

**NATIONAL CENTER FOR EARTHQUAKE
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State University of New York at Buffalo

**Experimental and Analytical Study of a
Hybrid Isolation System Using Friction
Controllable Sliding Bearings**

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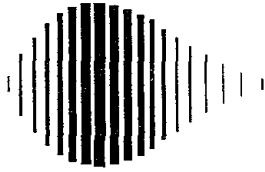
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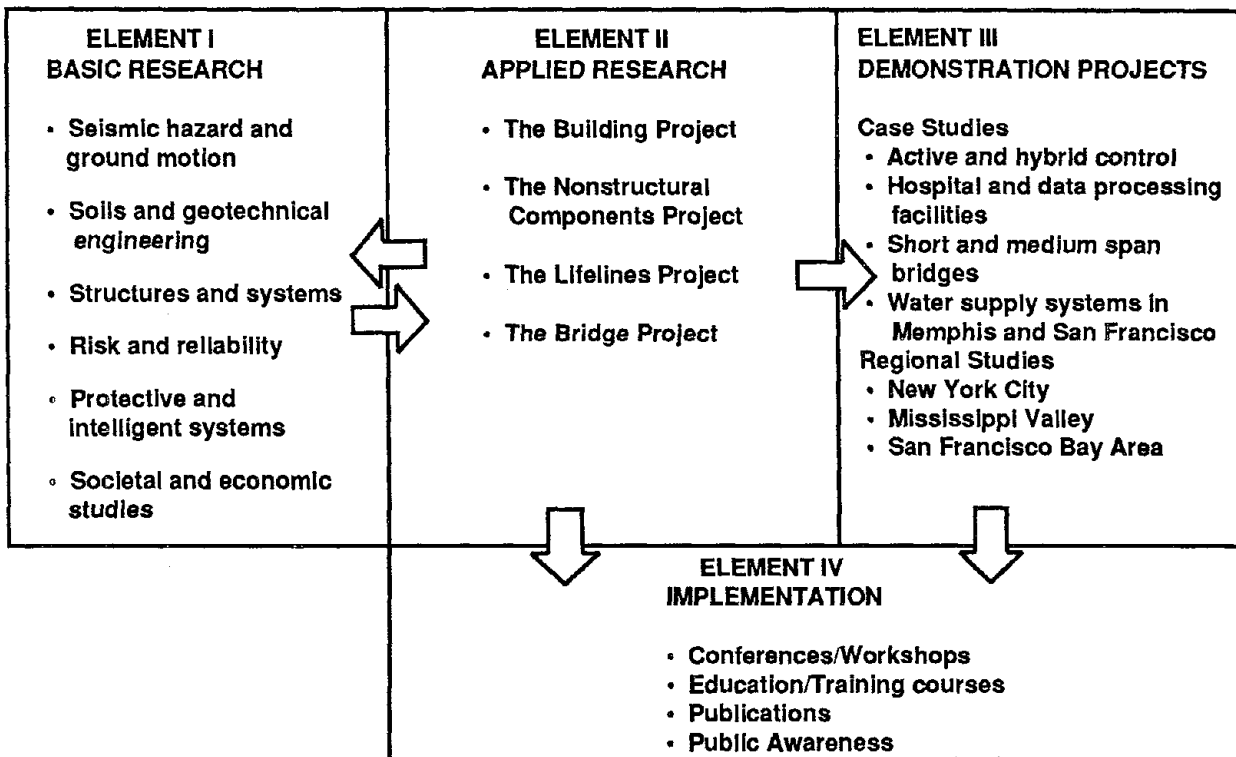
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PREFACE

The National Center for Earthquake Engineering Research (NCEER) was established to expand and disseminate knowledge about earthquakes, improve earthquake-resistant design, and implement seismic hazard mitigation procedures to minimize loss of lives and property. The emphasis is on structures in the eastern and central United States and lifelines throughout the country that are found in zones of low, moderate, and high seismicity.

NCEER's research and implementation plan in years six through ten (1991-1996) comprises four interlocked elements, as shown in the figure below. Element I, Basic Research, is carried out to support projects in the Applied Research area. Element II, Applied Research, is the major focus of work for years six through ten. Element III, Demonstration Projects, have been planned to support Applied Research projects, and will be either case studies or regional studies. Element IV, Implementation, will result from activity in the four Applied Research projects, and from Demonstration Projects.



Research in the **Building Project** focuses on the evaluation and retrofit of buildings in regions of moderate seismicity. Emphasis is on lightly reinforced concrete buildings, steel semi-rigid frames, and masonry walls or infills. The research involves small- and medium-scale shake table tests and full-scale component tests at several institutions. In a parallel effort, analytical models and computer programs are being developed to aid in the prediction of the response of these buildings to various types of ground motion.

Two of the short-term products of the **Building Project** will be a monograph on the evaluation of lightly reinforced concrete buildings and a state-of-the-art report on unreinforced masonry.

The **protective and intelligent systems program** constitutes one of the important areas of research in the **Building Project**. Current tasks include the following:

1. Evaluate the performance of full-scale active bracing and active mass dampers already in place in terms of performance, power requirements, maintenance, reliability and cost.
2. Compare passive and active control strategies in terms of structural type, degree of effectiveness, cost and long-term reliability.
3. Perform fundamental studies of hybrid control.
4. Develop and test hybrid control systems.

This report describes the effectiveness of a hybrid isolation system using friction controllable sliding bearings. Two control algorithms, bang-bang control and instantaneous optimal control, are developed to control the friction force. Their effectiveness is demonstrated by shaking table experiments and computer simulation.

The hybrid sliding isolation system using friction controllable bearings is physically developed. Shaking table experiments are performed using a structural model equipped with the hybrid system. These experiments demonstrate the advantage of using the hybrid system over the passive system.

Computer codes to simulate the structural response under passive and hybrid control are developed. The numerically simulated results show good agreement with the experimental results. These results verify that the analytical model developed adequately represents the actual system.

ABSTRACT

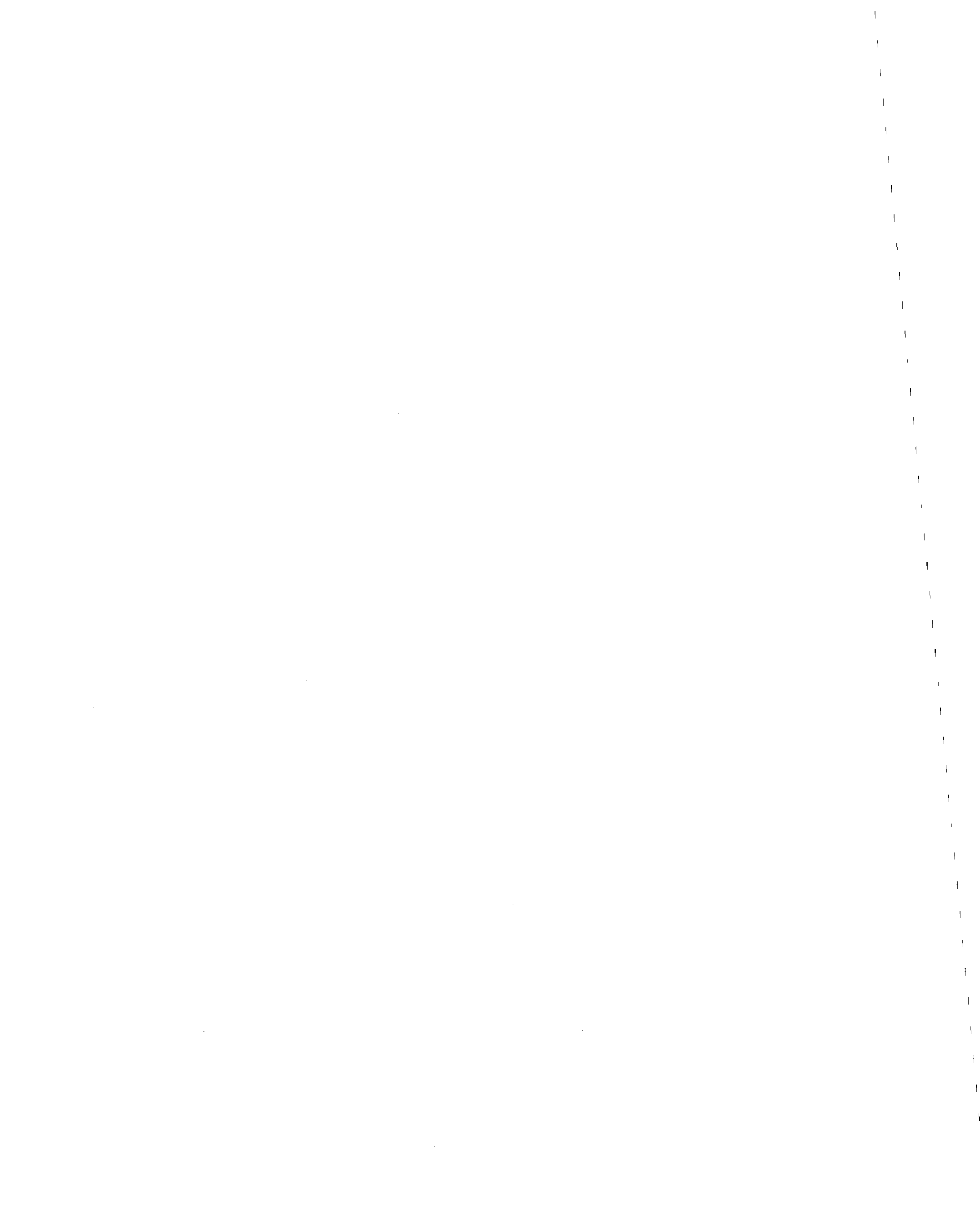
This study deals with a hybrid isolation system using friction controllable sliding bearings [1,2]. During earthquakes, this isolation system controls the friction force on the sliding interface between the supported structure and the ground, by adjusting the pressure in a bearing chamber, to confine the sliding displacement within an acceptable range, while keeping the transfer of seismic force to a minimum to obtain the best isolation performance. This is the advantage of the hybrid sliding isolation system that cannot be achieved by the passive sliding system.

