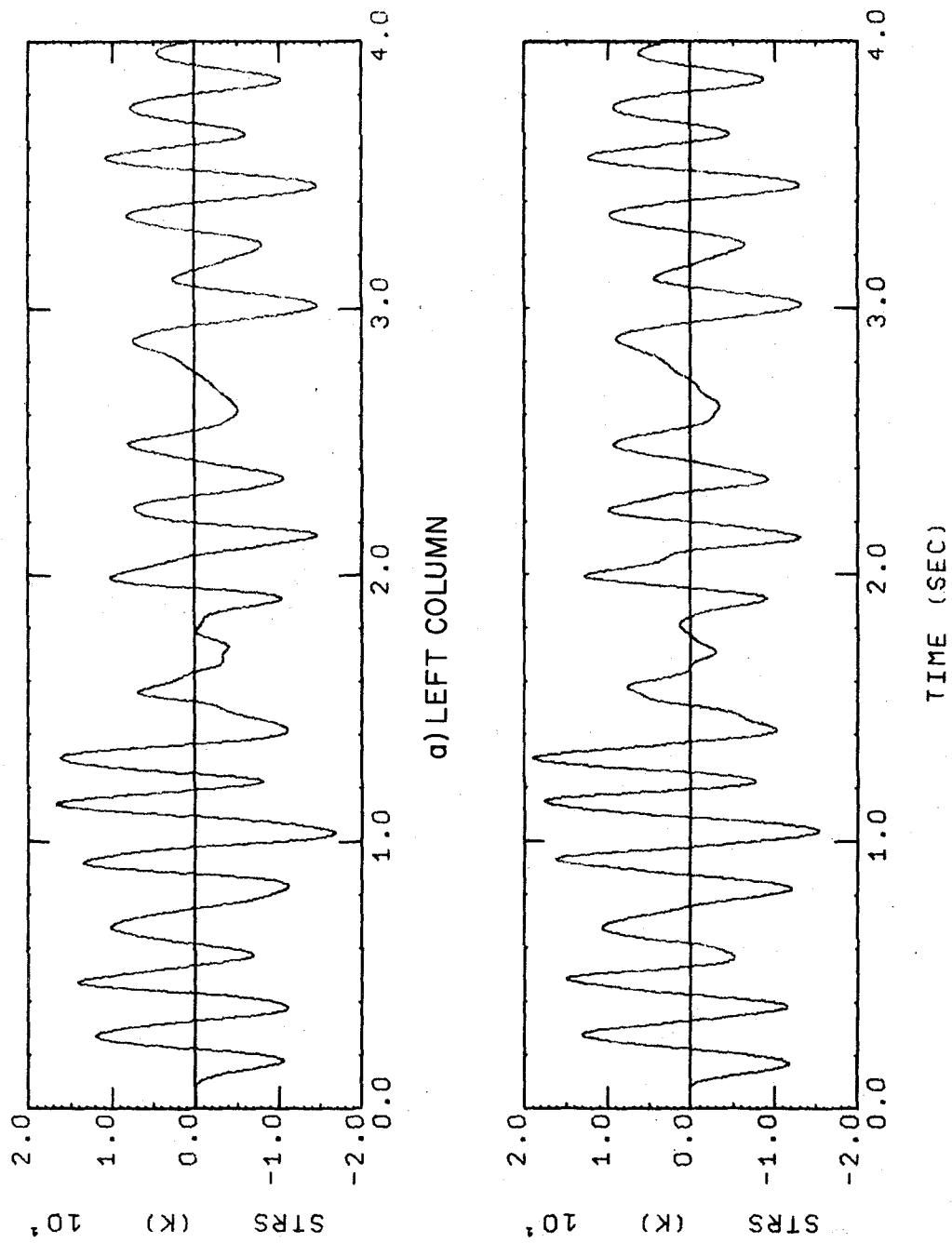


Fig. 25 Lateral column shears - Model B (non-linear)

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PROGRAM SKEM(INPUT,OUTPUT,TAPE1=OUTPUT,TAPE2=TAPE3,TAPE4=TAPE5,
TAPES,TAPE6,TAPE7,TAPE8,TAPE9,TAPE10,TAPE11,TAPE12)
C          C READ IN WALL DATA
C          NTW=0
SKEM.2      NTWEL=0
SKEM.3      READ 2,NTW
SKEM.4      IF(NTW .EQ. 0) GO TO 403
SKEM.5      READ 2,IWALL(I,I=1,NTW)
SKEM.6      WRITE(1,104),NTW,(IWALL(I),I=1,NTW)
SKEM.7      DO 302 I=1,NTW
SKEM.8      NTMEL=NWEL+IWALL(I)
SKEM.9      IWALL(I)=IWALL(I)
SKEM.10     NTMEL=NWEL+IWALL(I)
SKEM.11     302 CONTINUE
SKEM.12     403 CONTINUE
SKEM.13     C SOLVE FOR STATIC LOADING
SKEM.14     C CALL OVERLAY(6HFREED,7,0,0)
SKEM.15     SKEM.16     N15N15LAST
SKEM.17     N16N15A1*NUMEL
SKEM.18     N17N16I12*NUMEL
SKEM.19     N18N17*6MFR
SKEM.20     N18=N18-1
SKEM.21     IF(NTW .EQ. 0) GO TO 404
SKEM.22     DO 304 I=1,NTW
SKEM.23     IAM18E=I=IWALL(I)
SKEM.24     IAM18E=I=IWALL(I)
SKEM.25     304 CONTINUE
SKEM.26     404 CONTINUE
SKEM.27     CALL SECOND(S(8))
SKEM.28     C CALCULATE ELEMENT STRESS
SKEM.29     C CALL OVERLAY(6HFREED,8,0,6HRECALL)
SKEM.30     CALL SECOND(S(9))
SKEM.31     NLAST=N17
SKEM.32     N19=N18*NTW
SKEM.33     N20=N19*NTW*NTWEL
SKEM.34     N21N20*4*NTW
SKEM.35     N22=N21*NTW
SKEM.36     N23N22*NTW
SKEM.37     N24N23*NTW
SKEM.38     N25N24*NTW
SKEM.39     N26=N25+NTW-1
SKEM.40     IF(NTW .EQ. 0) GO TO 401
SKEM.41     C INITIALIZATION
SKEM.42     CALL OVERLAY(6HFREED,2,0,6HRECALL)
SKEM.43     DO 307 I=N21,N2E
SKEM.44     A(I)=0.
SKEM.45     307 CONTINUE
SKEM.46     CALL HOATA(I,N18),A(N19),A(N20),NTW,NTWEL)
SKEM.47     401 CCNTINUE
SKEM.48     N1=1
SKEM.49     N2=1+6*NURNP
SKEM.50     N3=2+4*NURATF
SKEM.51     N4=3+3*NUMEL
SKEM.52     N5=4+4*NEQA
SKEM.53     N6=5+NEQA
SKEM.54     N7=6+NEQA
SKEM.55     N8=7+NEQA
SKEM.56     N9=8+NEQA
SKEM.57     N10=N9*NECA
SKEM.58     N11=N10+NUMEL
SKEM.59     N12=N11*NSOLID
SKEM.60     N13=N12*NCMC
SKEM.61     N14=N13*NFR
SKEM.62     N15=N14*NSPRIN
SKEM.63     N26=N25+1
SKEM.64
SKEM.65
SKEM.66
SKEM.67
SKEM.68
SKEM.69
SKEM.70
SKEM.71
SKEM.72
SKEM.73
SKEM.74
SKEM.75
SKEM.76
SKEM.77
SKEM.78
SKEM.79
SKEM.80
SKEM.81
SKEM.82
SKEM.83
SKEM.84
SKEM.85
SKEM.86
SKEM.87
SKEM.88
SKEM.89
SKEM.90
SKEM.91
SKEM.92
SKEM.93
SKEM.94
SKEM.95
SKEM.96
SKEM.97
SKEM.98
SKEM.99
SKEM.100
SKEM.101
SKEM.102
SKEM.103
SKEM.104
SKEM.105
SKEM.106
SKEM.107
SKEM.108
SKEM.109
SKEM.110
SKEM.111
SKEM.112
SKEM.113
SKEM.114
SKEM.115
SKEM.116
SKEM.117
SKEM.118
SKEM.119
SKEM.120
SKEM.121
SKEM.122
SKEM.123
SKEM.124
SKEM.125

```

```

N27=N26+NFR
NINOT=0
C
C INITIALIZATION
C N26=N7+NEQA-1
DC 303 T=R27,N28E
A(I)=0
303 CONTINUE
C STORE NONLINEAR INFORMATION IN COMMON A
N28=N2E+1
N28A=N28+157*NFR
N29=N29A+157*NFR
N30=N29+NEQA
N31=N30+NEQA
N40=N31+9*NEQA
CALL NONLIN(A(N2),A(N3),A(N4),A(NS),A(N7),A(N8),A(N9),A(N33),
1 A(N15),A(N16),A(N17),A(N21),A(N28),A(N28A),
2 A(N31),A(N40),NUMEL,NEQA,KLIN,KTREL)
C
C PRINT OUT RESULT
CALL PRINTR(A(N1),A(N3),A(N4),A(NS),A(N6),A(N11),A(N12),A(N13),
1 A(N14),A(N15),A(N17),A(N21),A(N23),A(N22),A(N20),A(N21),A(N22),
405 CONTINUE
1 CALL NFORCE(A(N23),A(N24),A(N25),A(N26),NTW,NTHEL)
CALL SECOND(S(10))
C
C PRINT OUT RESULT
CALL PRINTR(A(N1),A(N3),A(N4),A(NS),A(N6),A(N11),A(N12),A(N13),
1 A(N14),A(N15),A(N17),A(N21),A(N23),A(N22),A(N20),A(N21),A(N22),
2 CALL RECORD(S(11))
C
C PRINT OUT RESULT
CALL PRINTR(A(N1),A(N3),A(N4),A(NS),A(N6),A(N11),A(N12),A(N13),
1 A(N14),A(N15),A(N17),A(N21),A(N23),A(N22),A(N20),A(N21),A(N22),
405 CONTINUE
1 CALL RECORD(S(11))
N32=N31+NEQA
N33=N32+NEQA
N34=N33+NEQA
N35=N34+NEQA
N36=N35+NEQA
N37=N36+NEQA
N38=N37+NEQA
N39=N38+NEQA
N40=N39+NEQA
N41=N40+NEQA
N42=N41+NEQA
N43=N42+NEQA
N44=N43+NEQA
N45=N44+NEQA
NLAST=NL15
NB1=1
NB2=NB3+NEQA+MBanca
NB3=NB2+NEQA+MBanca
NB4=NB5+NEQA+MEanca
NB5=NB6+NEQA+MBanca
NB6=NB7+NEQA+MBanca
NB7=NB8-1
CALL STEPA(N1),A(N2),A(N3),A(N4),A(N5),A(N6),A(N7),A(N8),A(N9),
1 A(N10),A(N11),A(N12),A(N13),A(N14),A(N15),A(N16),A(N17),A(N18),
2 A(N19),A(N20),A(N21),A(N22),A(N23),A(N24),A(N25),A(N26),A(N27),
3 A(N26),A(N27),A(N28),A(N29),A(N30),A(N31),A(N32),A(N33),
4 A(N34),A(N35),A(N36),A(N37),A(N38),A(N39),A(N40),A(N41),SKW.183,
5 A(N42),A(N43),A(N44),B(1),B(NB2),BNB(B),B(NB3),
6 BN(B6),NEQA,NTW,NTHEL,NUMEL,NUMP,KLIN,MBANDA)
C
C OUTPUT TIME HISTORY
CALL OVERLAY(6+THREED,11,0,0)
CALL SECOND(S(13))
DO 305 S(I)=S(I+1)-S(I)
305 TT=0.
306 TI=TT+S(I)
S(13)=TT
WRITE(11,L5) (S(I),I=1,13)
FORMAT(11,0F6.10)
1 FORMAT(11,2F6.10)
2 FORMAT(11,1,2A6//)
101 FORMAT(1M1,1,2A6//)
102 FORMAT(*1M1,1,2A6//)
103 FORMAT(*1M1,1,2A6//)
104 FORMAT(*1M1,1,2A6//)
105 FORMAT(*1M1,1,2A6//)
106 FORMAT(*1M1,1,2A6//)
END
SUBROUTINE LAM(LX,IND)
C
C CALCULATE LOCATION OF MASS MATRIX LM((36)) -LMC(36)
C LMA 2 THE RELATIVE DEGREE OF FREEDOM
C LMD 2 THE ELIMINATED D.O.F AFTER GUYAN REDUCTION
C
COMMON A(30000)
COMMON/ELEPAR/NUMNF,NUMEL,NETYPE,NEQA,NEOC,MBANDA,MBANDO,KLIN,MLAST,LM,10
COMMON/MATER/NUMATS,NUHATC,NUHATF,NUHATB,NUHNGE,NUHBC,MTYPE
COMMON/LNG/LMA(36),LMO(36),X(36),Y(36)
CCHMC/TOPEL/NFB(4,50),ITOPN
DIMENSION IX(111),IA(111)
EQUIVALENCE(IA,IA)
C
INITIALIZATION
DO 300 I=1,36

```

```

J1=J2-1
LMA(I)=0
LMO(I)=0
NN=6*NODE
NN0=NN-NOC
NI=NN+1
NID=NN+1
LAM.20
LAM.21
LAM.22
LAM.23
LAM.24
LAM.25
LAM.26
LAM.27
LAM.28
LAM.29
LAM.30
LAM.31
LAM.32
LAM.33
LAM.34
LAM.35
LAM.36
LAM.37
LAM.38
LAM.39
LAM.40
LAM.41
LAM.42
LAM.43
LAM.44
LAM.45
LAM.46
LAM.47
LAM.48
LAM.49
LAM.50
LAM.51
LAM.52
LAM.53
LAM.54
LAM.55
LAM.56
LAM.57
LAM.58
LAM.59
LAM.60
LAM.61
LAM.62
LAM.63
LAM.64
LAM.65
LAM.66
LAM.67
LAM.68
LAM.69
LAM.70
LAM.71
LAM.72
LAM.73
LAM.74
LAM.75
LAM.76
LAM.77
LAM.78
LAM.79
LAM.80
LAM.81
LAM.82
LAM.83
LAM.84
LAM.85
LAM.86
LAM.87
LAM.88
LAM.89
LAM.90
LAM.91
LAM.92
LAM.93
LAM.94
LAM.95
LAM.96
LAM.97
LAM.98
LAM.99
LAM.100
LAM.101
LAM.102
LAM.103
LAM.104
LAM.105
LAM.106
MULT.3
SUBROUTINE MLLT
C
*****MULTIPLY AND ADDITION OF C=C*A*B
*****TAKE ADVANTAGE OF DIAGONAL PROPERTY OF A AND SYMMETRY IN RESULTINGNU.6
*****MULT.4
*****MULT.5
*****MULT.6
*****MULT.7
*****MULT.8
*****MULT.9
*****MULT.10
*****MULT.11
*****MULT.12
*****MULT.13
*****MULT.14
*****MULT.15
*****MULT.16
*****MULT.17
*****MULT.18
*****MULT.19
*****MULT.20
*****MULT.21
*****MULT.22
*****MULT.23
*****MULT.24
*****MULT.25
*****MULT.26
*****MULT.27
*****MULT.28
*****MULT.29
*****MULT.30
*****MULT.31
*****MULT.32
*****MULT.33
*****MULT.34
*****MULT.35
*****MULT.36
*****MULT.37
*****MULT.38
C
HEXAHEDRON ELEMENT-SOLID ELEMENT-FRICTIONAL ELEMENT
NOD=6*NNUMP
N=12*NNUMP
N=3*NNUMP
N4=NN3*NNUMP
IF IX(4) .EQ. 0 GO TO 401
DO 301 I=1,IT
NODE=IX(I)
J3=3*I
J2=J3-1
J1=J2-1
NN=6*NODE
NN0=NN-NOC
NI=NN+1
NTO=NN0+
LMA(I)=IA(NN-5)
LMA(J2)=IA(NN-4)
LMA(J1)=IA(NN-5)
LMO(I)=IA(NN0-3)
LMO(J2)=IA(NN0-4)
LMO(J1)=IA(NN0-5)
IF IND .EQ. 0 GO TO 301
XX(I)=A(NN+NODE)
YY(I)=A(NN+NODE)
ZZ(I)=(NN+NODE)
301 CONTINUE LE. 31 GC TO 402
C
TOP SOL ELEMENT NEXT TO FRICTION JOINT
C
DO 303 I=1,4
NODE=NFB(I,ITOPN)
J3=3*I+24
J2=J3-1
J1=J2-1
NN=6*NODE
NN0=NN-NOC
NI=NN+1
NTO=NN0+
LMA(I)=IA(NN-3)
LMA(J2)=IA(NN-4)
LMA(J1)=IA(NN-5)
LMO(I)=IA(NN0-3)
LMO(J2)=IA(NN0-4)
LMO(J1)=IA(NN0-5)
303 CONTINUE
GO TO 402
C
ONE-DIMENSIONAL ELEMENT-BEAM COLUMN SPRING
401 CONTINUE
NO=2
IF IX(9) .EQ. 2 NODE=3
DO 302 I=1,IT
NODE=TX(I)
J6=6*I
J5=J6-1
J4=J5-1
J3=J4-1
J2=J3-1
LAM.20
LAM.21
LAM.22
LAM.23
LAM.24
LAM.25
LAM.26
LAM.27
LAM.28
LAM.29
LAM.30
LAM.31
LAM.32
LAM.33
LAM.34
LAM.35
LAM.36
LAM.37
LAM.38
LAM.39
LAM.40
LAM.41
LAM.42
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LAM.64
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LAM.96
LAM.97
LAM.98
LAM.99
LAM.100
LAM.101
LAM.102
LAM.103
LAM.104
LAM.105
LAM.106
MULT.3
SUBROUTINE MLLT
C
*****MULTIPLY AND ADDITION OF C=C*A*B
*****TAKE ADVANTAGE OF DIAGONAL PROPERTY OF A AND SYMMETRY IN RESULTINGNU.6
*****MULT.4
*****MULT.5
*****MULT.6
*****MULT.7
*****MULT.8
*****MULT.9
*****MULT.10
*****MULT.11
*****MULT.12
*****MULT.13
*****MULT.14
*****MULT.15
*****MULT.16
*****MULT.17
*****MULT.18
*****MULT.19
*****MULT.20
*****MULT.21
*****MULT.22
*****MULT.23
*****MULT.24
*****MULT.25
*****MULT.26
*****MULT.27
*****MULT.28
*****MULT.29
*****MULT.30
*****MULT.31
*****MULT.32
*****MULT.33
*****MULT.34
*****MULT.35
*****MULT.36
*****MULT.37
*****MULT.38
C
*****C STORED IN COMMON ELEMENT WISE
*****NC=LOCATION OF A1,B11,C11 IN COMMON BL.CCK A
*****NRA = NRB-NO. OF ROW OF MATRIX A AND B
*****NCA-NO. OF COLUMN OF MATRIX A EQUAL TO 1 IF DIAGONAL MATRIX
*****NCB-NC. OF COLUMN OF MATRIX B
*****K=INDICATOR OF RESULTING MATRIX
*****C IS UNSYMETRICAL
*****C IS BANDED AND SYMETRICAL
*****MBAND=HALF BAND WIDTH OF RESULTING MATRIX
*****LARGE B(220000)
*****COMMON A(30000),CCHMOMULT/NA,NE,NC,NRA,NRB,NCB,K,MBAND
*****LARGE C(8000)
*****MULTIPLICATION IN ROW WISE
*****MBAND=1
*****INITIALIZATION
*****DC 301 I=1,NRA
*****J=1
*****II=NC+I-1
*****ITBE=NC+I-1
*****SYMMETRICAL CASE
*****IF K .EQ. 2 J=I
*****1000 CONTINUE
*****JJ=NA+I-1
*****KK=NRB*(J-1)
*****IF NCA .EQ. 1 KK=KK*(J-1)
*****DO 302 N=,NCA
*****IF (B(J,J)) .EQ. 0.0 OR. B(KK) .EQ. 0.0 GO TO 403
*****TEP=B(J,J)*B(KK)
*****C (II)=C(II)+TEMP

```

