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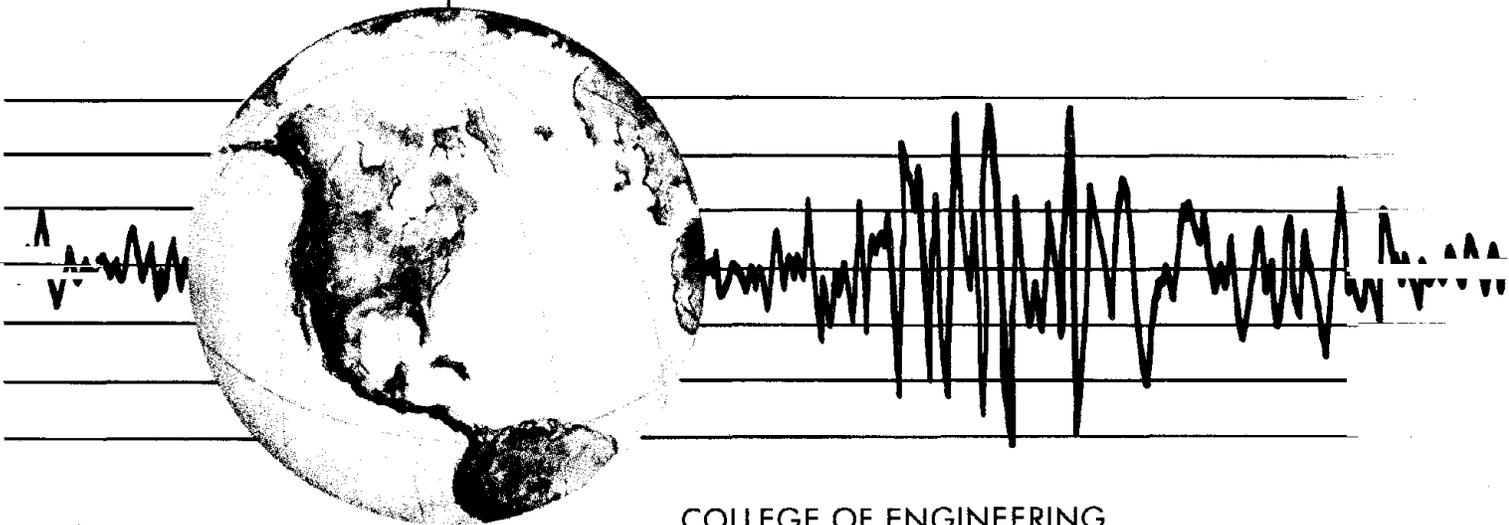
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# THEORETICAL AND EXPERIMENTAL STUDIES OF CYLINDRICAL WATER TANKS IN BASE ISOLATED STRUCTURES

by

MICHEL S. CHALHOUB  
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Report to the National Science Foundation



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Michel S. Chalhoub

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## ABSTRACT

This report presents the results obtained from an experimental study of two similar cylindrical water tanks, and a corresponding theoretical solution. One of the tanks was directly fixed to the earthquake simulator, the other was mounted on the base of a scaled nine-story steel structure. The structure was isolated on eight multilayered elastomeric bearings. Because the base accelerations were lower for the tank in the isolated structure, the dynamic pressure was reduced for this tank. Free surface water elevation was slightly higher because of the lower frequency that characterizes the motion of base isolated structures. This problem can be overcome by appropriate selection of the isolation system or by the addition of dampers at the locations of maximum water particle velocities. For the tank in the isolated structure, the accelerations and displacements at the tank rim were lower than for the tank directly fixed to the shake table. A theoretical solution developed from linear wave theory correlates very well with the experimental results. The advantages of using base isolation for large storage tanks are investigated.

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## CHAPTER ONE INTRODUCTION

Extensive work has been done on the response of internal equipment in structures subjected to ground motion [1]. An important advantage of base isolation is the protection of internal equipment in buildings located in seismic regions, against high accelerations transmitted from the ground and amplified in the structure. Studies have been made on oscillators simulating the contents of a building, where each oscillator was tuned to a chosen frequency [2].

Fluid containers are an important category of internal equipment for certain buildings and power plants. In the present research, the response of a cylindrical water tank fixed to the shake table is compared with that of a similar one installed at the bottom of a base isolated nine-story steel structure.

Quantities of interest in the response are the dynamic pressure at the tank walls, displacements and accelerations in the shell, and the water free surface deformation. These quantities were measured at certain locations. A detailed description of the experimental set-up is presented in the next chapter. From a structural viewpoint, it is of primary concern to study the stresses and deformations in the shell; however, in the present treatment we are mainly concerned with the loading exerted by the fluid on the container in terms of the dynamic pressure at the interface, and the effect of base isolation on the sloshing.

A theoretical solution is worked out and expressions for the dynamic pressure and the free surface elevation are proposed and compared with some experimental results. It is commonly accepted that the pressure at the interface consists of two components: an impulsive pressure and a convective pressure. The first component is due to the container wall accelerating against the fluid, the second component is due to the change in the fluid free surface elevation. The convective pressure is mainly attributed to the sloshing resulting from the formation of forced waves in the tank. Sloshing in tanks is a phenomenon of relatively low frequency (the first mode is predominant): for this reason base isolation might increase it, causing a slightly higher convective pressure component. However, base isolation has more effect on the impulsive component and thus leads to a much lower resultant dynamic pressure.

This study provides some insight into the advantages of mounting large tanks in seismic regions directly on base isolators, in order to reduce the hydrodynamic loading on the shell. The approach should be studied further using the earthquake simulator. Meanwhile, since the expressions presented in Chapter Three show good agreement with the experimental results, they can be used to predict the response of the fluid in a base isolated rigid tank.

Finally, the validity of the theoretical results and the ease of the numerical determination of the convolution integrals involved in the final expressions, suggest the direct use of the hydrodynamic solution instead of converting the problem into an equivalent mechanical analog that provides a modeling of only the overall behavior of the contained fluid.

Future research based on this work should include the effects of the tank flexibility on the response of the fluid. It should also include studies of the response of tanks independently isolated and of tanks isolated as groups on a simple isolation pad.

## CHAPTER TWO EXPERIMENTAL SET-UP AND TEST PROGRAM

### 1. Structural Model

The structural model used is the nine-story K-braced steel frame described in [3]. The structure was mounted on an isolation system consisting of eight elastomeric bearings provided by the Malaysian Rubber Producers' Research Association (MRPRA SET #1). Each bearing had a horizontal stiffness of about 1.1 kips/in. at 60% shear strain and a vertical stiffness of about 420 kips/in. The extremely high vertical stiffness provides a conventional support condition in the vertical direction and the high horizontal flexibility causes the entire structure to move like a rigid body at a very low frequency. This arrangement reduces drastically the ground accelerations transmitted to the structure. Since the total weight of the model was 91 kips, the natural frequency of the base isolated structure was 0.97 Hz.

### 2. Tanks

Two similar tanks were used for the purpose of comparison. One of them was directly fixed on the shake table, the other was fixed on the base of the isolated structure (Figures 2.1 and 2.2). They were cylindrical steel tanks, 1/25 in. thick, 2 feet in height and 4 feet in diameter. Each tank was fixed at its bottom by six 2×2×1/4 in. angles; two of the angles were along the diameter of excitation and the four others were installed by pairs at 45° from that diameter. The tank on the structure was mounted on a 1 in. thick steel plate welded to the bottom flanges of the two base beam girders.

For each tank, eight Piezo-electric pressure transducers were used to measure the dynamic water pressure. Six of them were installed along two vertical lines in the plane of excitation at the bottom, at mid-height and near the water free surface, while two others were installed at mid-height in a plane perpendicular to the plane of excitation. Two accelerometers were installed at the shell rim, at the north and west sides respectively, to measure horizontal accelerations. Two DCDTs were installed at the north and south sides of the shell rim, respectively, to measure rim displacements relative to the tank base. The locations of these instruments and their channel numbers are shown in Figures 2.3 to 2.6. A list of the channels with the symbols used and a brief description is given Table 2.1.

The free surface water elevation was measured at the shell wall at the north and south sides, using water level gages. The gages consisted of two parallel conductor wires connected to an electric bridge. The resistance in the circuit varied with the water elevation and thus was converted from  $m\Omega$  to inches.

### **3. Input Signals.**

The shake table and the signals used were derived from previously recorded ground motions, these are described in [3]. When the tanks were filled with water, these signals were applied at spans ranging from 50 to a maximum of 100 due to water spilling for spans larger than 100. The spilling problem can be solved by addition of a cover, however this problem is not pursued in the present work. The data processed for comparison with theoretical prediction corresponded to spans of 50 and 75. Table displacement and acceleration time histories are shown with the experimental results in Chapter 4.

### **4. Test Sequence**

Initially the model was loaded with a different mass distribution, and weighed about 120 kips. The bearings on which it was mounted were of the MRPRA SET #3 type, and had a horizontal stiffness of 0.8 kip/in. each at 60% shear strain. For this arrangement, the natural frequency of the isolated structure was of 0.7 Hz, which is very close to the frequency of the first sloshing mode in the tanks. A first test sequence was performed, starting with sine input signals at 1.5 Hz. The input frequency was then lowered to 0.7 Hz and 0.8 Hz. For this set of runs, overtopping of the water gages and sometimes spilling occurred because of resonance. The files corresponding to this set-up were used only for observation and were not compared with theoretical prediction.

The structural model was then lightened to 91 kips and the bearings replaced by eight MRPRA SET #1 bearings which had a horizontal stiffness of 1.1 kips/in. each at 60%. This resulted in a slightly higher base isolated natural frequency. Ground motions were used at various spans but spilling still occurred for some spans higher than 100. The files used for comparison with the

theoretical solution were chosen from the ones of spans lower than 75 and are listed in Table 2.2 under "last test sequence".

It is to be noted, however, that the time scaling was dictated by the model and was fixed to 2, and that if the tanks were to be tested without the structural model, the time scaling could be much higher.

Table 2.1 Channel numbering

CHANNEL NUMBERING FOR WATER TANK INSTRUMENTATION

Channel Name	Units	Remarks
84	ac1 tb g's	north accelerometer for tank on table
85	ac2 tb g's	west accelerometer for tank on table
86	ac1 md g's	north accelerometer for tank on model
87	ac2 md g's	west accelerometer for tank on model
88	ds1 tb inches	north DCDT for tank on table
89	ds2 tb inches	south DCDT for tank on table
90	ds1 md inches	north DCDT for tank on model
91	ds2 md inches	south DCDT for tank on model
92	wg1 tb inches	north water gage for tank on table
93	wg2 tb inches	south water gage for tank on table
94	wg1 md inches	north water gage for tank on model
95	wg2 md inches	south water gage for tank on model
96	pz1 tb psi	north bottom pressure for tank on table
97	pz2 tb psi	north mid-hi pressure for tank on table
98	pz3 tb psi	north top pressure for tank on table
99	pz4 tb psi	south bottom pressure for tank on table
100	pz5 tb psi	south mid-hi pressure for tank on table
101	pz6 tb psi	south top pressure for tank on table
102	pz7 tb psi	west mid-hi pressure for tank on table
103	pz8 tb psi	east mid-hi pressure for tank on table
104	pz1 md psi	north bottom pressure for tank on model
105	pz2 md psi	north mid-hi pressure for tank on model
106	pz3 md psi	north top pressure for tank on model
107	pz4 md psi	south bottom pressure for tank on model
108	pz5 md psi	south mid-hi pressure for tank on model
109	pz6 md psi	south top pressure for tank on model
110	pz7 md psi	west mid-hi pressure for tank on model
111	pz8 md psi	east mid-hi pressure for tank on model

Table 2.2 Test sequence.

TESTING OF WATER TANKS.  
TANK DIAMETER: 48"  
TANK HEIGHT : 24"  
WATER DEPTH : 17.5"  
ONE TANK MOUNTED ON THE ISOLATED BASE  
ONE TANK MOUNTED ON THE SHAKE TABLE

-----  
1st TEST SEQUENCE

BEARINGS USED : MRPRA #3, Kh=0.8 k/in  
TOTAL WEIGHT OF MODEL : 120 KIPS

FILENAME	SIGNAL	TIME	RATE	REMARKS
861009.01	sine	20 secs	.005	1.54 hz
861009.02	sine	10 secs	.005	0.9 hz
861009.03	sine	20 secs	.005	0.9 hz and stop
861009.04	sine	10 secs	.005	0.7 hz sph=80
861009.05	sine	10 secs	.005	0.75 hz sph=60
861009.06	sine	10 secs	.005	0.8hz sph=8
861009.07	sine	10 secs	.005	0.8 hz sph=10
861009.08	sine	10 secs	.005	0.8 hz sph=12
861009.09	random30.d	32 secs	.005	sph=300
861009.10	sine	10 secs	.005	3.0 hz sph=70
861010.01	sine	10 secs	.005	2.0 hz sph=90
861010.02	ec2	18 secs	.005	sph=150

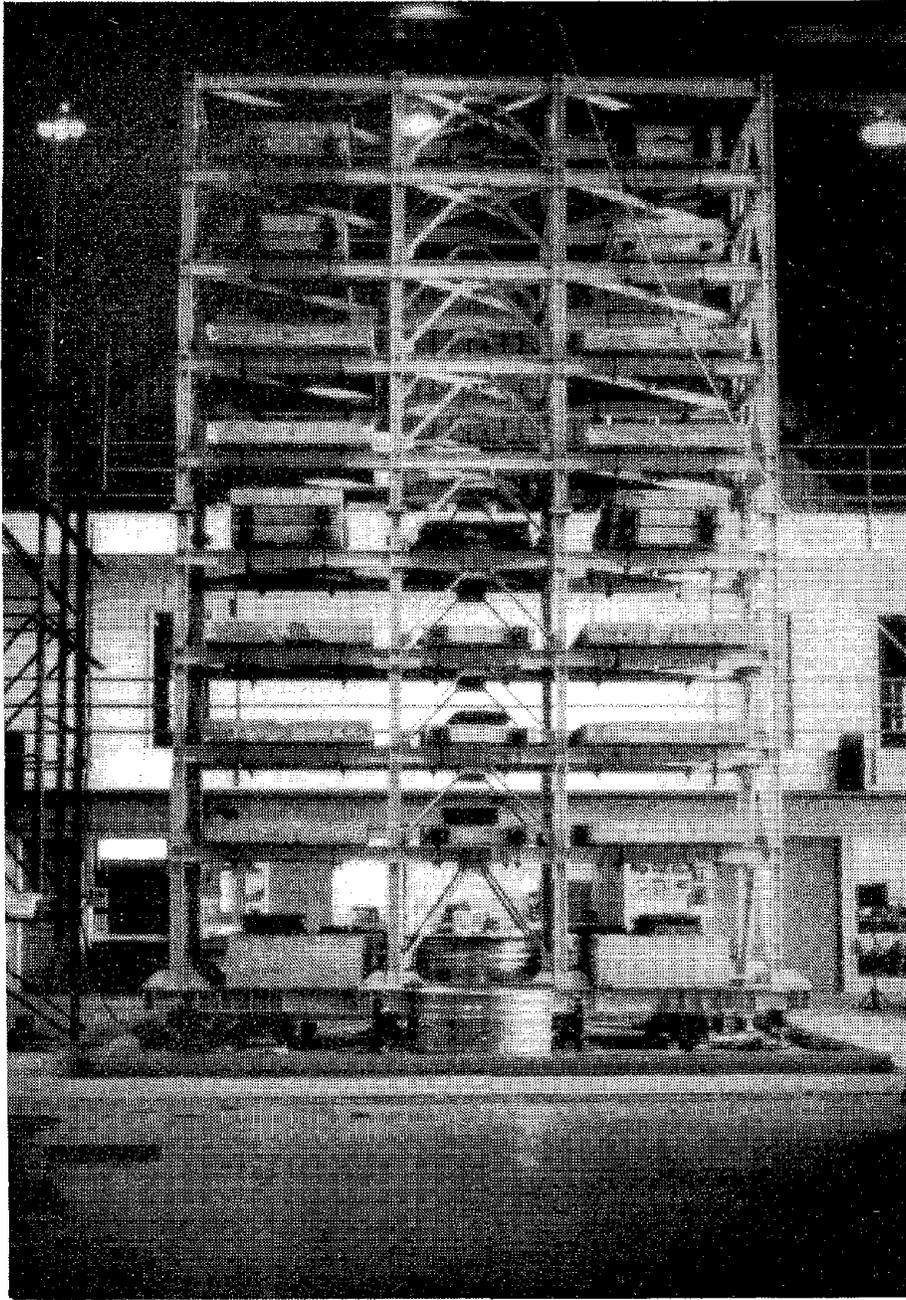
Excessive spilling occurred for both tanks since the sine inputs had frequencies close to the first sloshing mode frequency.

-----  
LAST TEST SEQUENCE

BEARINGS USED : MRPRA #1, Kh=1.1 k/in  
TOTAL WEIGHT OF MODEL : 91 KIPS

The files listed below were used for comparison with theoretical solution.

861023.01	ec2	19 secs	.005	sph=50	ts=1/4
861023.02	ec2	19 secs	.005	sph=75	ts=1/4
861023.03	ec2	19 secs	.005	sph=70	ts=1/4
861023.04	sf2	12 secs	.005	sph=50	ts=1/4
861023.05	sf2	12 secs	.005	sph=150	ts=1/4
861023.06	sf2	12 secs	.005	sph=75	ts=1/4
861023.07	pac2	12 secs	.005	sph=50	ts=1/4
861023.08	pac2	12 secs	.005	sph=75	ts=1/4
861023.09	park2	14 secs	.005	sph=50	ts=1/4
861023.10	park2	14 secs	.005	sph=75	ts=1/4
861023.11	taft2	19 secs	.005	sph=50	ts=1/4
861023.12	taft2	19 secs	.005	sph=75	ts=1/4
861023.13	bucl	12 secs	.005	sph=50	ts=1/4
861023.14	bucl	12 secs	.005	sph=25	ts=1/4
861023.15	sct	35 secs	.005	sph=25	ts=1/4
861023.16	sct	35 secs	.005	sph=50	ts=1/4



**Figure 2.1** 1/4 scale nine-story structural model and the tanks. Tank #1 fixed to the shake table, tank #2 fixed to the isolated base.

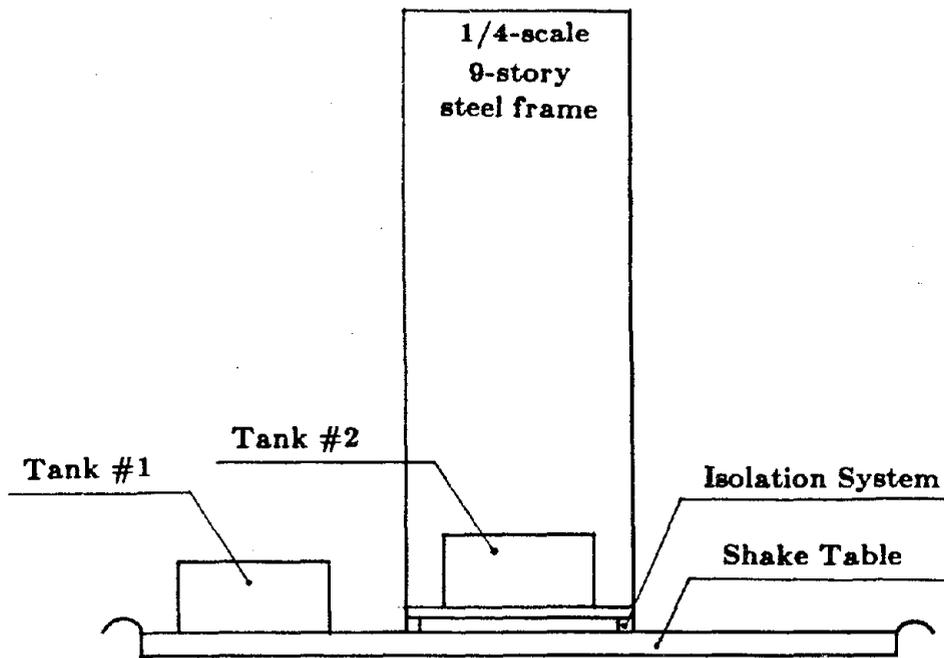
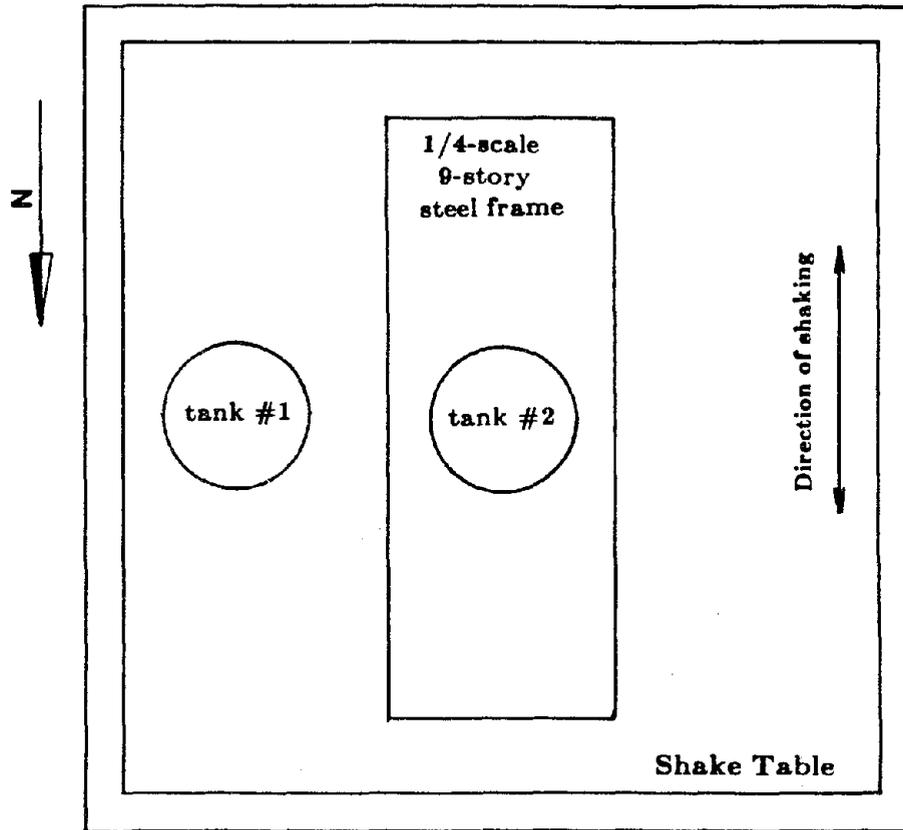


Figure 2.2 Plan view and section view of the experimental set-up.

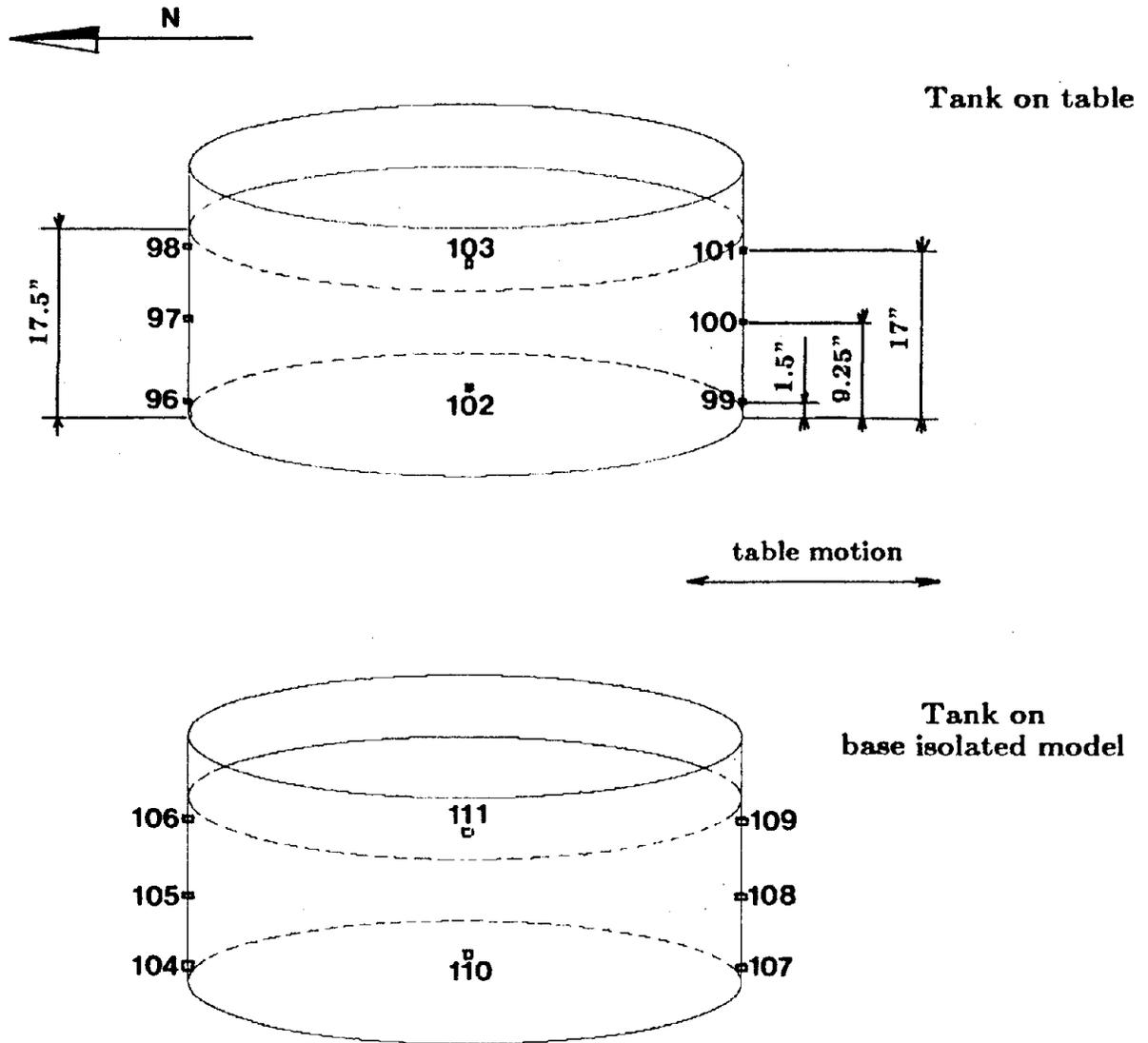


Figure 2.3 Location and channel numbering of pressure transducers.

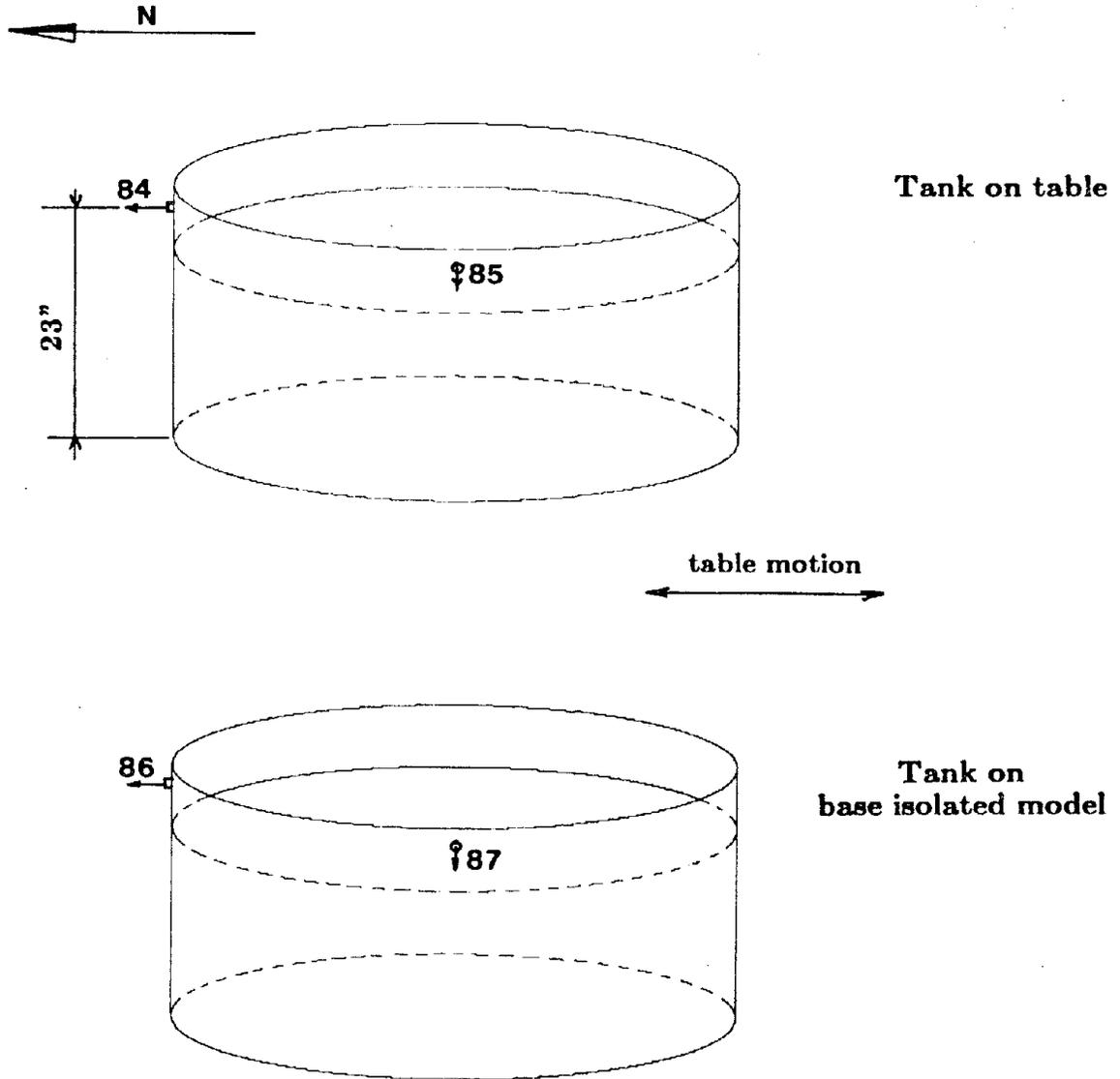


Figure 2.4 Location and channel numbering of accelerometers.