Instantaneous optimal control and bang-bang control algorithms are developed for controlling the friction force, since standard control theory is difficult to apply in a straightforward fashion in this case where the control force has a nonlinear feature. The effectiveness of the algorithms in controlling seismic response of a structural model is demonstrated by shaking table experiments and computer simulation.

A hybrid sliding isolation system using friction controllable bearings is physically developed, and shaking table experiments are performed using a rigid structural model equipped with such a hybrid system. The dynamic characteristics of the control system for bearing pressure and sliding friction is identified, and the advantage of the hybrid sliding isolation system over the passive system is demonstrated by experiments.

Computer codes for simulation of structural response under passive or hybrid control are developed. The numerically simulated results show good agreement with the experimental results, verifying that the analytical model developed represents the actual system very well.

Both experimental and analytical studies demonstrate the effectiveness of the hybrid sliding isolation system and suggest its advantageous use in civil structures.



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SECTION 1

INTRODUCTION

1.1 Background

Seismic base isolation techniques have been used with increasing popularity to protect the structures, together with their occupants, secondary systems and internal equipment, from the damaging effects of earthquakes. The sliding isolation systems, however, have the following problems:

1. A structure supported entirely by sliding bearings experiences forces at the sliding interface that are always bounded by the frictional force, regardless of the intensity and frequency content of the ground excitation. From this view point, the smaller the coefficient, the higher the isolation performance. However, because of unavoidable residual displacement and possibly excessive sliding displacement associated with such systems, particularly those with small coefficients of friction, the purely sliding isolation system is difficult to use.

2. For practice, most of the sliding isolation systems currently in use are equipped with some form of restoring devices [3, 4, 5]. For a passive sliding system with a restoring force, the design of coefficient of friction and stiffness of the restoring device is not a simple problem. If the coefficient of friction is relatively small, then the isolation performance is greatly influenced by the stiffness of the restoring device. As a result, the advantage of the purely sliding isolation system (mentioned above) cannot be realized. For this reason, the coefficient of friction μ is usually designed to the extent (e.g., $\mu = 0.15 \sim 0.20$) that its isolation performance is not unduly influenced by the stiffness of the restoring device [6, 7]. However such a system becomes totally ineffective for the earthquakes with peak ground acceleration less than or equal to μg , in spite of the fact that such small to medium earthquakes occur more often and they can cause damage to sensitive equipment, valuable items and secondary systems inside the building.
3. Since a passive sliding isolation system is usually designed and effective for large earthquakes with long return periods, the system seldom has the opportunity to be activated during the service life of the building. For example, the TASS system (Kawamura et al. 1988) was installed in an office building (called J Building) in Yokohama, Japan four years ago, but has never been activated, although many small earthquakes occurred in that area during this period. Whether or not the sliding system can maintain the initially designed value of coefficient of friction throughout the long period of inactivity remains to be an important practical problem.

1.2 Objectives and Hybrid Isolation System

The objectives of this research are then to physically develop a friction controllable sliding isolation system which can retain the advantages, while eliminating the disadvantages, of the purely sliding isolation system, and thus delivers a fundamentally superior isolation performance which cannot be achieved by the passive sliding isolation system (with a restoring device).

To be more specific, this system can intelligently control the friction force on the sliding interface between the supported structure and the ground so as to confine the sliding displacement in an acceptable range, to reduce the residual displacement, and at the same time, to minimize the transfer of seismic force to the structure. For small to medium earthquakes, the friction force can be controlled to a small level to obtain the best isolation performance. For large earthquakes, the sliding displacement can be confined within an acceptable range by controlling the friction force, while the minimum isolation performance is guaranteed by the maximum friction force of the system. Therefore, this friction controllable sliding isolation system is effective for all intensities of earthquakes, (ranging from small, medium or large), unlike the passive sliding isolation systems which are usually designed for large earthquakes.

Furthermore, the confidence in this system can be more easily established than the passive sliding isolation system for the reason that this system has more opportunities of getting activated even under small earthquakes which occur more frequently.

To achieve these objectives, this research attempts to:

1. propose and physically develop a friction controllable sliding bearing and a hybrid sliding isolation system using such bearings,

2. develop control algorithms for controlling the friction force which has nonlinear characteristics,
3. through experiments and computer simulation analysis, demonstrate the effectiveness of the hybrid isolation system using friction controllable sliding bearings in controlling the seismic responses of both a single-degree-of-freedom structural model and a multi-degree-of-freedom building.

The hybrid isolation system using friction controllable bearings (FCB's) to be investigated is conceptually depicted in Figs. 1.1 and 1.2, respectively with a building and a bridge structure resting on the bearings. Each bearing has a fluid chamber which is connected to a pressure control system composed of a servo valve, an accumulator and a computer. The friction on the interface between the bearing and the ground is controlled by adjusting the fluid pressure in the chamber. The computer calculates an appropriate signal to control the fluid pressure based on the observed structural response, such as response acceleration and sliding displacement, and sends it to the pressure control device as shown in Figs. 1.1 and 1.2.

The idealized section view of the friction controllable sliding bearing is given in Fig. 1.3. The bearing made of steel is of disk shape containing a fluid chamber inside which is sealed by a rubber O-ring around the circular perimeter just inside the sliding interface. A sliding material such as PTFE plate is placed on the sliding surface.

The word "HYBRID" is used for this friction controllable sliding isolation system, since it is a combination of the passive sliding isolation system and the active control device. The system can be a passive sliding isolation system as long as the pressure of the bearing chamber, and thus the friction is kept at a constant value. At the same time, the system can

be an active system as long as the pressure is controlled so that the friction is controlled.

In the analytical formulation involving such a hybrid system, the control force does not appear in the equation of motion as an independent term like the force from an actuator. The active control in this case is implemented through the friction term in the equation, and in that sense the system may also be interpreted as "Semi-Active".

This hybrid isolation system has the following general advantages: (a) Changing friction force through controlling pressure requires smaller amounts of energy and power than the corresponding actuator-driven control system, and as a consequence (b), the use of accumulators for the source of energy is possible, thus eliminating the necessity of emergency energy supply system.