```

403 CONTINUE
  K=K+1
  J=J+1
  302 CCNTINUE
    I=I+1
    IF (J .EQ. NCB) GO TO 401
    J=J-1
    I=I-1
    GO TO 1000
  401 CONTINUE
    IF (K .NE. 2) GO TO 301
    C CALCULATE HALF BANDWIDTH
    2000 CONTINUE
      IF (I(I)) .EQ. 0.0) GO TO 402
      MB=(I(I)-NC-I+1)/NRA+1
      IF (MB .GT. MEANDR) MEANDR=MB
      GO TO 301
  402 CONTINUE
    I=I-1
    NRA=I
    IF (I .LE. IIIBE) GO TO 301
    GO TO 2000
  301 CONTINUE
    RETURN
  END

SUBROUTINE WFORCE (ICOL,WALL,NELM,INFH,WGEOM,MFX,MFY,MFR,ZEOD,MFORCE,2
     NT,NWEL)
C CALCULATE THE TOTAL PERPENDICULAR FORCE AGAINST WALL, MFX,
C AND THE LINE OF ACT. HORIZONTAL SHEAR, MFZ, AND THE LINE OF ACT. HORIZONTAL
C POSITION
C CCMONITIME/JUMP,T,DT,MFR,MTE,KPRINT
C DIMENSTION WALL(6,1),INFH(NT,1),WGEOM(NT,1),MFX(NTM),
C 1WFY(1,1),MFZ(1),YB(1,1),ZEDG(1),RELW(1),ICCL(1)
C CALCULATE WALL BY WALL
C INITIALIZATION
DO 301 I=1,NTW
  HMY=0.0
  MFTZ=0.0
  MFX(1)=0.0
  MFY(1)=0.0
  MFZ(1)=0.0
  NEL=NELM(1)
  THETA=IGEOM(I,1)
  YBAR=AGE OM(1,2)
  ZBAR=AGE OM(1,3)
  AREA=AGE OM(1,4)
  AREA=ARE A144,0
  AREA=ARE AF144,0
  AF1=AF1AFLCAT(NEL)
  DO 302 I=1,NEL
    NEL=INFW(I,1)
    TICOL=TCOL(NGEL)
    MF=WL(I,1)
    NFX(I)=WFX(I)*MF
    NFX(I)=WL(I,1)*TICOL
    NFX(I)=WL(I,2)*TICOL
    NFX(I)=WL(I,3)*TICOL
    NFX(I)=WL(I,4)*TICOL
    NFX(I)=WFZ(I)+MF
    NFX(I)=MFZ(I)+MF
    RESULT=OF NORMAL FORCE ABOVE BASE
    YY=MALL(5,TICOL)
    MMY=MNY(MF1)*(YY-YBAR)
    ZZ=MALL(6,TICOL)
  302 CCNTINUE
    M=TDAT(N)
    IF (M .EQ. 0) GO TO 304
    D(I)=U(M)
    DO 304 I=1,6
      WRITE (1,113) N,(D(I),I=1,6)
    304 CCNTINUE
    M=TDAT(N)
    IF (M .EQ. 0) GO TO 304
    D(I)=U(M)
    DO 304 I=1,6
      WRITE (1,113) N,(D(I),I=1,6)
    302 CCNTINUE
    C NODAL VELOCITY
    YY=MALL(5,TICOL)
    IF (JUMP .EQ. 0) GC TO 404
    WRITE (1,504)
    JUMP=T
  404 CONTINUE
    WRITE (1,113) N,(D(I),I=1,6)
    PRINTR 47
    PRINTR 48
    PRINTR 49
    PRINTR 50
    PRINTR 51
    PRINTR 52
    PRINTR 53
  END

```