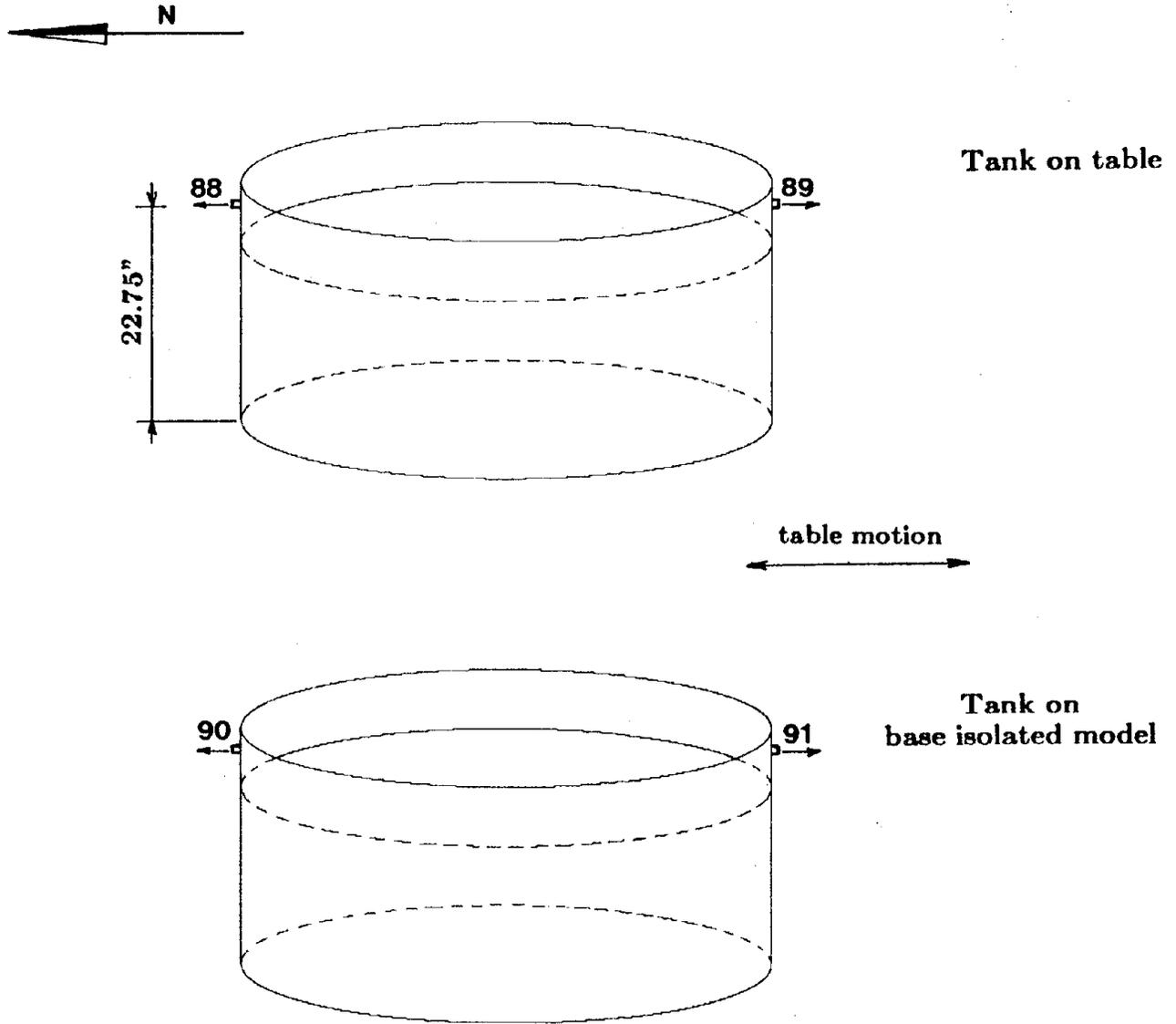


Figure 2.5 Location and channel numbering of DCDTs.

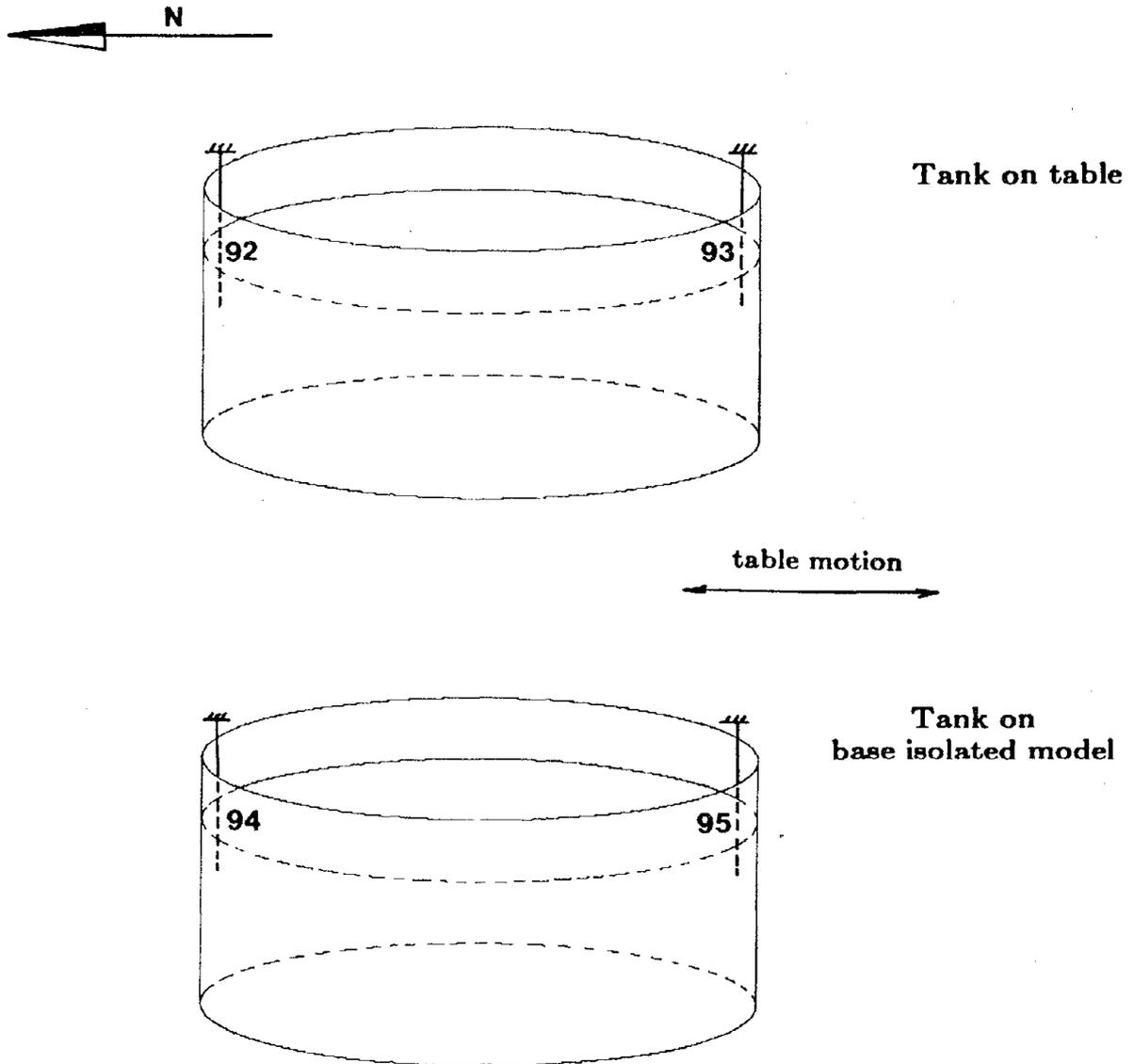


Figure 2.6 Location and channel numbering of water level gages.

### CHAPTER THREE THEORETICAL SOLUTION

#### 1. General.

The equations of fluid motion for many applications have been solved with various boundary conditions [4, 5]. Of particular interest were fluid containers of cylindrical shape because of their numerous applications ranging from ground supported containers to aircraft fuel tanks. The problem of forced oscillations of a fluid induced by the vibrations of its container is old and equivalent mechanical models have been proposed [6].

In the following treatment, the equations of motion relevant to our problem will be stated for completeness, then solved using commonly accepted boundary conditions for cylindrical fluid containers. It is found, however, that the solution can be carried out differently in order to yield an expression for the quantities of interest, such as dynamic pressure at the tank wall and water surface deformation, in terms of convolution integrals that represent the responses of single degree of freedom oscillators of frequencies equal to the ones of the fluid sloshing modes, and subjected to the ground motion. These integrals can be easily determined, and the exact solution involving the first few modes can be directly used. This approach is much closer to the real behavior than the mechanical analogs currently used in tank design. Furthermore, the expressions for the pressure and water elevation time histories proposed here are simple to evaluate at any location in the tank, and render the conversion of the hydrodynamics problem into a mechanical analogy unnecessary.

It is shown that the dynamic pressure at the tank wall is highly influenced by the ground acceleration, and hence can be drastically reduced by base isolation.

#### 2. Governing Equations - Boundary Conditions.

Consider a cylindrical tank of radius  $R_0$  and height  $H$  filled to a depth  $h$  with a fluid of density  $\rho$ , and subjected to a horizontal translation described by a function of time  $v_g(t)$ . A cylindrical coordinate system  $(r, \phi, z)$  is used and is shown in Figure 3.1.  $u$ ,  $v$  and  $w$  are the water particle velocities in the  $r$ ,  $\phi$  and  $z$  directions, respectively. Assuming that the fluid is nonviscous, denoting by  $P$  the total fluid pressure and by  $F$  an external force potential, the equations of motion for the water particles satisfy Euler's equations:

$$\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial r} + \frac{v}{r} \frac{\partial u}{\partial \phi} + w \frac{\partial u}{\partial z} - \frac{v^2}{r} = -\frac{1}{\rho} \frac{\partial P}{\partial r} - \frac{\partial F}{\partial r} \quad (3.1a)$$

$$\frac{\partial v}{\partial t} + u \frac{\partial v}{\partial r} + \frac{v}{r} \frac{\partial v}{\partial \phi} + w \frac{\partial v}{\partial z} + \frac{vu}{r} = -\frac{1}{\rho r} \frac{\partial P}{\partial \phi} - \frac{\partial F}{\partial \phi} \quad (3.1b)$$

$$\frac{\partial w}{\partial t} + u \frac{\partial w}{\partial r} + \frac{v}{r} \frac{\partial w}{\partial \phi} + w \frac{\partial w}{\partial z} = -\frac{1}{\rho} \frac{\partial P}{\partial z} - \frac{\partial F}{\partial z} \quad (3.1c)$$

For an irrotational fluid flow, the equations expressing the irrotationality condition in polar coordinates are

$$\frac{1}{r} \frac{\partial w}{\partial \phi} - \frac{\partial v}{\partial z} = 0 \quad (3.2a)$$

$$\frac{\partial u}{\partial z} - \frac{\partial w}{\partial r} = 0 \quad (3.2b)$$

$$\frac{\partial v}{\partial r} + \frac{v}{r} - \frac{1}{r} \frac{\partial u}{\partial \phi} = 0 \quad (3.2c)$$

If  $\vec{V}$  denotes the water particle velocity vector, then  $\text{curl } \vec{V} = \vec{0}$  and there exists a scalar function  $\Omega(r, \phi, z, t)$  that would define a velocity potential, or such that  $\vec{V} = \nabla \Omega$ . This equation is equivalent to three scalar equations, namely:

$$-\frac{\partial \Omega}{\partial r} = u \quad (3.3a)$$

$$-\frac{1}{r} \frac{\partial \Omega}{\partial \phi} = v \quad (3.3b)$$

$$-\frac{\partial \Omega}{\partial z} = w \quad (3.3c)$$

Substituting equations (3.2a,b,c) and (3.3a,b,c) into equations (3.1a,b,c), and integrating the three resulting equations with respect to  $r$ ,  $\phi$  and  $z$ , respectively, we have

$$-\frac{\partial \Omega}{\partial t} + \frac{P}{\rho} + F + \frac{1}{2} (u^2 + v^2 + w^2) = G(\phi, z, t)$$

$$-\frac{\partial \Omega}{\partial t} + \frac{P}{\rho} + F + \frac{1}{2} (u^2 + v^2 + w^2) = H(r, z, t)$$

$$-\frac{\partial \Omega}{\partial t} + \frac{P}{\rho} + F + \frac{1}{2} (u^2 + v^2 + w^2) = L(r, \phi, t)$$

where G, H and L are arbitrary functions. Noting that the left hand sides are the same expression and that r,  $\phi$  and z are independent variables, the arbitrary functions on the right hand side are all the same function of time only, G(t). Thus, the preceding treatment reduces to the Bernouilli equation

$$-\frac{\partial \Omega}{\partial t} + \frac{P}{\rho} + F + \frac{1}{2} (u^2 + v^2 + w^2) = G(t) \quad (3.4)$$

A function f(t) is defined such that  $G(t) = \frac{df(t)}{dt}$ , and when substituted in equation (3.4), a new velocity potential  $\Theta = \Omega + f(t)$  can be defined. The total pressure P(r, $\phi$ ,z,t) is then expressed as

$$P(r,\phi,z,t) = \rho \left( \frac{\partial \Theta}{\partial t} - \frac{1}{2} \{u^2 + v^2 + w^2\} - F \right)$$

Linear wave theory is developed for small velocity magnitude and neglects the term  $V^2$ ; thus,

$$P(r,\phi,z,t) = \rho \left( \frac{\partial \Theta}{\partial t} - F \right)$$

Furthermore the only body force acting on the fluid is gravity and F can be replaced by -gz where g is the gravitational constant:

$$P(r,\phi,z,t) = \rho \left( \frac{\partial \Theta}{\partial t} + gz \right) \quad (3.5)$$

Note that the total pressure is here expressed as the sum of the static component  $\rho gz$  and the dynamic component  $\rho \frac{\partial \Theta}{\partial t}$ .

From the definition of  $\Theta$ , equations (3.3a,b,c) can be rewritten as

$$-\frac{\partial \Theta}{\partial r} = u \quad (3.6a)$$

$$-\frac{1}{r} \frac{\partial \Theta}{\partial \phi} = v \quad (3.6b)$$

$$-\frac{\partial \Theta}{\partial z} = w \quad (3.6c)$$

For constant fluid density, the condition of continuity in cylindrical coordinates is written

$$\frac{\partial u}{\partial r} + \frac{u}{r} + \frac{1}{r} \frac{\partial v}{\partial \phi} + \frac{\partial w}{\partial z} = 0 \quad (3.7)$$

When equations (3.6a,b,c) are substituted in equation (3.7), the Laplace equation to be solved for the velocity potential is obtained:

$$\frac{\partial^2 \Theta}{\partial r^2} + \frac{1}{r} \frac{\partial \Theta}{\partial r} + \frac{1}{r^2} \frac{\partial^2 \Theta}{\partial \phi^2} + \frac{\partial^2 \Theta}{\partial z^2} = 0 \quad (3.8)$$

At the free surface, the pressure P is constant at all points. This is expressed by setting to zero the total derivative of P with respect to time:

$$\frac{DP}{Dt} = \frac{\partial P}{\partial t} + u \frac{\partial P}{\partial r} + \frac{v}{r} \frac{\partial P}{\partial \phi} + w \frac{\partial P}{\partial z} = 0 \quad (3.9)$$

at the level  $z=0$ . Note that if the free surface deflection is expressed by a function  $\eta(r,\phi,z,t)$  the above condition should be satisfied at  $z = \eta$ . Setting the condition at  $z=0$  is, however, a commonly accepted approximation. Substituting equation (3.9) into equation (3.8) yields the free surface boundary condition

$$\frac{\partial^2 \Theta}{\partial t^2} + g \frac{\partial \Theta}{\partial z} = 0 \quad \text{at } z = 0 \quad (3.10)$$

At the bottom of the tank, the fluid particles have zero vertical velocity. This results in the bottom boundary condition

$$\frac{\partial \Theta}{\partial z} = 0 \quad \text{at } z = -h \quad (3.11)$$

At the tank lateral surface the water particles and the tank wall must have the same velocity. Since the tank is considered rigid, its displacement in the x direction is described by the ground motion  $v_g(t)$ .

Along r the tank wall velocity is then  $v_{g,t}(t)\cos\phi$  and we can write

$$u = \frac{\partial \Theta}{\partial r} = v_{g,t}(t) \cos\phi \quad \text{at } r = R_o \quad (3.12)$$

### 3. Solution.

Equation (3.8) can be solved by separation of variables. If the velocity potential  $\Theta$  is considered as the product of three functions at any given time  $t$ , we have

$$\Theta(r,\phi,z) = R(r) \Phi(\phi) Z(z)$$

Substituting in equation (3.8) and dividing by  $R \Phi Z$  we have

$$\frac{\Phi_{,\phi\phi}}{\Phi} = -r^2 \frac{Z_{,zz}}{Z} - r^2 \frac{R_{,rr}}{R} - r \frac{R_{,r}}{R}$$

Since  $\Phi$  is a function of  $\phi$  only while  $Z$  and  $R$  do not depend on  $\phi$ ,  $\frac{\Phi_{,\phi\phi}}{\Phi}$  is equal to a constant. Note that, in order to determine the sign of this constant, we must have  $\Phi(2\pi) = \Phi(0)$ . Thus  $\Phi$  has to be a combination of trigonometric functions and the constant is then negative; and the above equation yields

$$\frac{\Phi_{,\phi\phi}}{\Phi} = -k^2$$

which has the solution

$$\Phi(\phi) = K_1 \cos k\phi + K_2 \sin k\phi$$

where  $K_1$  and  $K_2$  are arbitrary constants and  $k \neq 0$  since  $\Phi(\phi)$  must not be constant. Also we have

$$\frac{Z_{,zz}}{Z} = \frac{k^2}{r^2} - \frac{1}{r} \frac{R_{,r}}{R} - \frac{R_{,rr}}{R} = n^2 \quad (3.13)$$

where  $n$  is a constant by the same reasoning followed earlier. If  $n = 0$ , then

$$Z_0 = K_3 z + K_4$$

where  $K_3$  and  $K_4$  are arbitrary constants. The equation in  $R$ , for this case ( $n = 0$ ) is rewritten as

$$r R_{,r} + r^2 R_{,rr} - k^2 R = 0$$

for which  $R(r) = r^m$  is a general solution. When  $R(r) = r^m$  is substituted into the above expression and the indicial equation solved for  $m$ , we find  $m = \pm k$  and

$$R = K_5 r^k + K_6 r^{-k}$$

For the case where  $n \neq 0$ , equation (3.13) in  $Z(z)$  has the solution

$$Z(z) = K_7 \cosh nz + K_8 \sinh nz$$

and the equation in  $R(r)$  is a Bessel equation of order  $k$  and parameter  $n$

$$r^2 R_{,rr} + r R_{,r} + (n^2 r^2 - k^2)R = 0$$

Its general solution has the form

$$R(r) = K_9 J_k(nr) + K_{10} Y_k(nr)$$

and since for  $r = 0$  the velocity potential is finite,  $K_{10} = 0$ .

Finally, the general solution for the velocity potential can be written as

$$\Theta = \Theta_0 + \Theta_n$$

where  $\Theta_0$  and  $\Theta_n$  are the two solutions corresponding to the cases  $n = 0$  and  $n \neq 0$ , respectively:

$$\Theta_0(r, \phi, z) = (K_1 \cos k\phi + K_2 \sin k\phi)(K_3 z + K_4)(K_5 r^k + K_6 r^{-k})$$

$$\Theta_n(r, \phi, z) = (K_1 \cos k\phi + K_2 \sin k\phi)(K_7 \cosh nz + K_8 \sinh nz)(K_9 J_k(nr))$$

The Boundary Condition in equation (3.12) is satisfied if

$$(K_1 \cos k\phi + K_2 \sin k\phi)(K_3 z + K_4)(K_5 k R_0^{k-1} + K_6 k R_0^{-k-1}) = v_{g,t}(t) \cos \phi \quad (3.14a)$$

and

$$(K_1 \cos k\phi + K_2 \sin k\phi)(K_3 \cosh nz + K_4 \sinh nz) K_9 J_k'(nR_0) = 0 \quad (3.14b)$$

Equation (3.14a) yields

$$k = 1, \quad K_2 = K_3 = K_6 = 0,$$

$$K_1 K_4 K_5 = v_{g,t}(t), \quad \text{and} \quad \Theta_0 = v_{g,t}(t) r \cos \phi;$$

while equation (3.14b) yields after substitution of the determined coefficients

$$(K_1 \cos \phi)(K_3 \cosh nz + K_4 \sinh nz) K_9 J_1'(nR_0) = 0$$

Therefore,

$$J_1'(nR_0) = 0$$

The above equation has an infinite number of real roots for which the derivative of the Bessel function of the first kind and first order is zero. If we denote by  $n_i R_0 = \alpha_i$  the  $i^{\text{th}}$  root  $\alpha_i$  of  $J_1'(\alpha) = 0$ , we have (see Table 3.1)

$$\alpha_i = n_i R_0 = 1.84118, \quad 5.33144, \quad 8.53632, \quad 11.70600, \quad 14.86359, \\ 18.01553, \quad 21.16437, \quad 24.31133, \quad 27.45705, \quad 30.60192, \quad \text{etc} \dots$$

and the general expression for the velocity potential becomes

$$\Theta = \sum_{i=1}^{i=\infty} K_i J_1(n_i r) (K_7 \cosh n_i z + K_8 \sinh n_i z) \cos \phi + v_{g,t}(t) r \cos \phi \quad (3.14c)$$

The Boundary Condition for the bottom of the tank, in equation (3.11), applied to the above expression yields

$$K_7 \sinh n_i h = K_8 \cosh n_i h$$

Substituting for  $K_8$  in equation (3.14c) and rearranging the hyperbolic functions, we have

$$\Theta = \sum_{i=1}^{i=\infty} K_i J_1(n_i r) \frac{\cosh n_i (z+h)}{\cosh n_i h} \cos \phi + v_{g,t}(t) r \cos \phi \quad (3.15)$$

Note that up to this point in the preceding treatment, the equations were developed for a given time  $t$  and hence the constants can be functions of time. In order to determine  $K_i(t)$ , we apply the free surface boundary condition given by equation (3.10) to equation (3.15):

$$\frac{\partial^2 \Theta}{\partial t^2} = \sum_{i=1}^{i=\infty} K_{i,tt}(t) J_1(n_i r) \frac{\cosh n_i (z+h)}{\cosh n_i h} \cos \phi + v_{g,tt}(t) r \cos \phi$$

and

$$g \frac{\partial \Theta}{\partial z} = g \sum_{i=1}^{i=\infty} K_i(t) n_i J_1(n_i r) \frac{\sinh n_i (z+h)}{\cosh n_i h} \cos \phi$$

The sum of the two above expressions for  $z=0$  yields, after we simplify by  $\cos\phi$ ,

$$\sum_{i=1}^{i=\infty} \left[ K_{i,tt}(t) + (g n_i \tanh n_i h) K_i(t) \right] J_1(n_i r) + v_{g,tt} r = 0 \quad (3.16)$$

Equation (3.16) can now be easily solved for the free vibrations if the term  $v_{g,tt}(t)$  representing the forcing function is discarded.  $K_i(t)$  will then be a trigonometric function and the frequencies of free oscillations are  $(g n_i \tanh n_i h)^{1/2}$ . In the following, however, the forced oscillations solution will be pursued. By denoting the first expression in the brackets by  $T(t)$ ,

$$T(t) = \sum_{i=1}^{i=\infty} \left[ K_{i,tt}(t) + (g n_i \tanh n_i h) K_i(t) \right]$$

equation (3.16) can be solved for  $T(t)$  by noticing that the Bessel solutions satisfy certain Boundary Conditions that make them orthogonal on the interval  $\{0, R_o\}$ :

$$A_1 J_1(n_i 0) - B_1 J_1'(n_i 0) = 0 \quad \text{with } B_1 = 0$$

and

$$A_2 J_1(n_i R_o) - B_2 J_1'(n_i R_o) = 0 \quad \text{with } A_2 = 0$$

To use the orthogonality of the Bessel functions, expression (3.16) is multiplied by  $r J_1(n_j r)$ , so that

$$\sum_{i=1}^{i=\infty} T(t) J_1(n_i r) J_1(n_j r) r + v_{g,tt} r^2 J_1(n_j r) = 0$$

and integrated over the interval  $\{0 \text{ and } R_o\}$ . When the only non zero term in the summation is retained, we find

$$T(t) \int_0^{R_o} J_1^2(n_j r) r dr + v_{g,tt} \int_0^{R_o} r^2 J_1(n_j r) dr = 0$$

After the two integrals are evaluated, and the above expression solved for  $T(t)$ , we have

$$T(t) = v_{g,tt}(t) \chi_j$$

or

$$K_{j,tt}(t) + \lambda_j^2 K_j(t) = v_{g,tt}(t) \chi_j \quad (3.17)$$

where

$$\lambda_j = (g n_j \tanh n_j h)^{1/2} \quad \text{and} \quad \chi_j = \frac{2 R_o^2 J_2(n_j R_o) n_j}{J_1^2(n_j R_o) (1 - (n_j R_o)^2)}$$

The expression for  $\chi_j$  is further simplified by the use of the Bessel function identity

$$J_{v+1}(x) = \frac{v}{x} J_v(x) - J_v'(x)$$

and by noting that  $J_1'(n_j R_o) = 0$ . The expression for  $\chi_j$  becomes

$$\chi_j = \frac{2 R_o}{J_1(n_j R_o) (1 - (n_j R_o)^2)}$$

Equation (3.17) for  $K_j(t)$  is transformed to the Laplace domain and the initial conditions on  $v_g(t)$  are used. The expression is then solved algebraically for  $\bar{K}_j(s)$  where  $s$  is the Laplace transform variable.

The following two properties of the Laplace transform,

$$L\{f''(t)\} = s^2 L\{f(t)\} - s f(0) - f'(0)$$

and

$$L\{f'''(t)\} = s^3 L\{f(t)\} - s^2 f(0) - s f'(0) - f''(0)$$

are used, together with the assumption that the shake table, and hence the structural model motions, start from rest; so that  $v_g(0) = v_{g,t}(0) = v_{g,tt}(0) = 0$ . Since the water particle velocities are initially zero,  $\Theta(t=0) = 0$ , and since the dynamic pressure in the fluid is initially null,  $\frac{\partial \Theta}{\partial t}(t=0) = 0$ , and this yields  $K_j(0) = K_{j,t}(0) = 0$ . When the transform of equation (3.17) is solved for  $\bar{K}_j$ , it yields

$$\bar{K}_j(s) = \frac{s^3 \bar{v}_g(s)}{s^2 + \lambda_j^2} \chi_j \quad (3.18)$$

Equation (3.18) describes the response of a physical system to a given excitation, and the relation between the input and the output for zero initial conditions of the excitation  $v_g(t)$  has the form

$$\bar{K}_j(s) = \frac{\bar{v}_g(s)}{TR_j(s)}$$

where  $TR_j(s)$  is a transfer function. Furthermore since the present system is linear,  $TR_j(s)$  is simply the quotient of two polynomials in  $s$  :

$$TR_j(s) = \frac{s^2 + \lambda_j^2}{\chi_j s^3}$$

The Laplace transformation of the product of two functions of the same variable is expressed as

$$L\{f(t)\} L\{g(t)\} = L\left\{\int_0^t f(t-\tau)g(\tau) d\tau\right\} = L\{f(t) * g(t)\} \quad (\text{notation})$$

To apply the above theorem, the convolution or Faltung integral theorem, to equation (3.18), we rewrite it in the form

$$\bar{K}_i(s) = \left(\frac{s}{s^2 + \lambda_i^2}\right)(s^2 \bar{v}_g(s))\chi_i = L\{\cos \lambda_i t\} L\{v_{g,tt}(t)\} \chi_i$$

Then

$$K_i(t) = \int_0^t \chi_i \cos \lambda_i(t-\tau) v_{g,tt}(\tau) d\tau \quad (3.19)$$

Finally the velocity potential takes the form

$$\begin{aligned} \Theta(r, \phi, z, t) = \sum_{i=1}^{i=\infty} \chi_i \left[ \int_0^t \cos \lambda_i (t - \tau) v_{g,tt}(\tau) d\tau \right] J_1(n_i r) \frac{\cosh n_i (z + h)}{\cosh n_i h} \cos \phi \\ + v_{g,t}(t) r \cos \phi \end{aligned} \quad (3.20)$$

The dynamic pressure and the water surface deformation can now be expressed from the velocity potential. The dynamic pressure in the fluid is given by

$$\begin{aligned} P_d(r, \phi, z, t) = \rho \frac{\partial \Theta}{\partial t} \\ = \rho \cos \phi \sum_{i=1}^{i=\infty} \chi_i \left[ v_{g,tt}(t) - \lambda_i \int_0^t \sin \lambda_i (t - \tau) v_{g,tt}(\tau) d\tau \right] J_1(n_i r) \frac{\cosh n_i (z + h)}{\cosh n_i h} \\ + \rho v_{g,tt}(t) r \cos \phi \end{aligned} \quad (3.21)$$

The pressure at the tank wall in the plane of excitation is obtained for  $r = R_o$  and  $\phi = 0$ . Rearranging the terms in  $v_{g,tt}(t)$ , we have

$$\frac{P_d}{\rho R_o} = v_{g,tt}(t) \left( 1 + \sum_{i=1}^{i=\infty} C_i \right) - \sum_{i=1}^{i=\infty} C_i \lambda_i I_i(t) \quad (3.22)$$

where

$$C_i = \frac{2}{1 - (n_i R_o)^2} \frac{\cosh n_i (z + h)}{\cosh n_i h} \quad (3.23)$$

and

$$I_i(t) = \int_0^t \sin \lambda_i (t - \tau) v_{g,tt}(\tau) d\tau \quad (3.24)$$

The convolution integrals  $I_i(t)$  can be directly used without transforming the oscillating fluid into a mechanical equivalent system, since they each represent the undamped response of a single degree of

freedom oscillator of frequency  $\lambda_i$ , subjected to ground acceleration  $v_{g,tt}(t)$ . Two points of interest are to be noted:

(1) The coefficients  $C_i$  and  $\lambda_i C_i$  which control the rate of convergence of the solution decay very rapidly especially at larger depths  $z$ ; and the convolution integrals are bounded with the maximum value of an  $I_i(t)$  being the spectral pseudo-velocity response for a single degree of freedom oscillator of frequency  $\lambda_i$ . Table 3.2 lists those coefficients for the radius  $R_o = 24$  in. and for  $z$  varying from 0 in. at the water free surface to -17.5 in. at the tank bottom. These numerical values correspond to the tanks used in the experiment. The coefficients are highest at the water surface and decrease with depth; this decrease depends on the mode number. For the first mode,  $C_1$  at the bottom is 48% of  $C_1$  at the water surface, while for the second mode,  $C_2$  at the bottom is about 4% of  $C_2$  at the surface. On the other hand, for a given depth  $z$  the coefficients decay rapidly and the two or three first coefficients are enough to yield accurate results. The values of  $\lambda_i C_i$  were evaluated and presented in Table 3.3. At the water surface ( $z=0$ ) where the sloshing contributes most, the fourth mode coefficient  $\lambda_4 C_4$  is 4.7% of  $\lambda_1 C_1$  and is 3.8% of the sum of the first three constants. The  $C_i$ 's and the  $\lambda_i C_i$ 's are plotted in Figures 3.2 to 3.5.