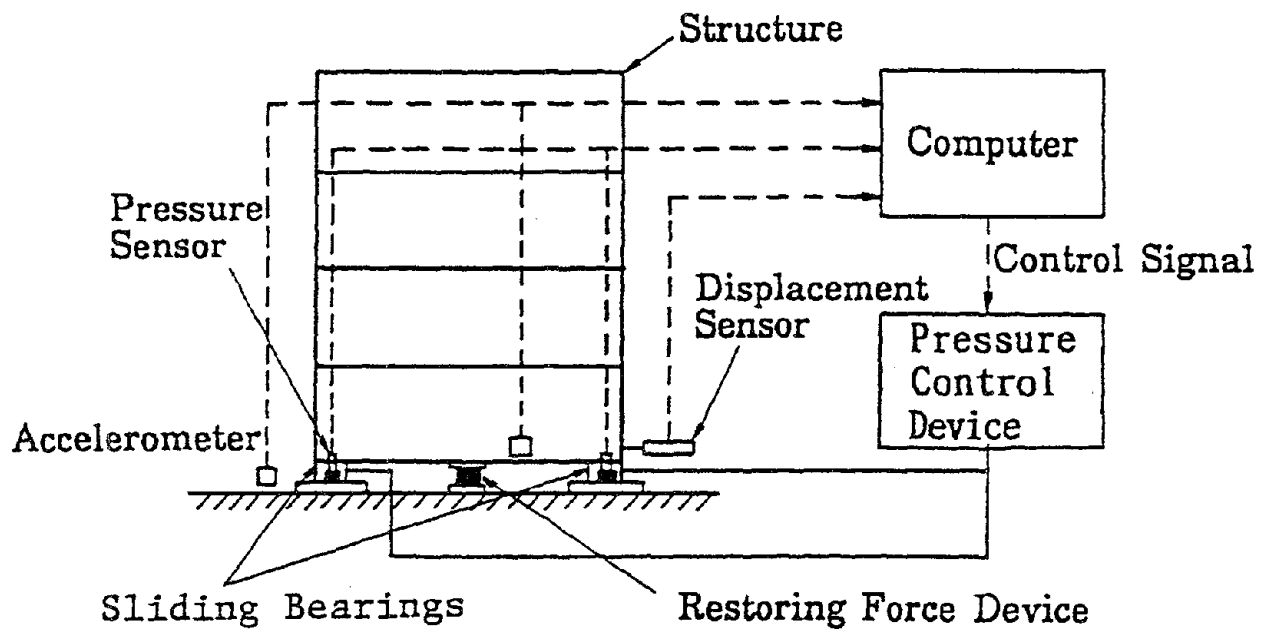


Figure 1.1: Concept of Hybrid Sliding Isolation System for Buildings

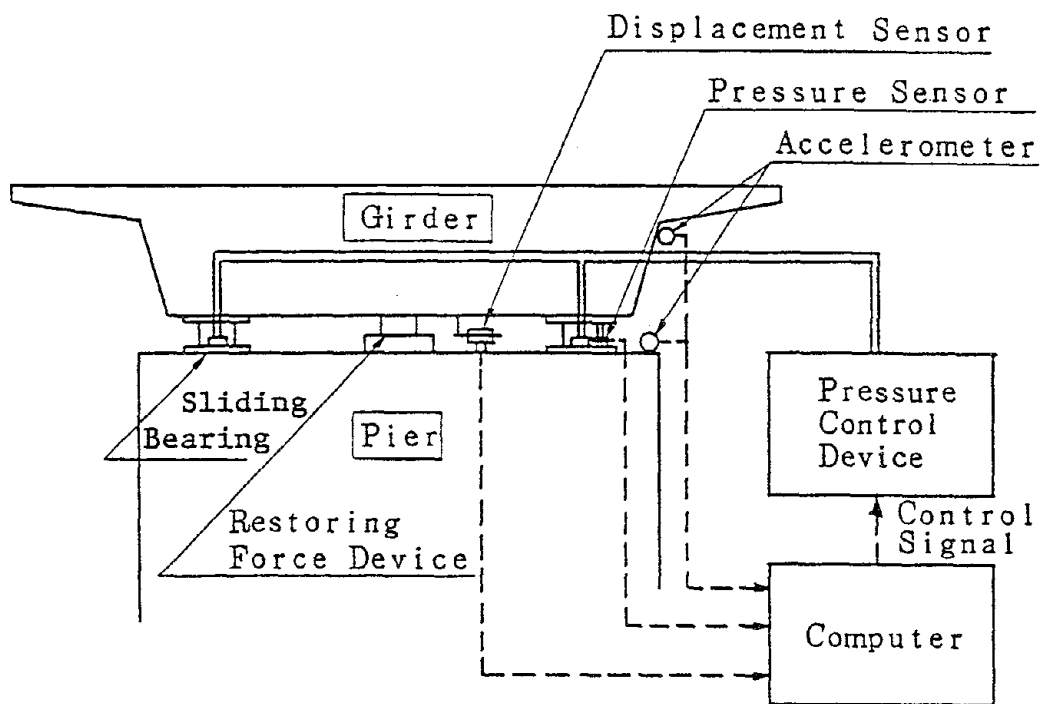


Figure 1.2: Concept of Hybrid Sliding Isolation System for Bridges

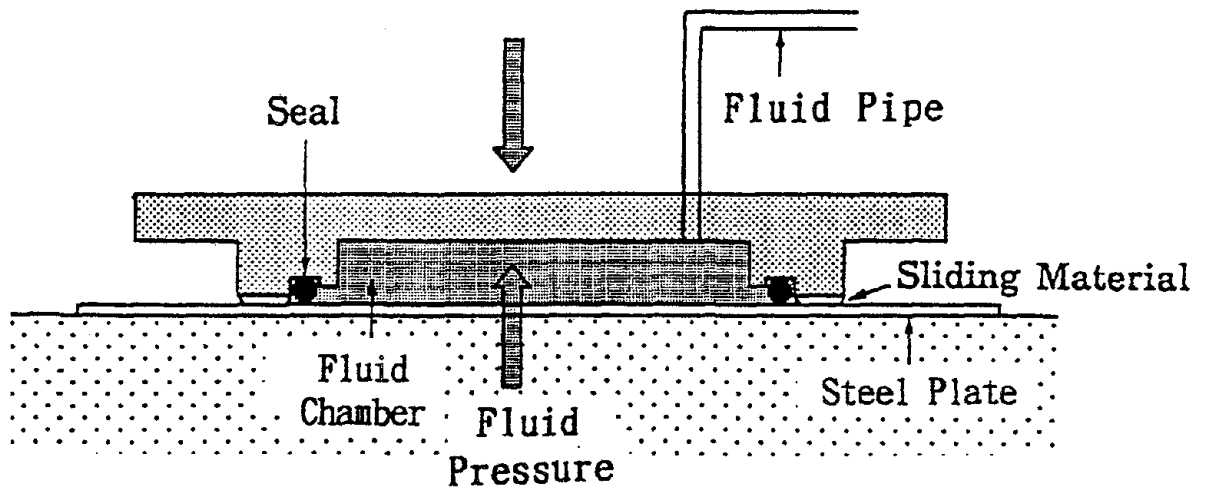


Figure 1.3: Idealized View of Friction Controllable Sliding Bearing

1.3 Outline of the Study

In Section 2, two kinds of control algorithms are newly developed for controlling the friction force in the sliding isolation system, since standard control theory can not be applied in a straightforward fashion in this case where the analytical model of the physical system is nonlinear involving the friction force. These control algorithms are either based on instantaneous optimal control theory or bang-bang control method. As to the instantaneous optimal control, two cases are studied: one takes into account the time delay of the friction control system, while the other does not. These control algorithms, whether based on instantaneous optimal control or bang-bang control, are relatively simple and yet robust for on-line control operations, and they effectively achieve the objectives mentioned above.

In Sections 3 and 4, the experimental study performed on a single-degree-of-freedom structure installed with a hybrid sliding isolation system is described. The sliding bearing and its control device are developed. A single-degree-of-freedom model structure is constructed. Computer codes using the C language for real-time on-line control operations are developed. The characteristics of the hybrid isolation system is identified by experiments. Then, the hybrid isolation system which can control the friction force in the sliding interface between the model structure and the ground (shaking table) is developed. It can act as a passive isolation system by maintaining the friction force at a certain value. Shaking table experiments of the model structure equipped with such a sliding isolation systems are carried out to evaluate its isolation performance under various earthquake excitations with differing levels of intensities. The isolation performance of the hybrid system is compared with that of the passive system, and the advantage of the proposed hybrid sliding isolation system is demonstrated.