```

      WRITE(1,102) VGXT,VGYT,VGZT
      DO 305 N=1,NUMNP
        D(I)=0.
        H=IDA(I,N)
        IF(H .EQ. 0) GO TO 307
        D(I)=VALM
        WRITE(1,103) N,(D(I),I=1,6)
      305 CONTINUE
      404 CONTINUE
      IF(JUMP .EQ. 0) GO TO 405
      C
      NODAL ACCELERATION
      WRITE(1,505)
      WRITE(1,101) JUMPT
      WRITE(1,102) OCCCX,OCCCY,OCCZ
      DO 308 N=1,NUMNP
        DO 310 I=1,6
        D(I)=0.
        M=IDA(I,N)
        IF(M .EQ. 0) GO TO 310
        D(I)=ACCA(M)
      310 CONTINUE
      WRITE(1,103) N,(D(I),I=1,6)
      308 CONTINUE
      405 CONTINUE
      C
      STRESS AT CENTER OF SOIL ELEMENT
      WRITE(1,506)
      IF(NSOLID .EQ. 0) GO TO 413
      DC 311 IN=1,NSOLID
      N=ISOLIN
      DO 313 I=1,6
      D(I)=SIG(I,N)
      313 CONTINUE
      WRITE(1,103) N,(D(I),I=1,6)
      311 CONTINUE
      413 CONTINUE
      C
      END FORCES OF BEAM OR COLUMN AT ENDS
      WRITE(1,507)
      IF(INCNC .EQ. 0) GO TO 414
      DO 314 IN=1,ACAC
        N=ICONIN
        DO 315 I=1,6
        D(I)=SIG(I,N)
      315 CONTINUE
      WRITE(1,103) N,(D(I),I=1,6)
      DO 316 I=7,12
        D(I)=SIG(I,N)
      316 CONTINUE
      WRITE(1,104) (D(I),I=1,6)
      314 CONTINUE
      414 CONTINUE
      C
      AXIAL FORCE AT BOUNDARY SPRING
      WRITE(1,508)
      IF(NSPIN .EQ. 0) GO TO 415
      DO 322 IN=1,NSPIN
        N=ISP(IN)

```

PRINTR.54 PRINTR.55 PRINTR.56 PRINTR.57 PRINTR.58 PRINTR.59 PRINTR.60 PRINTR.61 PRINTR.62 PRINTR.63 PRINTR.64 PRINTR.65 PRINTR.66 PRINTR.67 PRINTR.68 PRINTR.69 PRINTR.70 PRINTR.71 PRINTR.72 PRINTR.73 PRINTR.74 PRINTR.75 PRINTR.76 PRINTR.77 PRINTR.78 PRINTR.79 PRINTR.80 PRINTR.81 PRINTR.82 PRINTR.83 PRINTR.84 PRINTR.85 PRINTR.86 PRINTR.87 PRINTR.88 PRINTR.89 PRINTR.90 PRINTR.91 PRINTR.92 PRINTR.93 PRINTR.94 PRINTR.95 PRINTR.96 PRINTR.97 PRINTR.98 PRINTR.99 PRINTR.100 PRINTR.101 PRINTR.102 PRINTR.103 PRINTR.104 PRINTR.105 PRINTR.106 PRINTR.107 PRINTR.108 PRINTR.109 PRINTR.110 PRINTR.111 PRINTR.112 PRINTR.113 PRINTR.114 PRINTR.115

PRINTR.116 PRINTR.117 PRINTR.118 PRINTR.119 PRINTR.120 PRINTR.121 PRINTR.122 PRINTR.123 PRINTR.124 PRINTR.125 PRINTR.126 PRINTR.127 PRINTR.128 PRINTR.129 PRINTR.130 PRINTR.131 PRINTR.132 PRINTR.133 PRINTR.134 PRINTR.135 PRINTR.136 PRINTR.137 PRINTR.138 PRINTR.139 PRINTR.140 PRINTR.141 PRINTR.142 PRINTR.143 PRINTR.144 PRINTR.145 PRINTR.146 PRINTR.147 PRINTR.148 PRINTR.149 PRINTR.150 PRINTR.151 PRINTR.152 PRINTR.153 PRINTR.154 PRINTR.155 PRINTR.156 PRINTR.157 PRINTR.158 PRINTR.159 PRINTR.160 PRINTR.161 PRINTR.162 PRINTR.163 PRINTR.164 PRINTR.165 PRINTR.166 PRINTR.167 PRINTR.168 PRINTR.169 PRINTR.170 PRINTR.171 PRINTR.172 PRINTR.173 PRINTR.174 PRINTR.175 PRINTR.176 PRINTR.177

PRINTR.119 CONTINUE
 WRITE(1,103) N,(0(I),I=1,3)
 323 CONTINUE
 WRITE(1,103) N,(0(I),I=1,3)
 322 CONTINUE
 415 CONTINUE
 C
 FORCES AGAINST WALL
 IF(NIN .EQ. 0) GO TO 410
 WRITE(1,510)
 DO 324 IN=1,NTW
 WRITE(1,103) N,FX(N),WFY(N),WFZ(N)
 324 CONTINUE
 C
 WRITE INFORMATION ON TAPE AT EVERY MTAPE A
 PRINTR.70
 C
 NODAL FORCE AT FRICTIONAL ELEMENT
 WRITE(1,105)
 406 CONTINUE
 IF(NINTEL .EQ. 0) GO TO 408
 DO 337 IN=1,NFR
 N=IFR(IN)
 DO 338 I=1,4
 D(I)=IG(I,N)
 338 CONTINUE
 WRITE(1,105) N
 PRINTR.80
 PRINTR.81
 PRINTR.82
 PRINTR.83
 PRINTR.84
 PRINTR.85
 PRINTR.86
 PRINTR.87
 PRINTR.88
 PRINTR.89
 PRINTR.90
 PRINTR.91
 PRINTR.92
 PRINTR.93
 PRINTR.94
 PRINTR.95
 PRINTR.96
 PRINTR.97
 PRINTR.98
 PRINTR.99
 PRINTR.100
 PRINTR.101
 PRINTR.102
 PRINTR.103
 PRINTR.104
 PRINTR.105
 PRINTR.106
 PRINTR.107
 PRINTR.108
 PRINTR.109
 PRINTR.110
 PRINTR.111
 PRINTR.112
 PRINTR.113
 PRINTR.114
 PRINTR.115

PRINTR.116
 KJJ=JUMP/MTAPE
 IF(KJJ .NE. 0) JUMP GC 407
 C
 WRITE INFORMATION ON TAPE AT EVERY MTAPE STEPS
 NTIMEM=INT(ME)
 WRITE(6) (QUAL(I,I=1,NEGA),(ACCA(I),I=1,NEGA))
 WRITE(6) ((SIG(I,N),I=1,6),N=1,NUMEL)
 IF(NIN .EQ. 0) GO TO 407
 WRITE(10) (WFY(I),WFZ(I),YABV(I),ZEDG(I),I=1,NTW)
 407 CONTINUE
 RETURN
 C
 TIME NODE\*
 101 FORMAT(\* GROUND MOTION\*,6X,3E12.4)
 102 FORMAT(15X,F12.4)
 103 FORMAT(15X,6E12.4)
 104 FORMAT(20X,6E12.4)
 105 FORMAT(15X,15)
 501 FORMAT(\*STATIC ANALYSIS RESULT\*/)
 502 FORMAT(\*DYNAMIC ANALYSIS RESULT\*/)
 503 FORMAT(\* RELATIVE DISPLACEMENT-U//)
 1
 \* TIME NODE\*
 2
 3
 4
 5
 6
 504 FORMAT(5X,\*RELATIVE VELOCITY-V//)
 1
 \* TIME NODE\*
 2
 3
 4
 5
 6
 505 FORMAT(5X,\*ACCELERATION-AC\*/)
 PRINTR.116
 PRINTR.117
 PRINTR.118
 PRINTR.119
 PRINTR.120
 PRINTR.121
 PRINTR.122
 PRINTR.123
 PRINTR.124
 PRINTR.125
 PRINTR.126
 PRINTR.127
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 PRINTR.170
 PRINTR.171
 PRINTR.172
 PRINTR.173
 PRINTR.174
 PRINTR.175
 PRINTR.176
 PRINTR.177

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1      TIME NODE*, TIME NODE*, ACC-Y*, 6X*, ACC-Z*, 6X*, ACC-XX*,  

2      6X*, ACC-X*, 6Y*, ACC-ZZ*, 6Z*, SEC NO.*  

3      * STEP SEC NO.*  

4      5X*, IN/S--*, EX*, IN/S--*, EX*, IN/S--*, 5X*, RAD/S--*,  

5      5X*, RAD/S--*, EX*, RAD/S--*,  

6      5X*, RAD/S--*, EX*, RAD/S--*/  

506 FCRHAT(5X,*-RAD/S--*,EX*,PRAD/S--*)/ AT CENTER CF SCIL ELEMENT//  

1      12X* ELEMENT*,  

2      7X*, SIGX*, 7X*, SIGZ*, 7X*, SIGY*, 7X*, SIGXYZ*,  

3      7X*, SIGXZ*,  

4      6X*, NO*, 6X*, KSI**//  

507 FORMAT(5X,*-BEAM*,COLUMN FORCES,MOVENT AT I END*/  

1      AND J END//  

2      12X* ELEMENT*.  

3      4X*, AXIAL-X*, 4X*, SHEAR-Y*, 4X*, SHEAR-Z*, 4X*, BM-XX*,  

4      5X*, BM-Y*, 6X*, BM-ZZ*,  

5      16X*, NO*, 6X*, KIP-IN//  

508 FCRHAT(5X,* STRESS AT CENTER CF FRICITION ELEMENT//  

1      5X*, IT IS 1 STEP LATER THAN OTHER LINEAR ELEMENT//  

2      12X* ELEMENT NO. KSI**//  

3      4X*, SHEAR-X*, 4X*, SHEAR-Y*, 4X*, SHEAR-Z*, 4X*, YIELD//  

509 FORMAT(5X,*-AXIAL FORCE AT BOUNDARY SPRING//  

1      12X*, ELEMENT*,11X*,*,11X*,Y*,11X*,Z*/  

1      12X*, NO*, 6X*, KIN//)  

510 FORMAT(5X-*TOTAL FORCES ACTING ON THE WALL//  

1      15X* WALL*,4X*,NORMAL-X*,4X*,VERTIC-Y*,4X*,HORIZT-Z*,  

2      8X*, YBAR*, 8X*, ZEAR*,  

3      17X*, NO*,10X*,FT*,3K*,KIP//)  

END  

SUBROUTINE MCNLIN(PARAFL,IX,DVA,DAA,IFR,OSIG,SIG,  

1      WALL,NIND,KDUDSTIF,DSTIFF,TRAA,FFDA,NUMEL,  

2      NEDA,KLIN,NTREL)  

*****  

C FORM DIFFERENTIAL LOAD VECTOR(TOKY)(CL)  

C DUE TO CHANGE OF STIFFNESS OF FRICITION ELEMENT  

C *****  

C CMMNON/NIND,INDTO,OKU(200)  

CMMNON/TIME/JUMP(1),DT,MPTM,MTAPE,*PRINT  

C COMMON/LMC/LMA(36),LPO(36),XX(8)*,YY(8),ZZ(8)  

C COMMON/MAX1,MAX2  

C COMMON/STROUT/NSOLID,NCORG,NFR,NSPIN  

REAL MAX1,MAX2  

DIMENSION SS(758),ISS(758),DTU(12),F(3)*TT(3,3),H(5),DASA(12,12),  

1      SA(12,12),AS(12,12),AS(12,12),SA(12,12),SA(3,12),  

2      SA(12,12),SA(3,12),FSI(12,1),SC(3,12),  

3      DIMENSION TRA(11),TRA(11),DVA(11),DAA(11),WALL(6,1),DSTIF(11),  

1      DSTIFF(11),TRAA(11),DASP(12,12),TKDA(11),  

COMMON/BOUND/BCBLND(50),TBOUND(50),CENTER(50),WS(50),WT(50),DNH(50)  

COMMON/ITER/ITER,ITERAL,ERROR,SUSTEP  

COMMON/SEQDUR/ADDT  

COMMON/NDIM/NDIM,STEP,MYTEL,NSTEP,NCHANS,NINDPA(50)  

EQUIVALENCE(SS,ISS)  

NIND=0  

NIND=0  

REINC 11  

INITIALIZATION  

IF(JUMP .NE. 0) GO TO 414  

DO 322 I=1,50  

      J12=K12+J  

      J12=K12+J

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JI4=K14+J
DASAJ,I,J)=SS(J,I)
DASAP,I,J)=SS(J,I)
ASA1,I,J)=SS(J,I)
ASA2,I,J)=SS(J,I)
ASA4,I,J)=SS(J,I)
CONTINUE
KT=743
DO 308 I=1,3
   KI=KI+3*I
   DO 308 J=1,3
      JI=KI+J
      TT(J,I)=SS(J,I)
      PRINT 308
      * THE PROGRAM MUST BE STOP, D AT LOCAL AND GLOBAL IS*,*
      1 FORMAT(* THE PROGRAM MUST BE STOP, D AT LOCAL AND GLOBAL IS*,*
      1          25/
      2          * WHICH IS MISTAKENLY ELIMINATED, AT NONLIN=50*)
      401 CONTINUE
      IA=LMA(I)
      IF(IA .NE. 0) GO TO 402
      DTU(I)=0.
      GO TO 403
      402 CONTINUE
      DTU(I)=DUA(I,A)
      403 CONTINUE
      305 CONTINUE
      IF(KLKN .EQ. 0) GC TO 409
      C SEPARATION IN LOCAL DIRECTION
      W(1)=TT(3,1)*DTU(1)+TT(13,2)*DTU(2)+TT(3,3)*DTU(3)
      W(2)=TT(3,1)*DTU(4)+TT(13,2)*DTU(5)+TT(3,3)*DTU(6)
      W(3)=TT(3,1)*DTU(7)+TT(13,2)*DTU(8)+TT(3,3)*DTU(9)
      W(4)=TT(3,1)*DTU(10)+TT(13,2)*DTU(11)+TT(3,3)*DTU(12)
      W(NW)=W(1)+W(2)+W(3)+W(4)
      D(NW)=W(NW)
      C
      NT(NW)=NT(NW)+1
      IF(JUMP .NE. 0) GC TO 416
      C STORE STATIC SEPARATION AS NECESSARY GAP WITHOUT YIELD
      SHIFT YIELD BY CONNECTION
      C IS MANS SIGN IF INITIALLY SEPARATION
      EN=PARAFR2(MATYPE)
      CPARFR4(MATYPE)
      COCEN
      MS(NW)=M(5)
      INITIALLY COMPRESSION CASE SET BOUNDS
      IF(LS(NW)) .GT. (.0) GO TO 417
      C
      CBOUND(NNW)=0.
      CENTER(NNW)=MS(NNW)
      TBOUND(NNW)=-2.0*MS(NNW)
      WS(NW)=0.
      CBOUND(NNW)=CBUND(NNW)+CC
      CENTER(NNW)=CENTER(NNW)+CC
      TBOUND(NNW)=TBOUND(NNW)+CC
      WS(NNW)=WS(NNW)+CC
      GO TO 418
      C INITIALLY TENSION CASE SHIFT ZERO LINE
      IF(LS(NW)) .LT. (.0) GO TO 418
      CBOUND(NNW)=WS(NNW)
      CENTER(NNW)=3.0*WS(NNW)
      TBOUND(NNW)=3.0*WS(NNW)
      CENTER(NNW)=CENTER(NNW)+CC
      CBOUND(NNW)=CBUND(NNW)+CC
      TBOUND(NNW)=TBOUND(NNW)+CC
      WS(NNW)=WS(NNW)+CC
      GO TO 418
      C
      PRINT 204,CBUND(NNW),TBUND(NNW)
      204 FORMAT(1* BOUN*,ZE12.4)
      416 CONTINUE
      IF(JUMP .EQ. 0) GC TO 409
      DO 318 I=1,3
         F11=SIG(I,N)
      318 CONTINUE
      C FORM STRESS (LOCAL)-DISPLACEMENT (GLOBAL) MATRIX SA
      C
      C FORM PREVIOUS YIELD CONDITION
      IF(INID(NNW)) *ED, 0) GO TO 425
      IF(INID(NNW)) *ED, 1) GO TO 421
      IF(INID(NNW)) *ED, 2) GO TO 422
      IF(INID(NNW)) *ED, 3) GO TO 423
      IF(INID(NNW)) *ED, 4) GO TO 424
      421 CONTINUE
      C YIELD IN U1 DIRECTION
      DO 321 I=1,12
         SAI,I,J)=SA1(I,J)
      321 CONTINUE
      DC 351 I=1,12
      DC 351 I=1,12
      DC 351 I=1,12
      351 CONTINUE
      NORLN=133
      NORLN=134
      NORLN=135
      NORLN=136
      NORLN=137
      NORLN=138
      NORLN=139
      NORLN=140
      NORLN=141
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      NORLN=213
      NORLN=214
      NORLN=215
      NORLN=216
      NORLN=217
      NORLN=218
      NORLN=219
      NORLN=220
      C
      353 CONTINUE
      C CALCULATE SUMMATION OF DK(I)*DU(I) AS PART OF INTERNAL ELASTIC

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C FORCE
 426 CONTINUE
    DO 361 I=1,12
      XA=0.
      IF(IA .EQ. 0) GO TO 361
      DC 362 J=1,12
        XA=XADASAP(I,J)*DTU(J)
      362 CONTINUE
        FKA(IA)=FKDA(IA)*XA
      361 CONTINUE
C KEEP PREVIOUS CASE CONDITION
  NNDP=NIND(NNEL)
  NNDPA(NNEL)=NNDP
  IF(NNDP .NE. 0) NNDT=NNDP+1
  IF(NNDP .EQ. 0) GO TO 441
  IBGP=I+(NNEL-1)*157
  DSTIFP(IBGP)=FLOAT(NNEL)
  DO 354 I=1,12
    IL=LM(I)
    DSTIFP(IBGP+I)=FLOAT(IL)
  354 CONTINUE
C
  00 355 I=1,12
    IBP=IBGP+12*I
    DO 355 J=1,12
      IP=IBP+J
      DSTIFP(IP)=DASAP(I,J)
    355 CONTINUE
    441 CONTINUE
    410 CONTINUE
      CALL PFRCT(PARAF(1,1,1,MATYPE),SC,DTU,F,NND(NNEL),AS(NNEL),
      1 MT(NNEL),CBOND(NNEL),TBCC(NNEL),ENTER(NNEL),W(5))
C CHECK IF IT IS YIELD
C FORM PROPER DIFFERENTIAL STIFFNESS DASA
  IF(NIND(NNEL) .EQ. 0) GO TO 411
  IF(NIND(NNEL) .EQ. 1) GO TO 431
  IF(NIND(NNEL) .EQ. 2) GO TO 432
  IF(NIND(NNEL) .EQ. 3) GO TO 435
  IF(NIND(NNEL) .EQ. 4) GO TO 434
  431 CONTINUE
    YIELD IN U-1 DIRECTION
    DO 326 I=1,12
      DO 326 J=1,12
        ASA(I,J)=ASA1(I,J)
      326 CONTINUE
      GO TO 435
    432 CONTINUE
    C YIELD IN V-2 DIRECTION
    DO 324 I=1,12
      DO 324 J=1,12
        ASA(I,J)=ASA2(I,J)
      324 CONTINUE
      GO TO 435
    434 CONTINUE
    C YIELD U-V
    DO 328 I=1,12
      DO 328 J=1,12
        ASA(I,J)=ASA4(I,J)
      328 CONTINUE
      IF(NNDP .NE. 3) GO TO 440
      436 CONTINUE
      NONLIN=283
      NNDT=NIND+1
      C FORM DIFFERENTIAL STIFFNESS DASA
      C FORM DIFFERENTIAL LOAD VECTOR [K(I)*DTU(I-1)] CN RIGHT HAND
      C SIDE AS LOAD VECT(C,AS STARTING FOR ITERATION SCULTION
      DO 306 I=1,12
        XA=0.0
        NONLIN=226
        NONLIN=227
        NONLIN=228
        NONLIN=229
        NONLIN=230
        NONLIN=231
        NONLIN=232
        NONLIN=233
        NONLIN=234
        NONLIN=235
        NONLIN=236
        NONLIN=237
        NONLIN=238
        NONLIN=239
        NONLIN=240
        IA=LM(I)
        IF(IA .EQ. 0) GO TO 306
        DO 307 I=1,12
          XA=XADAS(I,J)*DTU(J)
        307 CONTINUE
        DKU(IA)=DKU(IA)+XA
      306 CONTINUE
C STORE FRICTION ELEMENT INFORMATION ON TAPE12,FOR ITERATION
  C
  NNDT=12
  IBEG=I+NNEL-1*157
  DSTIFP(IBEG)=FLOAT(NNEL)
  DO 329 I=1,12
    IL=LM(I)
    DSTIFP(IBG+I)=FLCAT(IL)
  329 CONTINUE
  DC 331 I=1,12
    IB=IBG+12*I
    DO 332 J=1,12
      IP=IB+J
      DSTIFP=ASA(I,J)
    331 CONTINUE
    C CALCULATE INCREMENTAL STRESS AND ADD UP AS TOTAL STRESS
    NNDP(NNEL)=0
    NNDP(NNEL)=NNDF
    NNDP(NNEL)=NNDF
    IF(KLN .EQ. 0) GO TO 440
  441 CONTINUE
C CHECK IF THE YIELD CONDITION CHANGED
  C EITHER TO THE TENSICN OR COMPRESSION TO TENSION
  C
  NNDH=NIND(NNEL)
  IF(NYIELD .EQ. 1) GO TO 438
  IF(NYIELD .EQ. 0) GO TO 436
  IF(NNDP .EQ. 3 ) AND. NNDW .EQ. 3 ) GO TO 436
  IF(NNDP .EQ. 3 ) AND. NNDW .NE. 3 ) GO TO 436
  TB=TBOUND(NNEL)
  TB=TBOUND(NNEL)
  IF(NNDPN .EQ. 3 ) AND. TW .LE. TB) GO TO 436
  IF(NNDPN .EQ. 3 ) AND. TW .GT. TB) GO TO 439
  439 CONTINUE
  C STEP SIZE
  DIVD=0.5*(TBOUND(NNEL)-CBOUND(NNEL))
  STEP=(CDIVD/
  STEP=ABS(STEP)
  IF(ISTEP .LT. 2.0) GO TO 436
  NCHAN=NCHANG+1
  NSTEP=IFX(STEP)
  IF(NSTEP .GE. MASTEP) MASTEP=NSTEP
  438 CONTINUE
  IF(NNDP .NE. 3) GO TO 440
  436 CONTINUE
  NONLIN=284
  NONLIN=285
  NONLIN=286
  NONLIN=287
  NONLIN=288
  NONLIN=289
  NONLIN=290
  NONLIN=291
  NONLIN=292
  NONLIN=293
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  NONLIN=302
  NONLIN=303
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  NONLIN=337
  NONLIN=338
  NONLIN=339
  NONLIN=340
  NONLIN=341
  NONLIN=342
  NONLIN=343
  NONLIN=344

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00 311 I=1,3
  DSIG(I,N)=0.
311  CONTINUE
  GO TO 412
440  CONTINUE
  DO 319 I=1,3
    DSIG(I,N)=0.0
    DO 310 I=1,12
      DSIG(I,N)=SIG(1,N)+SA(I,J)*DTU(J)
310  CONTINUE
    SIG(1,N)=SIG(1,N)+DSIG(I,N)
309  CONTINUE
412  CONTINUE
    SIG(*,N)=FLOAT(MIN(NMEL))
    WALL(1,NMEL)=SIG(3,N)
    WALL(2,NMEL)=SIG(2,N)
    WALL(3,NMEL)=SIG(1,N)
    WALL(4,NMEL)=XXA
    WALL(5,NMEL)=YYA
    WALL(6,NMEL)=ZZA
    C STORE STATIC MAXIMUM SHEAR STRESS
    IF (JUMP .NE. 0) GC TO 415
    IF (SHEAR1 .GT. MAX1MAX1=SHEAR1
    IF (SHEAR2 .GT. MAX2MAX2=SHEAR2
445  CONTINUE
    C CHECK IF IT IS NECESSARY TO RESET
    IF (INT(LD .NE. 0) GO TO 437
    IF (INCHANG .EQ. 0) GO TO 437
    CALL RESET(TRAU,TRVA,DUA,DVA,DAA,DSIG,SIG,NIND,DKU,DSITF,
1  CONTINUE
    END
    C SUBROUTINE RESET(TRAU,TRVA,DUA,DVA,DAA,DSIG,SIG,NIND,DKU,DSITF,
1  DSITFF,TRAA)
    C SET PRESENT VALUE TO ONE STEP BEFORE
    SET TIME STEP SIZE
    C COMMON/STROUT/NSOLID,NCONG,NFR,NSFRIN
    C COMMON/STRUT/NSOLID,NCONG,NFR,NSFRIN
    C COMMON/FRATON/NCR,NP,ROC,ITERATE,FRR, SUSTEP
    C COMMON/FRATON/NCR,NP,ROC,ITERATE,FRR, SUSTEP
    C COMMON/NOL/NINOT,NINDO,JKD(1200),KPRINT
    C COMMON/TIME/JUMP,T,DT,MPTM,MTAPE,KPRINT
    C COMMON/NDIV/MASITF,NVTEL,INSTP,NCHANG,NINPA(50)
    C COMMON/ELPAR/NUMNP,NUMEL,NETTYPE,NEQA,NEQO,MBANDO,KLIN,NLASTRESET
    C COMMON/NEVAL/NK,NTEL
    C COMMON/EG/DURANA,DTT
    C DIMENSION/BOUND/CSOUND(50),TSOUND(50),CENTER(50),WS(50),WT(50),DW(50)/RESET
    C T=T-DT
    C JUMP-JUMP-1
    C
00 301 I=1,NEQA
  TRUA(I)=TRUA(I)-DUA(I)
  TRVA(I)=TRVA(I)-DVA(I)
  TRAA(I)=TRAA(I)-DAA(I)
301  CONTINUE

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C      FORM RAYLEIGH DAMPING ALFA,BETA
IF (IDAMP .NE. 1) GO TO 401
READ 5,1,N2,X(1),X(2)
OMEGA(1)=OMEGA(1)
OMEGA(2)=OMEGA(2)
PRINT 1,05,X(1)*X(2)*OMEGA(1)*OMEGA(2),N1,N2
T1=X(2)*OMEGA(1)-X(1)*OMEGA(2)
T2=+.05*OMEGA(1)*OMEGA(2)
TC=OMEGA(1)*2-CM6A(2)**2
ALFA=T1*TC
T3=2.0*(X(1)*OMEGA(1)-X(2)*OMEGA(2))
BETA=T3*TC
GO TO 441
401 CONTINUE
C      DIRECT DAMPING BY PEZIEN,WILSON
IF (IDAMP .NE. 2) GO TO 403
CALL OVERLAY(GTHREED,9,0,0)
GO TO 441
403 CONTINUE
C      STRUCTURAL DAMPING
IF (IDAMP .NE. 3) GO TO 441
CALL OVERLAY(GTHREED,10,0,0)
441 CCNTINUE
C      INPUT ITERATION PARAMETER
INPUT 6,NORM,NPCC,TOTAL,ERROR,SUSTEP
WRITE(1,106) NORM,NPCC,ITERAL,ERROR,SUSTEP
C      FORM CONSTANTS FOR INTEGRATION METHOD
CALL CONSTINTEGR,D,THETA)
C      READ IN EARTHQUAKE ACCELERATION & STEPS EACH TIME
G=386.07
DO 301 I=1,9
U2G(I)=.
301 CONTINUE
READ 9,U2G(1),I=1,9
WRITE(1,109) U2G(I),I=1,9
ACCX=U2G(1)*EQHUL*G
DUR=7.0*DTEQ
DUR=8.0*DTEQ
C      INITIALIZATION WITH ACCX=-GROUND ACC AS INITIAL CONDITION
MX=IDA(1,1)
DO 302 I=1,NEQA
U(I)=TRVA(I)
TRUA(I)=0.
TRVA(I)=0.
TRAC(I)=0.
302 CONTINUE
DO 303 I=1,NUMNP
MX=IDA(1,I)
IF (MX .EQ. 0) GO TO 303
TRAF(MX)=-ACCX
303 CONTINUE
UGXT=0.
VGXT=0.
OACCX=ACCX
C      UGT=0.
UGZT=0.
VGYT=0.
VGZT=0.
307 CONTINUE
DACCY=0.0
DACCZ=0.
C      STORE TMASSA
REWIND 5
DO 307 I=1,NEQA
DO 307 J=1,MBANDA
TMASSA(I,J)=0.
EKAA(I,J)=0.
307 CONTINUE
READ (5,1) TMASSA(I,J),I=1,NEQA,J=1,NEQA
STORE TMASSA
READ (5,1) DMASSX(I),I=1,NEQA
STORE DMASSX
STORE EKAA
REWIND 2
READ (2,1) EKAA(I,J),I=1,NEQA,J=1,MBANDA
C      SAVE STIFFNESS MATRIX
DC 335 I=1,NEQA
DC 335 J=1,MBANDA
EKABAR1,J=EKAA(I,J)
335 CONTINUE
C      FORM EFFECTIVE MATRIX KBAR=EKAA(I,J)
C      IN CONSTANT ACCELERATION,RAYLEIGH METHOD KBAR=K+C1*I**2
C      FORM DAMPING MATRIX IF IT IS RAYLEIGH DAMPING
IF (IDAMP .NE. 1) GO TO 406
DO 333 J=1,MBANDA
CAX(J)=ALFA*TMASSA(I,J)+BETA*EKAA(I,J)
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STEP.161
C      WILSON THETA METHOD
IF (INTGR .NE. 1) GO TO 408
IF (KLLIN .NE. 0) DO 339 J=1,MBANDA
DO 339 I=1,NEQA
TAC=0.01*SUSTEP*PAOTMASSA(I,J)
TAC=D3*SUSTEP*PCA(I,J)
EKAA(I,J)=EKAA(I,J)+TAC+TAC
312 CONTINUE
GO TO 408
407 CONTINUE
C      WILSON THETA METHOD
IF (INTGR .NE. 1) GO TO 408
IF (KLLIN .NE. 0) DO 339 J=1,MBANDA
DO 339 I=1,NEQA
TAC=A1*SUSTEP*PCA(I,J)
EKAA(I,J)=EKAA(I,J)+TAC+TAC
339 CONTINUE
GO 311 I=1,NEQA
DC 311 J=1,MBANDA

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TAM=A*TMASSA(I,J)
TAC=A*TCAT(I,J)
EKAA(I,J)=EKAA(I,J)+TAM*TAC
311 CONTINUE
C 408 CONTINUE
C  TRIANGULATION EKA
CALL TRIAEKAA,NEQA,MBANDA)
IF (KLIN .EQ. 0 .OR. SUSTEP .EQ. 1.0) GO TO 473
CALL TRIAEKAA,NEQA,MBANDA)
473 CONTINUE
C INITIALIZATION
DO 306 I=1,NEQA
FIAT(I)=0.
306 CCNTINUE
C LOADING TERMS WITH MASS OR DAMPING
IF (INTGR .NE. 0) GO TO 433
C CONSTANT ACCELERATION
DO 330 I=1,NEQA
FMA(I)=0.
330 CCNTINUE
CALL MULTA(THMSSA),TRAA,FMA,NEQA,MBANDA)
GO TO 434
433 CCNTINUE
C WILSON METHOD
IF (INTGR .NE. 1) GO TO 434
CALL MULTA(THMSSA),TRAA,FMA,NEQA,MBANDA)
CALL MULTA(CC,TRAA,FMA,NEQA,MBANDA)
DO 331 I=1,NEQA
FMA(I)=2.*FMA(I)
FBM(I)=3*FBM(I)
331 CCNTINUE
DO 332 I=1,NEQA
FKDA(I)=0.
332 CCNTINUE
IF (KLIN .NE. 0) GO TO 428
DO 316 I=1,NEQA
OK(I)=0.
316 CONTINUE
428 CCNTINUE
CALL SETFL(550008)
***** START OF STEP-BY-STEP CYCLE *****
TIMEB(I)=0.
NSTEP=0
MASTER=0
NCHANG=0
NSTEP=0
IF (SUSTEP .EQ. 1.0) DDT=0.0
1.000 CONTINUE
IF (KLIN .EQ. 0 .OR. SUSTEP .EQ. 1.0) GO TO 474
C IF SUSTEP .EQ. 1, THERE IS NO VARIABLE STEP

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STEP•162 C IF(MASTER .EQ. 0) GO TO 454  
 STEP•163 C CALCULATE INTEGRATION CONSTANTS  
 STEP•164 DCT=D0/SUSTEP  
 STEP•165 CALL CONST(INTEG,GR,DT,THETA)  
 NYELO=NYELO+1  
 NSTEP=NFI/(SUSTEP)  
 IF(NYELD .GT. NSTEP) GO TO 454  
 456 CONTINUE  
 DT=DT/N  
 T=T+DT  
 GO TO 457  
 454 CONTINUE  
 C RECALCULATE CONSTANT IF RETURN TO NORMAL STEPS  
 DT=DT/N  
 CALL CONST(INTEG,GR,DT,THETA)  
 458 CONTINUE  
 NYELO=NYELO-1  
 MASTER=0  
 NCHANG=0  
 NSTEP=0  
 474 CONTINUE  
 T=T+DT  
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308 DUAL(I)=0. IN DYNAMIC
C 1ST STEP IN DYNAMIC .EQ. 0) GO TO 442
IF IJUMP .NE. 1 .OF. KLIN .EQ. 0)
DO 337 I=1,NEQA
  UOLD(I)=0.0
  337 CONTINUE
  C INCREMENTAL LOADING VECTOR
  DO 309 I=1,NEQA
    UAT(I)=FRA(I)+FIA(I)+FPA(I)+DKU(I)
  309 CONTINUE
  C INTEGRATION OF EQ ACCELERATION RECORD INTO VELO. AND DISP.
  C OF GROUND MOTION
  UGT=UGX+DT*UGXT+COM1*DACCX+COM2*ACCCX
  VGT=VGT+DT*(ACCCX*ACCA)
  DACCX=ACCX
  C SAVE INCREMENTAL LOAD VECTOR
  IF KLIN .NE. 0) GO TO 452
  IF INPROC .NE. 1) GO TO 451
  DO 341 I=1,NEQA
    PLORD(I)=DUA(I)
  341 CONTINUE
  GO TO 452
  451 CONTINUE
  DO 342 I=1,NEQA
    PLORD(I)=0.
  342 CONTINUE
  452 CONTINUE
  IF INYIELD .NE. 0) GO TO 461
  CALL BACK(KEKA,DUA,NEQA,MBANDA)
  IF KLIN .EQ. 0) GO TO 453
  NO ITERATION, IF NC YIELD
  IF (INDOT .EQ. 0) GO TO 412
  GO TO 463
  461 CONTINUE
  C SAVE NEW INCREMENTAL DISPLACEMENT VECTOR AFTER 1ST SOLUTION
  CALL BACK(KEKA1,DUA,NEQA,MBANDA)
  C NO ITERATION, IF NC YIELD
  IF INYIELD .EQ. 1 AND. NINDOT .EQ. 0) GO TO 412
  IF (INDOT .EQ. 0) GO TO 412
  463 CONTINUE
  DO 343 I=1,NEQA
    UNEW(I)=DUA(I)
  343 CONTINUE
  C BEGIN ITERATION .EQ. 1.0) CALL ITERATE(KEKA1,UOLD,UN1,PLCAD,NIND,DSIF)
  IF ISUSTEP .EQ. 1.0) CALL ITERATE(KEKA1,HOLC,UN1,PLCD,NIND,DSIF)
  C SAME CLO INCREMENTAL VECTOR FOR NEXT STEP
  DO 344 I=1,NEQA
    UOLD(I)=UNW(I)
  344 CONTINUE
  GO TO 453
  C CALCULATE TOTAL DISP, VELO, ACC AT THE END OF STEP
  412 CONTINUE
  DO 345 I=1,NEQA
    UOLD(I)=DUA(I)
  345 CONTINUE
  453 CONTINUE
  C CONSTANT ACCELERATION METHOD
  IF (INTEGR .NE. 0) GO TO 413
  DO 313 I=1,NEQA
    DISP=DUAL(I)
    VEL=DISP*DT3-BT1*TRA(I)
    DVA(I)=VEL
    ACCE=DISP*CON-A1*TVA(I)-BT1*TAA(I)
    DAI(I)=ACCE
    313 CONTINUE
    C UPDATE RELATIVE QUANTITIES
    DO 293 I=1,NEQA
      TRA(I)=TRA(I)+DVA(I)
      TVA(I)=TVA(I)+A1*TVA(I)+DVA(I)
      TAA(I)=TAA(I)+A1*TAA(I)+DVA(I)
    293 CONTINUE
    STEP•298
    STEP•299
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    STEP•408
    STEP•409
    IF (NXX .EQ. 0) GO TO 419
    U(NY)=TVA(NY)+US(NY)
    V(NY)=TVA(NY)
    ACCA(NY)=TAA(NY)
    416 CONTINUE
    N=IDA(2,1)
    IF (N2 .EQ. 0) GO TO 419
    U(N2)=TVA(N2)+US(N2)
    V(N2)=TVA(N2)
    ACCA(N2)=TAA(N2)
    419 CONTINUE
    NXX=IDA(1,1)
    IF (NXX .EQ. 0) GO TO 420
    IF (NXX .EQ. 0) GO TO 420
  
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STEP•410          FKDA(I)=FKDA(I)-OKU(I)
STEP•411          FTA(I)=FKDA(I)-FTA(I)
STEP•412          OTRUA(I)=TRUA(I)
STEP•413          OTRVA(I)=TRVA(I)
STEP•414          OTRAA(I)=TRA(A(I))
350 CONTINUE
466 IF(MASTER •NE• 0 •AND• NYIELD •EQ• 5) CALL CONST(JINTEGR, DTO, THETA(I))
STEP•415          STEP•416          STEP•417          STEP•418          STEP•419          STEP•420          STEP•421          C
STEP•415          CALCULATE PART OF LOADING TERM FOR NEXT STEP
STEP•415          C
STEP•415          CONSTANT ACCELERATION
STEP•415          IF(LINTEGR •NE• 0) GO TO 423
STEP•422          DO 319 I=1,NEQA
STEP•423          USE FMB TEMPORARY
STEP•424          FB(I)=A(I)*TRVA(I)+TRA(A(I))
STEP•425          CONTINUE
319 CALL MULBA(1,MASSA,FMB,FMANEQA,MBANDA)
CALL MULBA(ICA,TRAFA,TRVA,FMB,FNEQA,MBANDA)
GO TO 426
423 CONTINUE
STEP•426          C
STEP•426          WILSON THETA METHOD
STEP•427          TF(LINTEGR •NE• 1) GO TO 426
STEP•428          DO 320 I=1,NEQA
STEP•429          USE FMB,GRA TEMPORARILY
FRA(I)=0*TRVA(I)+A3*TRA(A(I))
FB(I)=A2*TRA(A(I)+2.0*TRA(A(I))
STEP•430          CONTINUE
320 CALL MULBA(MASSA,FMB,FMANEQA,MBANDA)
CALL MULBA(ICA,FGR,A,NEQA,MANLA)
426 CONTINUE
GO TO 1000
424 CONTINUE
RETURN
C
1 FORMAT(11246)