Similar results are obtained when these coefficients are multiplied by their respective convolution integrals. These integrals were evaluated for the first four modes ( $i=1,..4$ ) for the El Centro and San Francisco table motions at horizontal spans of 50. The forcing functions used were the shake table recorded acceleration and the isolated base recorded acceleration, respectively. Their time history plots are shown in Figures 3.6 and 3.7 for El Centro and in Figures 3.12 and 3.13 for San Francisco. In sum, the first two modes yield accurate results at higher depths and the first three modes at the water free surface.

(2) Equation (3.22) shows explicitly the different factors influencing the dynamic pressure. The first term

$$P_{d1} = \rho R_o v_{g,tt} \left( 1 + \sum_{i=1}^{i=\infty} C_i \right) \quad (3.25)$$

represents the direct effect of ground acceleration, and the term

$$P_{d_2} = \rho R_o \sum_{i=1}^{i=\infty} \lambda_i C_i I_i(t) \quad (3.26)$$

represents the effect of the sloshing modes.

To illustrate this effect,  $P_{d_1}$  and  $P_{d_2}$  were evaluated using two modes only and were plotted in Figures 3.8a to 3.9b for the El Centro record and in Figures 3.14a to 3.15b for the San Francisco record, at the location  $z=-16$  in. The resultant pressure time histories for the tank on the table and the tank in the structure were then obtained by summing these components. For the El Centro record, Figures 3.10 and 3.11 show the calculated dynamic pressure time histories using the recorded table acceleration and the recorded isolated base acceleration as forcing functions, respectively. Figures 3.16 and 3.17 show the same for the San Francisco record. Note the drastic reduction in the calculated dynamic pressure for the tank in the structure under the San Francisco record.

The free surface displacement is given by

$$\begin{aligned} \eta(r, \phi, z, t) &= \frac{1}{g} \frac{\partial \Theta}{\partial t} \quad \text{at } z = 0 \\ &= \frac{\cos \phi}{g} \sum_{i=1}^{i=\infty} \left[ v_{g,t} - \lambda_i \int_0^{R_o} \sin \lambda_i(t-\tau) v_{g,t}(\tau) d\tau \right] J_1(n_i r) + \frac{\cos \phi}{g} IV_{g,t} \end{aligned} \quad (3.27)$$

At the tank wall and along the diameter of excitation, we have

$$\frac{\eta}{R_o/g} = v_{g,t}(t) \left( 1 + \sum_{i=1}^{i=\infty} C_i' \right) - \sum_{i=1}^{i=\infty} C_i' \lambda_i I_i(t) \quad (3.28)$$

where the constants  $C_i'$  are the  $C_i$ 's at  $z=0$ . As mentioned above, the contribution of the sloshing at  $z=0$  is much more important than it is at other locations along the water depth. Nevertheless, only the first three modes are needed to yield accurate results.

In many design situations, only the Peak Ground Acceleration and the response spectrum of the selected design earthquake are given. By noting that the maximum value of  $\lambda_i I_i(t)$  is the spectral acceleration  $S_{a_i}$ , expressions (3.22) and (3.28) can be used to provide an upper bound of the fluid response. Equation (3.22), for instance yields

$$\frac{P_d}{\rho R_o} \leq \ddot{v}_{\max} \left( 1 + \sum_{i=1}^{i=\infty} C_i \right) + \sum_{i=1}^{i=\infty} C_i S_{a_i}$$

Furthermore, since it was found that a few modes yield good results, the sum can be limited to the first three terms ( $i=1,2,3$ ) and the root mean square values used similarly to a response spectra analysis for multi-degrees of freedom systems. The predictions of peak values can now be written as

$$\frac{P_d}{\rho R_o} \leq \ddot{v}_{\max} \left( 1 + \sum_{i=1}^{i=3} C_i \right) + \left( \sum_{i=1}^{i=3} C_i^2 S_{a_i}^2 \right)^{1/2} \quad (3.29)$$

for the dynamic pressure, and as

$$\frac{\eta}{R_o/g} \leq \ddot{v}_{\max} \left( 1 + \sum_{i=1}^{i=3} C_i' \right) + \left( \sum_{i=1}^{i=3} C_i'^2 S_{a_i}^2 \right)^{1/2} \quad (3.30)$$

for the water free surface deformation.

In the next section, the experimental results are presented and compared with the time histories obtained from expressions (3.22) and (3.28).

**Table 3.1** The first ten roots  $\alpha_i$  of  $J_1'(\alpha)$  and the sloshing frequencies  $\lambda_i$  in rd/sec and  $f_i$  in Hertz.

$\alpha_i$	$\lambda_i$ (rd/sec)	$f_i$ (Hz)
1.84118	5.08	0.81
5.33144	9.26	1.47
8.53632	11.72	1.87
11.70600	13.73	2.18
14.86359	15.47	2.46
18.01553	17.03	2.71
21.16437	18.46	2.94
24.31133	19.78	3.15
27.45705	21.03	3.35
30.60192	22.20	3.53

Table 3.2 Coefficients  $C_i$  for  $R_0=24$  in. and  $h=17.5$  in.

$z$	$C_1$	$C_2$	$C_3$	$C_4$	$C_5$	$C_6$
0.0	-0.836840	-0.072928	-2.782851e-2	-1.470259e-2	-9.093955e-3	-6.181247e-3
-0.5	-0.809449	-0.065268	-2.329463e-2	-1.152073e-2	-6.672215e-3	-4.246936e-3
-1.0	-0.783249	-0.058415	-1.949943e-2	-9.027484e-3	-4.895390e-3	-2.917927e-3
-1.5	-0.758202	-0.052282	-1.632257e-2	-7.073807e-3	-3.591737e-3	-2.004812e-3
-2.0	-0.734271	-0.046796	-1.366330e-2	-5.542933e-3	-2.635249e-3	-1.377440e-3
-2.5	-0.711420	-0.041887	-1.143731e-2	-4.343363e-3	-1.933476e-3	-9.463930e-4
-3.0	-0.689616	-0.037495	-9.573992e-3	-3.403396e-3	-1.418588e-3	-6.502361e-4
-3.5	-0.668827	-0.033567	-8.014278e-3	-2.666855e-3	-1.040814e-3	-4.467554e-4
-4.0	-0.649021	-0.030053	-6.708699e-3	-2.089710e-3	-7.636426e-4	-3.069509e-4
-4.5	-0.630171	-0.026910	-5.615854e-3	-1.637469e-3	-5.602829e-4	-2.108959e-4
-5.0	-0.612249	-0.024100	-4.701090e-3	-1.283099e-3	-4.110784e-4	-1.448995e-4
-5.5	-0.595227	-0.021587	-3.935404e-3	-1.005421e-3	-3.016074e-4	-9.955569e-5
-6.0	-0.579081	-0.019341	-3.294511e-3	-7.878376e-4	-2.212887e-4	-6.840144e-5
-6.5	-0.563787	-0.017333	-2.758086e-3	-6.173434e-4	-1.623592e-4	-4.699634e-5
-7.0	-0.549323	-0.015540	-2.309122e-3	-4.837481e-4	-1.191227e-4	-3.228964e-5
-7.5	-0.535667	-0.013939	-1.933382e-3	-3.790664e-4	-8.740021e-5	-2.218517e-5
-8.0	-0.522799	-0.012510	-1.618951e-3	-2.970418e-4	-6.412554e-5	-1.524270e-5
-8.5	-0.510701	-0.011235	-1.355857e-3	-2.327715e-4	-4.704908e-5	-1.047276e-5
-9.0	-0.499354	-0.010099	-1.135758e-3	-1.824141e-4	-3.452023e-5	-7.195498e-6
-9.5	-0.488742	-0.009087	-9.516751e-4	-1.429596e-4	-2.532800e-5	-4.943804e-6
-10.0	-0.478849	-0.008188	-7.977697e-4	-1.120498e-4	-1.858389e-5	-3.396746e-6
-10.5	-0.469661	-0.007390	-6.691622e-4	-8.783725e-5	-1.363604e-5	-2.333826e-6
-11.0	-0.461164	-0.006684	-5.617741e-4	-6.887478e-5	-1.000620e-5	-1.603544e-6
-11.5	-0.453346	-0.006060	-4.722002e-4	-5.402899e-5	-7.343533e-6	-1.101813e-6
-12.0	-0.446194	-0.005510	-3.976000e-4	-4.241253e-5	-5.390669e-6	-7.571229e-7
-12.5	-0.439700	-0.005029	-3.356078e-4	-3.333109e-5	-3.958849e-6	-5.203453e-7
-13.0	-0.433852	-0.004610	-2.842580e-4	-2.624187e-5	-2.909678e-6	-3.577315e-7
-13.5	-0.428642	-0.004248	-2.419221e-4	-2.072113e-5	-2.141748e-6	-2.461051e-7
-14.0	-0.424064	-0.003938	-2.072577e-4	-1.643889e-5	-1.580833e-6	-1.695559e-7
-14.5	-0.420109	-0.003677	-1.791655e-4	-1.313922e-5	-1.172715e-6	-1.171733e-7
-15.0	-0.416773	-0.003461	-1.567548e-4	-1.062488e-5	-8.779487e-7	-8.149133e-8
-15.5	-0.414050	-0.003288	-1.393149e-4	-8.745596e-6	-6.680420e-7	-5.742424e-8
-16.0	-0.411936	-0.003156	-1.262927e-4	-7.389038e-6	-5.227061e-7	-4.154179e-8
-16.5	-0.410428	-0.003062	-1.172752e-4	-6.474126e-6	-4.278933e-7	-3.158025e-8
-17.0	-0.409525	-0.003007	-1.119767e-4	-5.946174e-6	-3.744394e-7	-2.611983e-8
-17.5	-0.409223	-0.002988	-1.102290e-4	-5.773628e-6	-3.571775e-7	-2.438224e-8

Table 3.3 Coefficients  $\lambda_i C_i$  for  $R_0=24$  in. and  $h=17.5$  in.

$z$	$\lambda_1 C_1$	$\lambda_2 C_2$	$\lambda_3 C_3$	$\lambda_4 C_4$	$\lambda_5 C_5$	$\lambda_6 C_6$
0.0	-4.255306	-0.675378	-0.326239	-2.018417e-1	-1.406785e-1	-1.052720e-1
-0.5	-4.116024	-0.604440	-0.273088	-1.581602e-1	-1.032155e-1	-7.232897e-2
-1.0	-3.982798	-0.540975	-0.228596	-1.239321e-1	-7.572898e-2	-4.969481e-2
-1.5	-3.855435	-0.484178	-0.191353	-9.711142e-2	-5.556219e-2	-3.414367e-2
-2.0	-3.733746	-0.433373	-0.160178	-7.609510e-2	-4.076586e-2	-2.345899e-2
-2.5	-3.617550	-0.387911	-0.134082	-5.962703e-2	-2.990982e-2	-1.611789e-2
-3.0	-3.506677	-0.347237	-0.112238	-4.672287e-2	-2.194477e-2	-1.107408e-2
-3.5	-3.400965	-0.310860	-0.093953	-3.661140e-2	-1.610083e-2	-7.608629e-3
-4.0	-3.300253	-0.278318	-0.078647	-2.868819e-2	-1.181313e-2	-5.227639e-3
-4.5	-3.204401	-0.249211	-0.065836	-2.247968e-2	-8.667269e-3	-3.591739e-3
-5.0	-3.113268	-0.223187	-0.055112	-1.761478e-2	-6.359157e-3	-2.467763e-3
-5.5	-3.026712	-0.199915	-0.046136	-1.380273e-2	-4.665700e-3	-1.695519e-3
-6.0	-2.944610	-0.179115	-0.038622	-1.081568e-2	-3.423215e-3	-1.164935e-3
-6.5	-2.866840	-0.160519	-0.032334	-8.475082e-3	-2.511607e-3	-8.003882e-4
-7.0	-2.793291	-0.143914	-0.027070	-6.641043e-3	-1.842762e-3	-5.499204e-4
-7.5	-2.723851	-0.129088	-0.022665	-5.203941e-3	-1.352033e-3	-3.778325e-4
-8.0	-2.658417	-0.115854	-0.018979	-4.077882e-3	-9.919869e-4	-2.595962e-4
-8.5	-2.596899	-0.104046	-0.015895	-3.195559e-3	-7.278234e-4	-1.783601e-4
-9.0	-2.539200	-0.093526	-0.013315	-2.504236e-3	-5.340091e-4	-1.225455e-4
-9.5	-2.485238	-0.084154	-0.011157	-1.962593e-3	-3.918102e-4	-8.419723e-5
-10.0	-2.434933	-0.075828	-0.009352	-1.538254e-3	-2.874826e-4	-5.784951e-5
-10.5	-2.388212	-0.068438	-0.007845	-1.205857e-3	-2.109420e-4	-3.974707e-5
-11.0	-2.345005	-0.061900	-0.006586	-9.455344e-4	-1.547904e-4	-2.730973e-5
-11.5	-2.305251	-0.056121	-0.005536	-7.417267e-4	-1.136004e-4	-1.876483e-5
-12.0	-2.268883	-0.051028	-0.004661	-5.822524e-4	-8.339069e-5	-1.289446e-5
-12.5	-2.235861	-0.046573	-0.003934	-4.575796e-4	-6.124122e-5	-8.861928e-6
-13.0	-2.206124	-0.042693	-0.003332	-3.602565e-4	-4.501112e-5	-6.092476e-6
-13.5	-2.179632	-0.039340	-0.002836	-2.844660e-4	-3.313167e-5	-4.191382e-6
-14.0	-2.156353	-0.036469	-0.002430	-2.256782e-4	-2.445461e-5	-2.887683e-6
-14.5	-2.136242	-0.034052	-0.002100	-1.803792e-4	-1.814126e-5	-1.995562e-6
-15.0	-2.119278	-0.032052	-0.001838	-1.458616e-4	-1.358138e-5	-1.387868e-6
-15.5	-2.105432	-0.030450	-0.001633	-1.200622e-4	-1.033424e-5	-9.779844e-7
-16.0	-2.094682	-0.029227	-0.001481	-1.014390e-4	-8.085977e-6	-7.074924e-7
-16.5	-2.087014	-0.028357	-0.001375	-8.887881e-5	-6.619274e-6	-5.378388e-7
-17.0	-2.082422	-0.027847	-0.001313	-8.163092e-5	-5.792372e-6	-4.448431e-7
-17.5	-2.080887	-0.027672	-0.001292	-7.926216e-5	-5.525341e-6	-4.152504e-7

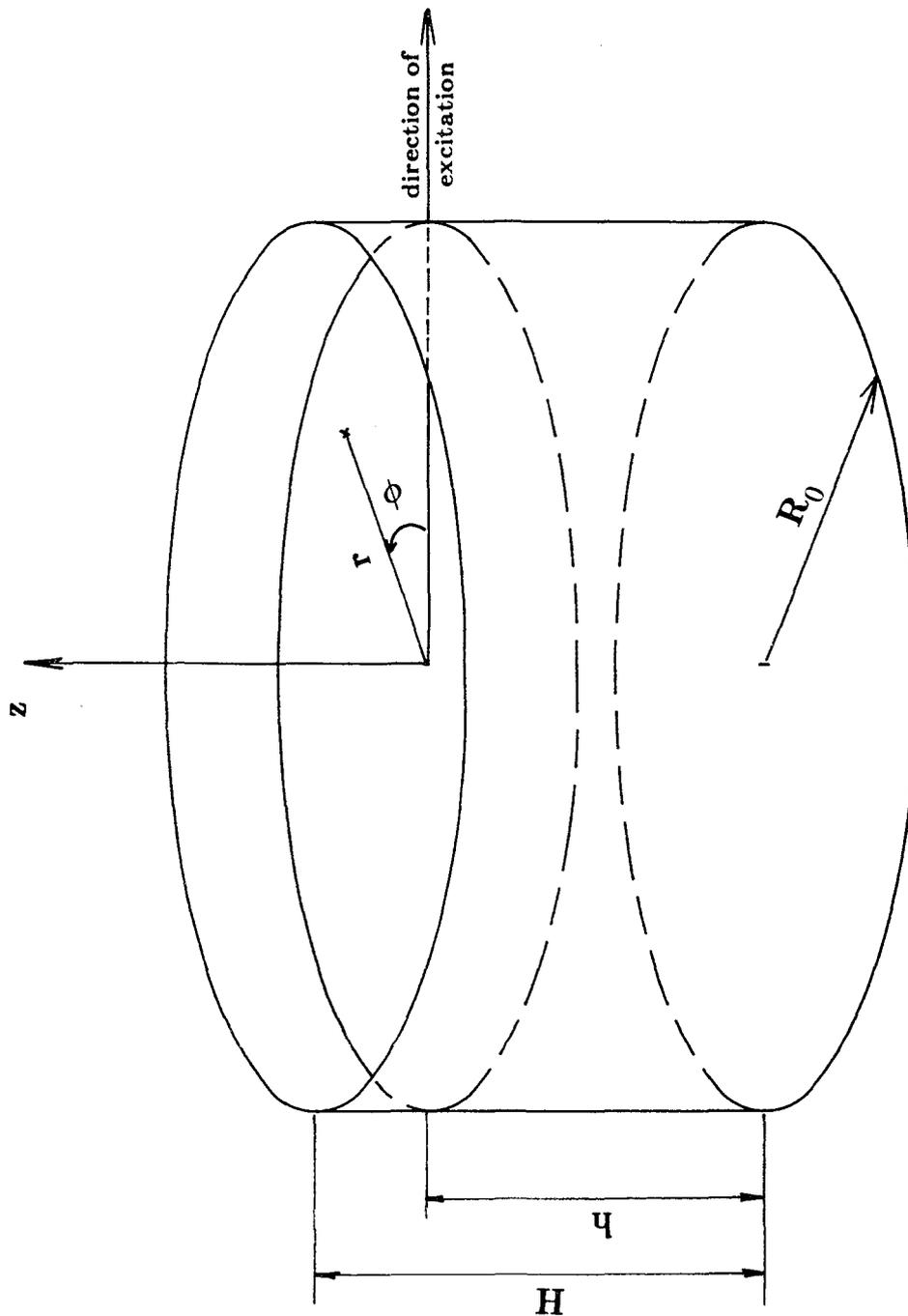


Figure 3.1 Cylindrical tank showing coordinates and dimensions.

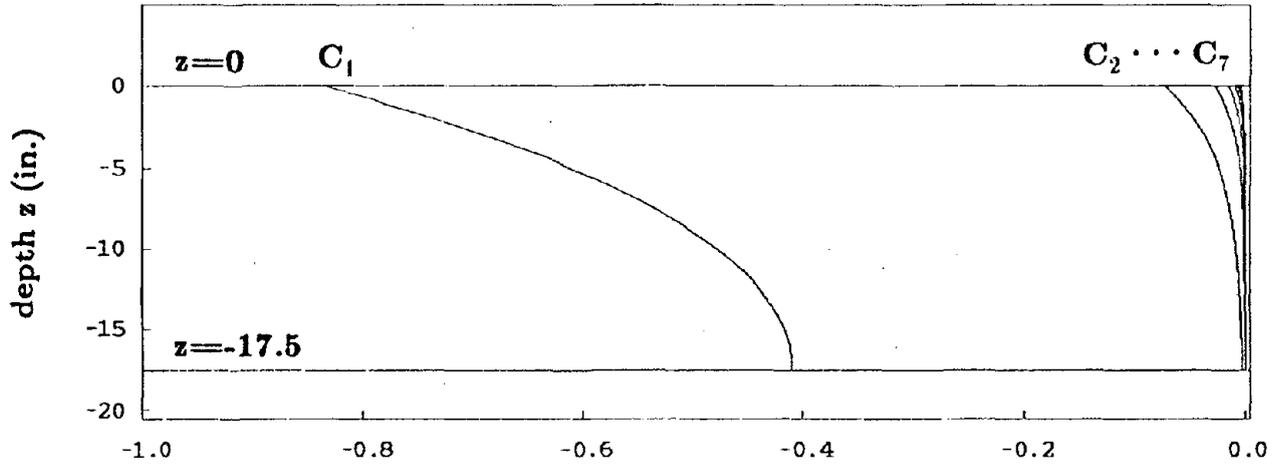


Figure 3.2 Coefficients  $C_i$  ( $i=1,\dots,7$ ) showing decrease with increasing depth and mode number.

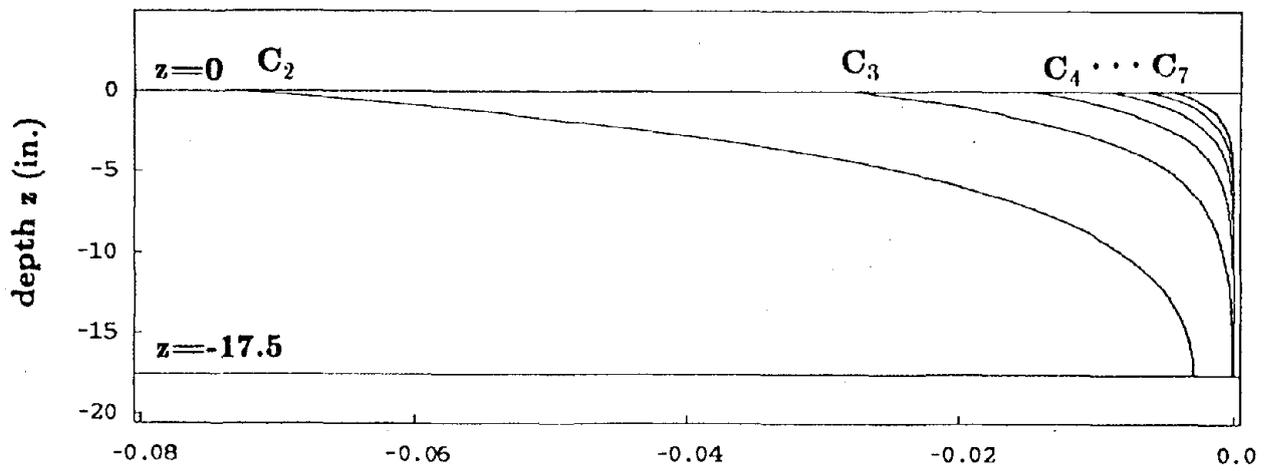


Figure 3.3 Coefficients  $C_i$  ( $i=2,\dots,7$ ) showing decrease with increasing depth and mode number.

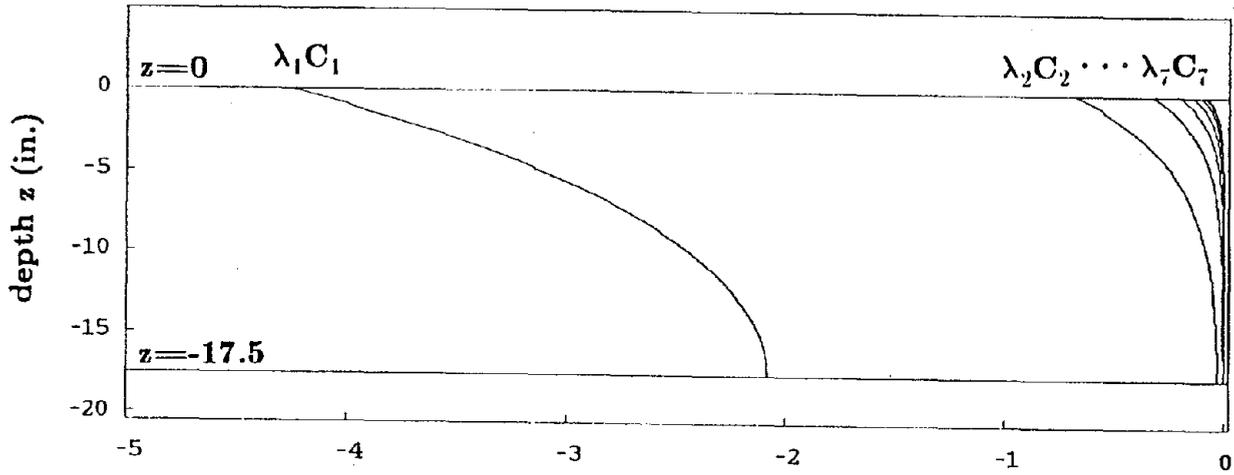


Figure 3.4 Coefficients  $\lambda_i C_i$  ( $i=1, \dots, 7$ ) showing decrease with increasing depth and mode number.

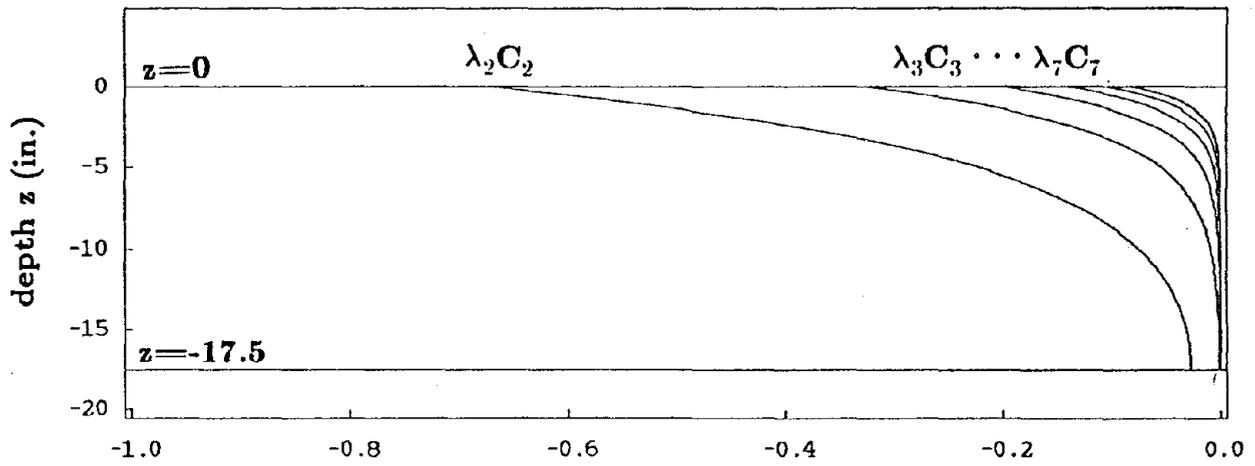


Figure 3.5 Coefficients  $\lambda_i C_i$  ( $i=2, \dots, 7$ ) showing decrease with increasing depth and mode number.