In Section 5, related analytical and numerical studies also performed. A computer pro-

gram for numerical evaluation of the response of the structure under passive or hybrid isolation is developed. The numerical simulation results show a good agreement with the experimental results, confirming the adequacy of the analytical model and the simulation method.

Finally, the conclusion of this report is given in Section 6.

SECTION 2

CONTROL ALGORITHMS

Control of structural response by the proposed hybrid sliding isolation system using friction controllable bearings presents an unique problem in developing control algorithms. The reason is that the control force in this sliding isolation system is the friction force, which depends on the direction of the sliding velocity and thus appears as a nonlinear term in the equation of motion. For controlling such a nonlinear force, standard control theory is difficult to apply, and not much research has been done to derive control algorithms. Fujita et al [8, 9] developed a semi-active seismic isolation system using controllable friction dampers. In this system, the frictional damping force is controlled by changing the pressure between the friction elements using an actuator. Linear optimal control algorithm in modern control theory was applied in a straightforward manner in this study. Since the frictional damping force is a nonlinear force depending on the direction of velocity, the optimal control force required by the linear control theory cannot be fully realized by the friction force, and thus the real "optimal control" can not be implemented. In this respect, Feng and Shinozuka [10, 11] were the first to derive a nonlinear control algorithm on the basis of instantaneous optimal

control [12, 13] for the system with such a nonlinear control force. By simulation study, they showed the effectiveness of the control algorithm. In their study, however, nonlinear differential equations are needed to be solved for control force at every control time instant, making real-time on-line control operations rather difficult.

In this study, two types of control algorithms, both of which are implementable in real-time on-line operations, are developed to control the nonlinear friction force in the proposed hybrid isolation system under earthquake loads. They are developed on the basis of the instantaneous optimal control theory and bang-bang control concept. As to the instantaneous optimal control, furthermore, two cases are studied: one with the time delay of the friction control system taken into consideration and the other without.

2.1 Analytical Model

A rigid structure supported by the friction controllable sliding bearings is considered. The motion of the structure can be modeled by a single-degree-of-freedom (SDOF) model as shown in Fig. 2.1. The equations of motion of the structure under earthquake excitation can then be written as follows.

1. Sticking Phase — Phase I

$$\dot{x} = 0, \quad x = \text{const.} \quad (2.1)$$

2. Sliding Phase — Phase II

$$\ddot{x} = -\ddot{z} - f \operatorname{sgn}(\dot{x}), \quad (2.2)$$

3. Criteria for transition from Phase I to Phase II

$$|\ddot{z}| > f \quad (2.3)$$

4. Criteria for transition from Phase II to Phase I

$$\dot{x} = 0 \quad (2.4)$$

$$|\ddot{x}| < 2f \quad (2.5)$$

where

- x : sliding displacement of mass relative to ground
- \ddot{z} : input earthquake acceleration
- μ : coefficient of friction on sliding interface
- f : normalized friction force defined as $f = \mu g$

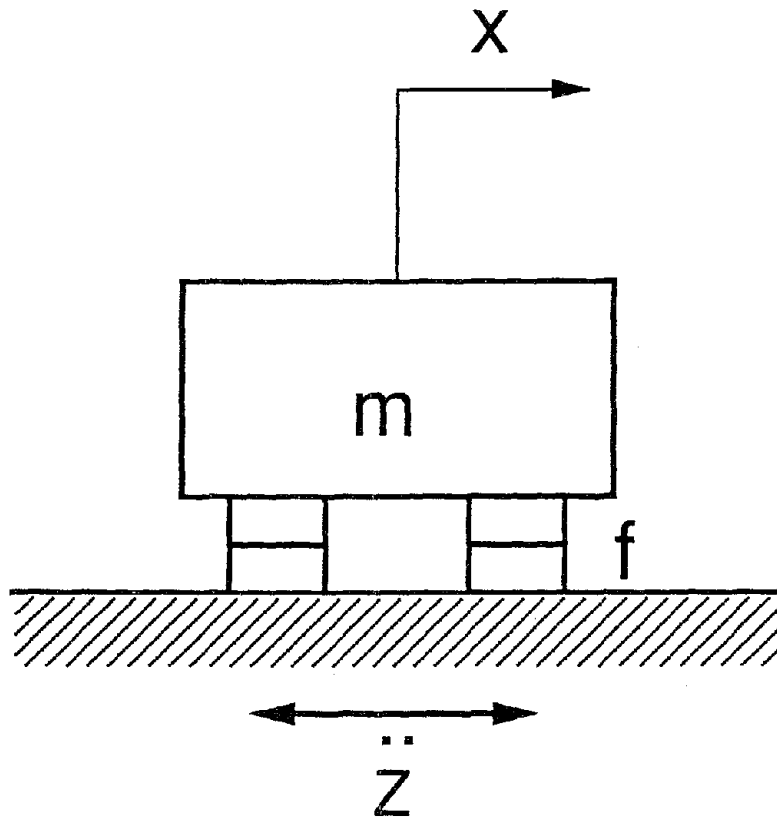


Figure 2.1: Analytical Model

In the sticking phase, Eq. 2.1 governs the motion of the structure until the Eq. 2.3 becomes true. As soon as the condition in Eq. 2.3 is met, the sliding phase starts and Eq. 2.2 governs the sliding motion. During the sliding phase, whenever \dot{x} becomes zero, the criterion, Eq. 2.4, is checked to determine the subsequent behavior. Validity of the inequality given by Eq. 2.4 is the condition for entering the sticking phase [14]. That is, if the inequality holds, the structure will stick to the ground and Eq. 2.1 applies. If the criterion given by Eq. 2.4 is not satisfied, Eq. 2.1 will continue to govern the the subsequent sliding motion.

On the other hand, the normalized friction force f on the sliding interface between the structure and the ground is controlled by changing the fluid pressure in the bearing chamber through a pressure control system consisting of a computer, servo valve, amplifier, etc. The dynamic characteristics of the pressure control system are assumed to follow the first order time delay model:

$$T\dot{p} + p = u \quad (2.6)$$

where

- p : pressure in fluid chamber of bearing
- u : pressure control signal from computer
- T : time constant

The normalized friction force f is negatively proportional to the pressure p in the bearing chamber:

$$f = -c_1p + c_2 \quad (2.7)$$

where c_1 and c_2 are constants.

2.2 Bang-Bang Control

Bang-bang control approach provides a simple and yet often effective algorithm. The particular algorithm used in this study facilitates the following control: when the sliding displacement and the velocity of the mass are in the same direction, the pressure control signal $u(t)$ will be decreased to a minimum value u_{min} to increase the friction force in order to put the brake on the sliding. On the other hand, when the sliding displacement and the velocity are in the opposite direction, the pressure control signal will be increased to a maximum value u_{max} to decrease the friction force in order to make the sliding easier as much as possible;

$$u(t) = \begin{cases} u_{max}, & \text{if } \text{sgn}(x) = -\text{sgn}(\dot{x}) \\ u_{min}, & \text{if } \text{sgn}(x) = \text{sgn}(\dot{x}) \end{cases} \quad (2.8)$$

in which the control parameter, u_{max} and u_{min} , should be determined carefully, since they directly influence the control performance.

u_{max} should be set at a level as large as possible in order to reduce the friction force to a minimum level, making the mass slide as much as possible. It is, however, limited by the maximum pressure which can be applied to the bearing chamber. This depends on the weight W of the structure supported by the bearing:

$$u_{max} < \frac{W}{S} \quad (2.9)$$

where S is the vertically projected area of the fluid chamber of bearing. u_{min} should be set at a small level in order to confine the sliding displacement within an acceptable range.

This bang-bang control algorithm is simple and easy to implement in real-time on-line control operations, since the control signal only switches between two values, and only the sliding displacement needs to be measured by a sensor and fed back to the control signal.