2 FORMAT(11246)
3 FORMAT(15F10.5,315)
4 FORMAT(15F10.5)
5 FORMAT(12.5,210.0)
6 FORMAT(13.5,210.0)
106 FORMAT(* ITERATION CONTROL DATA*/
1          * TYPE OF NORM•1 EUCLIDEAN NORM•2 MAXIMUM NORM•3
STEP•431          STEP•432          STEP•433          STEP•434          STEP•435          STEP•436          STEP•437          STEP•438          STEP•439          STEP•440          STEP•441          STEP•442          STEP•443          STEP•444          STEP•445          STEP•446          STEP•447          STEP•448          STEP•449          STEP•450          STEP•451          STEP•452          STEP•453          STEP•454          STEP•455          STEP•456          STEP•457          STEP•458          STEP•459          STEP•460          STEP•461          STEP•462          STEP•463          STEP•464          STEP•465          STEP•466          STEP•467          STEP•468          STEP•469          STEP•470          STEP•471
107 FORMAT(10F7.3)
101 FORMAT(1H1,12A6)
102 FORMAT(* EARTHQUAKE MULTIPLICATOR FACTOR EQUAL * ,F10.5/
1          * TIME INTERVAL ON DATA CARDS OTEQ * ,F10.5/
2          * TIME INCREMENT OF ANALYSIS DUREQ * ,F10.5/
3          * DURATION OF EQ INPUT DURANA * ,F10.5/
4          * DURATION OF ANALYSIS MPTN * ,F10.5/
5          * TIME INCREMENT OF OUTPUT/DT MTAP * ,F10.5/
6          * STEPS CAN WAITING TPE KPRINT * ,F10.5/
7          * STEP OF 1ST PRINTOUT
103 FORMAT(* CHOICE OF DAMPING/ IDAMP=1 RALEIGH, =2 PENZIEN, =3 STRUCTURAL*, 15//)
1          CALL CONSTINTEGR•DOT,THEJA)
1          CALL MULBA(EKAART,TRUA,ACCA,ISOL,ICCN,IFR,ISP,SIG,MFX,MFY,
1          MFZ,YABV,ZEDG,MINC,NTWEL)
104 FORMAT(* CHOICE OF DIRECT INTEGRATION MET100/*
1          * INTEGR=0 CONSTANT ACCELERATION, =1 WILSON THETA METHCC*, STEP•533
1          GC TO 466
465 CONTINUE
UGXT=PUGX1
VGX=PVGX1
OACC=PACCX
ODT=DTO,SUSTEP
CALL CONSTINTEGR•DOT,THEJA)
CALL MULBA(EKAART,TRUA,ACCA,ISOL,ICCN,IFR,ISP,SIG,MFX,MFY,
DC 350 I=1,NEQA

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405 CONTINUE
  IF(UNIT * LE, ITERAL) GO TO 1000
  WRITE(1,500) UNIT
  STOP
  RETURN
END
OVERLAY(1,0)
PROGRAM INPUTJ
C
***** READ AND GENERATE NODAL POINT DATA
C   CALCULATE EQUATION NO. AND BANCHEETH OF DEGREE OF FREEDOM TC BE
C   REMAED NEQA , MBANDA
C   AND THOSE TO BE ELIMINATED NEQD , MBANDO
C
C COMMON A(30000)
COMMON /ELPAR/NF,NMF,NUMEL,NETYPE,NEQA,NEQC,NBANDA,NBANDO,KLIN,NLASTINPNU
DIMENSION IA(1),ICODE(16)
EQUIVALENCE (IA,IA)
N1=1
NOO=6*NUMNP
N2=N1+12*NUMNP
N2E=N2-1
N3E=N3-1
N4E=N4-1
WRITE(1,501)
WRITE(1,501)
NOLD=0
1000 CCNTINUE
C
C NODE CODE 2 DEGREE OF FREEDOM TO BE FIXED
C   1
C   2
C   READ 1*N, (ICODE(I),I=1,6),X,Y,Z,KN
C   16=8*N-5
C   16=6*N
C   PUT INTO COMMON BLOCK A
C   IA(1)=ICODE(1)
C   IA((1+1)=ICODE(2)
C   IA((1+2)=ICODE(3)
C   IA((1+3)=ICODE(4)
C   IA((1+4)=ICODE(5)
C   IA((16)=ICODE(6)
C   A(NZE+N)=X
C   A(NZE+N)=Y
C   A(N4+N)=Z
C   WRITE(1,101) N,(A(I),I=1,16),(NZE+N),(NZE+N),A(NZE+N),KN
C   IF(NOLD .EQ. 0) GO TO 401
C   CHECK IF GENERATION REQUIRED
C   IF(KN .EQ. 0) GO TO 401
C   NUM=NO. OF INTERVAL BETWEEN INPUT NODE
C   NUM=NO. OF NODES TO BE GENERATED
C   DX,DY,DZ=THE LENGTH OF INTERVAL
C   NUM=N-NUL-1
C   IF(NUMN .LT. 1) GO TO 401
C
C   XNUM=NUM
D1=(A(N2E+N))-A(N2E+NOLD1)*XNUM
D2=(A(N3E+N))-A(N3E+NOLD1)*XNUM
  C
  GENERATE NEW NODAL POINT
  K=NOLD+2
  DO 301 J=1,NUMN
    KK=K
    INPUTJ.3
    INPUTJ.4
    INPUTJ.5
    INPUTJ.6
    INPUTJ.7
    INPUTJ.8
    INPUTJ.9
    INPUTJ.10
    INPUTJ.11
    INPUTJ.12
    INPUTJ.13
    INPUTJ.14
    INPUTJ.15
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    INPUTJ.17
    INPUTJ.18
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    INPUTJ.112
    INPUTJ.113
    INPUTJ.114
    INPUTJ.115
    INPUTJ.116
    INPUTJ.117
    INPUTJ.118
    INPUTJ.119
    C
    NODE CODE AS PREVIOUS ONE
    TA((6*(I-6)+1)=LAT6*KK-6+1)
    302 CONTINUE
    301 CONTINUE
    HOLD=N
    IF(N .NE. NUMNP) GO TO 1000
    C
    PRINT ALL NODAL POINT DATA
    WRITE(1,503)
    WRITE(1,EQ2)
    DO 304 NUMNP
    I1=6*I-5
    I6=6*N
    WRITE(1,102) N,(A(I),I=1,I6),A(NZE+N),A(NZE+N)
    304 CONTINUE
    C
    CALCULATE THE EQUATION NUMBERS FOR EVERY DEGREE OF FREEDOM
    NEQA=0
    NEQQ=0
    DO 303 N=1,NUMNP
    DO 303 I=1,6
    NH=6*I-6+1
    IF(TIA(NN)-1) 402,403,404
    403 CONTINUE
    NEQA=NEQA+1
    TAINN=NEQA
    A(NNN+100)=0
    GO TO 303
    404
    NEQQ=NEQQ+1
    TAINO=NEQQ
    A(NNO+NN)=NEQQ
    A(NNO)=0
    INPUTJ.101
    INPUTJ.102
    INPUTJ.103
    INPUTJ.104
    INPUTJ.105
    INPUTJ.106
    INPUTJ.107
    INPUTJ.108
    INPUTJ.109
    INPUTJ.110
    INPUTJ.111
    INPUTJ.112
    INPUTJ.113
    INPUTJ.114
    INPUTJ.115
    INPUTJ.116
    INPUTJ.117
    INPUTJ.118
    INPUTJ.119
    C
    WRITE EQUATION NUMBERS
    WRITE(1,504)
    DC 305 N=1,NUMNP
    I1=6*I-5
    I6=6*N
    WRITE(1,103) N,(A(I),I=1,I6)
    303 CONTINUE
    C

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3 * UNIT WEIGHT OF SOIL =WT LB/CU.FT// C NO STRESS OUTPUT REQUIRED
4 3H N,8X12HGS,* CCNCRETE PROPERTY DATA /* C 1 STIFFNESS AS PRECEDING ELEMENT
502 FORMAT///* CONCRETE PROPERTY DATA /* C NEED STRESS CUTOUT
1 * ELASTIC MODULUS=E KSI /* C 2 REQUIRE CALCULATION OF STIFFNESS
2 * POSSION RATIO=NU /* C NC STRESS CUTOUT
3 * UNIT WEIGHT =WT /* C 3 REQUIRE CALCULATION OF STIFFNESS
4 3H N,9X1HE,8X,2HNU,8X,2HNT///* C STRESS CUTOUT
503 FORMAT///* GEOMETRIC PROPERTY CF BEAM,COLUMN /*/
1 * GEOMETRIC PROPERTY NUMBER =N /* C TOP ELEMENT OF JOINT
2 * AXIAL AREA SO INCH =A /* C 4 STIFFNESS AS PRECEDING ELEMENT
3 * SHEAR AREA IN LOCAL 2-DIRECTION=A12/* C 5 CALCULATION OF STIFFNESS
4 * TORSIONAL INERTIA INCH =J /* C READ IN ELEMENT DATA
5 * FLEXURAL INERTIA ABOUT 2-AXIS =M2 /* C 1000 READ 1,M, IDE(I),I=1,14)
6 * /* C 2000 READ 1,M, IDE(I),I=1,14)
7 * /* C 3000 READ 1,M, IDE(I),I=1,14)
8 * 3H N,9X1HA,7X,3HAV3,9X,1HU,J,8X,2HM3,8X,2HM3//) C IF THE TOP SOIL ELEMENT OF FRICTION JOINT, ADDITIONAL 4 NODES OF
504 FORMAT///* FRICTION ELEMENT DATA /* C BOTTOM FRICTION ELEMENT ARE NEEDED
1 * SHEAR STIFF =KS KSI/* C IF IDE(11) .LE. 3) GO TO 407
2 * NORMAL STIFF =KN KSI/* C READ 1,FB,JF,B,KFB,LFB
3 * FRICTION COEFFICIENT=LU /* C 407 CONTINUE
4 * CONESSION =C KSI/* C 2000 N=N+1
5 * 3H N,9X12HKS,8X,2HKN,9X,1HU,9X,1HC//) C IF(M .EQ. N) GO TC 401
505 FORMAT///* BOUNDARY ELEMENT DATA /*/ C ELEMENT DATA GENERATION NEEDC
1 * LINEAR SPRING IN X DIRECTION =EX KIP-IN /*/ C 304 DO 304 I=1,8
2 * LINEAR SPRING IN Y DIRECTION =EY KIP-IN /*/ C N10=LAST+11*N-11
3 * LINEAR SPRING IN Z DIRECTION =EZ KIP-IN /*/ C NN10=N10+
4 * ROTATIONAL SPRING IN X DIRECTION =EX KIP-IN/RAD/*/ C IA(NN10)=IA(NN10-11)+1
5 * ROTATIONAL SPRING IN Y DIRECTION =EY KIP-IN/RAD/*/ C MATRO.120
6 * ROTATIONAL SPRING IN Z DIRECTION =EZZ KIP-IN/RAD/*/ C 304 CONTINUE
7 3H N,8X12HEY,8X,2HEY,8X,2HEZ,7X,3HEXX,*),3HEYY,7X, C IF IDE(12) .EQ. 0 AND. I .GE. 3) A(NN10)=0
8 3HEZZ//) C 305 I=9,11
END DO 305 I=9,11
OVERLAY3,01 N10=LAST+11*N-11
PROGRAM EDATA NN10=N10+I
***** COMMON A(30000) C STORE INPUT DATA INTO COMMON BLANK A
COMMON/ELPAR/NUMNF,NUMEL,NETYPE,NEQA,NBQG,MBANDA,PBANDO,KLIN,NLA,STLDATA,11 C 401 CONTINUE
COMMON/MATER/NUMATS,NUMATC,NUMATF,NUMATB,NUMGE,NUMBC,TYPE EDATA,12
COMMON/TOPL/MLA(36),LM0(136),XL(81),YL(81),ZL(61) EDATA,13
COMMON/TOEL/NFB(4,50),ITOPN EDATA,14
DIMENSION IDE(14) EDATA,15
DIMENSION IA(11) EDATA,16
EQUivalence (IA,IA) EDATA,17
WRITE(1501) EDATA,18
N=0 EDATA,19
NBC=0 EDATA,20
MBANDA=0 EDATA,21
NBANDO=0 EDATA,22
N11=LAST+11*NUMEL EDATA,23
ITOP=0 EDATA,24
0 301 I=1,14 EDATA,25
TOP(I1)=1 EDATA,26
301 CONTINUE EDATA,27
IDE(11) IS THE STIFFNESS AND STRESS CODE OF ELEMENT EDATA,28
C NOT TOP ELEMENT OF JOINT EDATA,29
C 0 STIFFNESS AS PRECEDING ELEMENT EDATA,30
ELDATA,31
ELDATA,32
ELDATA,33
ELDATA,34
ELDATA,35
ELDATA,36
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ELDATA,92
ELDATA,93

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408 CONTINUE
  IF(IODE(12) .EQ. 0) GO TO 406
C   CALCULATE BANDWIDTH
  DO 308 I=1,3
    N11=NBC+I
    NN11=N11*I+NBC-3
    TA(NN11)=ODE(11,I)
  308 CONTINUE
  406 CONTINUE
C   CALCULATE BANDWIDTH
  DO 309 L=1,36
    IF(ILM(L)) .EQ. 0 GO TO 309
    IF(ILM(L)) .GT. MAXA MAXA=MA(L)
    IF(ILM(L)) .LT. MINA MINA=MA(L)
  309 CONTINUE
  DO 310 L=1,36
    IF(ILM(L)) .EQ. 0 GO TO 310
    IF(ILM(L)) .GT. MAXD MAXD=LMD(L)
    IF(ILM(L)) .LT. MIND MIND=LMD(L)
  310 CONTINUE
  IF(MAXA .EQ. 0) MINA=1
  MB=MAYA-TIN+1
  IF(MBA .GT. MBANDA) MBANDA=MBA
  IF(MAO .EQ. 0) MIND=1
  MB=MAXD-TIN+1
  IF(HBO .GT. MBANDC) MBANDC=HBO
C   CONCRETE BEAM OR COLUMN ELEMENT
  IF(IODE(12) .EQ. 0) GO TO 402
  GO TO 403
  402 CONTINUE
  WRITE(11,101) N,(A(N10+I),I=1,11),MBA,MBO
  WRITE(11,101) N,(A(N10+I),I=1,11),MBA,MBO
  WRITE(11,101) N,(A(N10+I),I=1,11),MBA,MBO
  WRITE(11,101) N,(A(N10+I),I=1,11),MBA,MBO
  WRITE(11,101) N,(A(N10+I),I=1,11),MBA,MBO
  WRITE(11,101) N,(A(N10+I),I=1,11),MBA,MBO
  GO TO 404
  403 CONTINUE
  WRITE(11,102) N,(A(N10+I),I=1,11),(A(N11+I),I=1,3),MBA,MBO
  404 CONTINUE
  IF(N .EQ. NUMEL) GO TO 405
  IF(N .EQ. 1) GO TO 1000
  GC TO 2000
  405 CONTINUE
  NLAST=N11E+3*NURBC
  RETURN
  1 FORMAT(1I35,2I6)
  101 FORMAT(9X,13*15,2I10,15,2BX,2I5)
  102 FORMAT(9X,3,815,2I10,15,II0,2B,2I5)
  103 FORMAT(12,45)
  501 FORMAT(*,ELEMENT NO. I8 JB KB LB IT JT KT LT*,J BAND*,ELDATA,150)
  1           * EL TYPE MATEL NO INT GEOM NC. ENC REL I J
  2           * WIDTH//)
  END
  OVERLAY(4,0)
  PROGRAM LCAO
C   CALCULATE GRAVITY LOAD ELEMENT BY ELEMENT

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DO 304 NN=1,NUHEL
NN10=N10+11*N-3
C DETERMINE ELEMENT TYPE
C NTYPE=IA(NN10)
NTYPE=IA(NN10+1)
INTER=IA(NN10+2)
GO TO (N01+402,304),NTYPE
C 401 CONTINUE
C TYPE 1, SOLID ELEMENT
IF(INTER.EQ.0)OR.INTER.EQ.1
1.OR.INTER.EQ.4.OR.INTER.EQ.6) GO TO 403
DO 305 J=1,6
NN=10E*(N-1)+J
NODE=IA(NN)
NXA=6*NODE-5
NXG=NXA6*NUMNP
XX(J)=A(N2*NODE)
YY(J)=A(N3*NODE)
ZZ(J)=A(N4*NODE)
305 CONTINUE
CALL FSOLID(XX,YY,ZZ,PARASO(11,TYPE))
403 CONTINUE
C SAME AS PREVIOUS ELEMENT
DO 306 J=1,6
NN=10E*(N-1)+J
NODE=IA(NN)
NXA=6*NODE-5
NXG=NXA6*NUMNP
IA1=IA(NXA)
IA2=IA(NXA+1)
IA3=IA(NXA+2)
I01=IA(NXC)
I02=IA(NXC+1)
I03=IA(NXC+2)
306 CONTINUE
C Z DIRECTION
IF(I1A3.EQ.0) GO TO 404
MASS VECTOR OF D C F REMAINED AFTER GUYAN RECUCITION
C MASS VECTOR OF D C F REMAINED AFTER GUYAN RECUCITION
A(I03+N19E)=AMASS+A(I1A3+N18E)
GO TO 405
404 CONTINUE
C Z DIRECTION
IF(I1A2.EQ.0) GO TO 405
MASS VECTOR OF D C F ELIMINATED
A(I03+N19E)=AMASS+A(I1A2)
405 CONTINUE
C Y DIRECTION
IF(I1A2.EQ.0) GO TO 406
STATIC LOAD OF D C F REMAINED
A(N16E+IA2)=-WT*A(N12E+IA2)
C MASS OF D O F REMAINED
A(N16E+IA2)=AMASS+A(N16E+IA2)
GO TO 407
406 CONTINUE
C Y DIRECTION
IF(I1O2.EQ.0) GO TO 407
STATIC LOAD VECTOR OF D O F ELIMINATED
A(N13E+IO2)=-WT*A(N13E+IO2)
C MASS VECTOR OF D C F ELIMINATED
A(N17E+IO2)=AMASS+A(N17E+IO2)
407 CONTINUE
C X DIRECTION
LOAD=67
LOAD=68
LOAD=69
LOAD=70
LOAD=71
LOAD=72
LOAD=73
LOAD=74
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LOAD=190
C MASS VECTOR OF D C F REMAINED
A(N15E+IO1)=AMASS+A(N15E+IO1),
CONTINUE
IF(I1O1.EQ.0) GO TO 306
IF(I1O1.EQ.0) GO TO 304
TOP SOIL ELEMENT CF FRICITION JCINT, NEED TRANSFORM LOAD VECTOR
ITOP = I1O+1
ITOP = IA(NFB1,ITOP)
DO 311 I=1,4
NODE=IA(NFB1,ITOP)
NXA=6*NODE-5
NXG=NXA6*NUMNP
IA1=IA(NXA)
IA2=IA(NXA+1)
IA3=IA(NXA+2)
I01=IA(NXO)
I02=IA(NXC+1)
I03=IA(NXC+2)
CONTINUE
Z DIRECTION
IF(I1A3.EQ.0) GO TO 424
MASS VECTOR OF D C F REMAINED AFTER GUYAN RECUCITION
A(I1A3+N19E)=AMASS+A(I1A3+N18E)
GO TO 425
424 CONTINUE
IF(I1O3.EQ.0) GO TO 425
MASS OF D C F ELIMINATED
A(I1O3+N19E)=AMASS+A(I1O3+N18E)
425 CONTINUE
Y DIRECTION
IF(I1O2.EQ.0) GO TO 426
STATIC LOAD OF D C F ELIMINATED
A(I1O2+N19E)=WT*A(N12E+IA2)
MASS VECTOR OF D C F ELIMINATED
A(N17E+IO2)=AMASS+A(N17E+IO2)
426 CONTINUE
IF(I1O2.EQ.0) GO TO 427
STATIC LOAD OF D C F ELIMINATED
A(N17E+IO2)=-WT*A(N12E+IA2)
MASS VECTOR OF D C F ELIMINATED
A(N17E+IO2)=AMASS+A(N17E+IO2)
427 CONTINUE
X DIRECTION
IF(I1O1.EQ.0) GO TO 311
IF(I1O1.EQ.0) GO TO 311
CONTINUE
IF(I1O1.EQ.0) GO TO 311
CONTINUE
GO TO 304
C MASS VECTOR OF D C F REMAINED
A(N15E+IO1)=AMASS+A(N15E+IO1),
CONTINUE
311 CONTINUE
GO TO 304
C 402 CONTINUE

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DC 304 I=1,10
  DO 304 J=1,NUMATC
    DO 304 K=1,NUMGE
      D2(I,J,K)=0.0
  304  CONTINUE
  DC 305 I=1,6
  DO 305 J=1,NUMATB
    D4(I,J)=0.0
  305  CONTINUE
  DO 306 I=N23T,N24E
    A(I)=0.0
  306  CONTINUE
C   SCLID ELEMENTS (IL OR CONCRETE
  IF INUMATS .EQ. 0) GO TO 401
  DO 307 I=1,NUMATS
    M1=I5+(I-1)*3
    G=A(M1)
    ANU=ATHM+1
    COM=2.0*(1.0-ANU)/(1.0-2.0*ANU)
    D1(I,I)=COM*ANU*G/(1.0-ANU)
    D1(2,I)=COM*ANU*G/(1.0-ANU)
    D1(3,I)=G
  307  CONTINUE
C   CONCRETE COLUMN OR BEAM ELEMENT
  C   401 CONTINUE
  IF INUMATC .EQ. 0) GO TO 402
  N2E=I2*NUMMP
  N3E=N2E*NUMNP
  N4E=N3E*NUMNP
  DO 308 I=1,NUMGE
    BCL(I)=0.0
  308  CONTINUE
  DO 309 N=1,NUMEL
    N11=N10*(N-1)+11-8
    NTPE=IN(NN10)
    IF (NTPE .NE. 2) GO TO 309
C   CALCULATE BEAM CR COLUMN LENGTH INCH
  NBC=NBBC1
  NN1=NN1+(NBC-1)*3
  NG=(NN1)
  DO 310 J=1,2
    NED=N10E+1*(N-1)+J
    NED-TAINED
    NNX=N2E*NNED
    NN=NNJ*NEO
    NN2=N4E*INED
    XX(I)=ANNX
    YY(I)=ANNY
    ZZ(I)=ANNZ
  310  CONTINUE
    XL=XX(2)-XX(1)
    YL=YY(2)-YY(1)
    ZL=ZZ(2)-ZZ(1)
    TL=QRT(X**2+Y**2+ZL**2)
    BCL(NG)=TL+12.0
  309  CONTINUE
    M2=N6+(I-1)*3
    E=A(M2)

  ANU=A(M2+1)
    G=E/(I2+0.0+2.0*ANU)
    ESEPS.40
    ESEPS.41
    ESEPS.42
    ESEPS.43
    ESEPS.44
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  ESEPS.161
  ESEPS.162

  B3=(M2+1)*5
  PH12=12.0*E*83/(G*A2*(BCL(IJ)*2))
  PH21=1.0*PH12
  PH14=0.0*PH12
  PH13=18.0*E*B2/(G*A3*BCL(IJ)**2)
  T4=SA(M2+1)
  B2=(M2+1)*4
  PH14=1.0*PH13
  PH13=1.0*PH13
  PH131=1.0*PH13
  PH134=1.0*PH13
  EZ=E*83
  EY=E*82
  D2(1,I,J)=E*A1/BCL(IJ)
  D2(1,I,J)=12.0*E/(BCL(IJ)**3*PH12)
  D2(3,I,J)=12.0*E/(BCL(IJ)**3*PH13)
  D2(4,I,J)=6.*I*E/V(BCL(IJ))
  D2(5,I,J)=PH14*E/V(BCL(IJ)*PH13)
  D2(6,I,J)=PH124*E/Z(BCL(IJ)*PH12)
  D2(7,I,J)=6.0*E*Z(BCL(IJ)**2*PH12)
  D2(8,I,J)=6.0*E*Z(BCL(IJ)**2*PH12)
  D2(9,I,J)=(2.0-PH13)*E/(BCL(IJ)*PH13)
  D2(10,I,J)=(2.0-PH12)*E/Z(BCL(IJ)*PH12)
  311  CONTINUE
  C   402 CONTINUE
  IF INUMATF .EQ. 0) GO TO 403
  DO 312 I=1,NUMATF
    M3=I8+(I-1)*4
    D3(1,I)=A(M3)
    D3(2,I)=A(M3)
    D3(3,I)=A(M3+1)
    D3(4,I)=A(M3+2)
  312  CONTINUE
  C   403 CONTINUE
  C   BOUNDARY ELEMENT FORCE/DISPLACEMENT IN 6 DIRECTIONS
  DO 313 I=1,NUMATB
    M4=N9+(I-1)*6
    D4(1,I)=A(M4)
    D4(2,I)=A(M4+1)
    D4(3,I)=A(M4+2)
    D4(4,I)=A(M4+3)
    D4(5,I)=A(M4+4)
    D4(6,I)=A(M4+5)
  313  CONTINUE
  C   404 CONTINUE
  C   FORM STRESS-STRAIN MATRIX--A(1,23T TO N24E), ELEMENT BY ELEMENT
  C   IN COMMON BLOCK
  DO 314 N=1,NUMEL
    NN10=N0+11+N-3
    NTPE=TA(NN10)
    MAT=IA(NN10+1)
    NC=N23T+10*(N-1)-1
  314  CONTINUE
  ESEPS.94
  ESEPS.95
  ESEPS.96
  ESEPS.97
  ESEPS.98
  ESEPS.99
  ESEPS.100

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GO TO 405,406,407,408),NTYPE
C 405 CONTINUE
C SOLID ELEMENT,SCIL OR CONCRETE
A INC+1=D11, MAT1
A INC+2=D12, MAT1
A INC+3=D13, MAT1
GO TO 314
C 406 CONTINUE
C MATERIAL NO. AND GEOMETRIC NO.,-MAT,NG
N0C=N0C+1
N0C=N11+3*NBC-3
N0=IA1(N)
DO 315 I=1,10
A INC+1=D21, MAT, NG
A INC+2=D21, MAT, NG
A INC+3=D31, MAT1
A INC+4=D31, MAT1
A INC+5=D312, MAT1
A INC+6=D313, MAT1
GO TO 314
C 407 CONTINUE
C BOUNDARY ELEMENT
C FRICTIONAL ELEMENT
C 408 CONTINUE
C 316 CONTINUE
A INC+1=D41, MAT1
A INC+2=D411, MAT1
DO 316 I=1,6
EKA(I)=0
C 317 CONTINUE
C ASSEMBLE TOTAL ELASTIC STIFFNESS
CALL ESTIF
RETURN
END
SUBROUTINE ESTIF
***** FORM ELASTIC STIFFNESS MATRIX EKA(NEQA,NEACA),EKOD(NEQO,MBANDC),
EKA(NEQA,NEQO)
C AND STORE IN TAPE
EKA ? TAPE 2
EKA ? TAPE 3
C LARGE B(22000)
COMMON A(10000)
COMMON A(UNCO(U)(4),V(4))
COMMON ALPRNUMP,NUMEL,NETYPE,NEQA,NEQO,MBANDA,MBANDC,KLIN,LAStESTIF,16
COMMON MATER,NUMAT,NUMATB,NUMATC,NFR,NUMPR,TYPE
COMMON /NOL/NMA(36),M0136,XX(8),YY(8),ZZ(8)
COMMON /NOL/NINDT,NINDT,DRUO(120)
COMMON /NOL/NINDT,NINDT,DRUO(120)
COMMON /NOL/NFBL4,NFBL5,ITOP
COMMON /STRUT/ASLID,NCNC,NSPRIN
EQUIVALENCE(A,IA)
DIMENSION IA(11),ASA(36,36),TASA(36,36),SA(3,12)
C INITIALIZATION
C
C 163 ESEPS,164 ESEPS,165 ESEPS,166 ESEPS,167 ESEPS,168 ESEPS,169 ESEPS,170 ESEPS,171 ESEPS,172 ESEPS,173 ESEPS,174 ESEPS,175 ESEPS,176 ESEPS,177 ESEPS,178 ESEPS,179 ESEPS,180 ESEPS,181 ESEPS,182 ESEPS,183 ESEPS,184 ESEPS,185 ESEPS,186 ESEPS,187 ESEPS,188 ESEPS,189 ESEPS,190 ESEPS,191 ESEPS,192 ESEPS,193 ESEPS,194 ESEPS,195 ESEPS,196 ESEPS,197 ESEPS,198 ESEPS,199 ESEPS,200
C
C 164 NSOLID=0
N0C=0
NFR=0
NSPRIN=0
N6=1+**NUMNP*3*NUMATS
N7=16+**NUMATC
N10=N7.6*NUMGE*4*NUMATF+6*NUMATE
N11=N10*11*NUMEL
N20=N11*LAST
N21E=N20+E+1
N21T=N21E+1
N23E=N21E*NUMEL
N24E=N23E*NUMEL
N25E=N24E*NUMEL
N26E=N25E*NUMEL
NLAST=N26E
INITIALIZATION
DO 303 I=N21T,N26E
IA(I)=0
CONTINUE
N8=1
N9=N8+NEQO*MBANDC
NB2=N8-2-1
NB3=N8-2+NEQD*NECD
NB4=N8-3-1
NB5=N8-3+NEQA*NEQA-1
NBC=0
DC=30
B(I)=0.
CONTINUE
DO 304 I=1,NCCE
C(I)=0.
CONTINUE
DO 302 N=1,NUMEL
NN=10+N0+11*N-3
CONTINUE
C
C 165 DETERMINE ELEMENT TYPE,MATERIAL NC,
INTER INTEGRATION NO. TO DETERMETHER CALCULATION OF STIFFNESS
C OR STRESS IS NEEDED,AND WHETHER IT IS UPPER ELEMENT OF JOINT OR
C NOT,STRESS IS NOT NEEDED,AND WHETHER IT IS UPPER ELEMENT OF JOINT OR
C NOT,UPPER ELEMENT,NO STIFFNESS ,NO STRESS
C 1? C
C 2? C
C 3? C
C 4?UPPER ELEMENT ,NO STIFFNESS ,BUT STRESS
C 5 NEED STIFFNESS ,AND STRESS
C 6 NEED STIFFNESS ,NO STRESS
C 7 NEED STIFFNESS ,NO STRESS
C
NTYPE=TA(NN10)
MATYPE=TA(NN10+1)
INTER=A(NN10-2)
IXN10=11*NN-11
C
C 166 STRESS-STRAIN MATRIX IN COMMON BLOCK
NC=N20*10*(N-1)
GO TO (401+402+403+404),NTYPE
C
C 167 ESTIF,170 ESTIF,171 ESTIF,172 ESTIF,173 ESTIF,174 ESTIF,175 ESTIF,176 ESTIF,177 ESTIF,178 ESTIF,179 ESTIF,180 ESTIF,181 ESTIF,182 ESTIF,183 ESTIF,184 ESTIF,185 ESTIF,186 ESTIF,187 ESTIF,188

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401 CONTINUE
C SOLID ELEMENT,SIL OR CONCRETE
C LOCATION OF MASS
C CALL LAMIA(IIX),1)
IF(INTER.EQ.0 * OR. INTER .EQ. 1) GO TO 405
C FORM ELEMENT STIFFNESS
CALL SOLID(ASA,A(NC))
405 CONTINUE
IF(INTER.EQ.0 * OR. INTER .EQ. 2)
1.0R. INTER .EQ. 6 * OR. INTER .EQ. 7) GO TO 415
C FORM STRESS-DISPLACEMENT MATRIX AT CENTER OF VOLUMN
CC1=-NU,CC2=NU,CC3=0.5-NU,FACCE/(1-2*NU)*(1+NU),NU=POISSON RATIO
P=A(NC+1)/A(NC)
CC1=1.0/P
CC2=1.0-CC1
CC3=0.5-CC2
FAC=A(NC)/CC1
CALL STORE(CC1,CC2,CC3,FAC)
C ELEMENT NO. AND STRESS SEQUENCE TRANSFORMATION
NSOLID=NSOLID+1
IA(N2E*NSOLID)=N
415 CONTINUE
C
IF(INTER *LE. 3) GO TO 406
CALL EJECT(ASA,TASA)
ITOP=ITOP+1
ITOP=ITOP
CALL LAMIA(IIX),1)
CALL STORE(B(NB1),B(NB2),C(1),TASA,NEQA,NEQD,MBAND0,36)
GO TO 302
406 CONTINUE
C STORE STIFFNESS MATRIX KAA,KAO,*KOO
CALL STORE(B(NB1)*B(NB2),C(1),ASA,NEQA,NEQD,MBAND0,24)
GC TO 302
C CONCRETE BEAM OR COLUMN ELEMENT
402 CONTINUE
C LOCATION OF MASS
CALL LAMIA(IIX),1)
C FORM ELEMENT STIFFNESS
N6B=N6C+1
N6B=N6C*(NATYPE-1)*3
N10B=N11*(N-1)*11
N11B=N1B-1
N11B=N11+(N6C-1)*3
N6E=A(N1B)
N7B=N7*(N6E-1)*6
IF(INTER *EQ. 0) GO TO 407
CALL CONCTS(ASA,A(N6B),A(N7B),IA(IIX),IA(N1B),A(NC),N10,NN10,
1. IF(INTER .EQ. 2) GO TO 410
IA(N3E*NCONC)=N
410 CONTINUE
407 CONTINUE
C STORE STIFFNESS MATRIX
        CALL STORE(B(NB1),B(NB2),C(11),ASA,NEQA,NEQD,MBAND0,12)
        GO TO 302
403 CCNTINUE
C FORM ELEMENT STIFFNESS
NFR=NFR+1
IA(N2E*NFR)=NFR
IA(N2E*NFR)=N
NAA=121E-1N
NBB=124E-1NFR
IF(INTER .EQ. 0) GO TO 408
ETSTIF.*150
ETSTIF.*151
ETSTIF.*152
ETSTIF.*153
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ETSTIF.*207
ETSTIF.*208
ETSTIF.*209
ETSTIF.*210
ETSTIF.*211
ETSTIF.*212

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K=I-1
  DO 303 J=1,K
    S(I,J)=S(J,I)
  303 CONTINUE
C   CALCULATE FIXED END FORCES IN LOCAL COORDINATES
C   AX=COPROP(11)
  AX=A*XPRAC(3)
  L=0
  CALL FBCC1(XX,YY,ZZ)
C   MODIFY ELEMENT STIFFNESS AND ELEMENT FIXED END FORCES
C   FOR KNOWN ZERO MEMBER FORCES
C   IF(IJK(1)+J,(2)) .EQ. 0) GO TO 401
  DO 305 K=1,2
    KK=JK(K)
    KD=1000000
    I1=6*(K-1)+1
    I2=I1+5
    DO 305 I=I1,I2
      IF (KK .LT. KD) GO TO 305
      SII=(I,I)
      DO 306 N=1,12
        R(N)=S(I,N)
      306 CONTINUE
      DO 307 M=1,12
        SF(M)=SF(M)-C(M)*R(N)
      307 CONTINUE
      SF=SF(I)
      DO 308 N=1,12
        SF(M)=SF(M)-C(M)*SF(I)
      308 CONTINUE
      KK=KK-KD
      KO=KD/10
      C   OBTAIN SA(12,12) RELATING ELEMENT END FORCES (LOCALLY) AND JOINT
      C   DISPLACEMENT (GLOBAL)
      309 CONTINUE
      401 CONTINUE
C   ELEMENT STIFF ASA(112,12) AND FIXED END FORCES RRF (112)
C   IN GLOBAL COORDINATES
      DO 312 K=1,3
        X=K*S(I,K+HB)*T(K,M)
      311 CONTINUE
      SAI(1,J)=X
      310 CONTINUE
C   ELEMENT STIFF ASA(112,12) AND FIXED END FORCES RRF (112)
C   IN GLOBAL COORDINATES
      DO 312 K=1,1296
        ASAI(1)=
  CONCTS.69
  CONCTS.70
  CONCTS.71
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  CONCTS.192

  312 CONTINUE
    OC 313 LA=1,10,2
    LB=A-1
    DO 313 MA=1,10,3
      MB=IA+2
      DO 313 IL=1,3
        I=IL+LB
        J=MA+HB
        X=0.
        X=X*T(K,IL)*SA(I,K+(L,B),J)
      313 CONTINUE
      X=0.
      X=X*T(K,IL)*SF(I,K+(L,B),J)
    314 CONTINUE
    DO 315 LA=1,10,3
      LB=A-1
      DC 315 IL=1,3
      I=IL+LB
      X=0.
      X=X*T(K,IL)*SF(I,K+(L,B),J)
    315 CONTINUE
    IF ((JK(1)+JK(2)) .EQ. 0) GO TO 402
    C   MODIFIED STATIC LOAD VECTOR DUE TO END RELEASE
    C   NO=6*NHNP
    NO=6*NHNP
    DO 317 J=1,2
      NC0=IA(NN1,0)+J
      NX0=NODE-5
      NX0=NODE-NYA
      IA=IAN(XA)
      IA2=IAN(XA1)
      IA3=IAN(XA2)
      IA4=IAN(XA3)
      IA5=IAN(XA4)
      IA6=IAN(XA5)
      IA7=IAN(XO1)
      IA8=IAN(XC1)
      IA9=IAN(XO2)
      IA10=IAN(XC2)
      IA11=IAN(XC3)
      IA12=IAN(XC4)
      IA13=IAN(XO5)
      IA14=IAN(XC6)
      IA15=IAN(XC7)
      IA16=IAN(XC8)
      IA17=IAN(XC9)
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  CONCTS.182
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  CONCTS.187
  CONCTS.188
  CONCTS.189
  CONCTS.190
  CONCTS.191
  CONCTS.192

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407 CONTINUE
C      22 MOMENT
I1IA4=EQ. 0) GO TO 408
A(I1A6,N12E)=A(I1A6+N12E)-RFL(JM+6)+RRF(JM+6)
GO TO 402
408 CONTINUE
IF(I106 .EQ. 0) GO TO 402
A(I06,N13E)=A(I06+N13E)-RFL(JM+6)+RRF(JM+6)
317 CONTINUE
402 CONTINUE
412 CONTINUE
IF(INTER .EQ. 2) GO TO 413
N0NC=N0NC+1
      WRITE ELEMENT STIFFNESS INFORMATION ON TAPE 9
NS-12
NS-12
      WRITE(9) ND*NS,(LMA(I),I=1,ND),(LM0(I),I=1,NC),
     1 ((SA(I,J),I=1,NS),J=1,NO),(SF(I),I=1,NO)
1 413 CONTINUE
      RETURN
END
OVERLAY(6.0)
PROGRAM MODES
COMMON/TAPES/NMASS,NT
COMMON/TAPES/NSTIF,NMASS,NT
COMMON A(30000)
COMMON/DAMP2/NDAMP,NROOT,NPXP,XI(20),NXI(20),CMEGA(20)
COMMON/ELPAR/NUMF,NUMEL,NETYPE,NEQA,NEQC,HBAND,PKLIN,NLASTH
NTO=20000
NSTIF=6
NMASS=5
NT=7
NEQ=NEQA
MBAND=HBANDA
NBLOC=1
N2SE=NLAST
N10=N26E+1
NF=NROOT
      IF(NRND .GT. 20) WRITE(11,101)
101 FORMAT(' THE DIMENSION OF OMEGA IS NOT ENOUGH AT MODES')
      WRITE(11,103) NEC,BAND,NF
NIN=3
NC=NF*NM
NMA=NEQ*BAND
NM2=NMA*NA
NM3=NM2+NEQ
NM4=NM3+NEQ
NM5=NM4+NEQ
NM6=NM5+NEQ
NM7=NM6+NEQ*NC
NM8=NM7+NEQ*NC
NM9=NM8+NC
NM10=NM9+NC
NM11=NM10+NC
NM12=NM11+NC
NM13=NM12+NC
      IF(NTOT-NM13) 401,402,402
401 WRITE(11,102) NM13
      STOP
402 CONTINUE
      CALL SECANTOA(N18,A(NM2),A(NM3),A(NM5),A(NM6),A(NM7),
     1 A(NM8),A(NM9),A(NM10),A(NM11),A(NM12),NEQ,HBAND,NMA,NE,NC)
      MODES=41