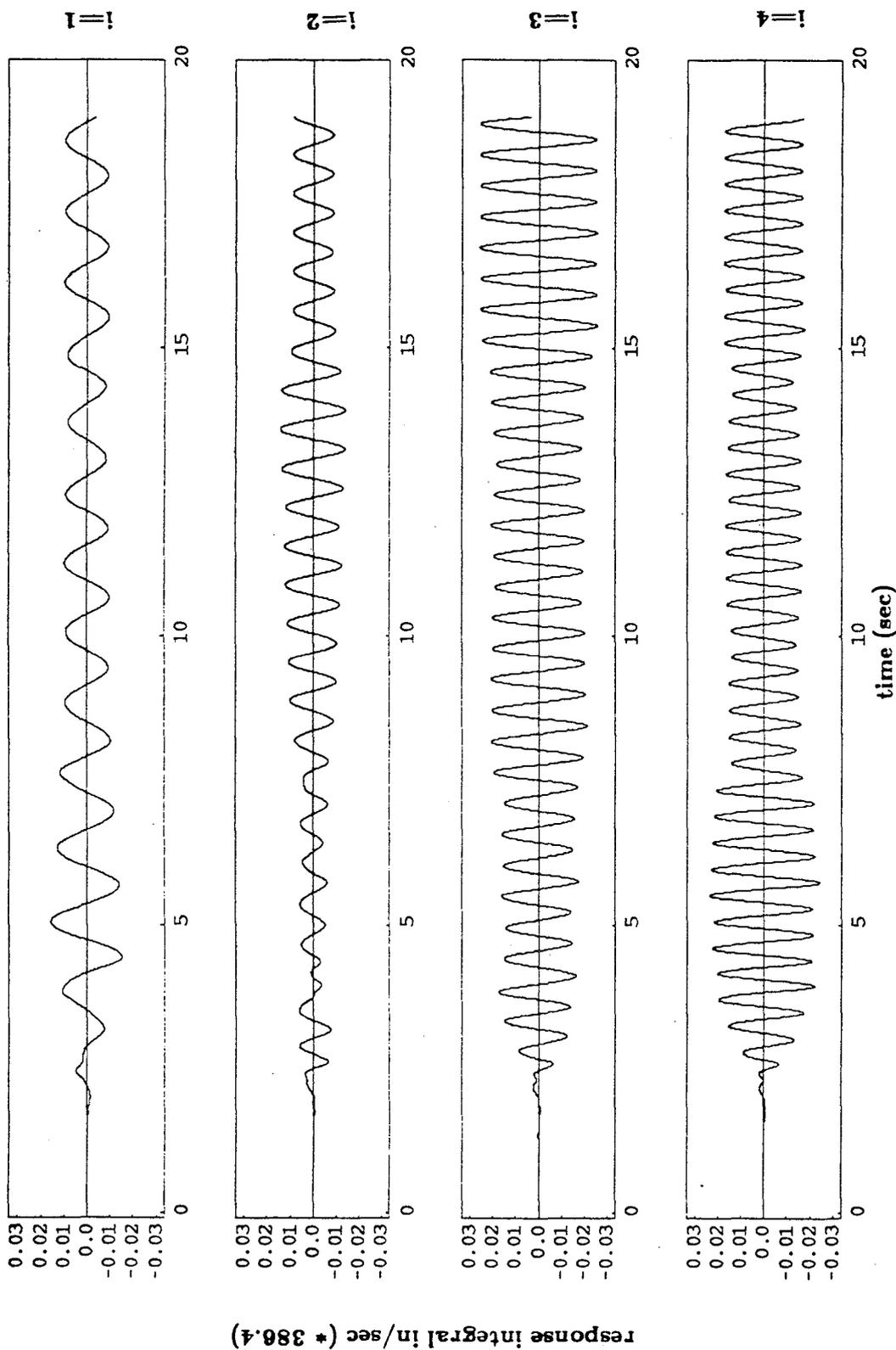


Figure 3.6 Calculated convolution integrals  $I_i(t)$  ( $i=1,2,3,4$ ) from equation (3.24). Forcing function: recorded table acceleration. Table input:  $\sqrt{4}$  time scaled El Centro horizontal span 50, PTA=0.114 g.

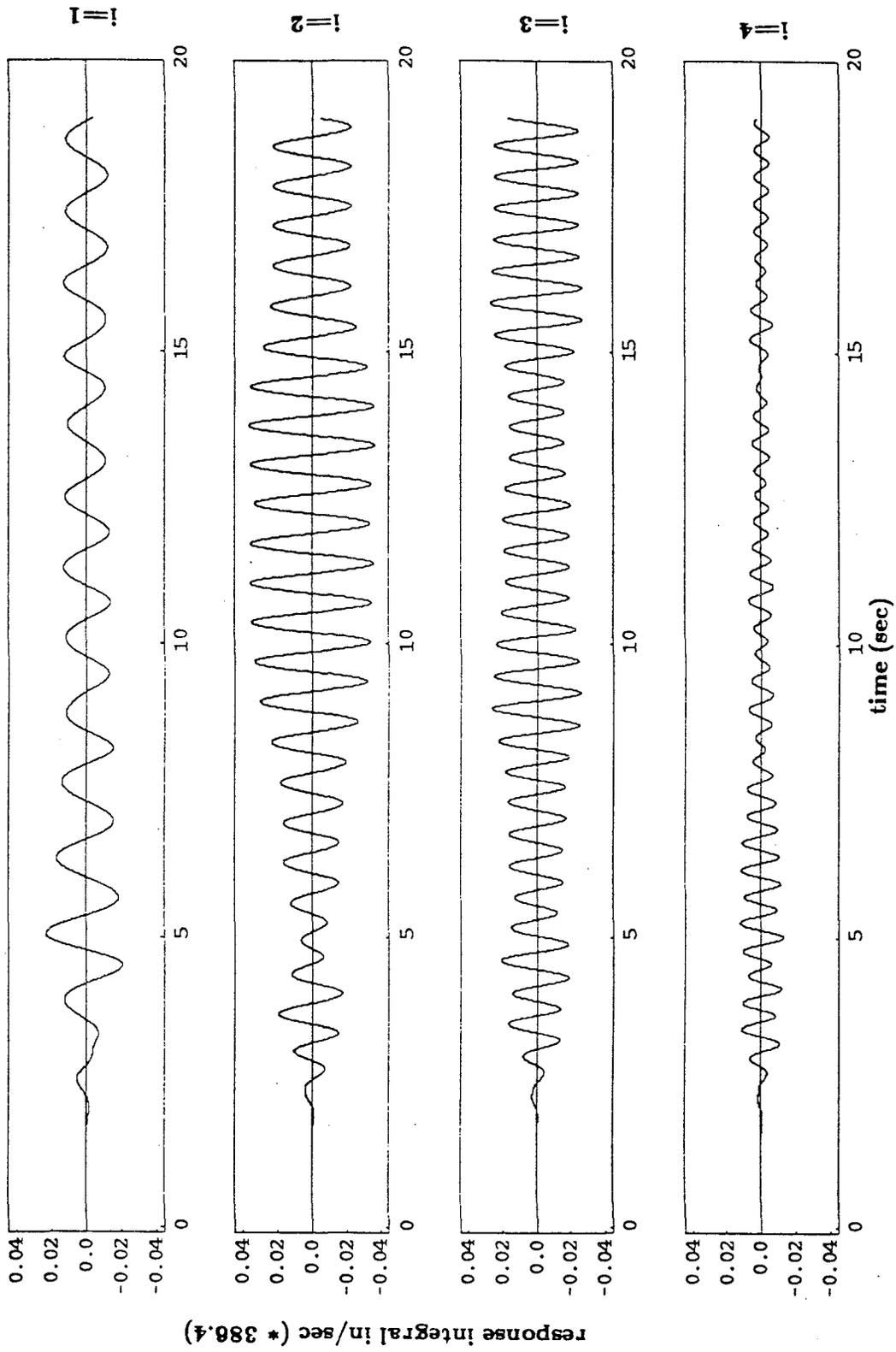


Figure 3.7 Calculated convolution integrals  $I_i(t)$  ( $i=1,2,3,4$ ) from equation (3.24). Forcing function: recorded structure's isolated base acceleration. Table input:  $\sqrt{4}$  time scaled El Centro horizontal span 50, PTA=0.114 g.

Theoretical solution using 2 modes

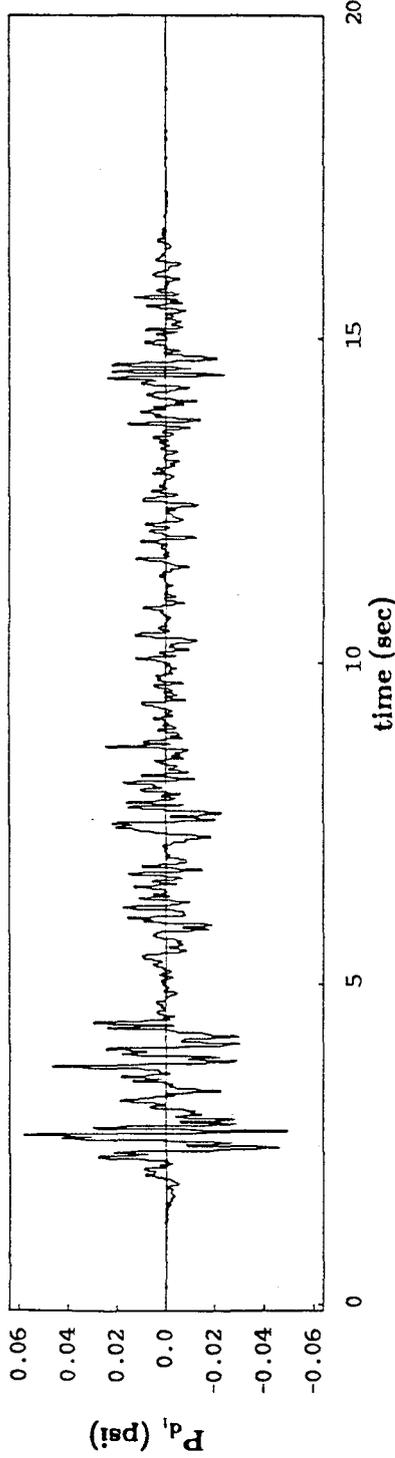


Figure 3.8a Calculated  $P_{d_1}$  component from equation (3.25) using two modes. Location:  $z=-16$  in. Forcing function: recorded table acceleration. Table input:  $\sqrt{4}$  time scaled El Centro horizontal span 50, PTA=0.114 g.

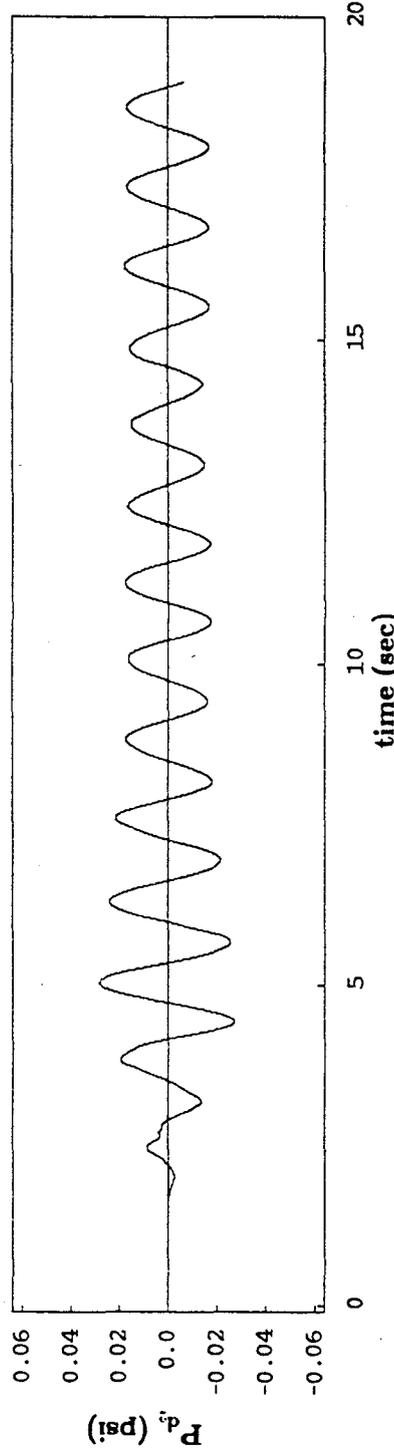


Figure 3.8b Calculated  $P_{d_2}$  component from equation (3.26) using two modes. Location:  $z=-16$  in. Forcing function: recorded table acceleration. Table input:  $\sqrt{4}$  time scaled El Centro horizontal span 50, PTA=0.114 g.

Theoretical solution using 2 modes

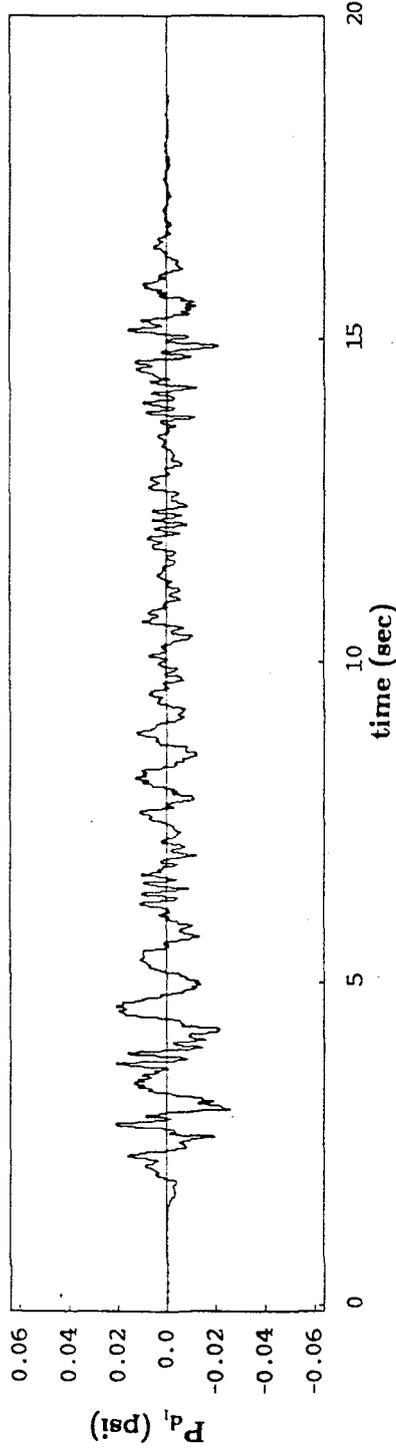


Figure 3.9a Calculated  $P_{d_1}$  component from equation (3.25) using two modes. Location:  $z=-16$  in. Forcing function: recorded structure's isolated base acceleration. Table input:  $\sqrt{4}$  time scaled El Centro horizontal span 50, PTA=0.114 g.

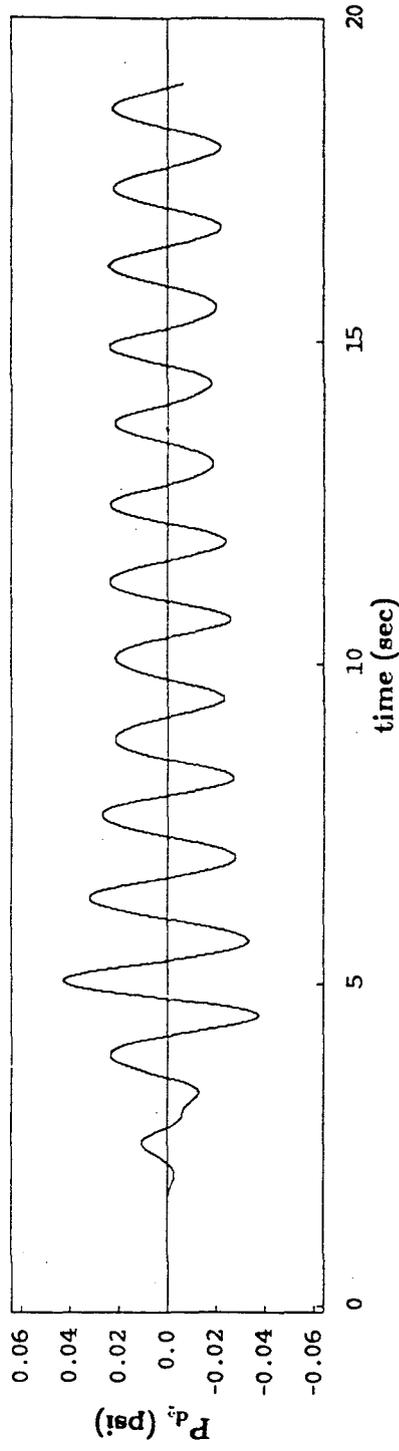


Figure 3.9b Calculated  $P_{d_2}$  component from equation (3.26) using two modes. Location:  $z=-16$  in. Forcing function: recorded structure's isolated base acceleration. Table input:  $\sqrt{4}$  time scaled El Centro horizontal span 50, PTA=0.114 g.

Theoretical solution using 2 modes

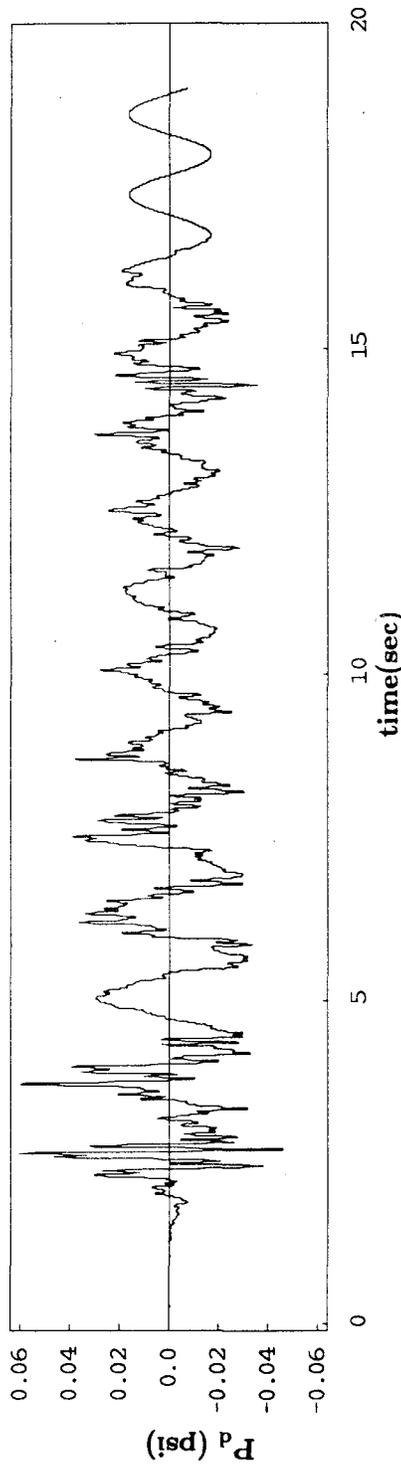


Figure 3.10 Calculated dynamic pressure  $P_d = P_{d_1} + P_{d_2}$  component using two modes. Location: z=-16 in. Forcing function: recorded table acceleration. Table input:  $\sqrt{4}$  time scaled El Centro horizontal span 50, PTA=0.114 g.

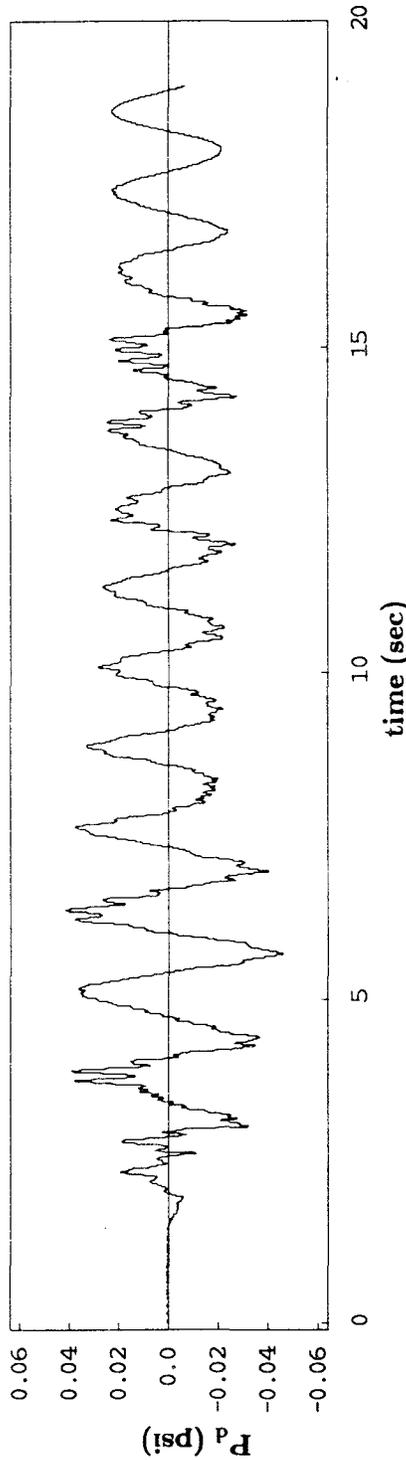


Figure 3.11 Calculated dynamic pressure  $P_d = P_{d_1} + P_{d_2}$  using two modes. Location: z=-16 in. Forcing function: recorded structure's isolated base acceleration. Table input:  $\sqrt{4}$  time scaled El Centro horizontal span 50, PTA=0.114 g.

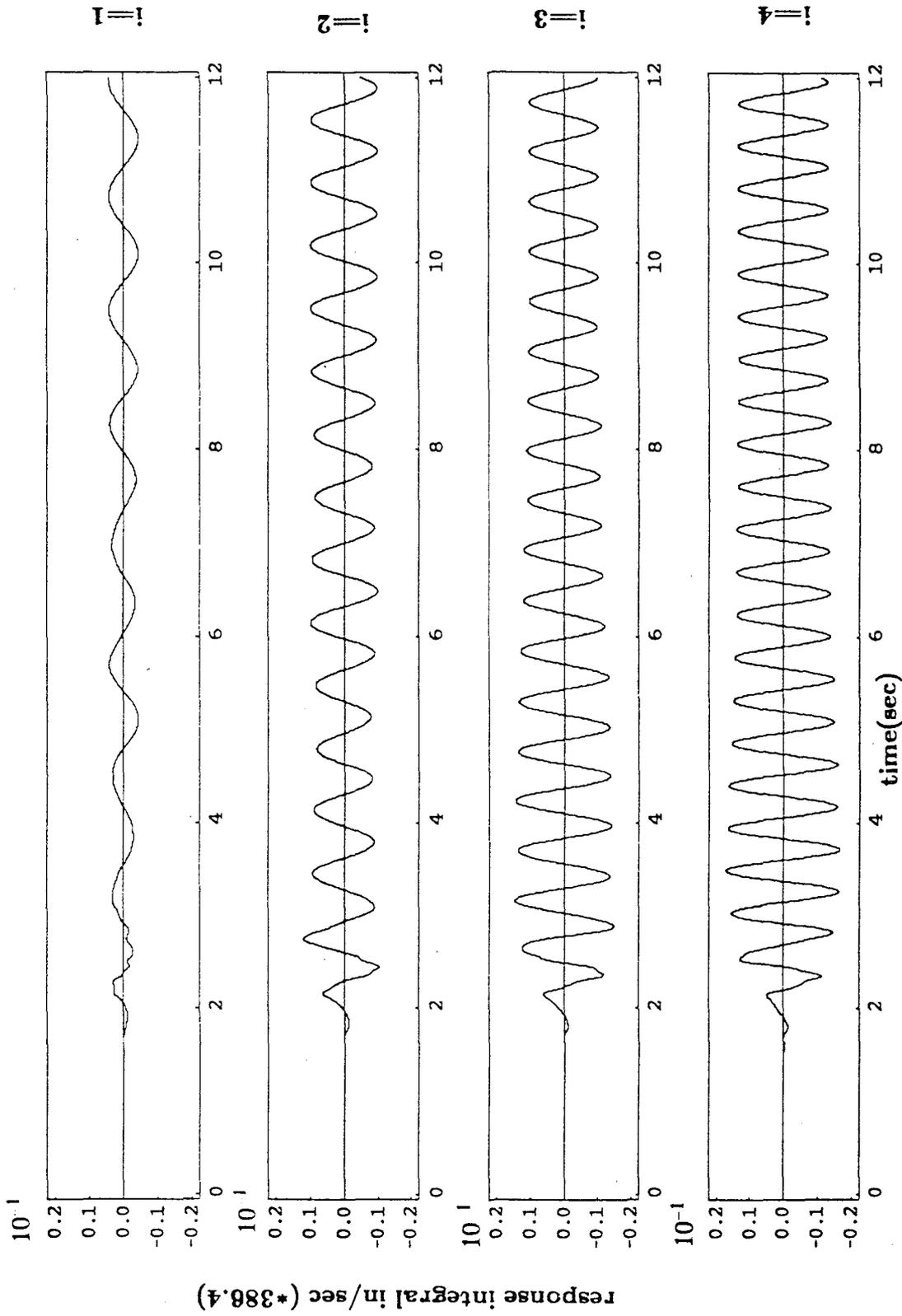


Figure 3.12 Calculated convolution integrals  $I_i(t)$  ( $i=1,2,3,4$ ) from equation (3.24). Forcing function: recorded table acceleration. Table input:  $\sqrt{4}$  time scaled San Francisco horizontal span 50, PTA=0.331 g.

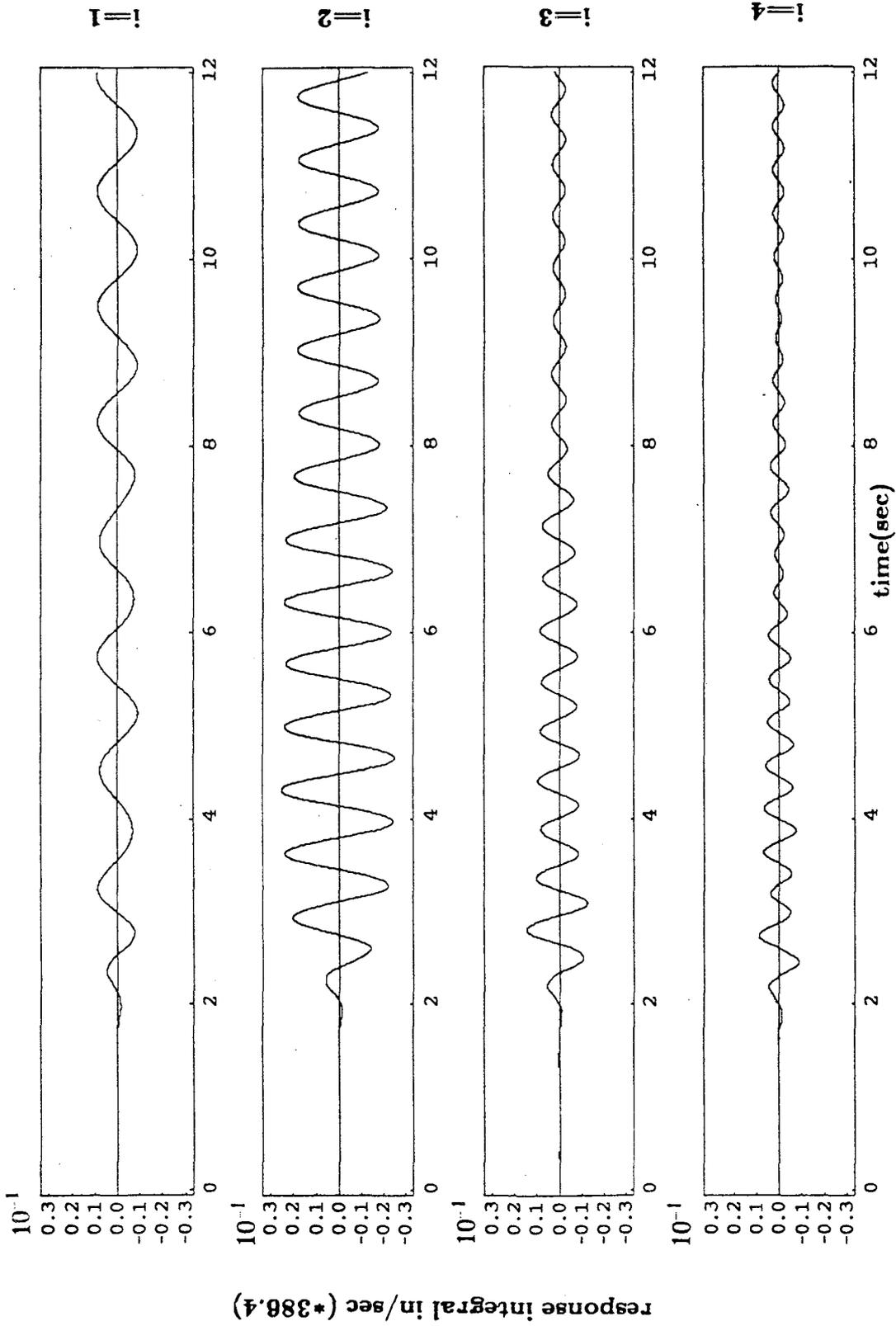


Figure 3.13 Calculated convolution integrals  $I_i(t)$  ( $i=1,2,3,4$ ) from equation (3.24). Forcing function: recorded structure's isolated base acceleration. Table input:  $\sqrt{4}$  time scaled San Francisco horizontal span 50, PTA=0.331 g.