Notice that $\text{sgn}(\dot{x})$ can be obtained by the displacement signal and does not need to measure the velocity

However, this algorithm has the following difficulty: In order to confine the sliding displacement within an acceptable range, the smaller value of u_{min} is needed. The smaller u_{min} , however, will lead to the larger seismic response acceleration of the structure and thus degrade the isolation performance. This conflict can only be alleviated, it appears, by taking advantage of pertinent optimal control algorithms.

2.3 Instantaneous Optimal Control

2.3.1 Formulation

As mentioned earlier, control theory has not been well developed for such systems in which the control force has the nonlinearity, unique to the friction controllable sliding isolation device. For this reason, an optimal control algorithm for control of the nonlinear friction system is developed on the basis of the instantaneous optimal control theory originally proposed by Yang et al. [12, 13].

The optimal pressure control signal $u(t)$ is determined by minimizing the following time dependent objective function $J(t)$ at every time instant t for the entire duration of an earthquake.

$$J(t) = q_d x^2(t) + q_f f^2(t) + r u^2(t) \quad (2.10)$$

in which the normalized friction force f equivalently represents the amount of response acceleration and also serves as a measure of the transfer of seismic force to the structure. The weighting coefficient q_d and q_f are non-negative and r is positive. They indicate the relative importance in the control objectives of the sliding displacement, response acceleration and pressure control signal, respectively. The basic objectives of the control is to make the structure slide as much as possible within an acceptable range and at the same time to ensure the transfer of seismic force to a minimum.

The following control algorithm is derived under the assumption that the structural motion is always in the sliding phase. The equation of motion given by Eq. 2.2 should be used as a constraint when minimizing the objective function $J(t)$. The first order time delay relationship between the control signal and the pressure described in Eq. 2.6, as well

as the linear relationship between the friction and the pressure shown in Eq. 2.7, are also constraints. In the present formulation, however, these equations will be solved numerically using the Newmark's β method with $\beta = 1/6$ as shown below and these numerical solutions will be used as constraints:

$$\Delta \dot{x}(t) = \ddot{x}(t) \Delta t + \Delta \ddot{x}(t) \frac{\Delta t}{2} \quad (2.11)$$

$$\Delta x(t) = \dot{x}(t) \Delta t + \ddot{x}(t) \frac{\Delta t^2}{2} + \Delta \ddot{x}(t) \frac{\Delta t^2}{6} \quad (2.12)$$

$$\Delta f(t) = \dot{f}(t) \Delta t + \Delta \dot{f}(t) \frac{\Delta t}{2} \quad (2.13)$$

From the above equations, one obtains:

$$x(t) = x(t - \Delta t) + \dot{x}(t - \Delta t) \Delta t + \ddot{x}(t - \Delta t) \frac{\Delta t^2}{2} + [\ddot{x}(t) - \ddot{x}(t - \Delta t)] \frac{\Delta t^2}{6} \quad (2.14)$$

$$f(t) = f(t - \Delta t) + \dot{f}(t - \Delta t) \Delta t + [\dot{f}(t) - \dot{f}(t - \Delta t)] \frac{\Delta t}{2} \quad (2.15)$$

Furthermore,

$$x(t) = a f(t) \operatorname{sgn}(\dot{x}(t)) + b \ddot{z}(t) + d_1 (t - \Delta t) \quad (2.16)$$

$$f(t) = -c u(t) + d_2 (t - \Delta t) \quad (2.17)$$

where

$$a = b = -\frac{\Delta t^2}{6}, \quad c = \frac{c_1 \Delta t}{2T + \Delta t} \quad (2.18)$$

$$d_1(t - \Delta t) = x(t - \Delta t) + \dot{x}(t - \Delta t) \Delta t + \frac{1}{3} \ddot{x}(t - \Delta t) \Delta t^2 \quad (2.19)$$

$$d_2(t - \Delta t) = \frac{2T}{2T + \Delta t} \left(\frac{c_2 \Delta t}{2T} + f(t - \Delta t) + \frac{1}{2} \dot{f}(t - \Delta t) \Delta t \right) \quad (2.20)$$

The numerical solution to Eq. 2.2 given in Eq. 2.16, and the numerical solution to Eqs. 2.6 and 2.7 expressed in Eq. 2.17 are used as constraints as mentioned above. Thus,

the following generalized objective function is established by introducing the Lagrangian multipliers λ_1 and λ_2 :

$$\begin{aligned}
H(t) = & q_d x^2(t) + q_f f^2(t) + r u^2(t) \\
& + \lambda_1 [x(t) - a f(t) \operatorname{sgn}(\dot{x}(t)) - b \ddot{z}(t) - d_1 (t - \Delta t)] \\
& + \lambda_2 [f(t) + c u(t) - d_2 (t - \Delta t)]
\end{aligned} \tag{2.21}$$

The necessary conditions for minimizing the objective function $J(t)$ are:

$$\frac{\partial H}{\partial x} = 0, \quad \frac{\partial H}{\partial f} = 0, \quad \frac{\partial H}{\partial u} = 0, \quad \frac{\partial H}{\partial \lambda_1} = 0, \quad \frac{\partial H}{\partial \lambda_2} = 0, \tag{2.22}$$

Substituting Eq. 2.21 into Eq. 2.22 yields the optimal pressure control signal:

$$u(t) = F_f f(t) + F_d x(t) \operatorname{sgn}(\dot{x}(t)) \tag{2.23}$$

where, the control feedback gains F_f and F_d are calculated by

$$F_f = \frac{c q_f}{r}, \quad F_d = \frac{c a q_d}{r} \tag{2.24}$$

and the displacement $x(t)$ and friction $f(t)$ are used for feedback purpose. Again, notice that $\operatorname{sgn}(\dot{x})$ can be obtained from the displacement signal without the need to measure the velocity. The friction is difficult to measure by a sensor, but the signal from the acceleration sensor ($\ddot{x}(t) + \ddot{z}(t)$) can be used instead of f in the SDOF structure. Therefore, the control signal becomes:

$$u(t) = F_f |\ddot{x}(t) + \ddot{z}(t)| + F_d x(t) \operatorname{sgn}(\dot{x}(t)) \tag{2.25}$$

In the development of the optimal control algorithm shown above, the time delay of the control device shown in Eq. 2.6 has been incorporated. In this case, the control is referred to as “instantaneous optimal control with time delay”.

If the response of the control device is so fast that the time delay can be ignored, the relationship between the pressure and the control signal is given by:

$$p(t) = u(t) \quad (2.26)$$

The control algorithm is also developed under this condition, in which case the objective function and Hamiltonian become respectively:

$$J(t) = q_d x^2(t) + q_f f^2(t) + r u^2(t) \quad (2.27)$$

and

$$\begin{aligned} H(t) = & q_d x^2(t) + q_f f^2(t) + r u^2(t) \\ & + \lambda_1 [x(t) - a f(t) \operatorname{sgn}(\dot{x}(t)) - b \ddot{z}(t) - d_1 (t - \Delta t)] \\ & + \lambda_2 [f(t) + c_1 u(t) - c_2] \end{aligned} \quad (2.28)$$

The control based on Eq. 2.26 is referred to as “instantaneous optimal control without time delay”. By letting the following partial derivatives equal to zero,

$$\frac{\partial H}{\partial x} = 0, \quad \frac{\partial H}{\partial f} = 0, \quad \frac{\partial H}{\partial u} = 0, \quad \frac{\partial H}{\partial \lambda_1} = 0, \quad \frac{\partial H}{\partial \lambda_2} = 0, \quad (2.29)$$

the following control signal is obtained:

$$u(t) = F + F_d x(t) \operatorname{sgn}(\dot{x}(t)) \quad (2.30)$$

where

$$F = \frac{c_1 c_2 q_f}{r + q_f c_1^2}, \quad F_d = \frac{c_1 a q_d}{r + q_f c_1^2} \quad (2.31)$$

In this case, only the sliding displacement $x(t)$ needs to be measured and fed back.

Such instantaneous optimal control algorithms are also applicable to deal with other types of nonlinearity, by establishing a time dependent objective function and using numerical solutions of nonlinear equations as constraints to minimize objective function.

2.3.2 Sufficient Condition For Optimal Control

In the original derivations for the instantaneous optimal control algorithms shown above, the optimal control signals $u(t)$ are obtained from the necessary conditions, such as Eq. 2.22 and Eq. 2.29. In fact, the derived optimal control signals $u(t)$ also satisfy the sufficient conditions of optimality as will be proved in the following.