```

```

NM8=NM8-1
DO 301 I=1,NROOT
  OMEGA(I)=6.28318/A(NM8+E)
301 CONTINUE
      RETURN
103 FORMAT('1PROBLEM INFORMATION
   1      * NO. OF EQUATIONS
   2      * NO. OF BANDWIDTH OF A
   3      * NO. OF FREQUENCYSEG
   4      * FOR EXECUTION NEED TO INCREASE NTOT TO *,16)
      END
      SUBROUTINE SECANT(A,B,V,MAXP,N,MM,RCOT,TIP,ERRV,ERRR,
     NITE,NA,MA,NA,MA,NA,MA,NA,MA,NA,MA,NA,MA,NA,MA,NA,MA,NA)
      COMMON/TAPES/NSTIF,NMASS,NT
      DIMENSION A(NMA),B(NV),V(NV),VS(11),W(N),VV(N,NC),WM(N,NC),
     1 TINC,ERRVN,ERRRNC
      * INTEGER KITE(NC),MAXA(NC)
      ACTOL=1.0E-04
      RCBTOL=1.0E-06
      RTCL=1.0E-10
      ROTOL=1.0E-12
      SCALE=1.0*900
      NNROOT=NRCOT
      C
      NTFS
      ITEM=10
      NITEM=10
      MODES=3
      MODES=2
      MODES=3
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      MODES=99
      MODES=100
      W(I)=B(I)
      DO 100 I=1,N
      100 RT=1.0
      ITT=0
      KK=0
      KITE=11111
      110 ITT=11111
      00 120 I=1,N
      V(I)=W(I)
      120 CALL BANDET(A,B,V,MAXA,N,MM,RA,NM,SC,DETA,ISC,KK)