Theoretical solution using 2 modes

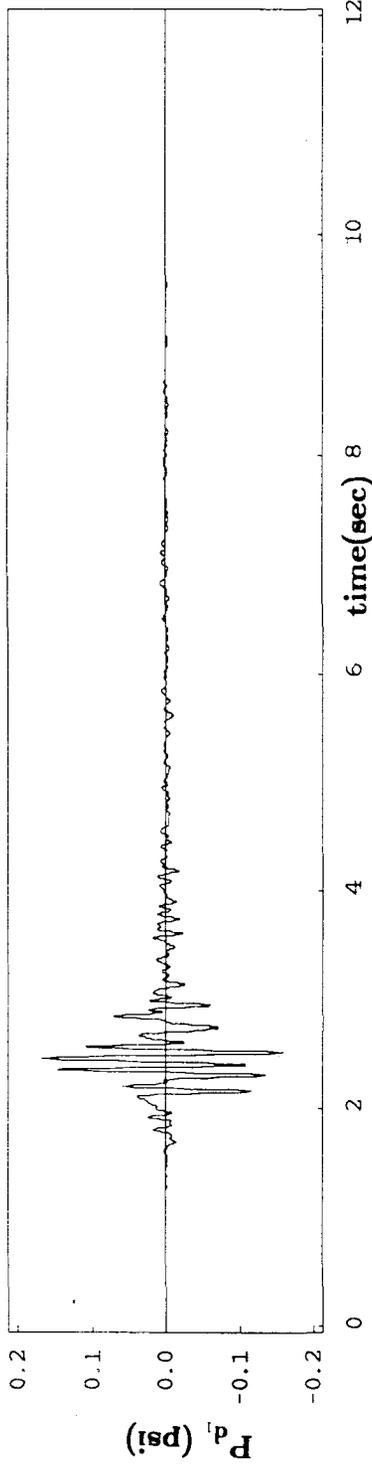


Figure 3.14a Calculated  $P_{d_1}$  component from equation (3.25) using two modes. Location:  $z=-16$  in. Forcing function: recorded table acceleration. Table input:  $\sqrt{4}$  time scaled San Francisco horizontal span 50, PTA=0.331 g.

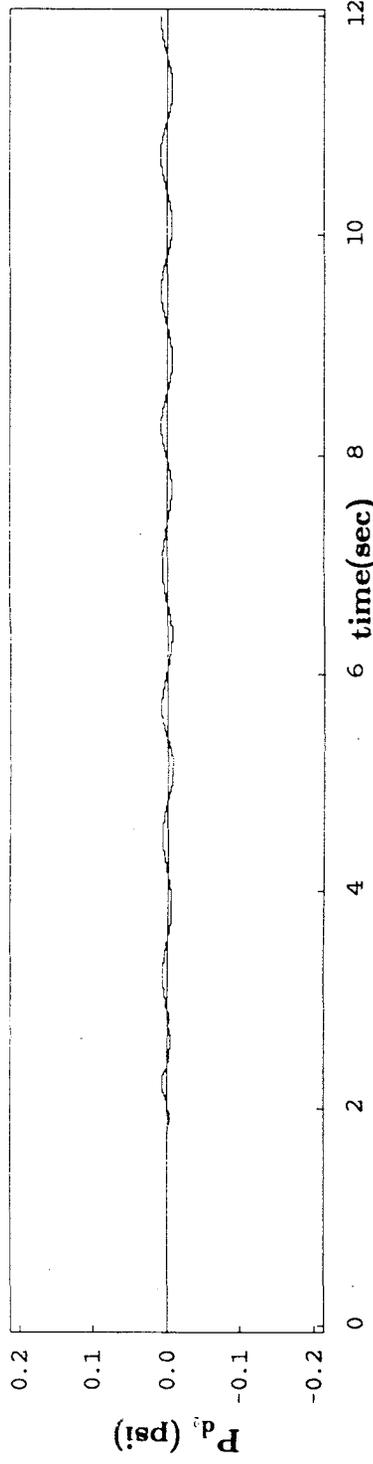


Figure 3.14b Calculated  $P_{d_2}$  component from equation (3.26) using two modes. Location:  $z=-16$  in. Forcing function: recorded table acceleration. Table input:  $\sqrt{4}$  time scaled San Francisco horizontal span 50, PTA=0.331 g.

Theoretical solution using 2 modes

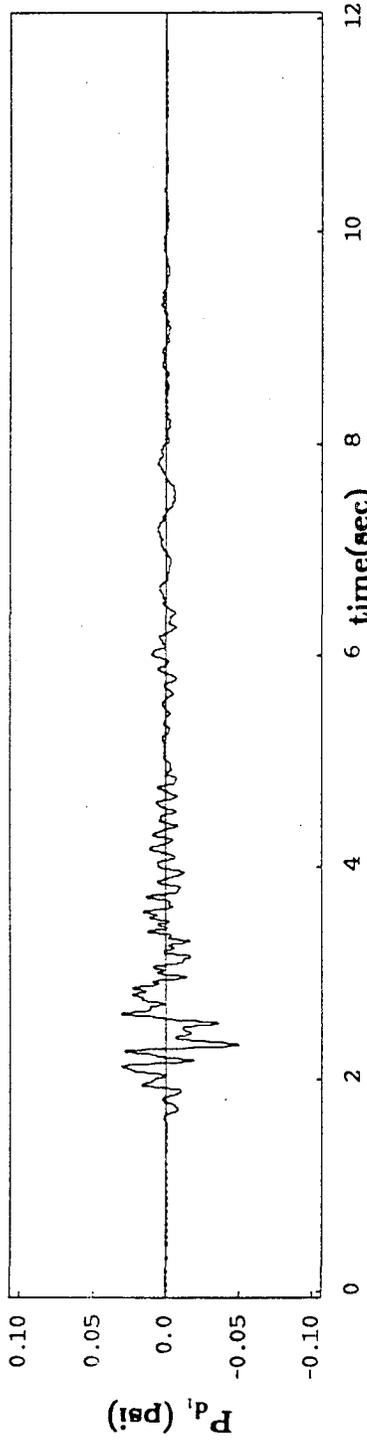


Figure 3.15a Calculated  $P_{d_1}$  component from equation (3.25) using two modes. Location:  $z=-16$  in. Forcing function: recorded structure's isolated base acceleration. Table input:  $\sqrt{4}$  time scaled San Francisco horizontal span 50, PTA=0.331 g.

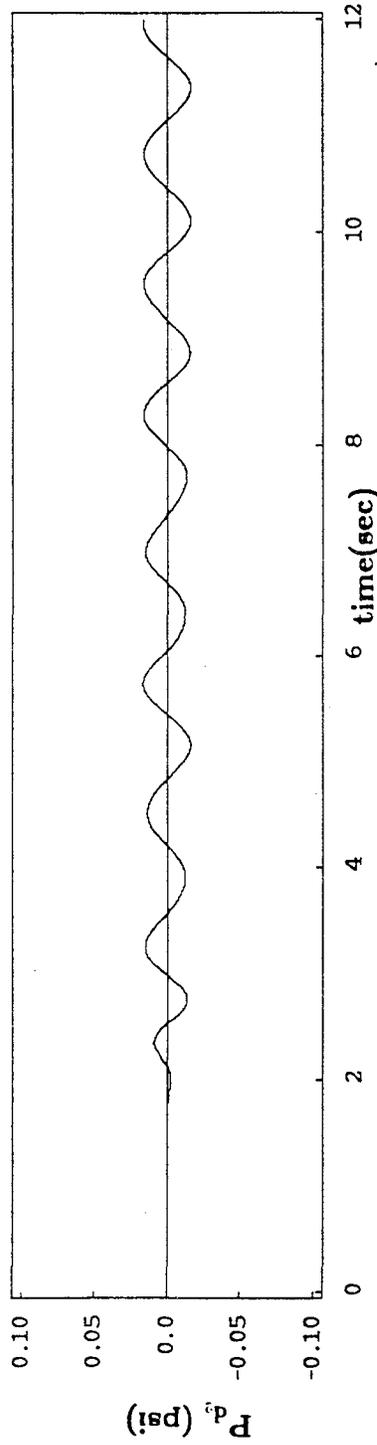


Figure 3.15b Calculated  $P_{d_2}$  component from equation (3.26) using two modes. Location:  $z=-16$  in. Forcing function: recorded structure's isolated base acceleration. Table input:  $\sqrt{4}$  time scaled San Francisco horizontal span 50, PTA=0.331 g.

Theoretical solution using 2 modes

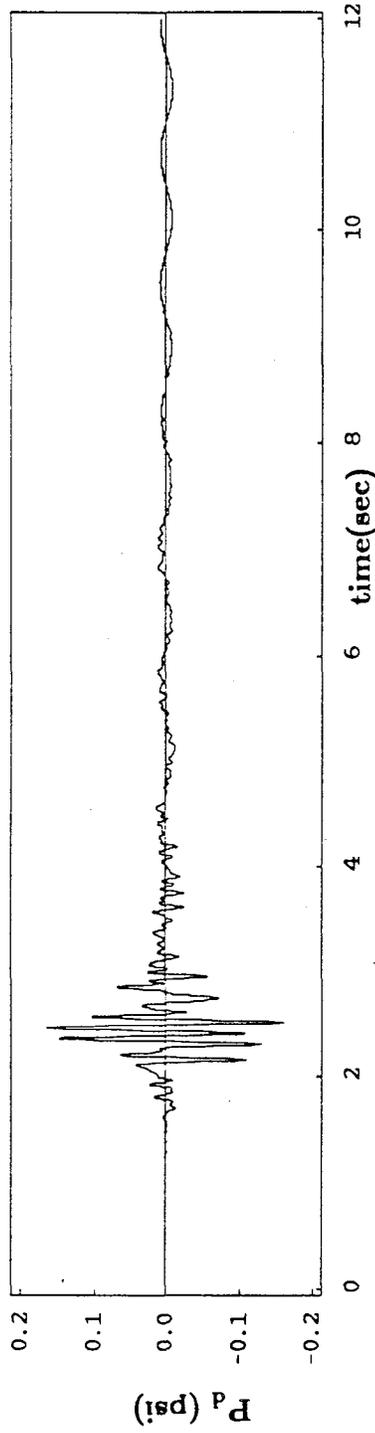


Figure 3.16 Calculated dynamic pressure  $P_d = P_{d_1} + P_{d_2}$  component using two modes. Location: z=-16 in. Forcing function: recorded table acceleration. Table input:  $\sqrt{4}$  time scaled San Francisco horizontal span 50, PTA=0.331 g.

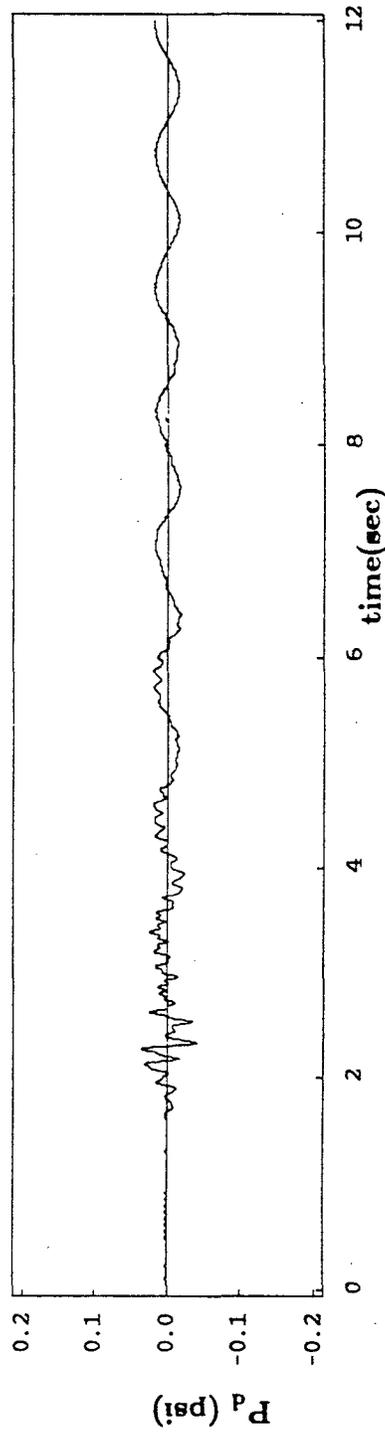


Figure 3.17 Calculated dynamic pressure  $P_d = P_{d_1} + P_{d_2}$  using two modes. Location: z=-16 in. Forcing function: recorded structure's isolated base acceleration. Table input:  $\sqrt{4}$  time scaled San Francisco horizontal span 50, PTA=0.331 g.

## CHAPTER FOUR TEST RESULTS AND CORRELATION

### 1. Some Experimental Results.

Seven different ground motion records were applied to the shake table at various spans ranging from 25 to 100. In the following, five records  $\sqrt{4}$  time scaled and applied at a horizontal span of 50 will be presented then compared with the theoretical solution in the next section. The main concern is the response of the fluid. Because of the size of the tank models, the shell deformations were not studied in great detail and are not presented here. Shell deformations will be studied on a larger scale model in the next earthquake simulator water tank test.

For each table input, the displacement and acceleration at the tank support along with the north and south water surface deformation measured by the water level gages are shown. For the tank on the table, the table displacement and table acceleration are provided while for the tank on the structure, the structure's base absolute displacement and the structure's base acceleration are shown.

In general, the tanks on the model showed more sloshing and less pressure. The input acceleration was more reflected in the response of tank #1 than it was in the response of tank #2. This is due to the filtering provided by the isolation system.

The calculated frequencies of the fluid surface free vibrations,  $\lambda_i = (g_n \tanh n_i h)^{1/2}$ , were compared with the experimental ones by using the FFT of the measured water elevation time histories for both tanks under different ground motions. This is illustrated in Figure 4.1 for the FFTs of tank #1 and tank #2 north water gage time histories.

### El Centro

For the El Centro record at 50 horizontal span, the Peak Table Acceleration (PTA) was 0.114 g. The reduction in the accelerations transmitted to the structure's base was of the order of 2. This is shown in Figure 4.2 where the Peak Base Acceleration (PBA) is around 0.05 g. The Peak Table Displacement (PTD) was 0.25 in. while the Peak Base Absolute Displacement was 0.37 in. The sloshing was slightly higher in the tank on the structure; the ratio of the positive peak values in the

water surface deformation is about 1.2.

Figure 4.3 shows the dynamic pressure time histories measured at the north bottom transducer for both tanks. The behavior of this transducer is descriptive of all the others with some slight differences. Note the reduction in the pressure for the tank on the structure, and its lower frequency. The ratio of the peak pressure at the bottom of tank #2 to that of tank #1 is 0.76.

### **San Francisco**

For the San Francisco record at 50 horizontal span, the PTA was 0.331 g. This record is of relatively high frequency and high acceleration. For this reason it did not excite the sloshing for the tank on the table while it yielded higher sloshing for the tank in the structure since the whole system was responding at low frequency. The ratio of the peak sloshing values is 1.47 for tank #2 to tank #1. The base acceleration was drastically reduced in this case, and the reduction factor is 3.4 (Figure 4.4).

Figure 4.5 shows a drastic reduction in the dynamic pressure for the tank in the structure. The ratio of the peak values for tank #1 to tank #2 is 4.3. That was expected since equation (3.22) predicts a high dependence of the pressure on the tank support acceleration.

### **Pacoima Dam**

For Pacoima Dam, the reduction in the acceleration at the structure base was not noticeable because this signal at a span of 50 caused a very low PTA of 0.076 g, and the isolation system was not quite activated. The lowering of the input frequency caused slightly higher sloshing while the reduction in the pressure was still present, and the ratio of peak pressure in tank #1 to peak pressure in tank #2 is 1.75 (Figures 4.6 and 4.7).

## **Parkfield**

For the Parkfield record at 50 horizontal span, conclusions similar to the ones for Pacoima Dam can be made. The only difference is that the table displacement here consists of a large sway in one direction followed by small cycles and this sway corresponds to a spike in the acceleration. This behavior was changed by base isolation, the displacement and the acceleration were transformed into more distributed, equally spaced cycles at lower frequency. This is reflected in the pressure time history but to a lesser extent; the few spikes in the dynamic pressure for tank #1 are reduced and distributed for tank #2.

The ratio of Peak Water Elevation in tank #1 to tank #2 is 0.78 while the ratio of peak pressure at  $z=16$  in. in tank #1 to tank #2 was 1.2 (Figures 4.8 and 4.9).

## **Taft**

For Taft there were no major differences in the response, except the slight reduction in the dynamic pressure for the tank on the structure, the reduction factor was around 1.4 (Figures 4.10 and 4.11).

The peak values of tank support motions and the fluid response are summarized in Table 4.1.

## **2. Comparison with Theoretical Solution.**

The measured pressure time histories were compared with the ones given by equation (3.22) using the first two modes ( $i=1,2$ ), for the five table motions and for both tanks, and good agreement was found. These are shown in Figures 4.12a to 4.21b. On the average, there is a difference of about 15% between calculated and measured peak pressures. This level of discrepancy is unavoidable because of the difficulty in achieving perfect measurements on one hand, and because of the assumptions made in the theoretical solution on the other hand.

Correlation coefficients between measured and calculated pressure time histories were obtained for most of the table motions and they were 97% for tank #1 and 94% for tank #2 with the El Centro record, 97% for tank #2 with the San Francisco record, 98% for both tanks with the Pacoima Dam record, and 98% for tank #2 with the Parkfield and Taft records.

The water elevation at the tank wall was calculated for El Centro using the first three modes ( $i=1,2,3$ ) for both tanks. The comparison is shown in Figure 4.22 for the tank on the table and Figure 4.23 for the tank on the structure. Good correlation (97%) was found for both. Obviously, the higher frequencies in the water surface deformation are absent in the calculated time history but the behavior is still well represented and equation (3.28) using three modes can predict spilling or overtopping accurately.

The preceding comparisons show that the expressions for the dynamic pressure and the water surface deformation can be directly used for a prototype to predict the response of the contained fluid under a given ground motion. Furthermore, this formulation gave insight into the advantages of using the approach to isolate a large tank. The pressure and the acceleration are reduced, and the sloshing is slightly increased. If the isolation system is chosen such that it is not in resonance with the first two sloshing modes, the effects on the water elevation will be small. The present experimental results do not represent the behavior of such a tank because the structural model had a fundamental period very close to the first sloshing mode of the 4 ft. diameter tank. Also, the scaling would be very different if the tanks were to be tested without the structural model. In other words, the accelerograms were time scaled by a factor of two because the geometric scale was dictated by the structure: if the 4 ft. diameter tank is used to represent a prototype of, for example, 80 ft. in diameter, the accelerograms should be time scaled by  $\sqrt{20}$ . This would lead to a ground motion of much higher frequency and thus yield a much better behavior for the isolated tank.

The simplicity of the solution presented renders the transformation of the problem into a mechanical analog unnecessary for tanks under the effect of ground motion. A mechanical analog is needed for tanks that are part of a more complex system, like airplane fuel containers [7], where the

tanks need to be incorporated in an overall response analysis of the system.

The equations governing the oscillations of a fluid are similar in nature to those for a spherical pendulum. It has been shown, however, that the motion of a spherical pendulum under an excitation in a vertical plane can become unstable near resonance and deviate from a planar trajectory because of nonlinear coupling [8]. When nonlinear wave theories are used to study forced waves in a fluid container they reveal coupling between the longitudinal modes of oscillation and the transverse modes that linear wave theory ignores. For certain forcing frequencies and certain damping levels in the system, the motion becomes chaotic [9]. These problems are of interest for applications where the tank is subjected to excitations of very long duration, and where there is interaction with the supporting structure, such as for tanks in moving vehicles or in aircrafts. However, for small tanks in structures or large ground supported tanks subjected to earthquake excitation this problem is of less significance and the present research shows that linear wave theory yields accurate results.

Table 4.1 Peak values of dynamic pressure and water elevation.

		El Centro span 50	San Francisco span 50	Pacoima Dam span 50	Parkfield span 50	Taft span 50
tank #1 (on table)	Peak Table Acceleration (g)	0.114	0.331	0.076	0.070	0.088
	Peak Table Displacement (in)	0.25	0.25	0.26	0.25	0.26
	Peak Water Elevation at tank wall (in) north gage	1.77	0.70	0.64	0.89	0.93
	Peak Dynamic Pressure (psi) at z=-16 in.	0.071	0.188	0.049	0.035	0.053
tank #2 (on structure)	Peak Base Acceleration (g)	0.052	0.098	0.055	0.038	0.061
	Peak Base Absolute Displacement (in)	0.37	0.48	0.37	0.25	0.31
	Peak Water Elevation at tank wall (in) north gage	2.15	1.03	0.87	1.14	0.96
	Peak Dynamic Pressure (psi) at z=-16 in.	0.054	0.044	0.028	0.029	0.037

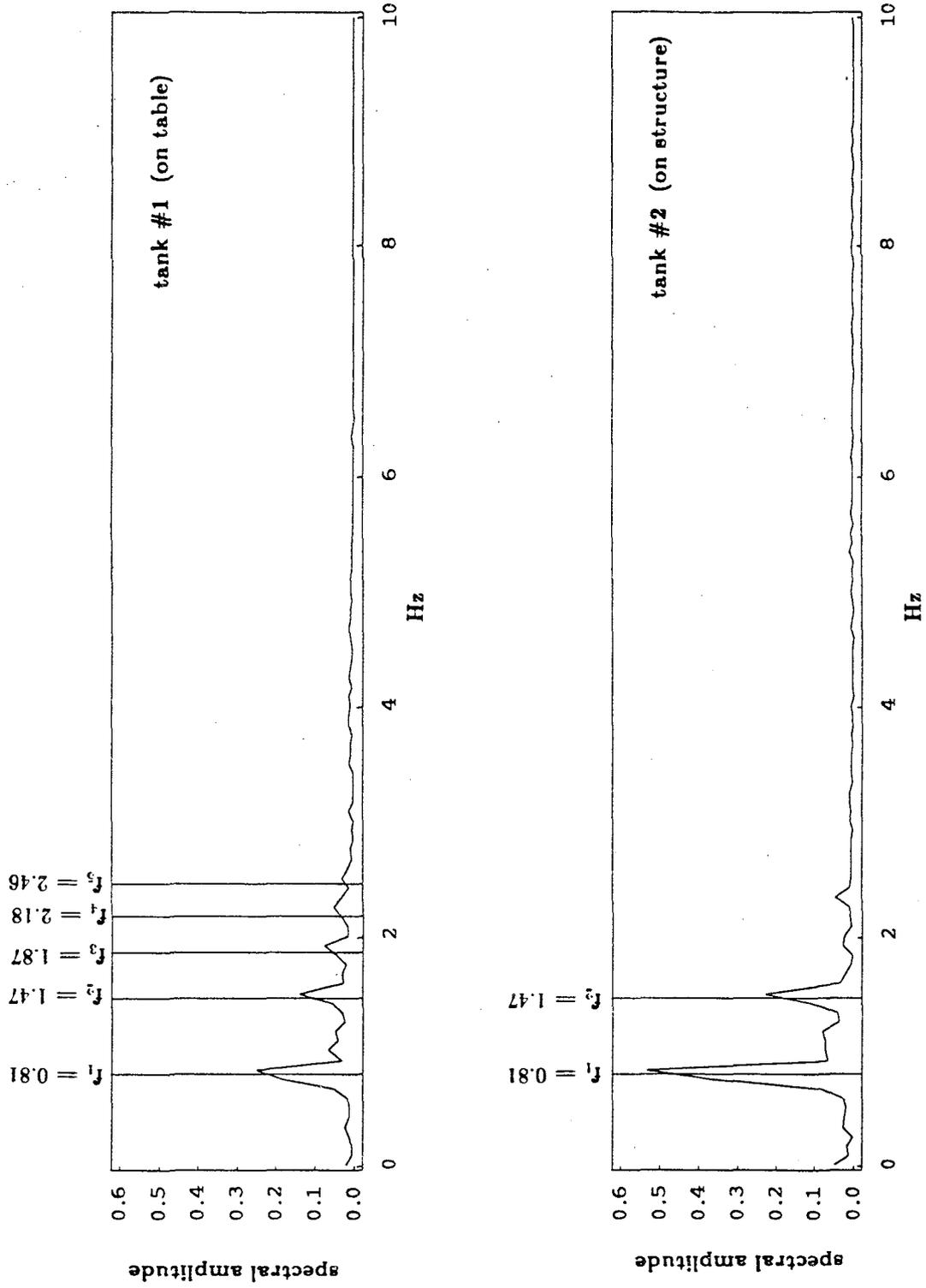


Figure 4.1 FFT of measured water elevation at tank wall (north gage) for tank #1 and tank #2 under San Francisco span 50, compared to the calculated frequencies of free vibration  $F_i = \lambda_i/2\pi = (g\eta_i \tanh \eta_i h)^{1/2}/2\pi$ .

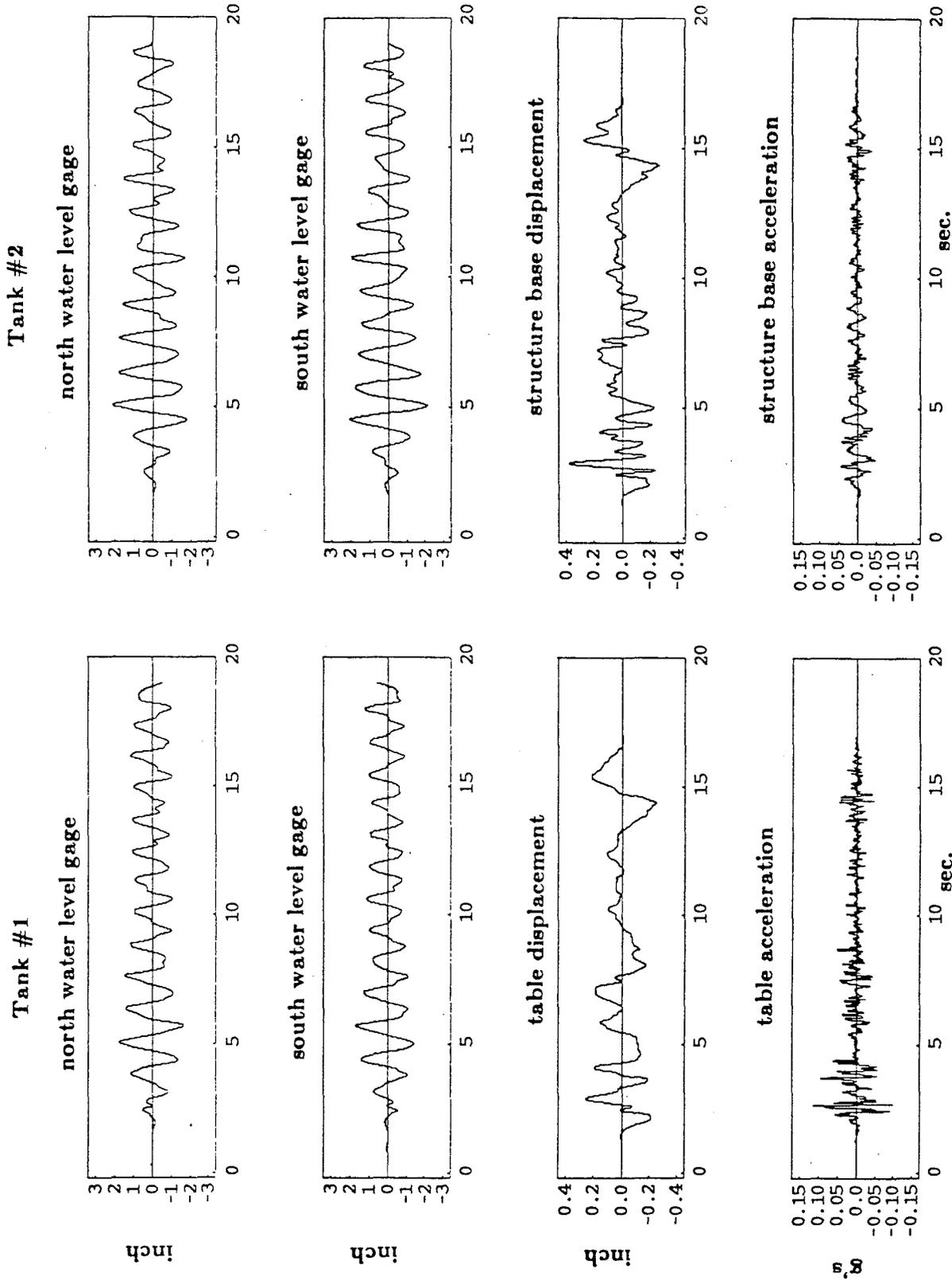


Figure 4.2 Water elevation at tank wall. Tank #1 and tank #2. Table input:  $\sqrt{4}$  time scaled El Centro horizontal span 50, PTA=0.114 g.

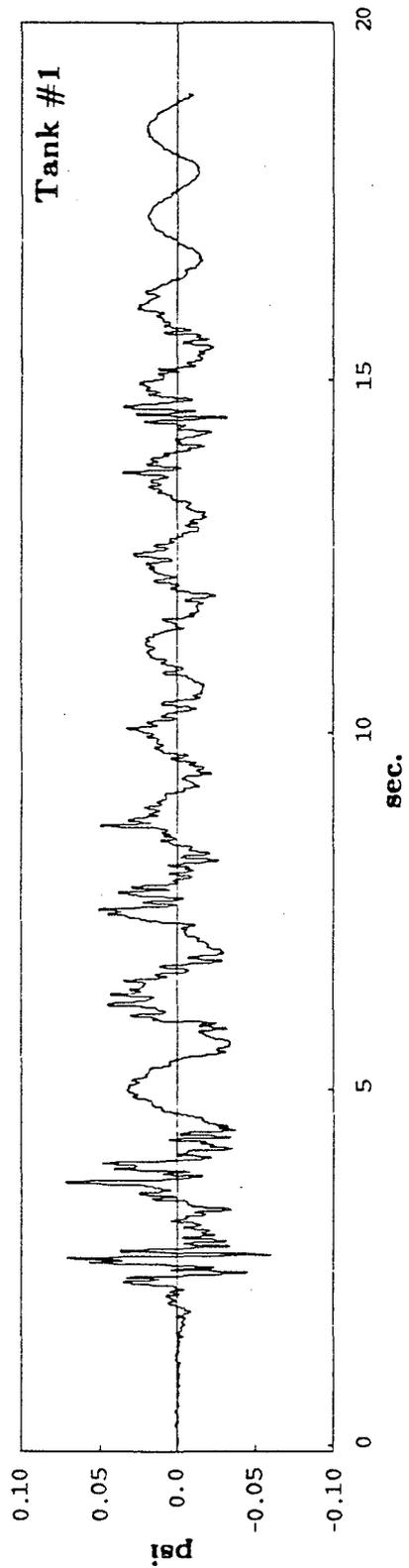


Figure 4.3a Measured pressure, north bottom transducer in tank #1. Table input:  $\sqrt{4}$  time scaled El Centro horizontal span 50, PTA=0.114 g.