The sufficient condition for the optimal solution shown in Eq. 2.23 is given by [15, 16]:

$$[\delta x \ \delta f \ \delta u] \begin{bmatrix} \frac{\partial^2 H}{\partial x^2} & \frac{\partial^2 H}{\partial x \partial f} & \frac{\partial^2 H}{\partial x \partial u} \\ \frac{\partial^2 H}{\partial f \partial x} & \frac{\partial^2 H}{\partial f^2} & \frac{\partial^2 H}{\partial f \partial u} \\ \frac{\partial^2 H}{\partial u \partial x} & \frac{\partial^2 H}{\partial u \partial f} & \frac{\partial^2 H}{\partial u^2} \end{bmatrix} \begin{bmatrix} \delta x \\ \delta f \\ \delta u \end{bmatrix} > 0 \quad (2.32)$$

Taking derivatives of H in Eq. 2.21 obtains:

$$\begin{aligned} \frac{\partial^2 H}{\partial x^2} &= 2q_d, & \frac{\partial^2 H}{\partial f^2} &= 2q_f, & \frac{\partial^2 H}{\partial u^2} &= 2r, \\ \frac{\partial^2 H}{\partial x \partial f} &= \frac{\partial^2 H}{\partial f \partial x} = 0, & \frac{\partial^2 H}{\partial x \partial u} &= \frac{\partial^2 H}{\partial u \partial x} = 0, & \frac{\partial^2 H}{\partial f \partial u} &= \frac{\partial^2 H}{\partial u \partial f} = 0 \end{aligned} \quad (2.33)$$

Substitution of Eq. 2.33 into the left hand side of Eq. 2.32 leads to the following expression:

$$2 [q_d (\delta x)^2 + q_f (\delta f)^2 + r (\delta u)^2] > 0 \quad (2.34)$$

which is true since q_d and q_f are non-negative and r is positive, Eq. 2.32 is greater than zero. Thus, the sufficient condition for the optimal solution, Eq. 2.32, is satisfied.

Similarly, it can be shown that the optimal signal derived under the condition of no time delay, Eq. 2.30, also satisfies the sufficient condition.

SECTION 3

SHAKING TABLE EXPERIMENT

In order to examine the performance of the proposed hybrid isolation system and to study the feasibility of its applications in structural engineering, experimental studies have been performed. For this purpose, a prototype hybrid isolation system with the friction controllable sliding bearings has been developed. A rigid structural model supported on the hybrid system was experimented on a shaking table at Taisei Technology Research Center in Yokohama, Japan. This three dimensional shaking table has a maximum stroke of 40 cm in each horizontal direction and maximum loading capacity of 20 tonf weight.

3.1 Structure Model and Isolation Device

The structure model used for experiments is shown in Fig. 3.1. The model, representing a rigid structure, consists of a steel frame and steel weights. The total weight of the model is 12 tonf.

Figures 3.2 and is a photograph of the structure model in the experiments. The model is supported equally by four friction controllable sliding bearings on the shaking table, as

shown in Figs. 3.1 and 3.2. The bearing developed for the experiments is shown in Fig. 3.3 and Fig. 3.4. Figure 3.5 is a photograph of the bearing being used in the experiments. The bearing, with a brass sheet of 1 mm thickness attached to be used as sliding surface, slides on a stainless steel plate fixed on steel I-bars bolted down on the shaking table, as shown in Figs. 3.1 and 3.2. Furthermore, a rubber O-ring of 5.7 mm in diameter acts as seal for the fluid in the fluid chamber. The area of the sliding surface is 86.0 cm^2 , and the vertically projected area of the fluid chamber is 57.7 cm^2 . No restoring force device is used in order to study the effect of friction force only. A servo valve is located at the center of the experimental structure from which the pressurized fluid is distributed to each sliding bearing as described in Fig. 3.1.