```

```

KK=2
ROT=0.0
DO 130 I=1,N
  ROT=RQ*(W(I))*V(I)
130
DO 180 I=1,N
  W(I)=B(I)*V(I)
180
RQ=0.0
DO 140 I=1,N
  RQB=RQ*(W(I))*V(I)
140
  RQRQ=RQB
  WRITE(1,1004) RQ
  BS=SQR(RQB)
  TOL=ABS((RQ-RT)/RQ
  IF ((TOL.LT.RGTOL)) GO TO 150
  DO 160 I=1,N
    M(I)=W(I)/BS
160
    R=ROT
    IF ((IITE.LT.IITER)) GO TO 110
    IF (V(I)=V(I))
170
    RB=IC*(1.0-AMIN1(0.0,100*TOL))
    IS=0
    CALL BANDET (A*B,V,MAXA,N,NWA,RB,N SCH,DE TB,ISC,1)
    WRITE(1,1020) RB,N SCH
    FB=DETB
    IF (NSCH.EQ.0) GC TO 300
    IS=IS+1
    IF ((ISLE.NNTF)) GC TO 240
    WRITE(1,1030)
    STOP
    RB=RB/N SCH+1)
240
  GO TO 230
C   ITERATION FOR INDIVIDUAL ROOT
C
  STOP WHEN REQUIRED NO. OF ROOTS SMALLER THAN RC AND NOV=0 FOUND
C
  C 300  WRITE(1,1040)
  NITE(JR)=1
  WRITE(1,1050) JR,NITE(JR),RA,ETA,FA,ETA,ISC
  NITE(JR)=2
  WRITE(1,1050) JR,NITE(JR),RB,DETB,FB,ETA,ISC
C
  C 310  IF (NSCH.GE.NRC01) GO TO 900
  C
  DIF=FB-FA
  IF ((DIF.LE.0.0)) GO TO 320
  DO 900
    DEL=FB-(RB-RA)/DIF
    RC=RB-ETA*DEL
    TOL=ROBTOL*RC
    IF ((ABS((RC-RB).GT.TOL)) GO TO 330
    WRITE(1,1070)
    ROOT(JR)=RB
    GO TO 400
C   CALL BANDET (A,B,V,MAXA,N,NWA,RB,N SCH,DE TB,ISC,1)
330
  FG=DETB
  NITE(JR)=NITE(JR)+1
  IF ((JR.EQ.1)) GO TO 340
  IF ((IS.EQ.1)) GO TO 340
  MODES=104
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  MODES=226
  MODES=227

J=J+1,JJ
DO 350 K=1,JJ
  FCFC/(RC-ROOT(K))
  WRITE(1,1050) JR,NITE(JR),RC,ETC,FC,ETA,ISC
350
C   START INVERSE ITERATIONS
C
  NES=0
  IF ((JR.EQ.1)) GC TO 380
  DO 360 I=1,JJ
    IF ((ROOT(I)).LT.RC)) NES=NES+1
    NOV=N SCH-NES
    IF ((NOV.EQ.0)) GO TO 370
    WRITE(1,1080) NOV
    ROOT(JR)=RC
    IF ((NOV.GT.1)) NES=1
    GO TO 400
360
  380
C   RRFA
  FAFB
  DET=DETA
  RA=RB
  FA=FB
  DETB=DETB
  RB=RC
  FB=FC
  DETB=DETC
  MODES=104
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NITE(JR)=NITE(JR)+1
440
  DO 450 I=1,N
    W(I)=B(I)*V(I)
450
  V(I)=W(I)
  IS=0
  CALL BANDET (A,B,V,MAXA,N,NWA,RC,N SCH,DE TB,ISC,KK)
  IF ((IS.EQ.1)) GO TO 460
  IF ((IS.EQ.1)) GO TO 460

```



```

      FR=FR/(RR-NROOT(L))
      IF (RROT(JR).LE.FC) NOV=NOV-1
      JR=JR+1
      NITE(JR)=0
      ROOT(JR)=RC
      IF (NOV.GT.0) GO TO 400
      NSK=0
      ETA=2.0
      GO TO 300
C      MODES=352
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      MODES=473
      MODES=474
      MODES=475

      C      FORMAT (IH,12E11.4)
      1002 FORMAT (IH,12E11.4)
      1004 FORMAT (IH,6E22.12)
      1006 FORMAT (IH,6I20)
      1008 FORMAT (IH,6F20.2)
      1010 FORMAT (IH,3HINVERSE INTERN GIVES FOLLOWING APPROXIMAT TC LCMESMODES=19
      1T EIGENVALUE )
      1015 FORMAT (4HINTERN ABANDON INTERN BECAUSE NO CF INTERN IS I3,9H FOR EGMODES=421
      1T I3 )                                             MODES=422
      1020 FORMAT (3HORB = E20.12.7H NSCH = I4)
      1030 FORMAT (3H0ME BETTER CHECK THE MATRICES )
      1040 FORMAT (IH1*XX*4HROOT*X,4HNNIE,18X*2HRC,15X*12HDET (A=RC*B)*15X,MODES=125
      /2HPC*15X,3HETAB*4,3HNSC)                                MODES=425
      1050 FORMAT (IH0*4X*14*XX*IX,*EX*3E22*14*F7*2,I6)
      1060 FORMAT (42HOTHE DEFLATED POLYNOMIAL HAS NO MORE ROOTS )
      1070 FORMAT (429H0IRCRB) IS SMALLER THAN TOL )
      1080 FORMAT (4HONE JUMPED OVER I4,16H UNKNOWN ROOT(S) )
      1090 FORMAT (1H1,36H MORE STORAGE FOR VECTORS WE QUIT )
      1100 FORMAT (IH0*34*4HROOT*18X*2HRC,18X*4HNOR=,12)
      1110 FORMAT (IH0*4X*14*XX*IX,*EX*2E22*14)
      1120 FORMAT (120HNTIME FOR INV TIEFA F5.2)
      1130 FORMAT (20HOTHE EIGENVALUES ARE )
      1140 FORMAT (42H0N OF ITERATIONS FOR EACH EIGENVALUE ARE /)
      1150 FORMAT (30HOTHE USED FOR EACH EIGENVALUE /)
      1160 FORMAT (43HOFOLLOWING ARE ERROR BOUNDS ON EIGENVALUES )
      1170 FORMAT (1H1,62HNE ACCEPT FOLLOWING FREQUENCIES AND MODES ARRANGE MODES=339
      . IN ORDER )
      2000 FORMAT (35H****PRINT OF FREQUENCIES AND PERIODS //
      6X*4HMODE,9X,11HFREQUENCIES,13X*7HPERIODS /
      6X*4HNC,3X,11H (RAD/SEC),13X*7H (SEC) )
      2001 FORMAT (110,2F20.4)
      3000 FORMAT (//46H EIGENVALUE AND EIGENVECTOR ARE STORED IN TAPE,I3,MODES=445
      C      END
      $      SUBROUTINE EANCET (A,B,V,MMAXA>NN,MMA,RA,NSCH,DET,ISCALE,KK)
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      MODES=471
      MODES=472
      MODES=473
      MODES=474
      MODES=475

      C      DIMENSION A(MMA),B(I1*V(1)),MAXA(1)
      C      COMMON TAPES$NSTIF,NMASS
      C      TRIANGULARIZE BANDED STIFFNESS MATRIX
      NR00=NSCH
      NR00=NR00
      DO 960 I=1,NROOT
      IF (RROOT(I)*LE.0.0) GO TO 960
      RROOT(I)=SORT(RCCT(I))
      CONTINUE
      IF (IS.GT.0) GO TO 910
      C      WRITE(I,1170)
      NR00=NSCH
      NR00=NR00
      DO 960 I=1,NROOT
      IF (RROOT(I)*LE.0.0) GO TO 960
      RROOT(I)=SORT(RCCT(I))
      CONTINUE
      WRITE(I,3000) NT
      IF (INT.NE.7) NT=7
      REMIN NT
      WRITE (INT) (RCCT(I),I=1,NROOT)
      C      PRINT FREQUNCIES AND MODE SHAPES
      C      WRITE(I,2000)
      DO 990 I=1,NROOT
      PERIOD=6.2831857/ROOT(I)
      WRITE(I,2001) I,ROOT(I),PERIOD
      ROOT(I)=PERIOD
      C      WRITE (INT) ((VV(I,J),I=1,N),J=1,NROOT)
      RETURN
C      MODES=412
      MODES=413
      PIV=A(N)
      MODES=475