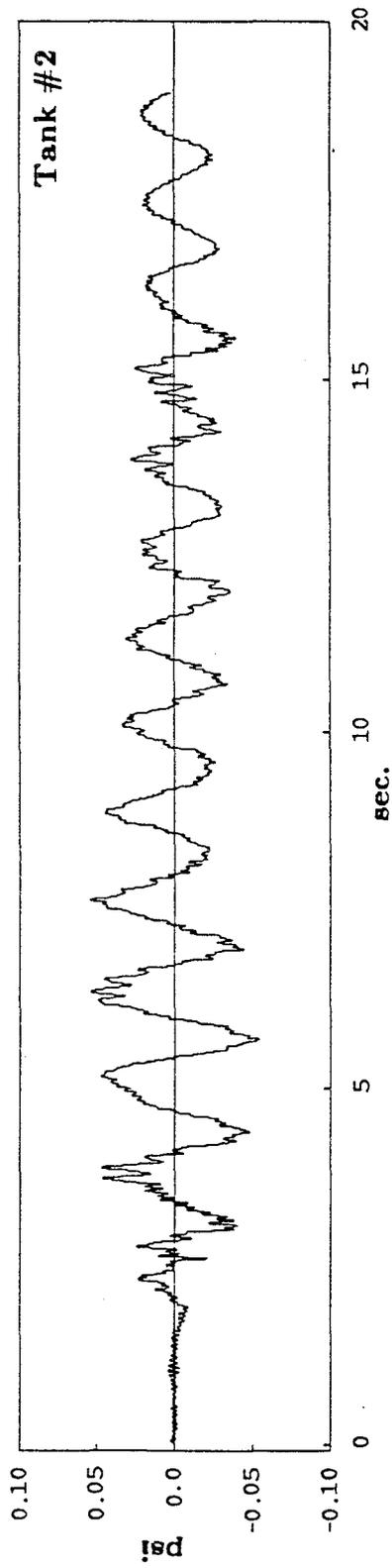


Figure 4.3b Measured pressure, north bottom transducer in tank #2. Table input:  $\sqrt{4}$  time scaled El Centro horizontal span 50, PTA=0.114 g.

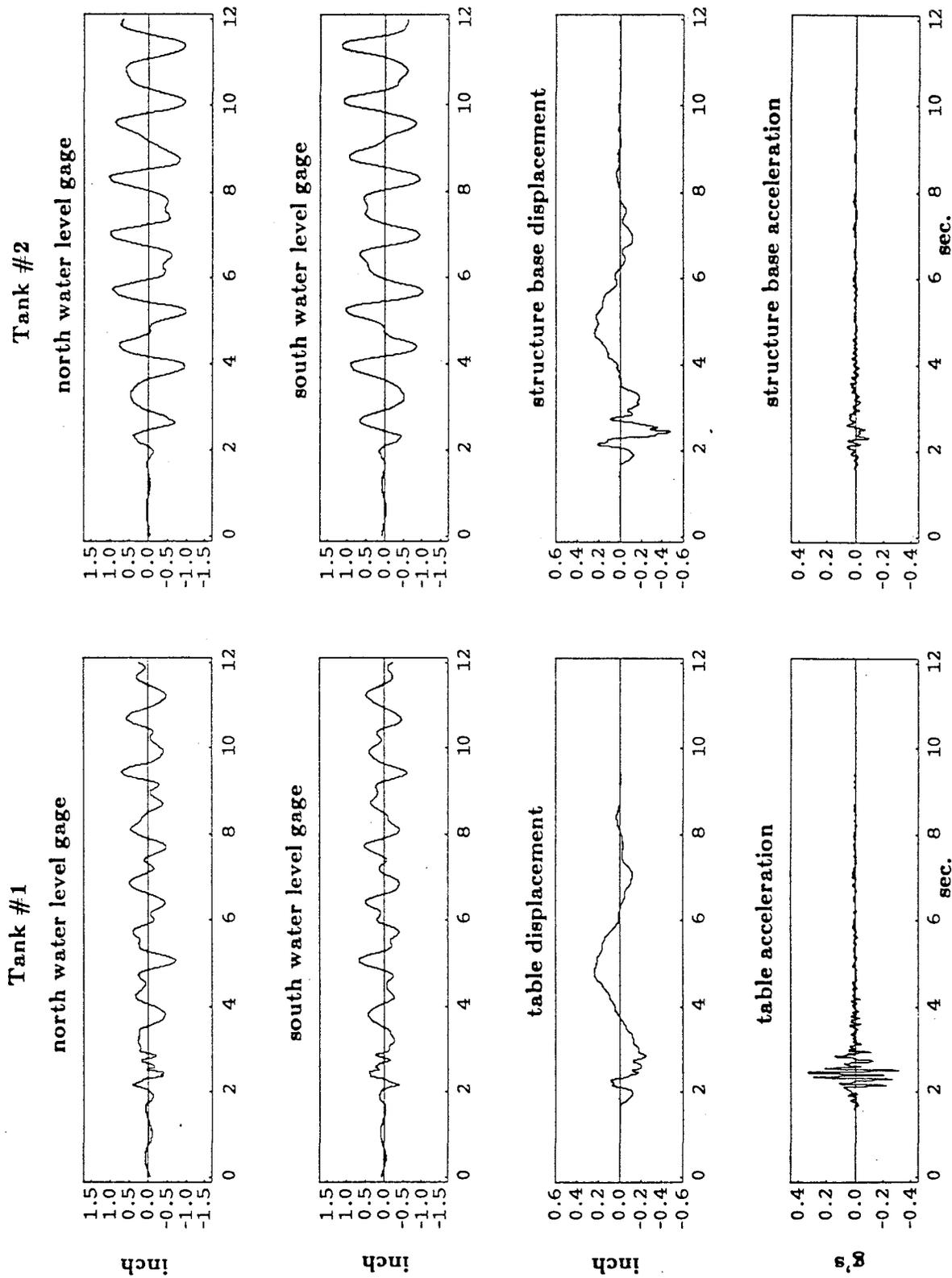


Figure 4.4 Water elevation at tank wall. Tank #1 and tank #2. Table input:  $\sqrt{4}$  time scaled San Francisco horizontal span 50, PTA=0.331 g.

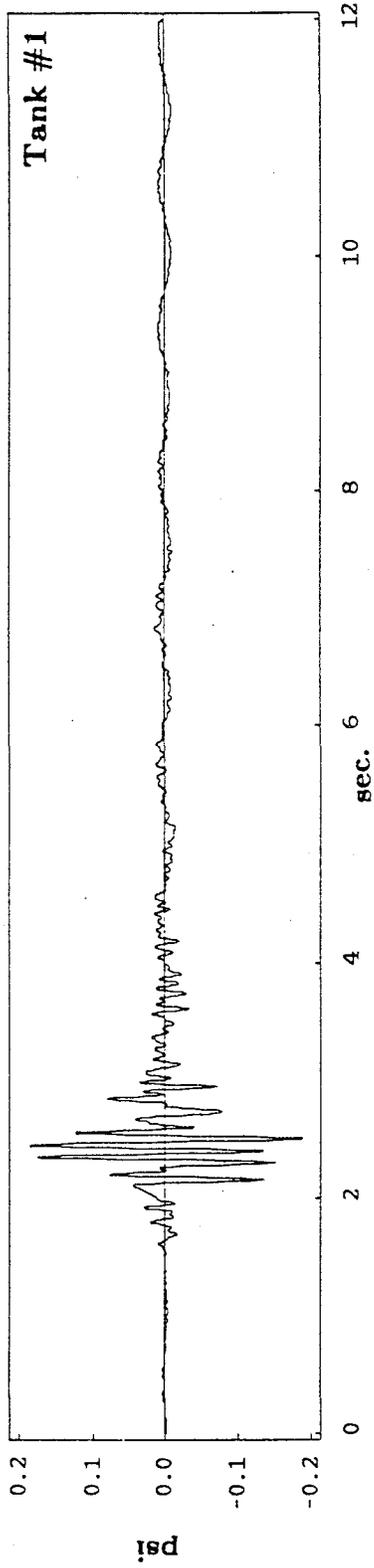


Figure 4.5a Measured pressure, north bottom transducer in tank #1. Table input:  $\sqrt{4}$  time scaled San Francisco horizontal span 50, PTA=0.331 g.

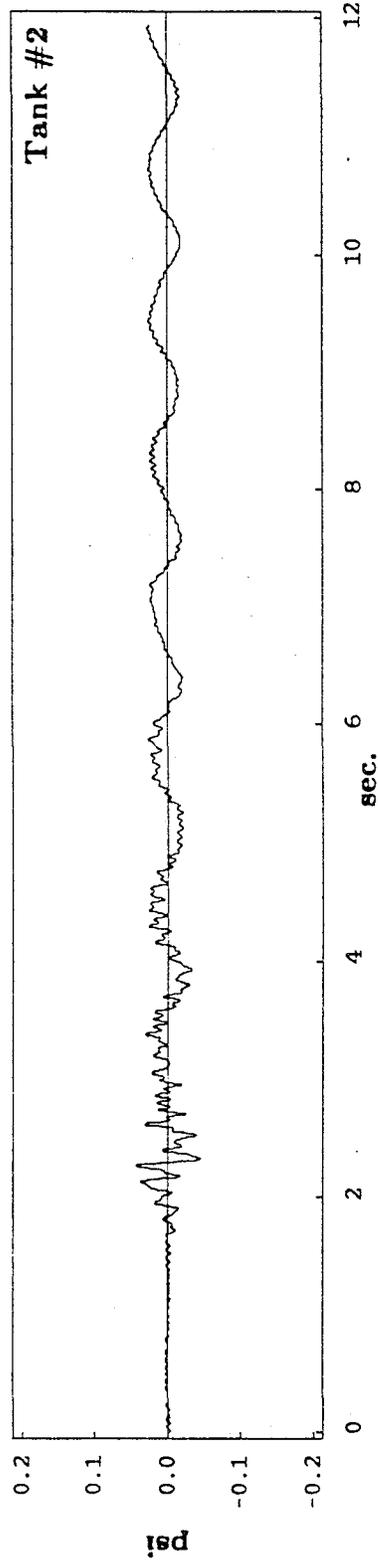


Figure 4.5b Measured pressure, north bottom transducer in tank #2. Table input:  $\sqrt{4}$  time scaled San Francisco horizontal span 50, PTA=0.331 g.

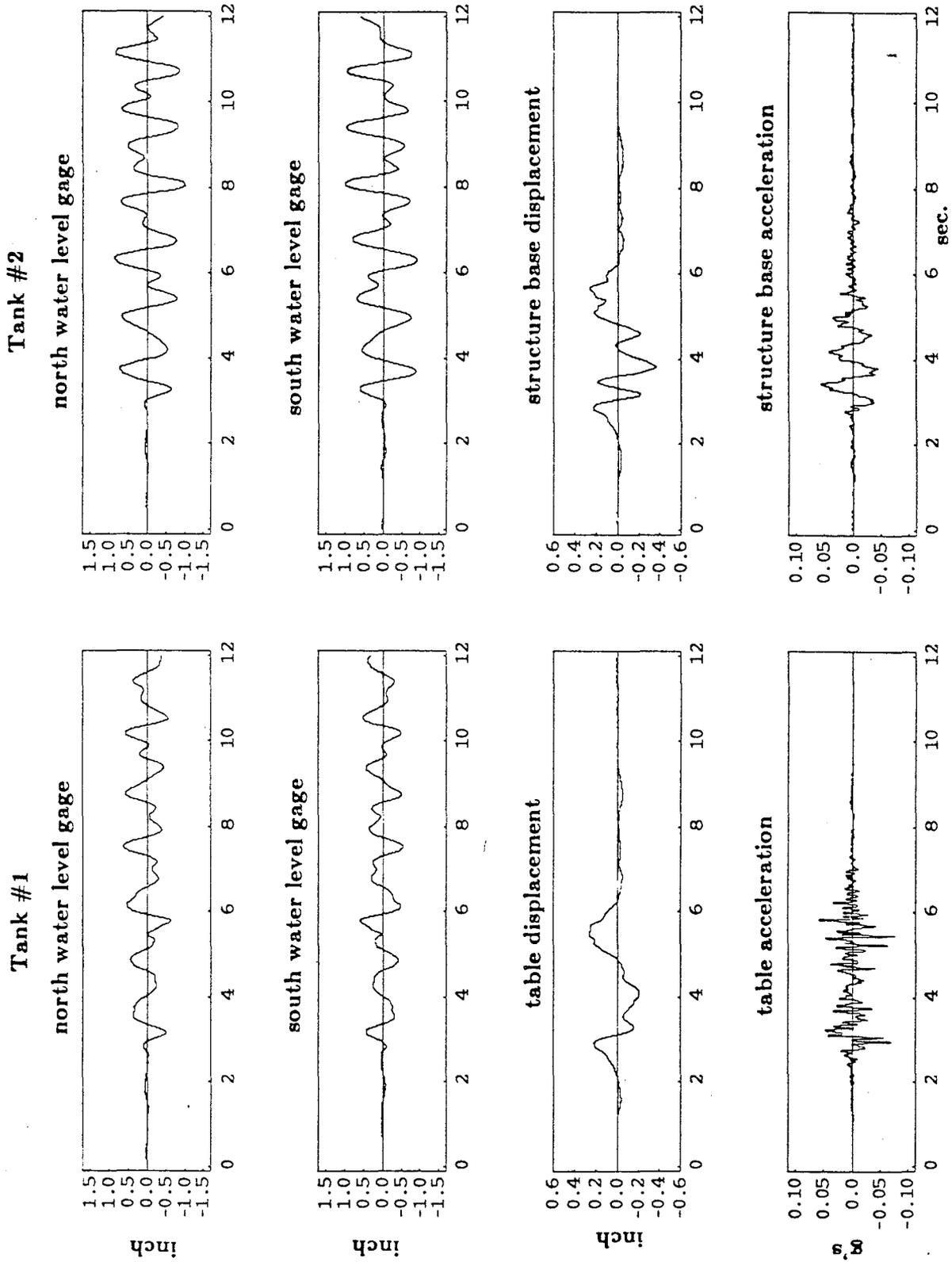


Figure 4.6 Water elevation at tank wall. Tank #1 and tank #2. Table input:  $\sqrt{4}$  time scaled Pacoima Dam horizontal span 50, PTA=0.076 g.

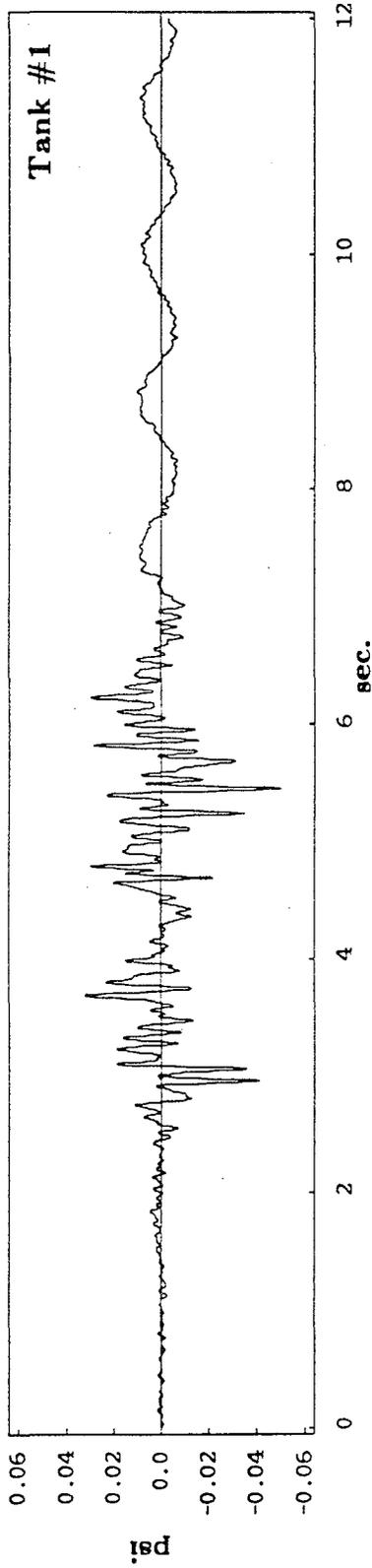


Figure 4.7a Measured pressure, north bottom transducer in tank #1. Table input:  $\sqrt{4}$  time scaled Pacoima Dam horizontal span 50, PTA=0.076 g.

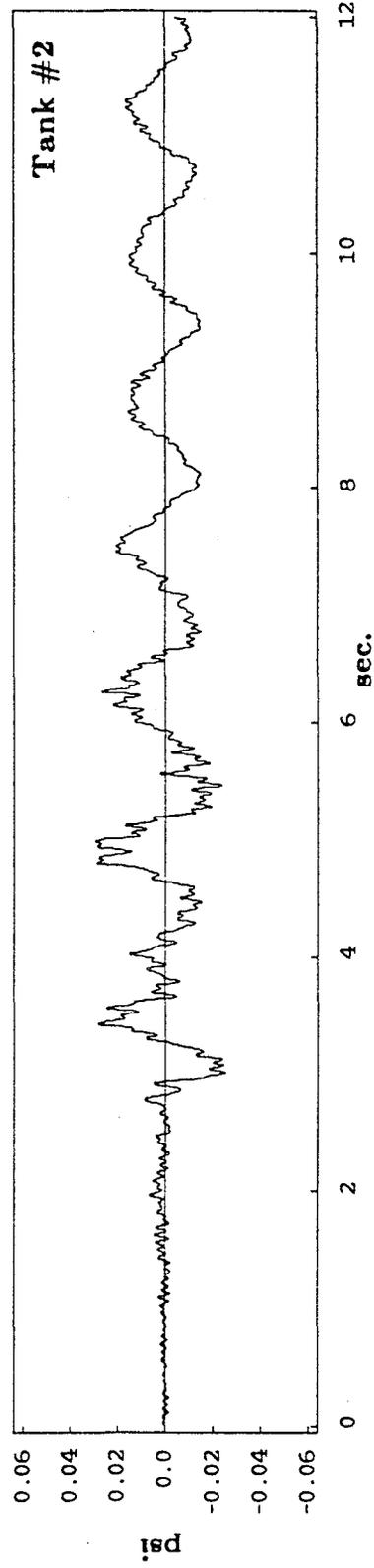


Figure 4.7b Measured pressure, north bottom transducer in tank #2. Table input:  $\sqrt{4}$  time scaled Pacoima Dam horizontal span 50, PTA=0.076 g.

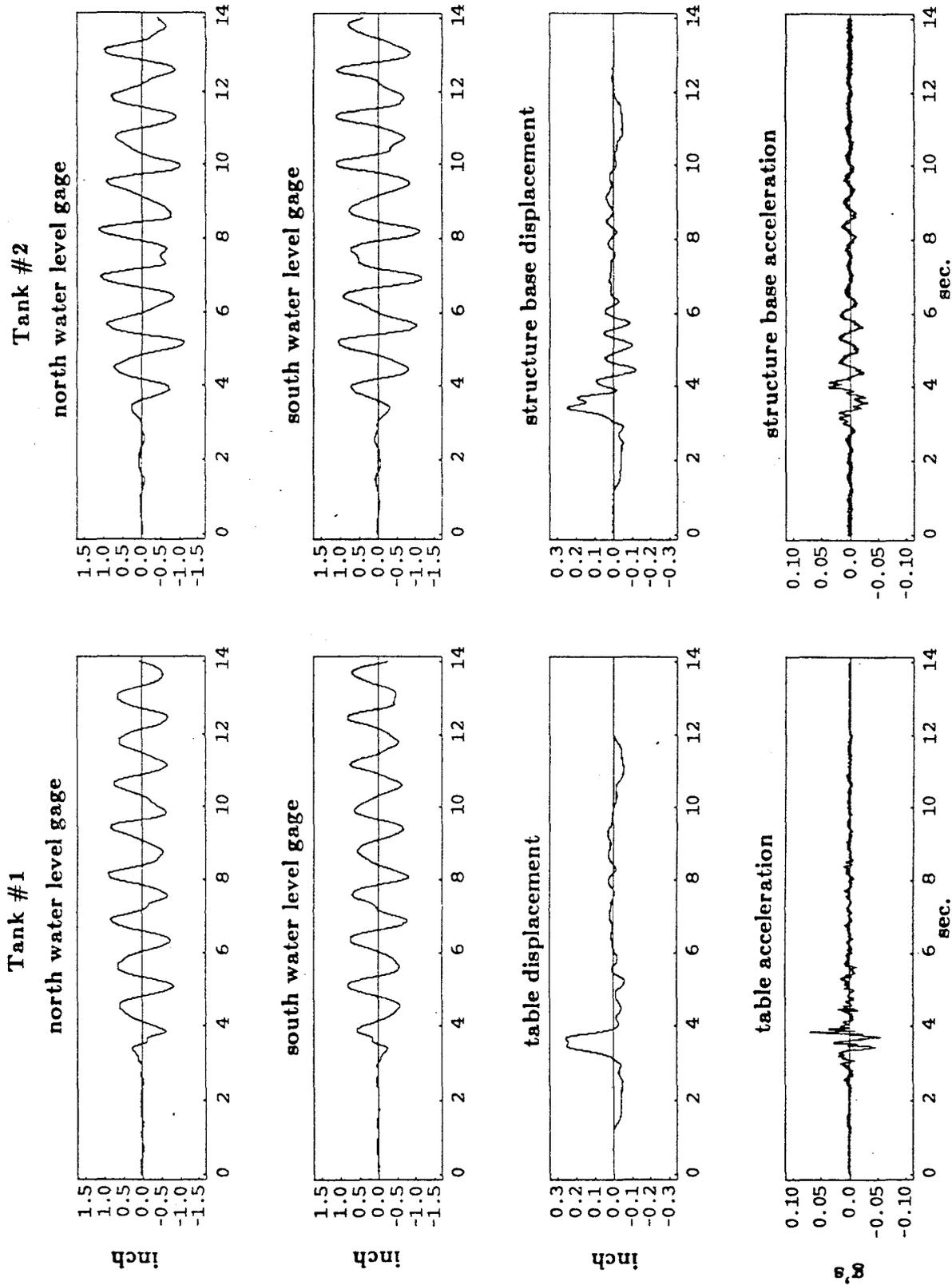


Figure 4.8 Water elevation at tank wall. Tank #1 and tank #2. Table input:  $\sqrt{4}$  time scaled Parkfield horizontal span 50, PTA=0.070 g.

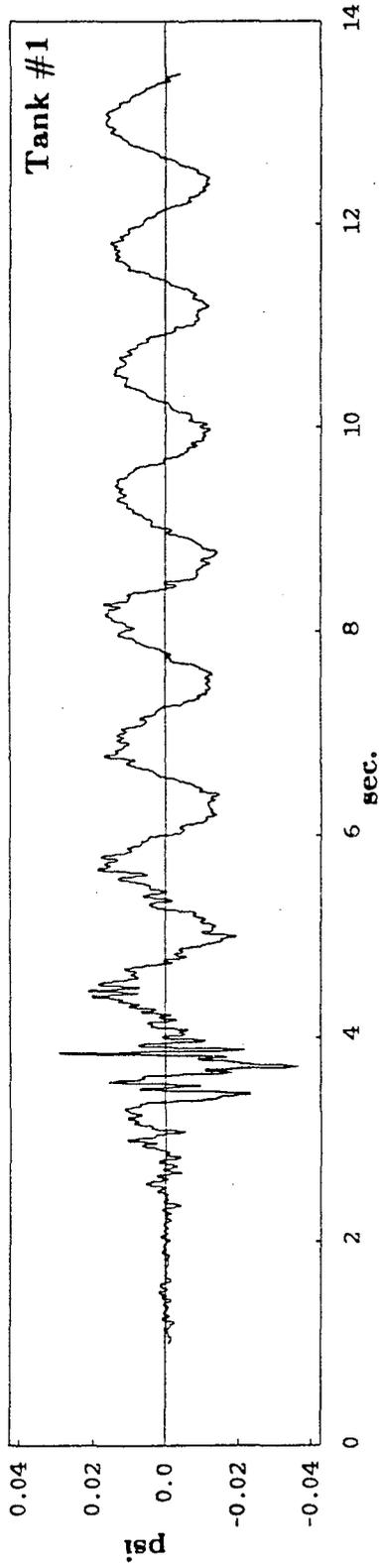


Figure 4.9a Measured pressure, north bottom transducer in tank #1. Table input:  $\sqrt{4}$  time scaled Parkfield horizontal span 50, PTA=0.070 g.

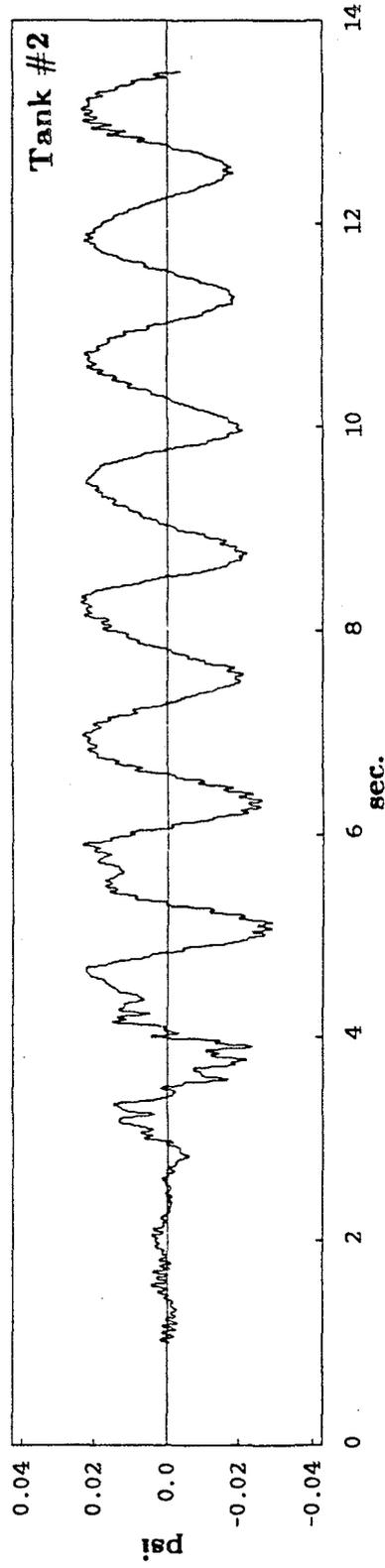


Figure 4.9b Measured pressure, north bottom transducer in tank #2. Table input:  $\sqrt{4}$  time scaled Parkfield horizontal span 50, PTA=0.070 g.

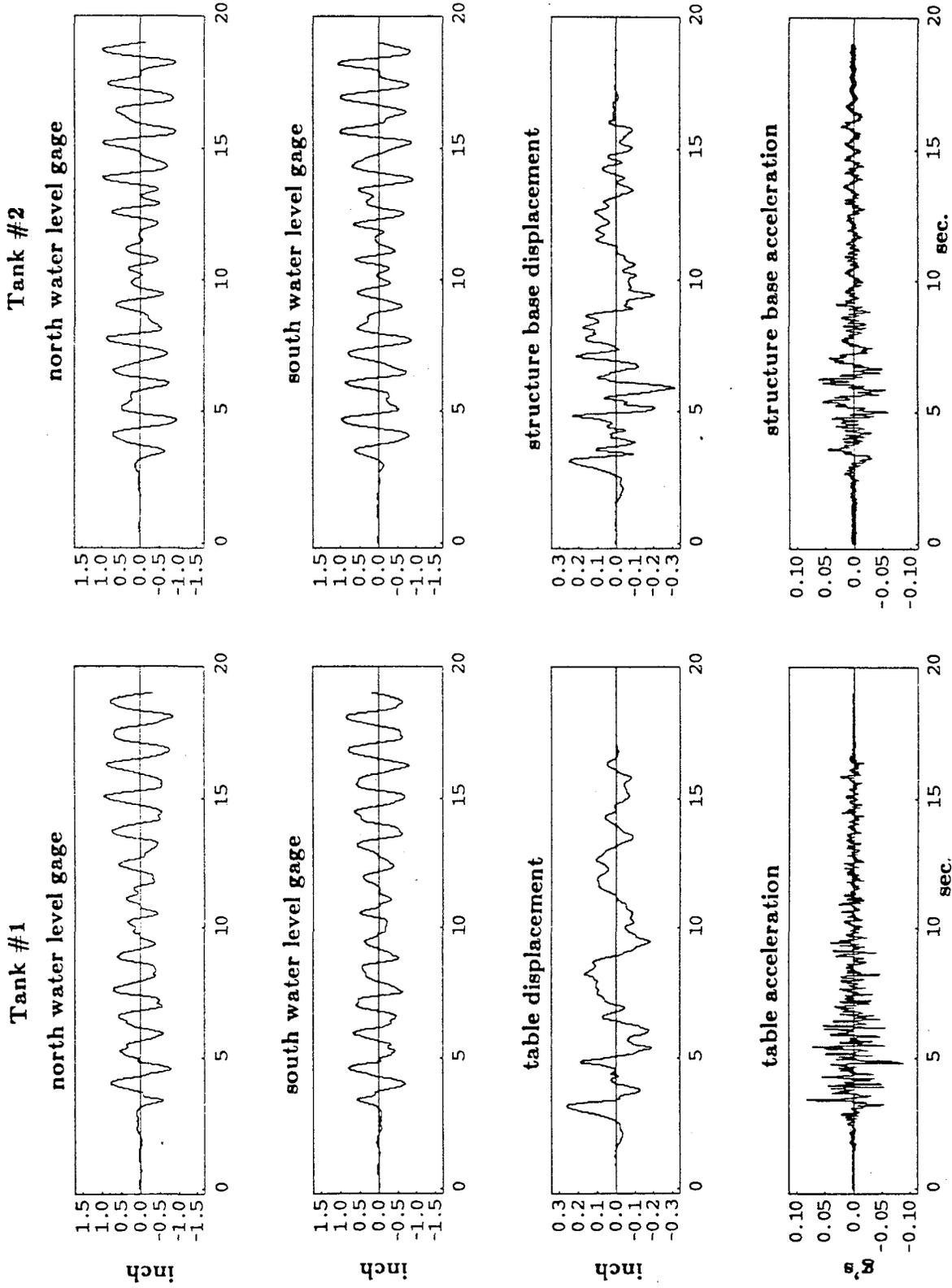


Figure 4.10 Water elevation at tank wall. Tank #1 and tank #2. Table input:  $\sqrt{4}$  time scaled Taft horizontal span 50, PTA=0.088 g.