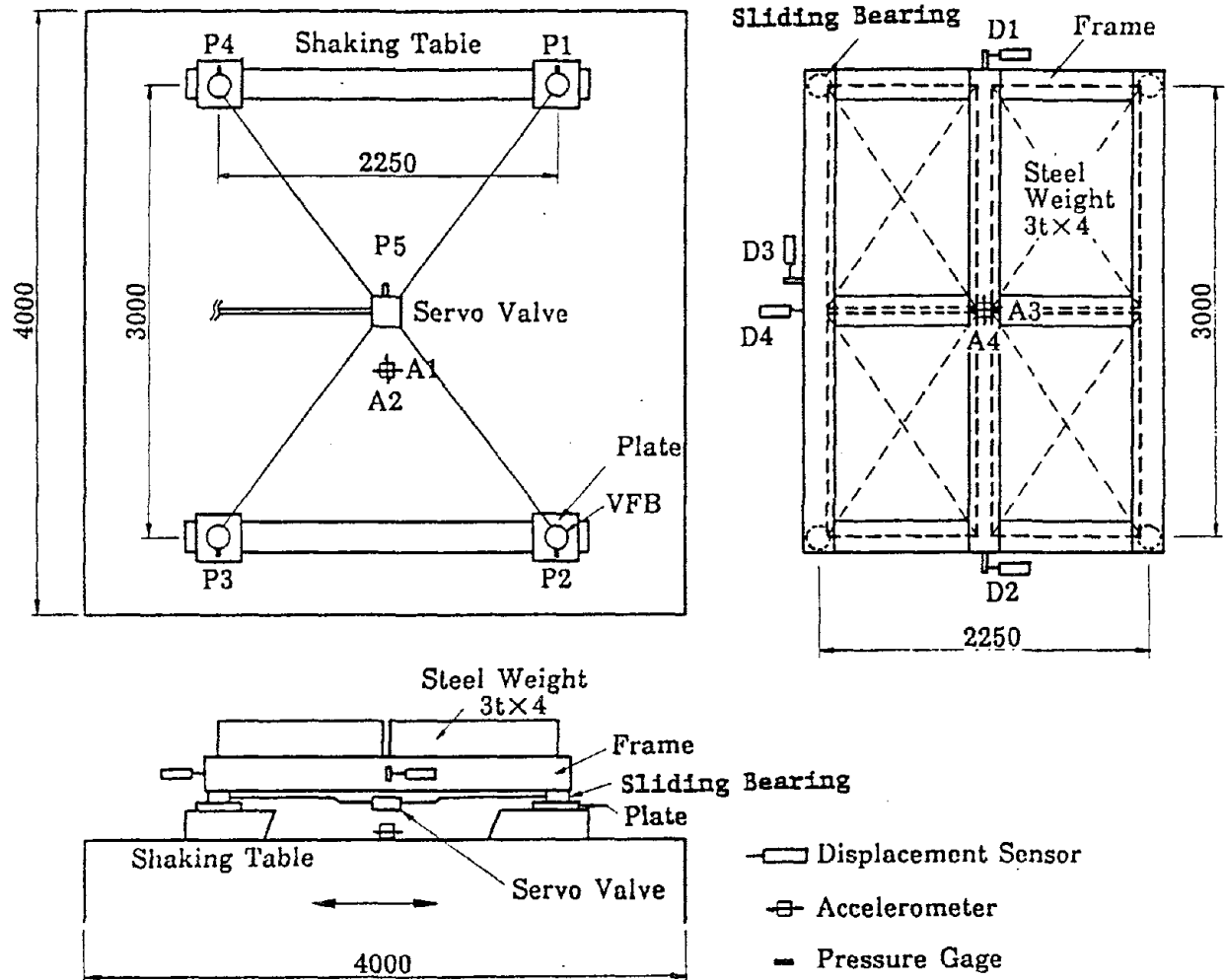


Figure 3.1: Structure Model with Hybrid Sliding Isolation Device

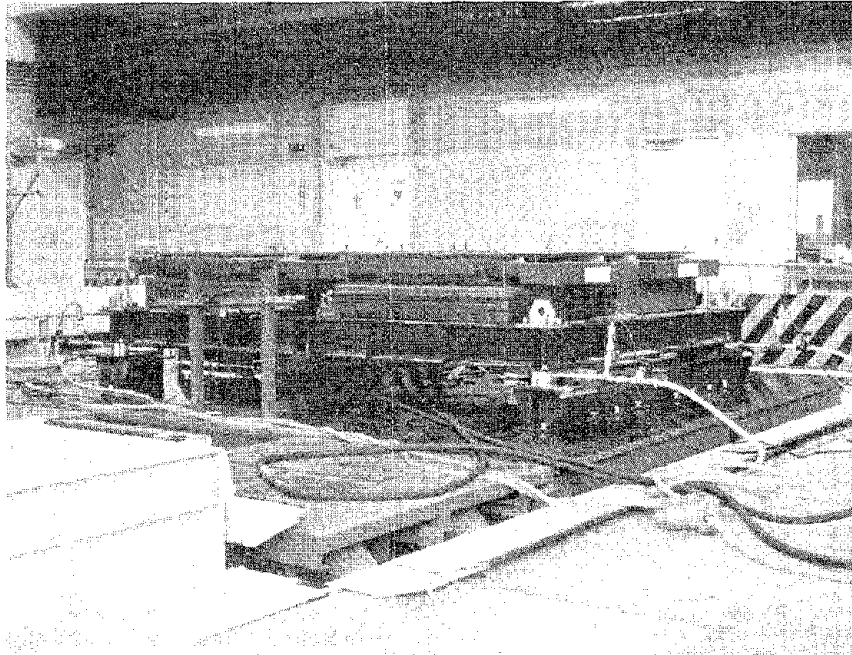


Figure 3.2: Structure Model with Hybrid Sliding Isolation Device in Experiment

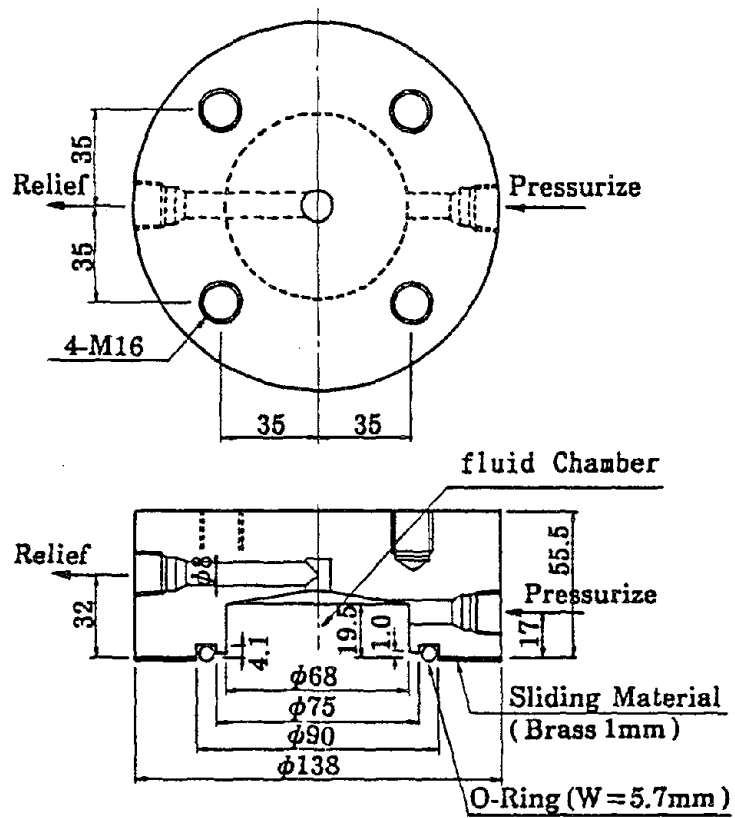


Figure 3.3: Friction Controllable Sliding Bearing

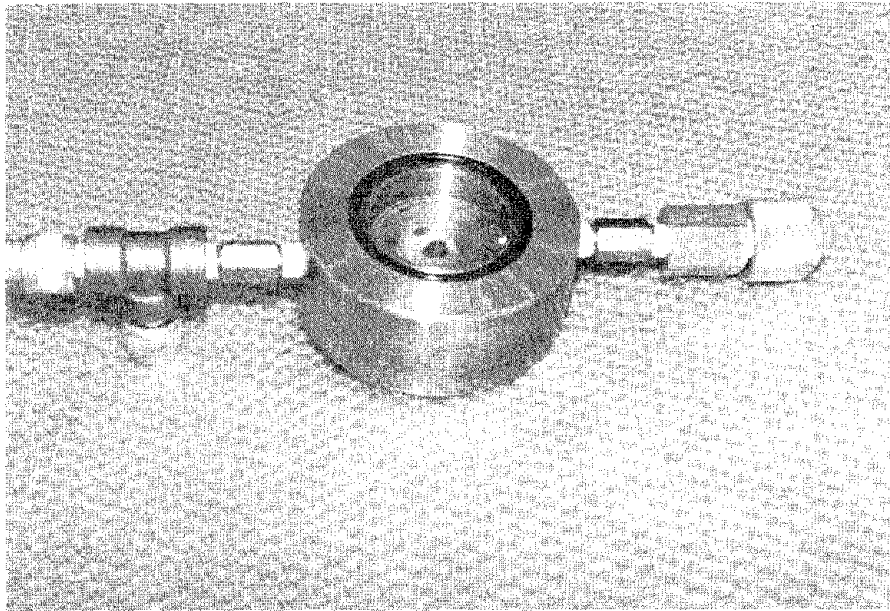


Figure 3.4: Friction Controllable Sliding Bearing for Experiment

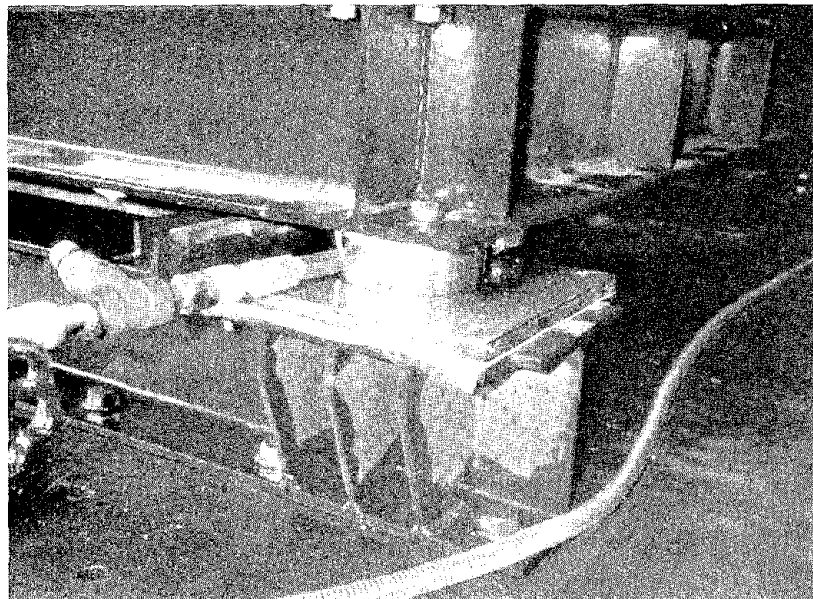


Figure 3.5: Friction Controllable Sliding Bearing in Experiment

3.2 Control System and Instrumentation

A block diagram of the control system is shown in Fig. 3.6 [17, 18]. The controller is a 16 bit micro-computer (80286) with a numerical co-processor (80287) to facilitate faster computation. The response signals for feedback purpose are measured by sensors and sent to the micro-computer through 12 bit A/D converter. Then, the control signal is calculated according to one of the feedback control algorithms described in the previous section, and sent to the servo valve and servo amplifier through a 12 bit D/A converter to control the fluid pressure in the bearing chamber.

A computer code for control implementation in experiments is developed using the C language.

Sensors are placed to monitor (1) accelerations on the shaking table and on the structural model, (2) relative displacement between the shaking table and the model, and (3) fluid pressure at each fluid chamber and at the servo valve. The locations of the sensors are also shown in Fig. 3.1.

The measurement performed by the displacement sensor D4 and the accelerometer on the structural model A4 are used for the feedback control purpose, while the pressure at the servo valve P5 is used for the analog regulation of the pressure in the servo valve.