```



```

N14=N14E+1
N15=N14+NEQA
N15E=N15-1
N16=N15+NEQO
N16E=N16-1
N17=N16+NEQA
N17E=N17-1
N18=N17+NEQO
N18E=N18-1
N19=N18+NEQA
N19E=N19-1
N13=N13E+1
N12=N13-1
N12E=N12-1
N10P=N12-11*NUEL-3*NUMBC
N8=N10P-6*NUMATE-4*NUMATF
N8EP=N8-1
N12P=N12E
N21T=N21E+1
N2A=N20E*(MBANDO+NEQA)
N2B=N20+NAV*
NAB=NEQA*NEQO*(MEANDO-2)*NEQC+2)
N2M=N21BNAB
MBANDO=NEO-1
SOLVE FOR CO, AND STORE IN TAPE 6
WHERE KOO-60--(KAO)
THE DIMENSIONS ARE (NEQ*MBANC)*(NEQC+NEQA)=(NEQC*NEQA)
CHECK DIMENSION B,C REQUIRED
NBD1=NBD1,NBD2,NCD
PRINT 1,NBD1,NBD2,NCD
1 FFORMAT*, LARGE CORE FOR B IS NBD1,NBD2*,210/
C NCD*,110/
1 * CHECK LARGE STATEMENT IN STATIC AND RFL CARD*
CALL SESOL(B(N20A),C(1),B(N21B),NEQO,MBANDO,NEGA,1,NEQA,NAV,MI,
1 4,10,6,8)
REIND FOR GO
REIND 6
C REIND FOR KAA,STCRE IN C(N22T),(KAO) IN E(N21T)
REIND 2
C
N24T=NEOT*NEQA*NECO
N24E=N21T-1
N25T=1
N26E=N21E*NEQA*NEGO
N27T=N22T*NEQA*NEGA
N28E=N23-1
NA=N21T
NC=N20T
NCA=NECO
NRA=NEA
NCE=NEQA
NFB=NEQO
K=2
K=2
MULTIPLICATION AND SUMMATION KAABAR=KAA*KAO*GO
THE DIMENSION ARE
(NEQA*MBAD)=NEQA*MBANDO)+(NEQA*NEQO)*(NEQO*NEQA)
DO 300 I=N22T,N23E
C(I)=0.
300 CONTINUE
N23E=N23-1

```

21 FORMAT\*, BAND WIDTH OF KAABAR IS\*,I5)
 READ(12,101)
 READ(16,102)
 CALL MULT
 STORE KAAEAR INTO TAPE 2
 PRINT 21,REANK
 MBAND=MBANDR
 MBAND=MBAND
 WRITE(12,101)
 WRITE(12,101)
 MULTIPICATION INSIDE LARE CORE,SLVA,GLXA
 DO 308 I=1,NEQA
 C(I)=AN12E+1)
 B(NB3+I)=A(N14E+1)
 308 CONTINUE
 C
 SLVO,DL,XO
 DO 309 I=1,NEQO
 B(NB2+I)=A(N13E+1)
 B(NB4+I)=A(N15E+1)
 309 CONTINUE
 C
 CALCULATE CONDENSED LOAD VECTR
 PABAR=PA(GO)\*PO
 THE DIMENSIONS ARE (NEQA\*)=(NEQA\*1)+(NEQA\*NEQO)\*(NEQO\*1)
 TTRANSPOSE (GO) TO (GO) AND STORE IN A(N21T)
 STATIC.72
 STATIC.73
 STATIC.74
 STATIC.75
 STATIC.76
 STATIC.77
 STATIC.78
 STATIC.79
 STATIC.80
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 STATIC.171
 STATIC.172
 STATIC.173
 STATIC.174

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N24E=N23E*NEGA*NEQA
N24T=N24E*I
READ(51,B(I),I=1,NB2E)
DO 302 I=N22T,N23E
C11=0,
CONTINUE
N1=N102
NB=N101
NC=N102T
CALL MULT
NCA=1
NRA=NEDO
NCS=NEQA
NRE=NEQD
K=1
C
N2ET=N20T-1
N22E=I+22-1
MH=N3A*NQ0
DO 600 I=*,MM
BIN20ET+I=CIN2ET+I
CONTINUE
N3E=0
N24E=N23E*NEGA*NEQA
N27T=N23E*I
DO 303 I=N23T,N24E
C11=0.
CONTINUE
DC 304 I=1,NQQA
CIN23E+I)=B(I)
CONTINUE
NAN=21T
NC=N23T
CALL MULT
STORE IN TAPE 5 MAABAR(NEQA,MEANC)
PRINT 22, MBAND
FORMAT*, END WITH MAABAR'S,IS)
IF (MBAND .GT. MBAND) MBAND=MBAND
IF (MBAND .LT. MBAND) MBAND=MBAND
N24E=N23E*NEGA*NEBAND
REIND 5
WRITE(65) C(I),I=N23T,N24E)
C
CALCULATE
RMASS(X)=MAA(X)+GOIT*HOC(X)
AND WRITE IN TAPE5, AFTER MAABAF, FOR DYNAMIC INERTIA FORCE
THE DIMENSION ARE
INEQA=NEGA+NECA*(NEQD*1)
N1=N21T
NB=NB4
NC=1
DO 601 I=1,NEQA
C(I)=B(NB4E+I)
CONTINUE
NCA=NEQD
STATIC.175
NRA=NEQA
NC8.1
NRB=NEQD
STATIC.176
STATIC.177
K=1
CALL MULT
WRITE(15) C(I),I=1,NB2E)
C
THE DISPLACEMENT FOR DO F REMAINED AND ELIMINATE WILL REPLACE
THE DISPLACEMENT FOR DO F REFINED AND ELIMINATE WILL REPLACE
LCAD VECTOR IN A (N12),A(N13)
UNSYMETRICAL SOLUTION
KABARUA=PAABAR
REIND 2
N24E=N23E*NQ0
READ(2) (B(I),I=201,N2LE)
CALL TRIA(B(N201),NEQA,MBANDK)
CALL BICKS(B(N201),A(N12),NEQA,MBANDK)
STORE DISPLACEMENT AT RELATIVE NODE UA IN COMMON BLANK A
NEW SEQUENCE IN COMMON A
N1=N16+*NUMNP
N2E=N2-1
STATIC.185
STATIC.186
STATIC.187
STATIC.188
STATIC.189
STATIC.190
STATIC.191
STATIC.192
STATIC.193
STATIC.194
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STATIC.292
STATIC.293
STATIC.294
STATIC.295
STATIC.296
STATIC.297
STATIC.298
NN=N3*((I-1)*3
IA(NN3)=IA(NN1+6)
IA(NN31)=IA(NN10+9)
IA(NN3+2)=IA(NN10+10)
401 CONTINUE
C
SHIFT ELEMENT TYPE, MATERIAL TYPE AND INTEGRATION NO. INTO IX(3)
DO 310 I=1,NUEL
SHIFT ICOL TO A(N10)
IF INTM .EQ. 01 60 TO 401
NN1D=N10P*(I-1)*1
IA(N10+I)=IA(N21EP+I)
315 CONTINUE
406 CCNTINUE
C
STRESS OUTPUT ELEMENT TRANSFORMATION VECTOR
DO 310 I=1,NUEL
SHIFT ICOL TO A(N10)
IF INTM .EQ. 01 60 TO 401
NN1D=N10P*(I-1)*1
IA(N10+I)=IA(N21EP+I)
401 CONTINUE
C
SHIFT ELEMENT TYPE, MATERIAL TYPE AND INTEGRATION NO. INTO IX(3)
NN=N3*((I-1)*3
IA(NN3)=IA(NN1+6)
IA(NN31)=IA(NN10+9)
IA(NN3+2)=IA(NN10+10)

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310 CONTINUE
C SHIFT ISOL,ICOM,IFR,ISP
C IF (NSOLID .EQ. 0) GO TO 402
DO 311 I=1,NSOLID
IA(N11E+I)=IA(N22EP+I)
311 CONTINUE
402 CONTINUE
C
C IF (NSPRIN .EQ. 0) GO TO 403
DC 312 I=1,NCONC
IA(N12E+I)=IA(N23EP+I)
312 CONTINUE
403 CONTINUE
C
C IF (NINEL .EQ. 0) GO TO 404
C
C DO 313 I=1,NFR
IA(N13E+I)=IA(N24EP+I)
313 CONTINUE
404 CONTINUE
C
C IF (NSPRIN .EQ. 0) GO TO 405
C
C DO 314 I=1,NSPRN
IA(N14E+I)=IA(N25EP+I)
314 CONTINUE
405 CONTINUE
C
N15E=N15+1
A(IJ)=I=N4,N10
DO 315 I=N4,N10
A(IJ)=0.
315 CONTINUE
C STATIC DISPLACEMENT AS INCREMENT
N4=E=N4-1
N7=E=N7-1
DO 316 I=1,NEQA
A(N4E+I)=A(N12P+I)
A(N7E+I)=A(N12P+I)
316 CONTINUE
RETURN
END
OVERLAY10+0
PROGRAM STRESS
*****
***** COMPONENTS AT TWO ENDS FOR FEAM OF COLUMN ELEMENT *****
***** REQUIRED 12 COMPONENTS AT THE TOTAL STRESS OF THE ELEMENT *****
***** CALCULATE INCREMENAL STRESS AND THEN TOTAL STRESS *****
***** AT CENTER OF VOLUMN AND CENTER OF IEJBKE LE FACE FOR SOIL ELE HNSTRESS.7 *****
***** 3 FORCES, 1 NORMAL AND 2 SHEAR FOR FRICTION ELEMENT *****
***** 6 COMPONENTS FOR BOUNDARY ELEMENT *****
***** OVERLAY10+0 *****
COMMON/STROUT/NSCLD,NCONC,NFR,NSPRIN
COMMON/ELPAR/NUNP,NUMEL,NETYPE,NEQA,NEOC,MBANDA,PBANDA,PKLIN,NLASTSTRESS,12
COMMON/HATER/NUHATS,NUMATC,NUMATE,NUMATE,NUMGE,NUMBC,MTYPE
COMMON/VTIME/JUMP,TDT,MPRTM,NTAPE,KRINT
COMMON/NEWAL/NIN,NINEL
DIMENSION SS(13),XYZ(3,3),XYZEL(3,3),LMA(24),LHO(24),
ISAI(1,2,24),SA21(2,2),SA4(6),ISS(11),IA11,
EQUVALENCE(ISS,SS)
C
N15=NLAST
N16=N15+12*NUMEL

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IF(I .GE. 13) GO TO 406
JJB=LM(J+24)
NQUB=NFE+JJB
TODISPA(NQUB)+A(INDUB)
A(INDSIG)=A(INDSIG)+SAI(I,I)*TODISP
GO TO 308
406 CONTINUE
A(INDSIG)=A(INDSIG)+SAI(I,J)*A(NUCA)
308 CONTINUE
A(INDSIG)=A(INDSIG)+A(INDSIG)
307 CONTINUE
GO TO 302
402 CONTINUE
C TYPE 2: BEAM,COLUMN ELEMENT
READ(19) (SS(I),I=1,12)
SET UP STRESS-DISPLACEMENT MATRIX SA2(12,12)
C KK=14
DO 309 I=1,12
K1=KK+12*I
DO 309 J=1,12
J1=I+J
SA2(I,J)=SS(J1)
309 CONTINUE
C SET UP LOCATION OF MASS
DO 310 I=1,12
LMAIT=ISS(2*I)
LMDIT=ISS(14+I)
310 CONTINUE
CALCULATE INCREMENTAL STRESS A(N19) OR A(INDSIG)
C AND THE TOTAL STRESS A(N19) OR A(INDSIG)
DO 311 I=1,12
NOSIG=N15EL+I
NSIG=N16EL+I
A(INDSIG)=0.
DO 312 J=1,12
JJ=LMA(J)
IF(I,JA,EQ,0) GO TO 312
NQUB=NFE+JJA
A(INDSIG)=A(INDSIG)+SA2(I,J)*A(NUDA)
311 CONTINUE
GO TO 302
404 CONTINUE
C TYPE 4: BOUNDARY ELEMENT
READ(19) (SS(I),I=1,20)
SET UP STRESS-DISPLACEMENT MATRIX SA(6)
KK=4
DO 317 I=1,6
S44(I,1)=SS(I+KK)
317 CONTINUE
C LOCATION OF MASS
DO 318 I=1,6
LMA(I)=ISS(I+2)
318 CONTINUE
C CALCULATE INCREMENTAL STRESS A(N19) OR A(INDSIG)
C AND THE TOTAL STRESS A(N19) OR A(INDSIG)
DO 319 I=1,6
NDSIG=N15EL+I
NSIG=N16EL+I
STRESS.06
STRESS.07
STRESS.08
STRESS.09
STRESS.10
STRESS.11
STRESS.12
STRESS.13
STRESS.14
STRESS.15
STRESS.16
STRESS.17
NQUDAT=MA(I)
IF(I,JA,EQ,0) GO TO 319
NQUDAT+=JJA
A(INDSIG)=A(INDSIG)+SAI(I)*A(NUDA)
A(INDSIG)=A(INDSIG)+A(INDSIG)
319 CONTINUE
302 CONTINUE
RETURN
END
OVERLAY(11,0)
PROGRAM DIDAMP*****
C ***** DIRECT DAMPING MATRIX C(I,J) ****
C ASSEMBLE ***** C
C ***** ***** ****
COMMON A(30000)
COMMON DAMP/NDAMP, NROOT, NFP, XII(20), MXI(20), OMEGA(20),
COMMON ELPAR/NUINF, NUHML, NETTYPE, NEQR, NEOC, MBANDA, FBANDA, KLIN, LASTDIAH, 9
DIDAMP.0
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DIDAMP.53
A(11)=0.
302 CONTINUE
NA=N37
NB=N38
NC=N39
NCA=N39A
NRA=N39A
NCB=DAMP
NRB=NEQA
CALL MULT
TRANPOSE VV=A(N38) INTO (VV) T=A(N40)
301 CONTINUE
THE(I,J) MASS NORMALIZED MODE SHAPE MATRIX INT C (N39)
DO 302 I=N39,N40 E
A(T1)=0.
302 CONTINUE
NA=N37
NB=N38
NC=N39
NCA=N39A
NRA=N39A
NCB=DAMP
NRB=NEQA
K=1
CALL MULT
TRANPOSE VV=A(N38) INTO (VV) T=A(N40)

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      K=N40      C   MULTIPLE THETA*BETA*(THETA) TRANSPOSE INTO A (N38)
      DO 303 I=1,NEQA
      L=NA+I-1
      DO 303 J=1,NDAMP
      A(K)=AL(J)
      L=L+NEQA
      K=K+1
  303 CONTINUE
      C   NORMALIZED MASS(I,J) INTO A (N38)
      N38=N4E*NDAMP*NEQA
      DO 304 I=N38,N39E
      A(I)=0.
  304 CONTINUE
      NA=N40
      NB=N39
      NC=N38
      NCA=NEQA
      NRA=NDAMP
      NCB=NDAMP
      NRB=NEQA
      K=1
      CALL MULT
      CONDENSED AMASS(I,J) TO A(MASS(I,J)) AT A (N38)
      DO 313 I=1,NDAMP
      NR=(38+I-1)*NDAMP+(I-1)
      NDIA=N38E+I
      A(INDIA)=A(NR)
  313 CONTINUE
      C   DIAGONAL MATRIX BETA INTO A (N40)
      DO 305 I=1,NDAMP
      NBETA=N40+I-1
      K=NX1(I)
      NMASS=N38+I-1
      A(BETA)=2.0*X1(I)*OMEGA(K)/A(NMASS)
  305 CONTINUE
      N4E=NA0+1
      C   TRANSPOSE THETA A (N39) INTO A (N38)
      NA=N39
      K=N8
      DO 306 I=1,NEQA
      L=NA+I-1
      DO 306 J=1,NDAMP
      A(K)=AL(J)
      L=L+NEQA
      K=K+1
  306 CONTINUE
      N4E=NA0*NEQA*NDAMP
      C   MULTIPLE BETA*THETA) TRANSPOSE INTO A (N41)
      N4E=N4E+1
      DO 307 I=N41,N42E
      A(I)=0.
  307 CONTINUE
      NA=N40
      NB=N38
      NC=N41
      NRA=NDAMP
      NCB=NEQA
      NRB=NDAMP
      K=1
      B(K)=RATIO
      CALL MULT
      DIDAMP=54
      DIDAMP=55
      DIDAMP=56
      DIDAMP=57
      DO 308 I=N4 1
      A(I)=0.
      308 CONTINUE
      NA=N39
      NB=N41
      NC=N42
      NRA=NDAMP
      NCB=NEQA
      NRB=NDAMP
      K=1
      DIDAMP=59
      DIDAMP=60
      DIDAMP=61
      DIDAMP=62
      DIDAMP=63
      DIDAMP=64
      DIDAMP=65
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      DIDAMP=137
      DIDAMP=138
      DIDAMP=139
      DIDAMP=140
      DIDAMP=141
      DIDAMP=142
      STDAMP=2
      STDAMP=3
      STDAMP=4
      STDAMP=5
      STDAMP=6
      STDAMP=7
      STDAMP=8
      STDAMP=9
      STDAMP=10
      STDAMP=11
      STDAMP=12
      STDAMP=13
      STDAMP=14
      STDAMP=15
      STDAMP=16
      STDAMP=17
      STDAMP=18
      STDAMP=19
      STDAMP=20
      STDAMP=21
      STDAMP=22
      STDAMP=23
      STDAMP=24
      STDAMP=25
      STDAMP=26
      STDAMP=27
      STDAMP=28
      STDAMP=29
      STDAMP=30
      STDAMP=31
      STDAMP=32
      STDAMP=33
      STDAMP=34
      STDAMP=35
      STDAMP=36
      C   FORMATTED STATEMENT
      *****
      1 FORMAT(1Z15.5F5.0)
      2 FORMAT(1Z15.5F5.0)
      101 FORMAT(* NO. OF HIGH ZERO MODE DAMPING RATIO*,IS//)
      102 FORMAT(* NO. OF HIGHEST MODE WITH NONZERO DAMPING RATIO*,IS//)
      501 FORMAT(*1INPUT DAMPING DATA *//)
      102 FORMAT(1Z15.3)
      RETURN
      END
      OVERLAY(12,0)
      PROGRAM STDAMP
      *****
      C   FORMULATE STRUCTURAL DAMPING C(I,J)
      *****
      C   ASSEMBLE STRUCTURAL DAMPING
      COMMON A(30000)
      DIMENSION B(300)
      C   READ IN VISCOS DAMPING RATIO CF EACH NODE
      N37=N1LAST
      N38=N37+3*NEQA*NECA*7*NEQA
      N32=N37-3*NEQA
      N32E=N32-1
      N42=N42-5*NEQA
      N4E=N42-1
      NOLD=0
      1000 CONTINUE
      READ 1, NODE, RATIO, KN
      1 FORMAT(1Z15.10,0,5)
      PRINT 101, NODE, RATIO, KN
      500 FORMAT(*1INPUT DAMPING RATIO AT EACH NODE//)
      101 FORMAT(1Z15.3,15)
      IF(NOLD .EQ. 0) SC TO 401
      C   NUMN=NO. CF NODES TO BE GENERATED
      NUMN=NODE-NOLD
      NUMN=NUM-1
      IF(NUMN .LT. 1) GO TO 401
      C   GENERATE NEW NODE POINT
      K=NOLD
      DO 301 J=1,NUMN
      KK=K
      B(K)=RATIO
      301 CONTINUE
      END
  
```