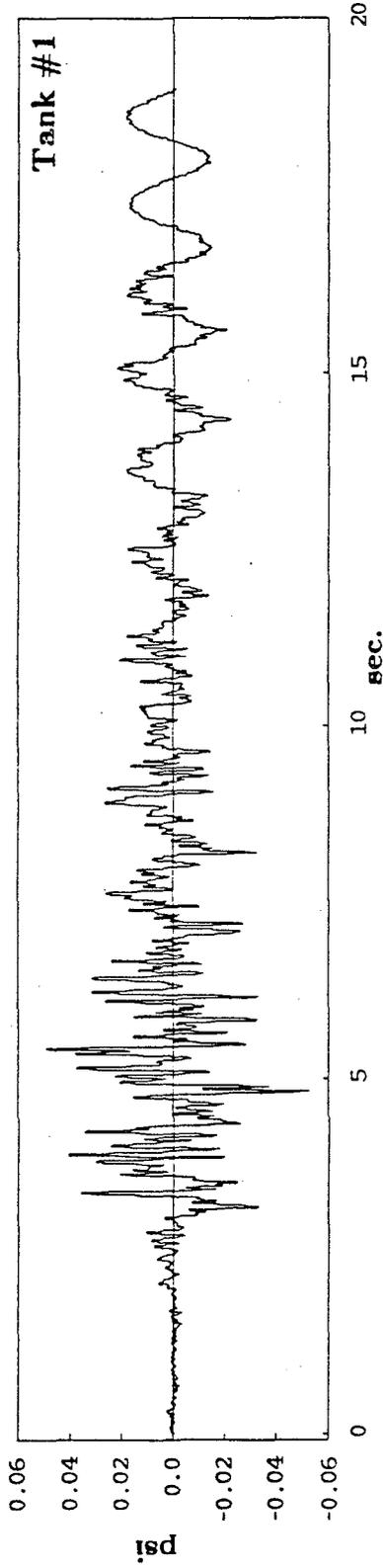


Figure 4.11a Measured pressure, north bottom transducer in tank #1. Table input:  $\sqrt{4}$  time scaled Taft horizontal span 50, PTA=0.088 g.

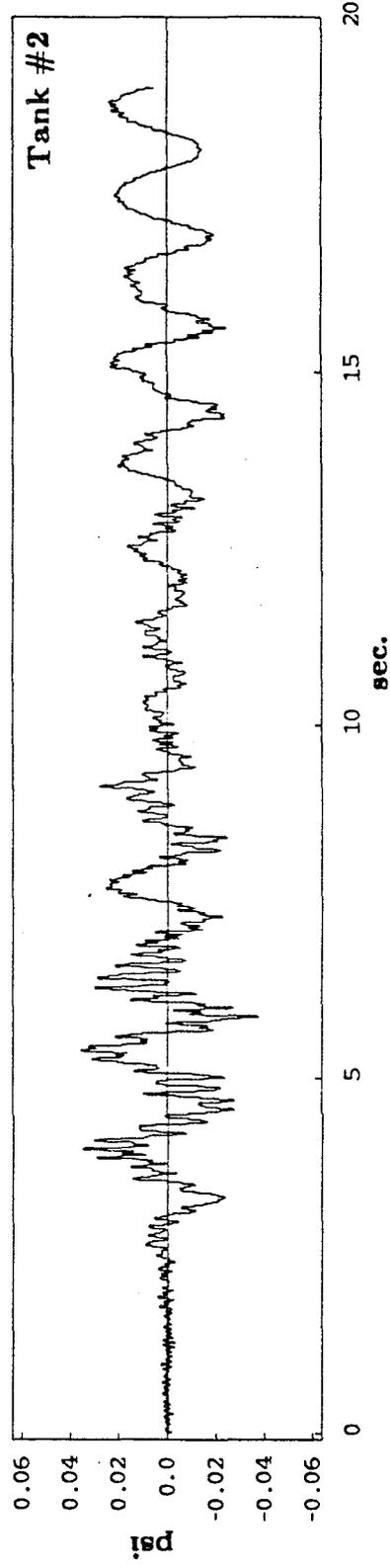
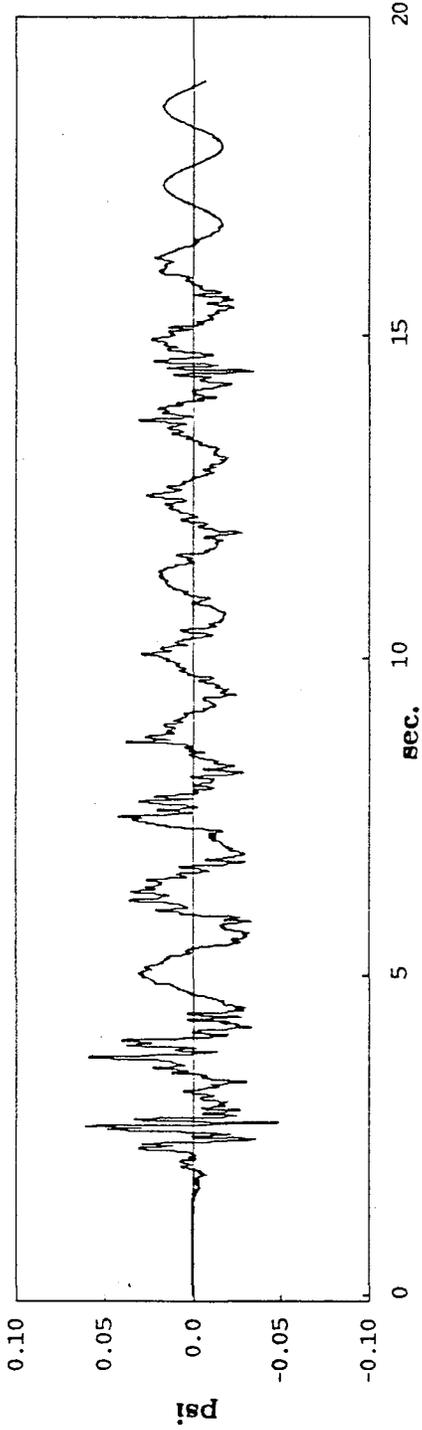
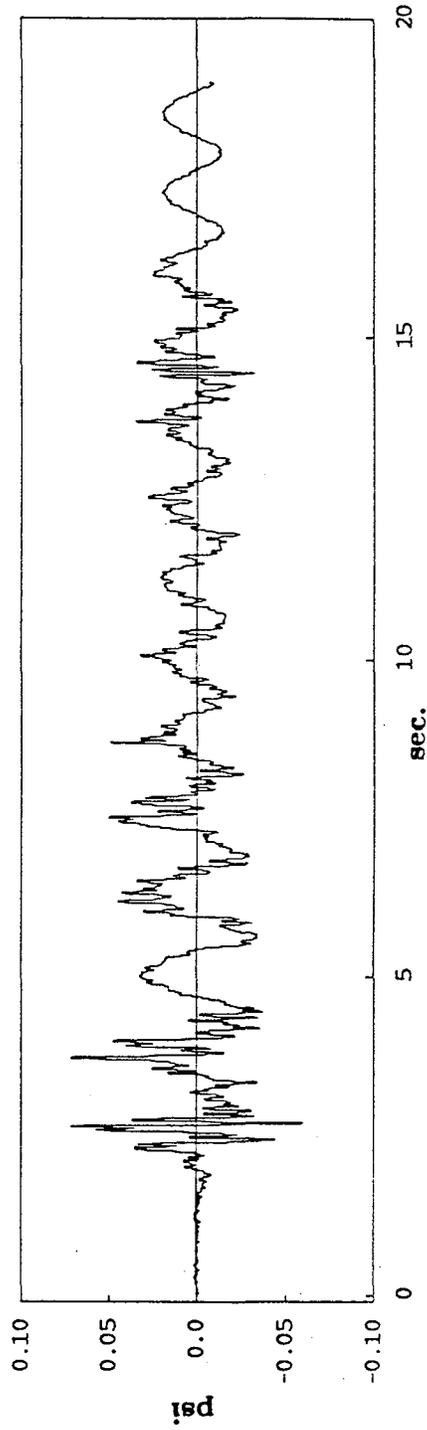


Figure 4.11b Measured pressure, north bottom transducer in tank #2. Table input:  $\sqrt{4}$  time scaled Taft horizontal span 50, PTA=0.088 g.



**Figure 4.12a** Calculated dynamic pressure  $P_d$  using two modes. Location:  $z=16$  in. Forcing function: recorded table acceleration. Table input:  $\sqrt{4}$  time scaled El Centro horizontal span 50, PTA=0.114 g.



**Figure 4.12b** Measured dynamic pressure at north bottom transducer in tank #1. Location:  $z=16$  in. Table input:  $\sqrt{4}$  time scaled El Centro horizontal span 50, PTA=0.114 g.

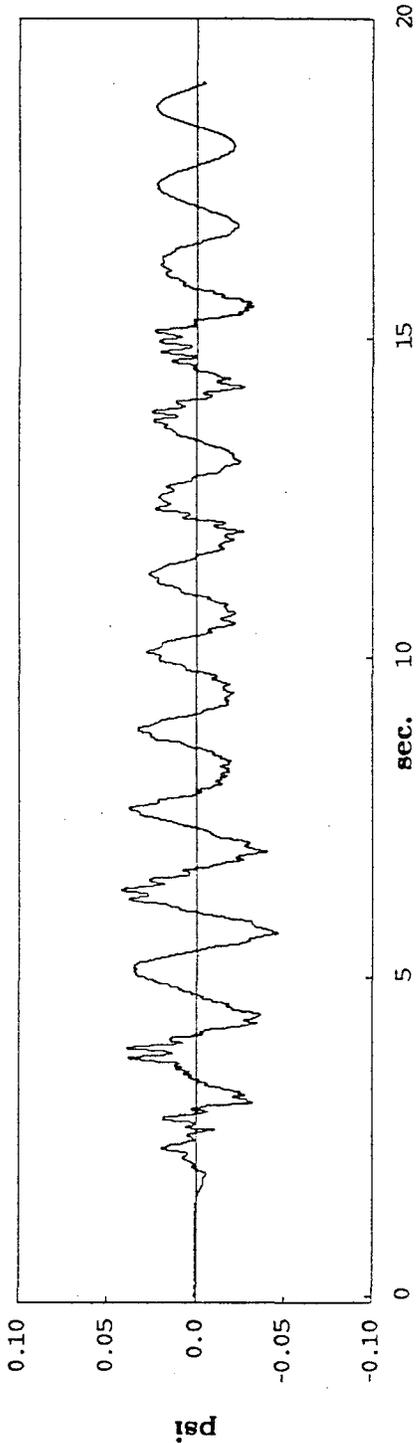


Figure 4.13a Calculated dynamic pressure  $P_d$  using two modes. Location:  $z=16$  in. Forcing function: recorded structure's isolated base acceleration. Table input:  $\sqrt{4}$  time scaled El Centro horizontal span 50, PTA=0.114 g.

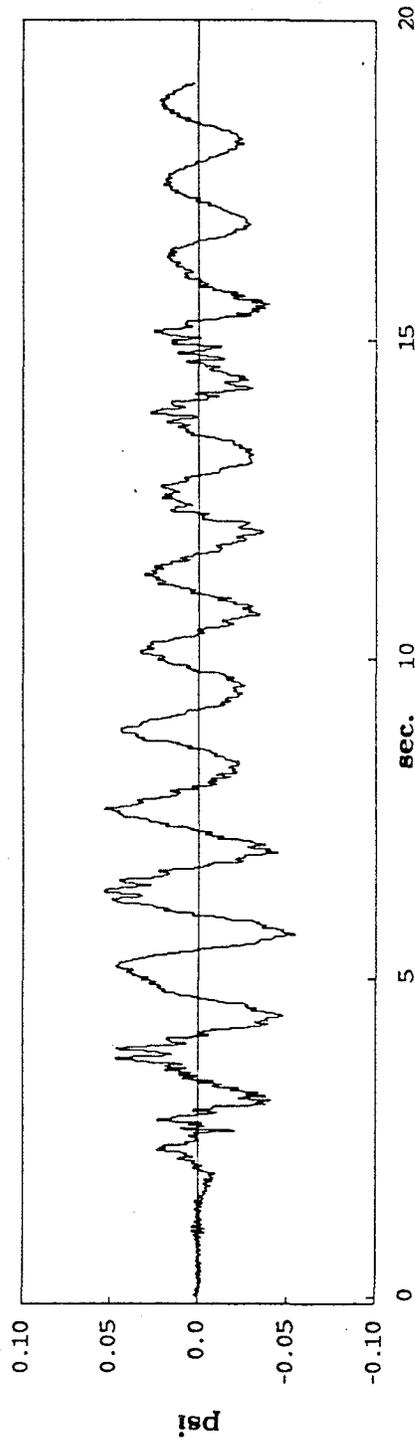


Figure 4.13b Measured dynamic pressure at north bottom transducer in tank #2. Location:  $z=16$  in. Table input:  $\sqrt{4}$  time scaled El Centro horizontal span 50, PTA=0.114 g.

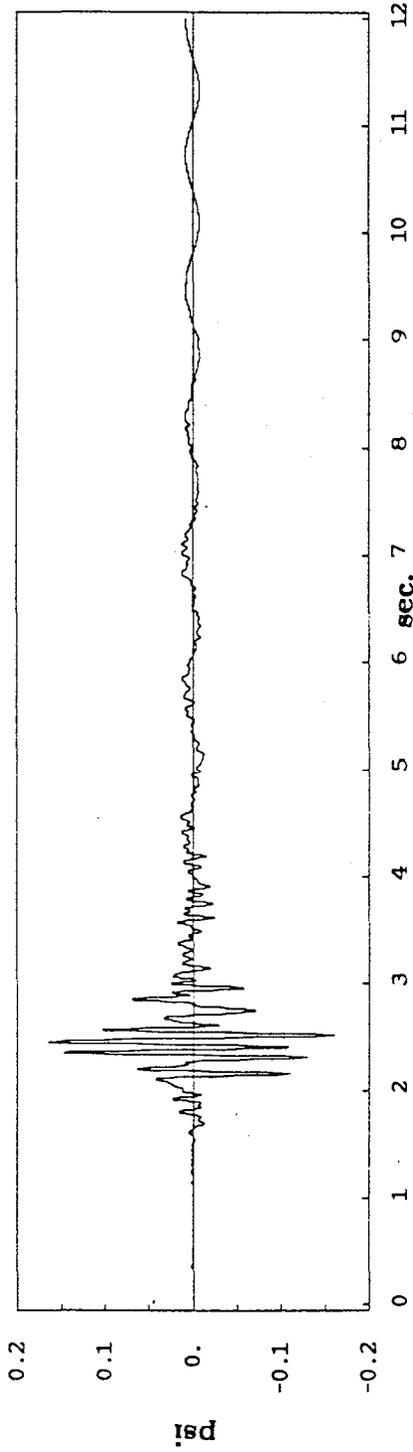


Figure 4.14a Calculated dynamic pressure  $P_d$  using two modes. Location:  $z=-16$  in. Forcing function: recorded table acceleration. Table input:  $\sqrt{4}$  time scaled San Francisco horizontal span 50, PTA=0.331 g.

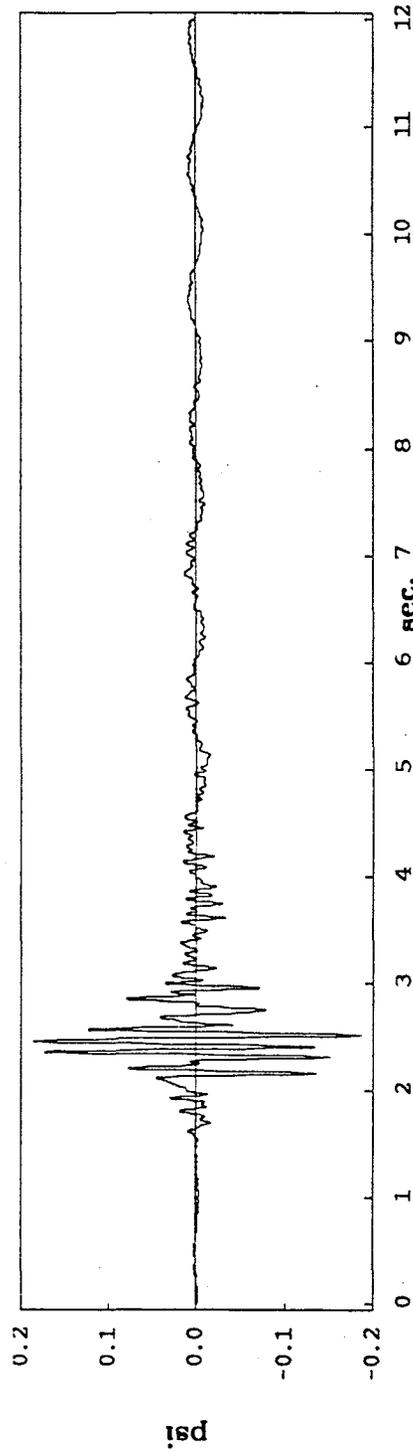


Figure 4.14b Measured dynamic pressure at north bottom transducer in tank #1. Location:  $z=-16$  in. Table input:  $\sqrt{4}$  time scaled San Francisco horizontal span 50, PTA=0.331 g.

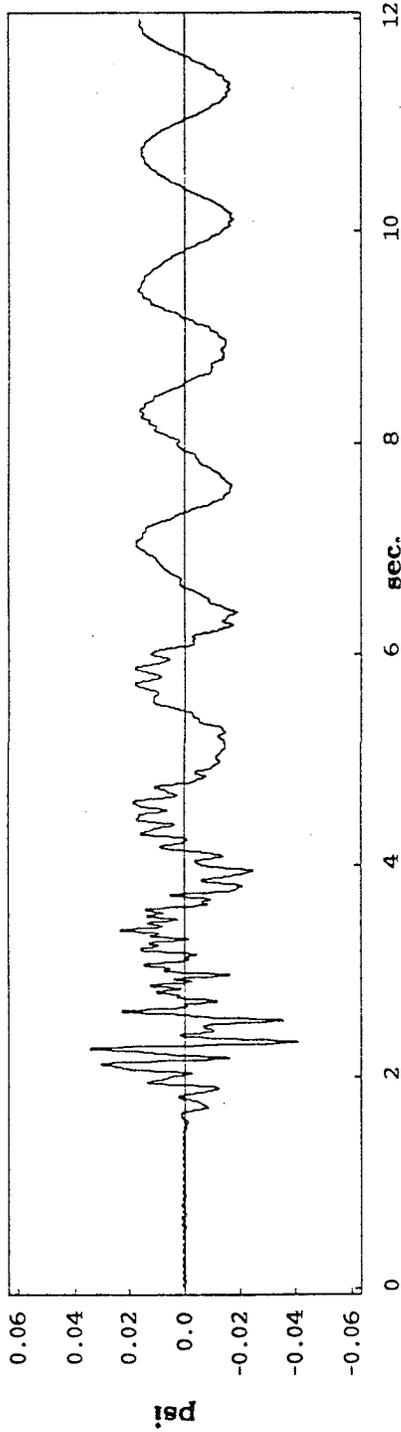


Figure 4.15a Calculated dynamic pressure  $P_d$  using two modes. Location:  $z=-16$  in. Forcing function: recorded structure's isolated base acceleration. Table input:  $\sqrt{4}$  time scaled San Francisco horizontal span 50, PTA=0.331 g.

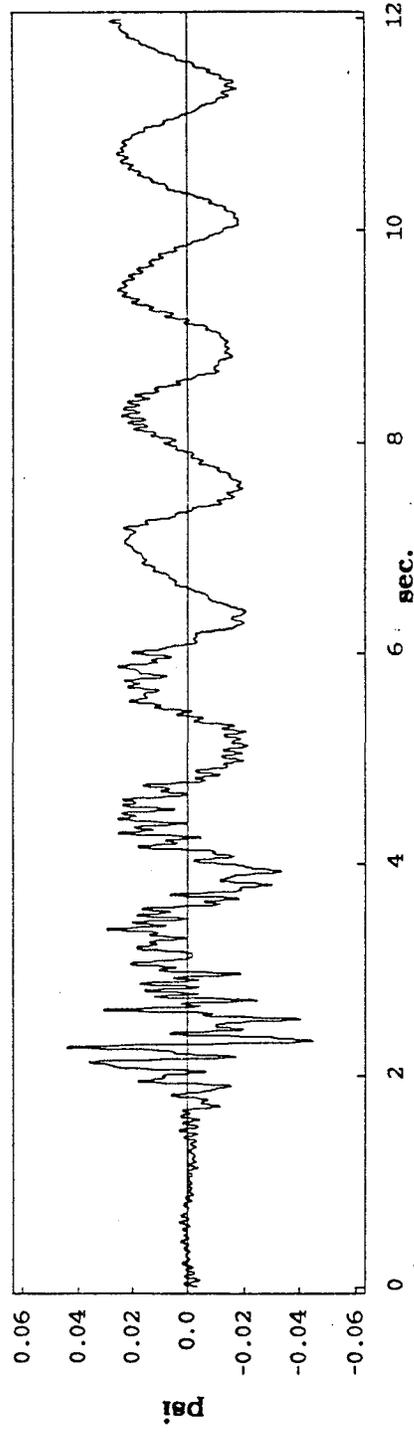


Figure 4.15b Measured dynamic pressure at north bottom transducer in tank #2. Location:  $z=-16$  in. Table input:  $\sqrt{4}$  time scaled San Francisco horizontal span 50, PTA=0.331 g.

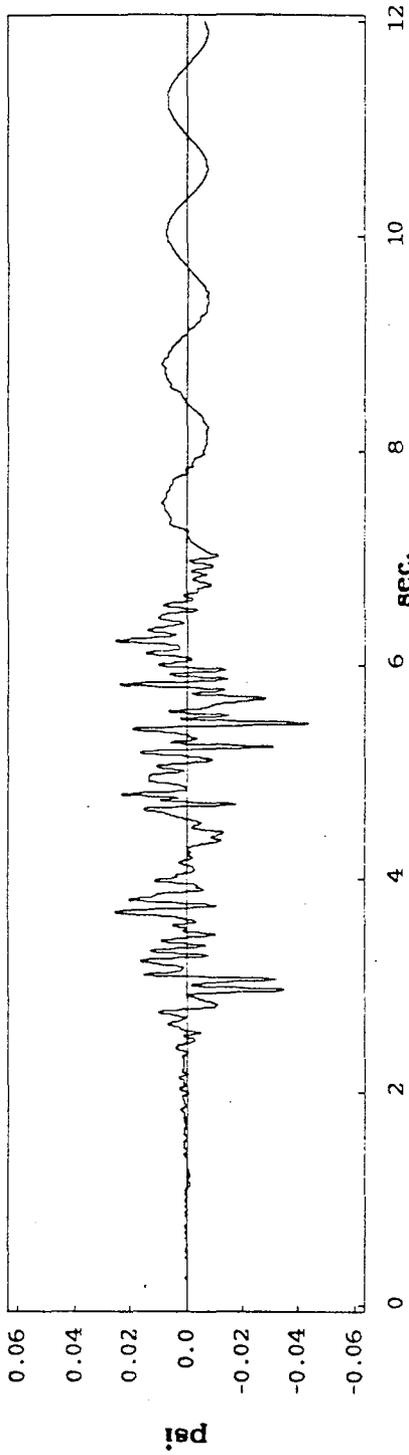


Figure 4.16a Calculated dynamic pressure  $P_d$  using two modes. Location:  $z=-16$  in. Forcing function: recorded table acceleration. Table input:  $\sqrt{4}$  time scaled Pacoima Dam horizontal span 50, PTA=0.076 g.

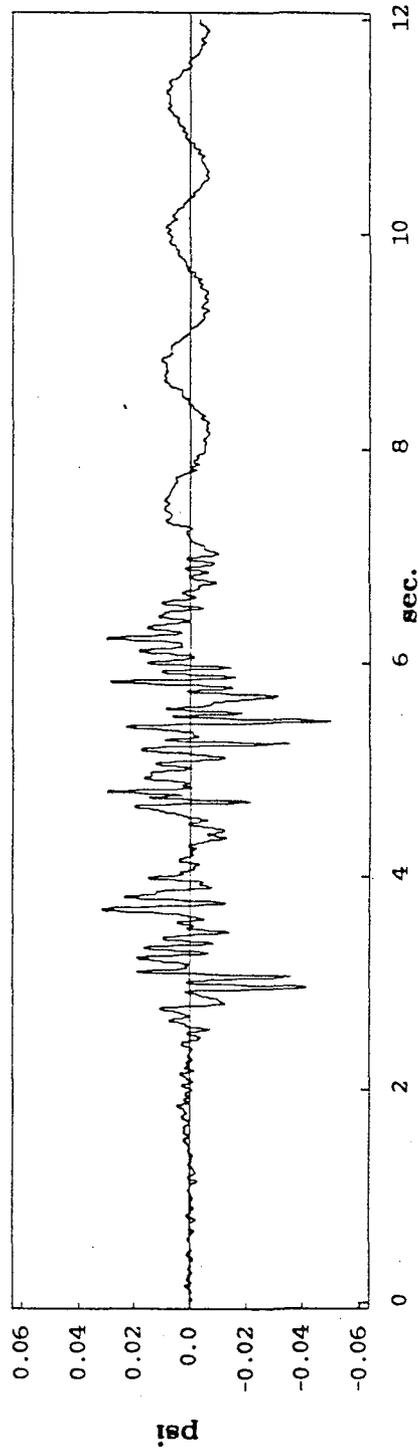


Figure 4.16b Measured dynamic pressure at north bottom transducer in tank #1. Location:  $z=-16$  in. Table input:  $\sqrt{4}$  time scaled Pacoima Dam horizontal span 50, PTA=0.076 g.

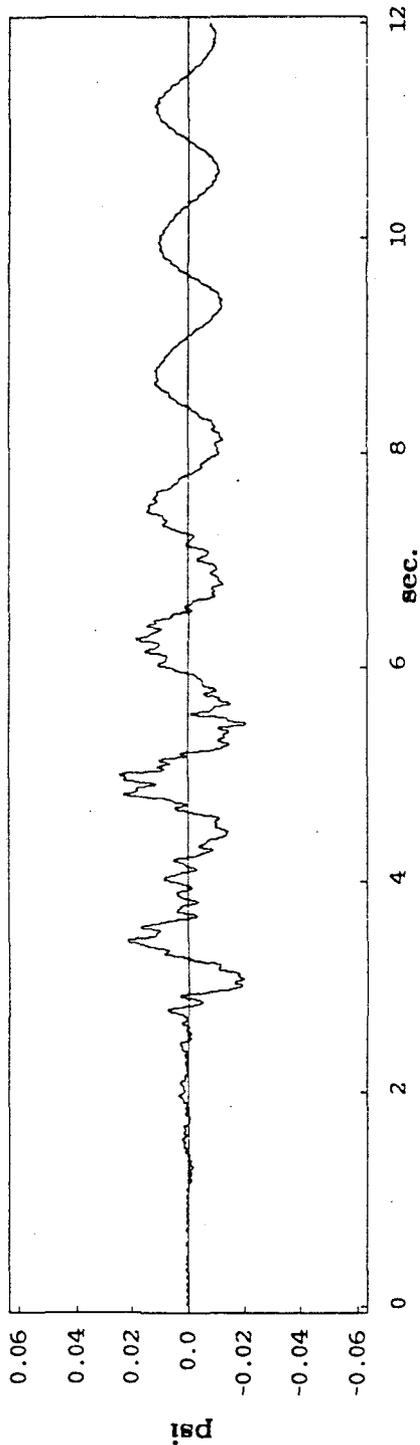


Figure 4.17a Calculated dynamic pressure  $P_d$  using two modes. Location:  $z=-16$  in. Forcing function: recorded structure's isolated base acceleration. Table input:  $\sqrt{4}$  time scaled Pacoima Dam horizontal span 50, PTA=0.076 g.