```

401 CONTINUE
  NOLD=NODE
  IF(NODE .LT. NUMNP) GO TO 1000
  C PRINT OUT ALL NODAL DATA
  WRITE(1,501)
  501 FORMAT(* NODE VISCOS DAMPING RATIO DATA */
           *//)
  DO 302 N=1,NUMNP
    * NODE DAMPING RATIO
    WRITE(1,102) N,FATIO
  102 FORMAT(1X,F15.3)
  302 CONTINUE
  C FORM DAMPING FORCE VECTOR STORE IN A(N44)
  P=3.46159
  DO 303 I=1,NEQA
    VEL=A(N32+E*I)
    FORCE=A(N12+E*I)-B(I)*P**SIGN(FORCE,VEL)
  303 CONTINUE
  END
  OVERLAY(13,0)
  PROGRAM OUTPUT
  **** OUTPUT THE SELECTED RESULTS IN TIME HISTORY
  C **** COMMON/VELDAR/NUMP,NUMEL,NETYPE,NEQA,NEQD,HBANDA,HBANDO,KLIN,NLASTOUTP,*
  C **** COMMON/NEAL/ATM,ATMEL
  COMMON/NEAL/ATM,ATMEL
  COMMON/TIME/NTIME,NB66
  COMMON/A(30000)
  CCOMMON/BLCK/NSC,ASS,NSK,KK1,KK2,KK3,MOTS,MSTR,MHAL,NBD,NBS,NBW
  COMMON/QJ/UNA,0DT
  LARGE B(22000)
  C INPUT SPECIFICATION FOR OUTPUT OF RESPONSE TIME HISTORY
  REINTD 2
  REINTD 3
  REINTD 5
  REINTD 7
  REINTD 9
  REINTD 11
  REINTD 12
  NEQ=NEQA
  NH=1
  NH2=NH1+6+NUMNP
  NH3=NH2+NEQ
  NH4=NH3+6*NUMEL
  NH5=NH4+5*NTH
  CALL INOUT(A(NH1),A(NH2),A(NH3),A(NH4),NUMNP,NUMEL,NEQ,NTM)
  C PACK TIME HISTORY IN BLOCK
  MHTOT=50000
  NDS=NTIME
  ND5B=(MMTOT-NHS5-NEQ*2)/(MDIS*2)
  MDISB=(MMTOT-NHS5-9*NDIS)/(MDIS*2)
  IF(NDISB .GT. MDISB) NDISB=MDISB
  IF(NDISB .LT. MDISB) NOISB=NDISB
  NBD=(NDIS-1)/NOISB+1
  NSTRB=(MMTOT-NHS5-6*NUMEL)/(MSTR*2)
  MSTRB=(MMTOT-NHS5-5*NDIS)/(PSTR*2)
  NSTRB=NDS-1/NDS+1
  C IF(MHAL .EQ. 0) GC TO 402
  MWF0=MMTOT-NHS5*NTM/MHAL
  MWF0B=MMTOT-NHS9*NDIS/MHAL
  IF(MWF0 .GT. MWF0B) NMFB=MWF0B
  IF(MWF0B .GT. MWF0) NMFB=MWF0
  NB=(NDS-1)/NMFB+1
  C IF(MHAL .EQ. 1) NMF0=NMFB
  C IF(MHAL .EQ. 2) NMF0B=NMFB
  C IF(MHAL .EQ. 3) NMF0=NMFB
  C IF(MHAL .EQ. 4) NMF0B=NMFB
  C IF(MHAL .EQ. 5) NMF0=NMFB
  C IF(MHAL .EQ. 6) NMF0B=NMFB
  C IF(MHAL .EQ. 7) NMF0=NMFB
  C IF(MHAL .EQ. 8) NMF0B=NMFB
  C IF(MHAL .EQ. 9) NMF0=NMFB
  C IF(MHAL .EQ. 10) NMF0B=NMFB
  C IF(MHAL .EQ. 11) NMF0=NMFB
  C IF(MHAL .EQ. 12) NMF0B=NMFB
  C IF(MHAL .EQ. 13) NMF0=NMFB
  C IF(MHAL .EQ. 14) NMF0B=NMFB
  C IF(MHAL .EQ. 15) NMF0=NMFB
  C IF(MHAL .EQ. 16) NMF0B=NMFB
  C IF(MHAL .EQ. 17) NMF0=NMFB
  C IF(MHAL .EQ. 18) NMF0B=NMFB
  C IF(MHAL .EQ. 19) NMF0=NMFB
  C IF(MHAL .EQ. 20) NMF0B=NMFB
  C IF(MHAL .EQ. 21) NMF0=NMFB
  C IF(MHAL .EQ. 22) NMF0B=NMFB
  C IF(MHAL .EQ. 23) NMF0=NMFB
  C IF(MHAL .EQ. 24) NMF0B=NMFB
  C IF(MHAL .EQ. 25) NMF0=NMFB
  C IF(MHAL .EQ. 26) NMF0B=NMFB
  C IF(MHAL .EQ. 27) NMF0=NMFB
  C IF(MHAL .EQ. 28) NMF0B=NMFB
  C IF(MHAL .EQ. 29) NMF0=NMFB
  C IF(MHAL .EQ. 30) NMF0B=NMFB
  C IF(MHAL .EQ. 31) NMF0=NMFB
  C IF(MHAL .EQ. 32) NMF0B=NMFB
  C IF(MHAL .EQ. 33) NMF0=NMFB
  C IF(MHAL .EQ. 34) NMF0B=NMFB
  C IF(MHAL .EQ. 35) NMF0=NMFB
  C IF(MHAL .EQ. 36) NMF0B=NMFB
  C IF(MHAL .EQ. 37) NMF0=NMFB
  C IF(MHAL .EQ. 38) NMF0B=NMFB
  C IF(MHAL .EQ. 39) NMF0=NMFB
  C IF(MHAL .EQ. 40) NMF0B=NMFB
  C IF(MHAL .EQ. 41) NMF0=NMFB
  C IF(MHAL .EQ. 42) NMF0B=NMFB
  C IF(MHAL .EQ. 43) NMF0=NMFB
  C IF(MHAL .EQ. 44) NMF0B=NMFB
  C IF(MHAL .EQ. 45) NMF0=NMFB
  C IF(MHAL .EQ. 46) NMF0B=NMFB
  C IF(MHAL .EQ. 47) NMF0=NMFB
  C GO TO 403
  402 CCNTINUE
  NMFB=0
  NB=0
  403 CCNTINUE
  NH5=NH5-NEQ
  NH7=NH6-NEQ
  NH8=NH7*MOTS*NDISB
  NH10=NH5*E*NUMEL
  NH12=NH5*F*NTM
  NH13=NH12+MHAL*NWF0
  CALL REPACK(A(NH1),A(NH2),A(NH3),A(NH4),A(NH5),A(NH6),A(NH7),
             A(NH8),A(NH9),A(NH10),A(NH11),A(NH12),NDS,NEQ,MOTS,
             NDISB,NUMEL,MTRR,NSTRB,NTM,MHAL,NFB,BD,NBS,NBM,
             NUMNP)
  1
  2
  3
  C
  MT=0
  FILE=0
  IF(KM1 .EQ. 2) NFILE=NFILE+NDSD
  IF(KM2 .EQ. 2) NFILE=NFILE+NDSD
  IF(KM3 .EQ. 2) NFILE=NFILE+NDSD
  IF(FILE .EQ. 0) EO TO 401
  MT=4
  REWIND MT
  WRITE(MT) I
  WRITE(MT) NFILE,NDS,DDT
  401 CONTINUE(* OUTPUT*,215/(10E12*4))
  C
  OUTPUT.11
  OUTPUT.12
  OUTPUT.13
  OUTPUT.14
  OUTPUT.15
  OUTPUT.16
  OUTPUT.17
  OUTPUT.18
  OUTPUT.19
  OUTPUT.20
  OUTPUT.21
  OUTPUT.22
  OUTPUT.23
  OUTPUT.24
  OUTPUT.25
  OUTPUT.26
  OUTPUT.27
  OUTPUT.28
  OUTPUT.29
  OUTPUT.30
  OUTPUT.31
  OUTPUT.32
  OUTPUT.33
  OUTPUT.34
  OUTPUT.35
  OUTPUT.36
  OUTPUT.37
  OUTPUT.38
  OUTPUT.39
  OUTPUT.40
  OUTPUT.41
  OUTPUT.42
  OUTPUT.43
  OUTPUT.44
  C
  OUTPUT SELECTED ABSOLUTE DISPLACEMENT TIME HISTORY
  REWIND 2
  CALL CUTTHIS(A(NH1),A(NH2),A(NH3),A(NH4),A(NH5),A(NH6),A(NH7),
               NUMP,NUMEL,NTM,NDIS,NDISB,NSD,NDSD,ND0,1,KK1,2,11,MT,
               IFF,NEQ)
  1
  2
  C
  OUTPUT SELECTED ACCELERATION TIME HISTORY
  REWIND 2
  CALL CUTTHIS(A(NH1),A(NH2),A(NH3),A(NH4),A(NH5),A(NH6),A(NH7),
               NUMP,NUMEL,NTM,NDIS,NDISB,NSD,NDSD,ND0,2,KK1,2,12,MT,
               IFF,NEQ)
  1
  2
  C
  OUTPUT.95
  OUTPUT.96
  OUTPUT.97
  OUTPUT.98
  OUTPUT.99
  OUTPUT.100
  OUTPUT.101
  OUTPUT.102
  OUTPUT.103
  OUTPUT.104
  OUTPUT.105
  OUTPUT.106

```

```

C OUTPUT SELECTED STRESS TIME HISTORY
C REWIND 3
C CALL QUTHIS(A(NH1),A(NH2),A(NH3),A(NH4),A(NH5),A(NH6),A(NH7),
1      NUMN,NUREL,NTH,NSTR,NSTRE,NSS,NBS,3,KK2,3,7,MT,
2      IFF,NEQ)
C CASE OF NO WALL
C IF(NTH.EQ.0) GO TO 404
C OUTPUT SELECTED WALL FORCES TIME HISTORY
C REWIND 5
C CALL QUTHIS(A(NH1),A(NH2),A(NH3),A(NH4),A(NH5),A(NH6),A(NH7),
1      NUMN,NUMEL,NTN,NUSE,NUAL,NUFE,NSK,NEW,4,KR3,5,9,MT,
2      IFF,NEQ)
404 CONTINUE
C RETURN
END
SUBROUTINE QUTHIS(ND,DISR,ISR,INALL,TA,X,UH,NUMNF,NUMEL,NTN,
1      NDS,NDIN,NDJ,NDN,NNB,KKK,IT,JT,MT,IFF,NEQ)
C ****
C OUTPUT RESPONSE TIME HISTORY ON SPECIFIC DISPLAY MEDIUM
C COMMON/TIME/LUMP,T,DT,MPTIM,MTAPE,KPRINT
COMMON/ED/DIGANA,COT
DIMENSION TM(8),X(8),ID(6,NUMNP),ISTR(6,NUMEL),ITALL(5,NTN),TA(11)OUTHS.11,
1      COMMONTATETIME,NTIME,NCDE,LARGE,B220001,NBSE
DIMENSION KOT(4,8)
C TAPE IT INPUT TAPE STORE XH(NDIS,NDISB),X2HH(NDIS,NDISB)
C TAPE JT INPUT TAPE STORE XH(NDIS,ASTRB),XFH(NDIS,NDISB)
C IF(NOB.EQ.0) RETURN
DO 301 M=1,NOB
IFF=IFF+1
REWIND 11
READ(11) KOT,L
DO 302 I=1,8
TM(I)=0.0
XM(I)=0.0
302 CONTINUE
C PRINT APPROPRIATE TITLE
C GO TO 4014-4034,4044 KKK
401 CONTINUE
WRITE(11,501) M,IFF
WRITE(11,501) (KD(1,I),KD(2,I),I=1,L)
GC TO 405
402 CONTINUE
WRITE(11,502) M,IFF
403 CONTINUE
WRITE(11,503) M,IFF
WRITE(11,502) (KD(1,I),KD(2,I),KD(3,I),I=1,L)
GC TO 405
404 CONTINUE
WRITE(11,504) M,IFF
C OUTPUT TIME HISTORY IN OUTPUT FORM
405 CONTINUE
MPTIM=1-TAPE
N=0
DO 303 NB=1,NMB
OUTPUT.113
OUTPUT.114
OUTPUT.115
OUTPUT.116
OUTPUT.117
OUTPUT.118
OUTPUT.119
OUTPUT.110
OUTPUT.111
OUTPUT.112
OUTPUT.110
READ(JT) KUH
DO 304 J=1,K
N=N+1
MPTIM=MPTIM+1
TT=B(MPTIM+N56E)
DC=3.05 T=1.0
GO TO 411
T=1.0
GO TO (411,412,413,414) KKK
411 CONTINUE
JJ=KD(3,I)
II=DIS(JJ)
XX=UH(II,J)
GO TO 415
412 CONTINUE
JJ=KU(3,J)
II=DIS(JJ)
XX=UH(II,J)
GO TO 415
413 CONTINUE
LL=KD(2,I)
II=KD(3,I)
II=ISTR(II,LL)
XX=UH(II,J)
GO TO 415
414 CONTINUE
LL=KD(1,I)
AX=ABS(XX)
II=KD(2,I)
II=TMALL(II,LL)
XX=UH(II,J)
C ABSOLUTE MAX
415 CONTINUE
X(1,I)=XX
TA(N)=TT
IF (AX>X(1,I)) 416,417
417 CONTINUE
X(1,I)=AX
TM(1)=TT
OUTHS.26
OUTHS.21
OUTHS.16
OUTHS.15
OUTHS.14
OUTHS.13
OUTHS.12
OUTHS.11
OUTHS.10
OUTHS.9
OUTHS.8
OUTHS.7
OUTHS.6
OUTHS.5
OUTHS.4
OUTHS.3
OUTHS.2
OUTHS.1
OUTHS.0
OUTHS.17
OUTHS.16
OUTHS.15
OUTHS.14
OUTHS.13
OUTHS.12
OUTHS.11
OUTHS.10
OUTHS.9
OUTHS.8
OUTHS.7
OUTHS.6
OUTHS.5
OUTHS.4
OUTHS.3
OUTHS.2
OUTHS.1
OUTHS.0
OUTHS.18
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OUTHS.1
OUTHS.0
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OUTHS.88
OUTHS.89
OUTHS.90
OUTHS.91
OUTHS.92
OUTHS.93
OUTHS.94
OUTHS.95
OUTHS.96
OUTHS.97
OUTHS.98
OUTHS.99
OUTHS.100
OUTHS.101
OUTHS.102
OUTHS.103
OUTHS.104
OUTHS.105
OUTHS.106
OUTHS.107
OUTHS.108
C ARRANGE TIME HISTORY IN OUTPUT FORM
418 CONTINUE
DO 306 N=1,NDIS
WRITE(11,104) TA(N),(X(I,N),I=1,L)
306 CONTINUE
WRITE(11,105) (XM(I),I=1,L)
WRITE(11,106) (YM(I),I=1,L)
GC TO 420
419 CONTINUE
IF (KK1.EQ.2) WRITE(MT) IFF,KKK,L,KD,XX,*
GO TO 410
420 CONTINUE
304 CONTINUE

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RETURN
101 FORMAT(1BH TIME=2X,B(I8,1H)=I2,X)
102 FORMAT(1BH TIME=2X,B(I7,2H)=I3,I4H-V12)
103 FORMAT(1BH TIME=2,X,I8,1H-V12,X)
104 FORMAT(F8.4,X,BE12.4)
105 FORMAT(* MAXIMUM ABSOLUTE VALUES */,* MAXIMUM * ,BE12.4)
106 FORMAT(* MAXIMUM VALUE TIME=F11.5)
501 FORMAT(* TIME HISTORY FOR SELECTED DISPLACEMENT COMPONENTS*,  

      1      5H*** I3.37X,*FILE NO.* I5/  

      2      20*X*NODE NUMBERS AND DISPLACEMENT COMPONENTS*)  

502 FORMAT(* TIME HISTORY FOR SELECTED ACCELERATION COMPONENTS*,  

      1      5H*** I3.37X,*FILE NO.* '13//'
      2      20*X*NODE NUMBERS AND ACCELERATION COMPONENTS*)
503 FORMAT(* TIME HISTORY FOR SELECTED STRESS COMPONENTS*,  

      1      5H*** I5.44X,*FILE NO.* '3/
      2      20*X* ELEMENT TYPE ELEMENT NO.-STRESS 1A TO E1+YIELD (4)* /)
504 FORMAT(* TIME HISTORY FOR SELECTED WALL FORCE*,  

      1      5H*** I3.47X,*FILE NO.* '13//'
      2      20*X* WALL NO.-COMPONENTS,1-U,2-V,3=N,4=YBAR,5=ZBAR */)
END

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