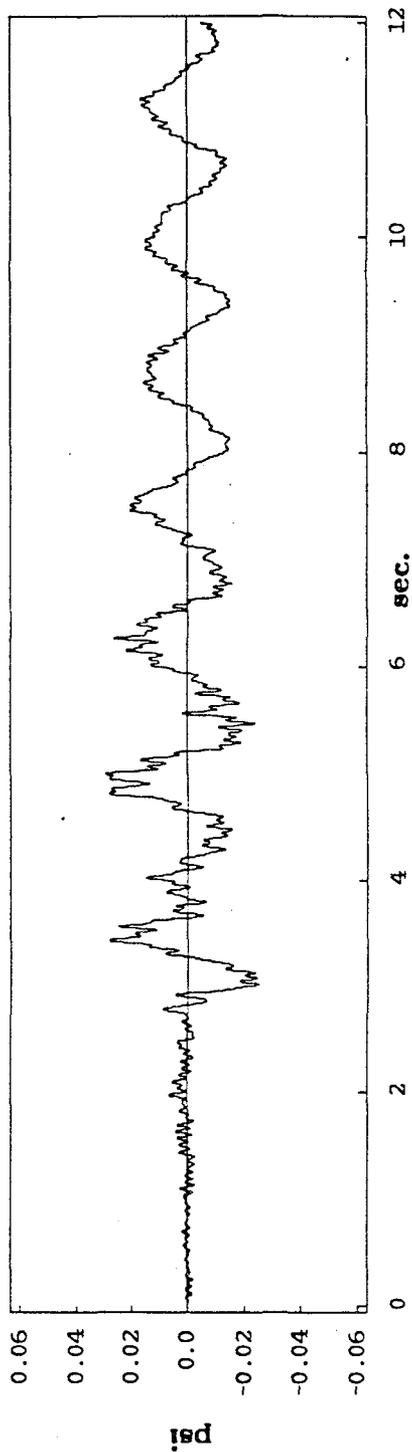


Figure 4.17b Measured dynamic pressure at north bottom transducer in tank #2. Location:  $z=-16$  in. Table input:  $\sqrt{4}$  time scaled Pacoima Dam horizontal span 50, PTA=0.076 g.

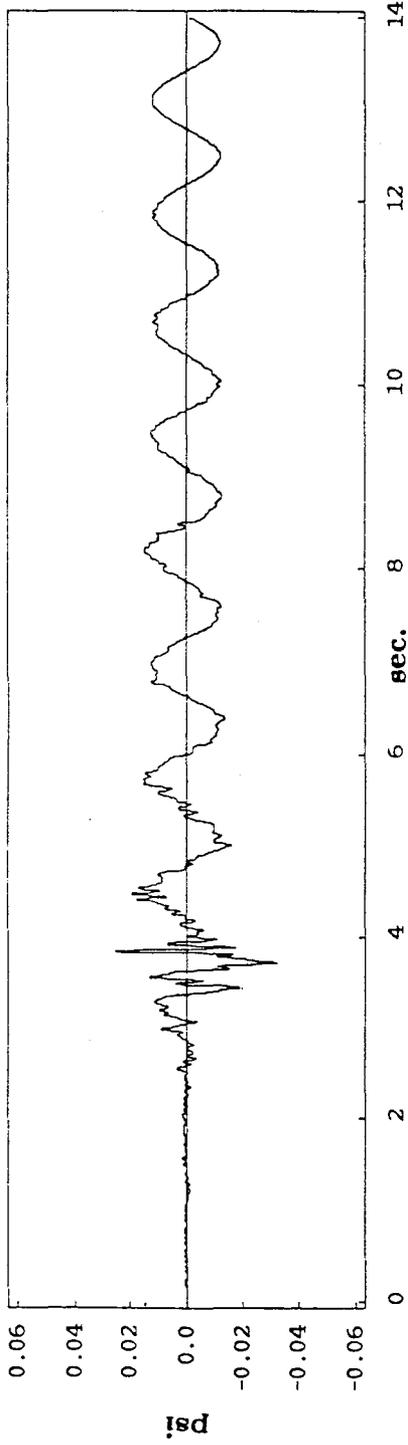


Figure 4.18a Calculated dynamic pressure  $P_d$  using two modes. Location:  $z=-16$  in. Forcing function: recorded table acceleration. Table input:  $\sqrt{4}$  time scaled Parkfield horizontal span 50, PTA=0.070 g.

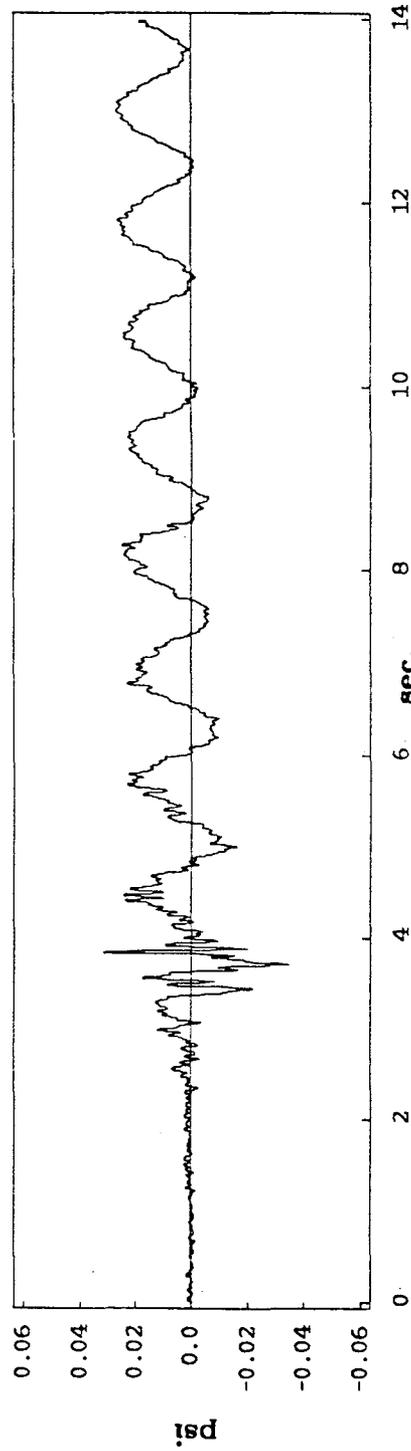


Figure 4.18b Measured dynamic pressure at north bottom transducer in tank #1. Location:  $z=-16$  in. Table input:  $\sqrt{4}$  time scaled Parkfield horizontal span 50, PTA=0.070 g.

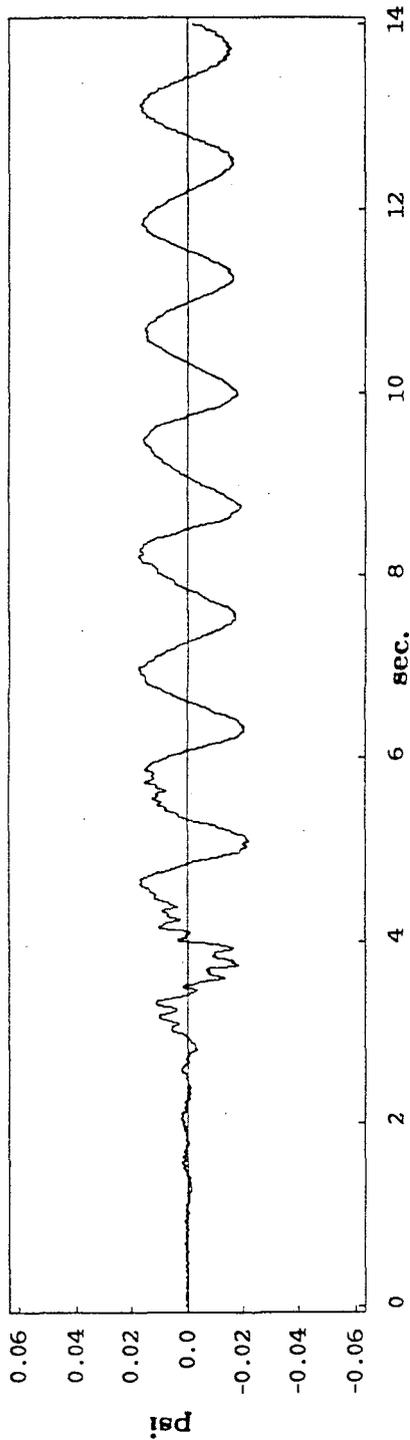


Figure 4.19a Calculated dynamic pressure  $P_d$  using two modes. Location:  $z=-16$  in. Forcing function: recorded structure's isolated base acceleration. Table input:  $\sqrt{4}$  time scaled Parkfield horizontal span 50, PTA=0.070 g.

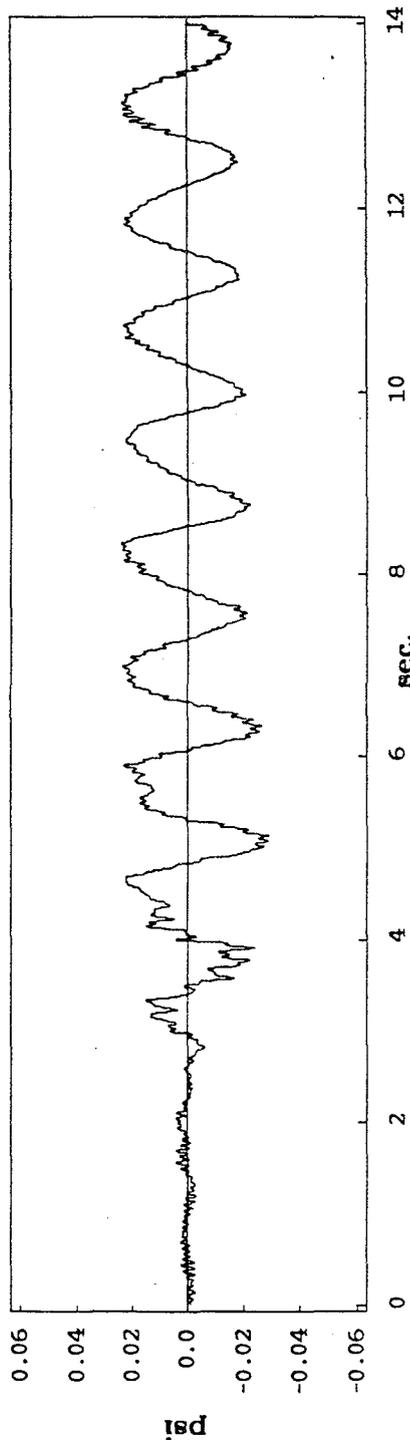


Figure 4.19b Measured dynamic pressure at north bottom transducer in tank #2. Location:  $z=-16$  in. Table input:  $\sqrt{4}$  time scaled Parkfield horizontal span 50, PTA=0.070 g.

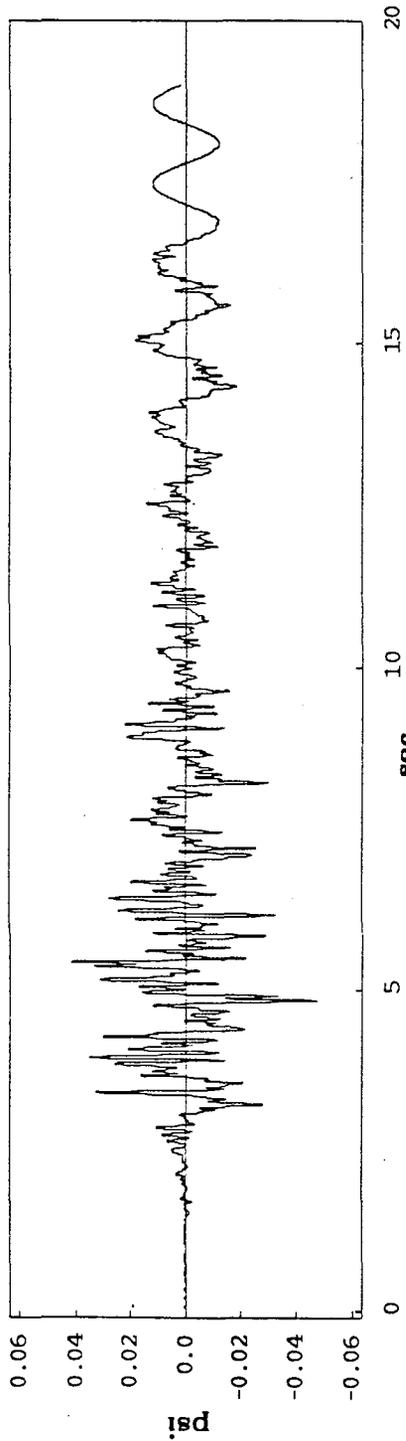


Figure 4.20a Calculated dynamic pressure  $P_d$  using two modes. Location:  $z=-16$  in. Forcing function: recorded table acceleration. Table input:  $\sqrt{4}$  time scaled Taft horizontal span 50, PTA=0.088 g.

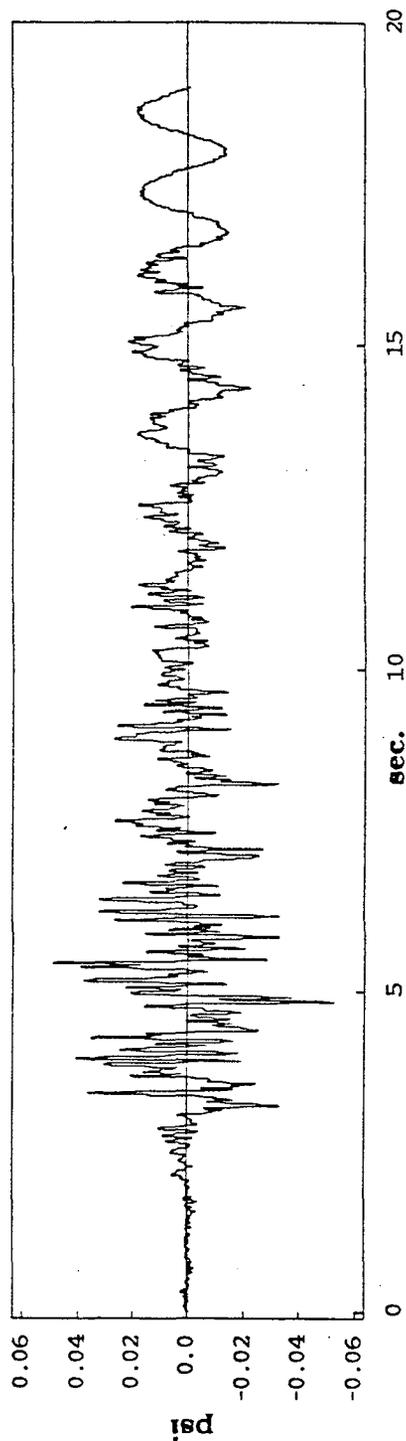


Figure 4.20b Measured dynamic pressure at north bottom transducer in tank #1. Location:  $z=-16$  in. Table input:  $\sqrt{4}$  time scaled Taft horizontal span 50, PTA=0.088 g.

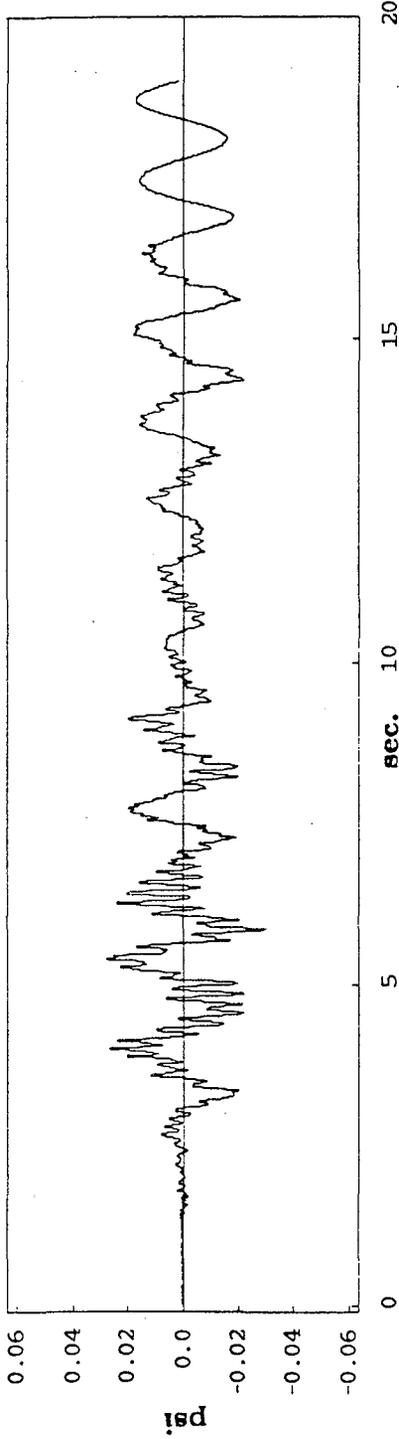


Figure 4.21a Calculated dynamic pressure  $P_d$  using two modes. Location:  $z=-16$  in. Forcing function: recorded structure's isolated base acceleration. Table input:  $\sqrt{4}$  time scaled Parkfield horizontal span 50, PTA=0.088 g.

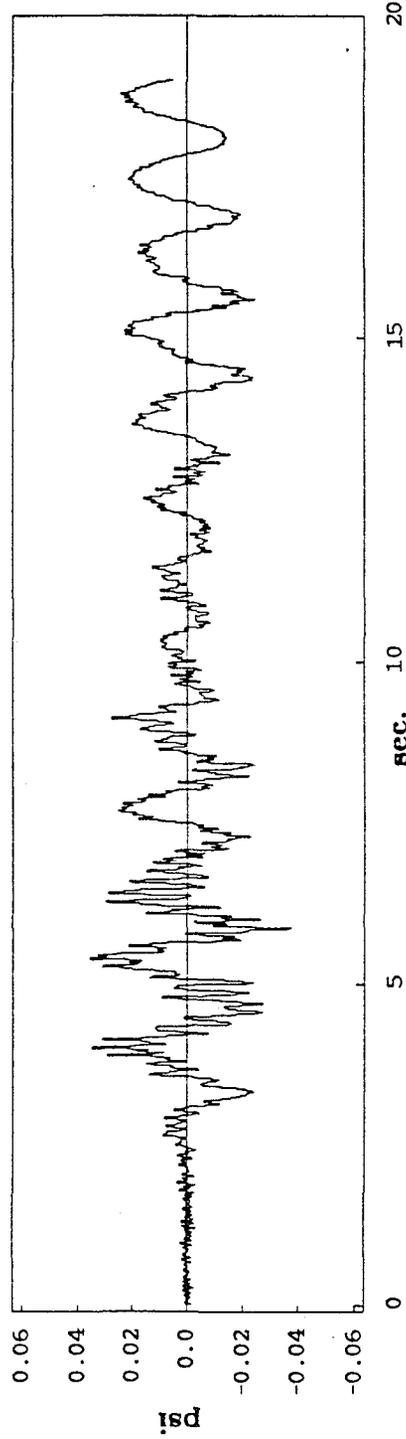


Figure 4.21b Measured dynamic pressure at north bottom transducer in tank #2. Location:  $z=-16$  in. Table input:  $\sqrt{4}$  time scaled Taft horizontal span 50, PTA=0.088 g.

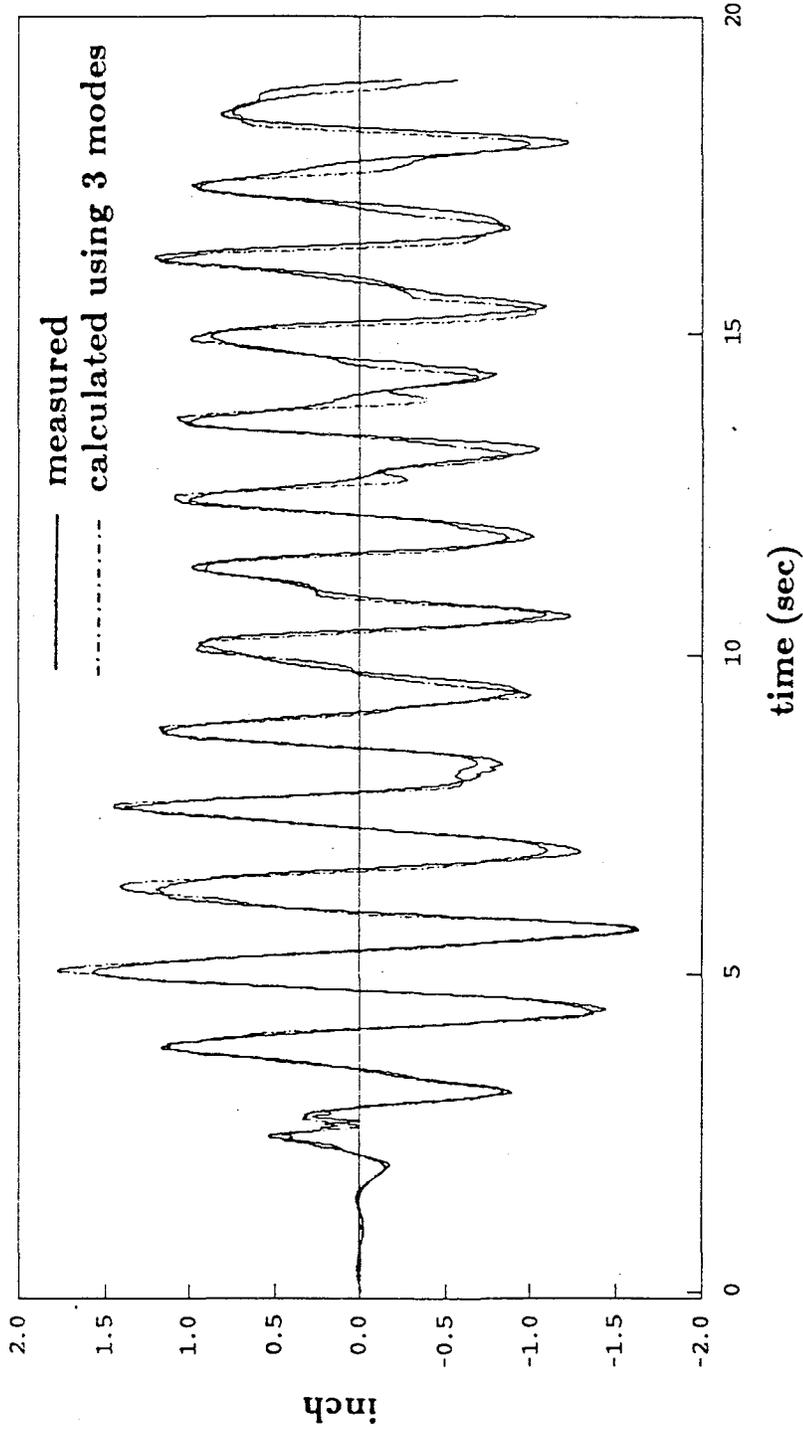


Figure 4.22 Comparison of measured water elevation time history to the calculated one, at tank wall for tank #1 (on table). Table input:  $\sqrt{4}$  time scaled El Centro horizontal span 50, PTA=0.114 g.

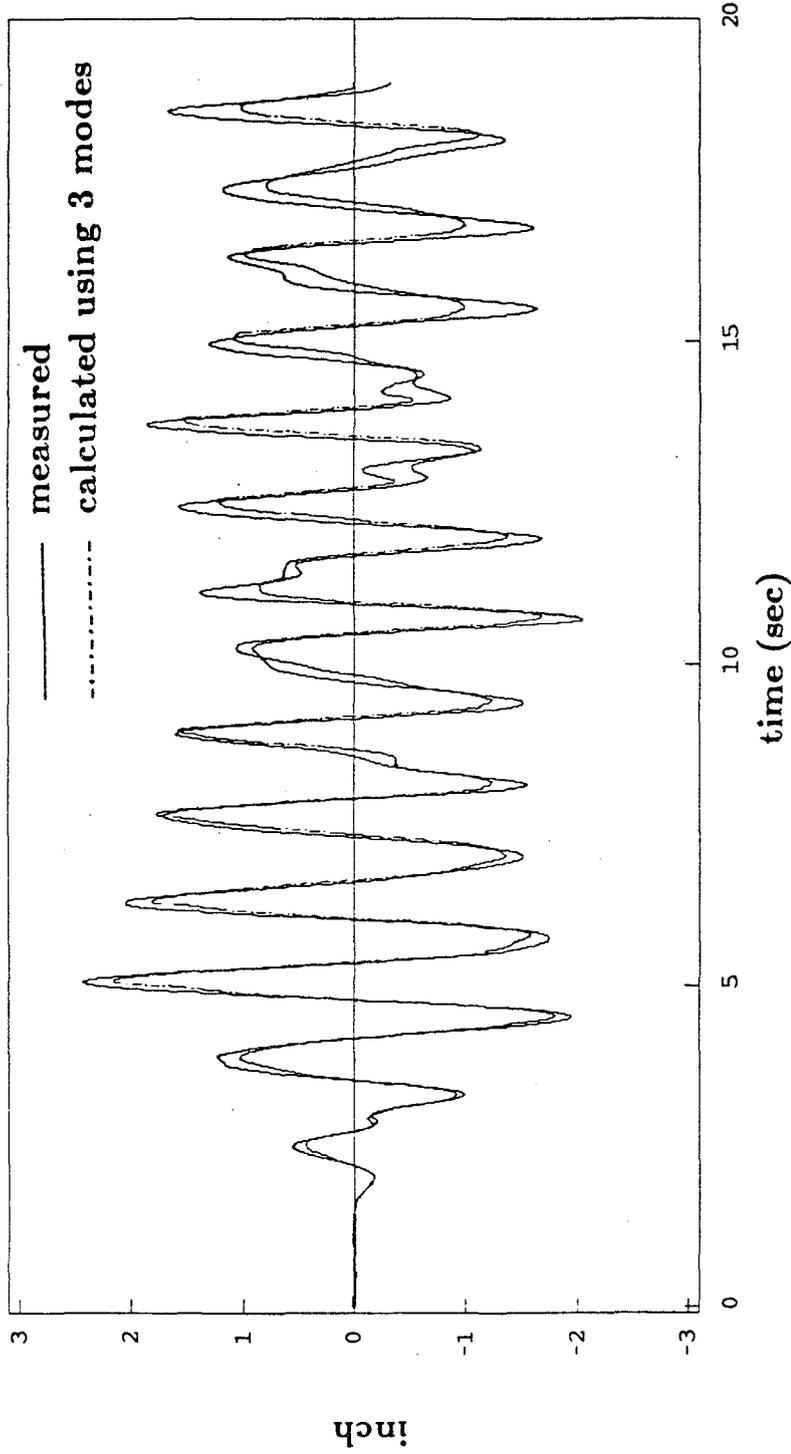


Figure 4.23 Comparison of measured water elevation time history to the calculated one, at tank wall for tank #2 (on structure). Table input:  $\sqrt{4}$  time scaled El Centro horizontal span 50, PTA=0.114 g.

## CHAPTER FIVE CONCLUSIONS

Damage to tanks in recent earthquakes has shown the need for new design techniques. In this report, a specific application was pursued. Tanks were treated as internal equipment in a structure mounted on a base isolation system. The findings, however, can be easily applied to tanks supported independently on the ground, or tanks sharing a common rigid slab.

Base isolated structures are characterized by their low frequency motion. There is concern about the effect of this motion on fluid containers housed in such structures. In this study water tanks were mounted on the earthquake simulator and on the structure when a solely rubber isolation system was used. The effect of base isolation on the tanks treated as internal equipment is summarized by a slight increase in sloshing and a reduction in the dynamic pressure exerted by the fluid on the tank wall. A linear wave theory was used in the theoretical solution. The final expressions for the dynamic pressure and the water elevation consist of a term that directly depends on the ground acceleration and a series of convolution integrals each corresponding to an oscillation mode of the fluid. These expressions were compared with the measured pressure and water elevation time histories and very good agreement was found by using only the first few modes. These expressions can be now directly applied to a prototype subjected to an arbitrary support excitation.

Mechanical analogies for this problem have been proposed [6]. The simplicity of the solution presented in this report renders the transformation of the system into a mechanical analog unnecessary. Also, unlike the mechanical models that provide only the overall behavior, the present solution provides the desired quantities at any location in the fluid by using only a few modes. Moreover, the mechanical model proposed in [6], which consists of a number of horizontal single degree-of-freedom oscillators, ignores the effects of vertical support excitation while the proposed formulas accommodate vertical support excitations by replacing the gravitational field acceleration by the resultant vertical acceleration time history.

From the expressions for the pressure and the water elevation time histories, expressions for upper bounds of these quantities are derived and peak response prediction can be easily achieved whenever the design ground motion is provided in terms of its peak acceleration and its response spectra.

A major limitation of the experimental set-up was the scaling problem. It becomes more important to include the deformational characteristics of a tank when its dimensions are larger. The interaction of the shell vibrations with the fluid oscillations can affect the response. In this case, however, the difference between the measured and calculated pressure time histories was the presence of very high frequencies in the measured one. The amplitudes of these superposed high frequencies were very small and did not really affect the response. The earthquakes were time-scaled by a factor of two because of the geometric scaling of the structural model. If the tanks were tested independently, the time-scale factor could be increased to as high as 10 in order to extrapolate the results to realistic, commonly-used tank dimensions.

An important topic of future research is the study of a tank directly mounted on a flexible foundation, and its response determined for different isolation systems of various stiffnesses and damping capacities. Moreover, the use of different water depths in the tank would lead to the formation of waves of different lengths and traveling at different frequencies.

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