Figure 3.7 is a photograph showing the computer, some of the measuring and recording equipment used in the experiments.

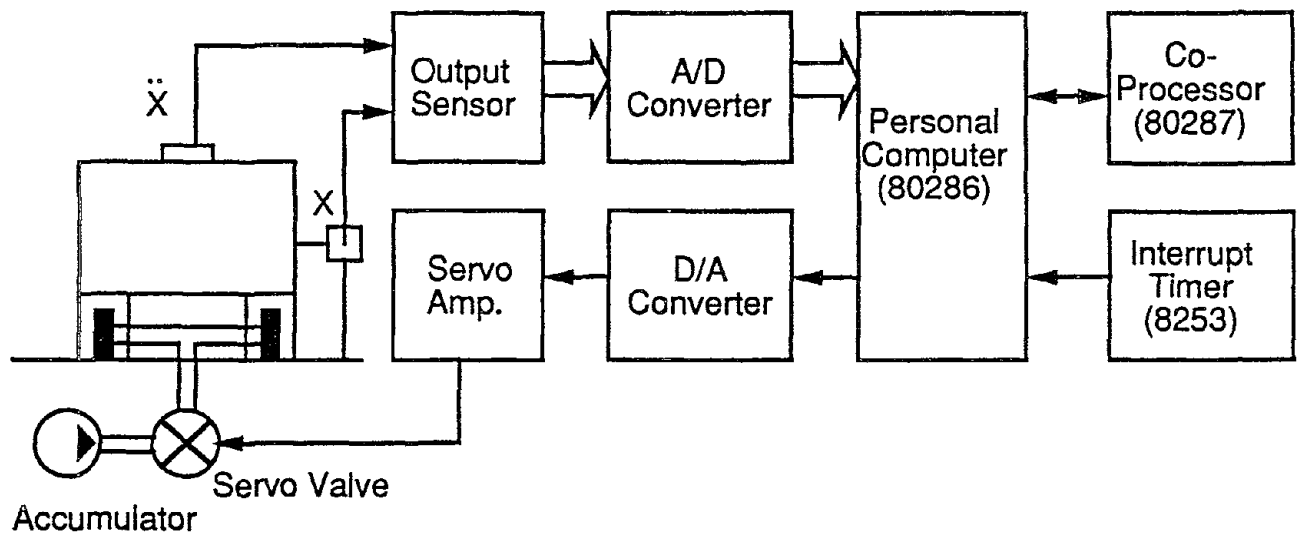


Figure 3.6: Block Diagram of Control System

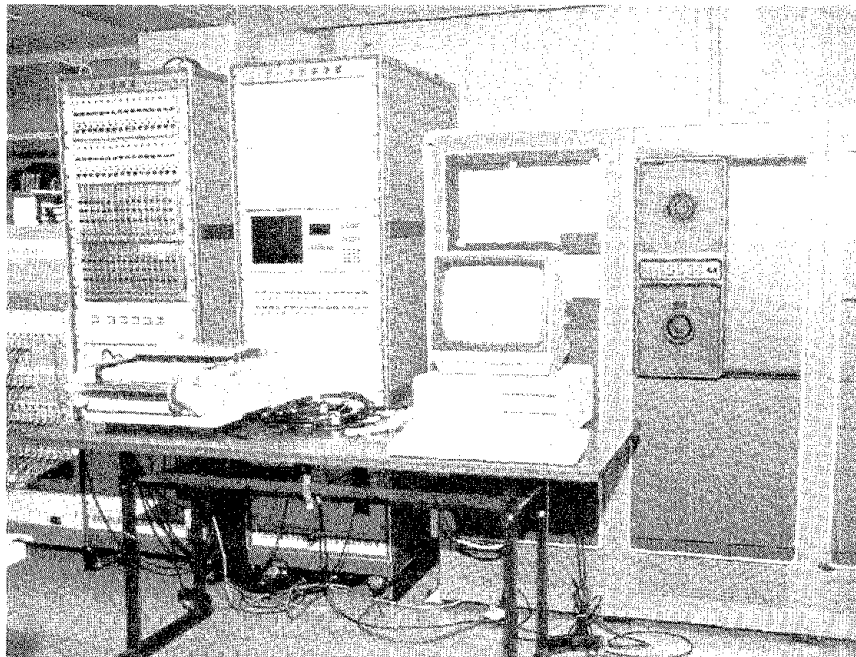


Figure 3.7: Computer and part of control system in experiment

3.3 Experimental Program

The experimental program, as described in Table 3.1, involves the following three isolation types;

1. Passive isolation

The pressure control signal is kept constant at a certain value for each shaking table experiment (at 10, 20, 30, 40 and 45 kgf/cm²).

2. Hybrid isolation using bang-bang control

The pressure control signal is switched between the following two values: $u_{max} = 45$ kgf/cm² and $u_{min} = 10$ kgf/cm².

3. Hybrid isolation using instantaneous optimal control

- 3a. algorithm with time delay

- 3b. algorithm without time delay

The pressure control signal is bounded by $u_{max} = 45$ kgf/cm² and $u_{min} = 10$ kgf/cm² for both cases. In case 3a, both response acceleration and sliding displacement are used for the feedback purpose with the feedback gains $F_f = 1.0$ kgfs²/cm³ and $F_d = -45.0$ kgf/cm³ in Eq. 2.25. In case 3b only sliding displacement is used, as indicated in Eq. 2.30, with the feedback gains $F = 16.5$ kgf/cm² and $F_d = -16.4$ kgf/cm³. In this case, the pressure control signal is nominally equal to the minimum value at the sliding displacement of 2.5 cm. The value of $u_{max} = 45.0$ kgf/cm² is determined by the maximum pressure which can be applied to the bearing chamber, depending on the

Table 3.1: Experimental Series

SERIES	PURPOSE	INPUT MOTION
1. Passive Isolation	1. Isolation performance 2. Identification of friction -pressure relationship	El Centro (NS) 100,200,300,400,450 gal Hachinohe (NS), 300 gal Taft (EW), 300 gal
2. Bang-Bang Control	1. Isolation performance 2. Identification of time delay of control device	El Centro (NS), 300gal Sinusoidal waves, 1 Hz 150 gal and 250 gal
3. Optimal Control without time delay	1. Isolation Performance 2. Study of time intervals 3. Effect of feedback gains 4. Effect of window comparator	El Centro (NS) 100,200,300 gal Hachinohe (NS), 300 gal Taft (EW), 300 gal
4. Optimal Control with time delay	1. Isolation performance 2. Study of window comparator	El Centro (NS) 100,200,300 gal Hachinohe (NS), 300 gal Taft (EW), 300 gal

vertical load identical to the structural weight equally shared by the four bearings:

$$u_{max} = \frac{W}{4S} * \lambda = 45.0 \text{ kgf/cm}^2 \quad (3.1)$$

where, S is the vertically projected area of the fluid chamber of each bearing with the value of 57.7 cm^2 , and λ is a safety coefficient with the value $\lambda = 0.87$ providing some margin of safety. On the other hand, the value of $u_{min} = 10 \text{ kgf/cm}^2$ is set because the experiment indicates that the pressure response p to the control signal u is too slow for the control to be effective if it is below 10 kgf/cm^2 .

The relationship between the pressure and the coefficient of friction is identified from the passive isolation experiments which are referred to as Experimental Series 1. The dynamic response characteristics of pressure and friction to the control signal is identified from the

bang-bang control experiment in Experimental Series 2. The time intervals for measurement and control are examined using the optimal control experiments without time delay (Experimental Series 3), and the appropriate values of the window comparator are studied using the optimal control experiments with and without time delay (Experimental Series 4). More importantly, the results of these experiments are used to examine and compare the isolation performance of each isolation type under different input motions at differing levels of intensity. These experimental series and their purpose are summarized in Table 3.1.

The shaking table experiments were conducted under one-dimensional horizontal motion (as shown in Fig. 3.1). The El Centro (NS, 1940) record was used as ground input motion for most of the experiment cases, by linearly adjusting the maximum acceleration to 100, 200, 300 400 and 450 gal as shown in Table 3.1, where it is also shown that Hachinohe NS, Taft EW , and sinusoidal waves were also used in some experiment cases.



SECTION 4

EXPERIMENTAL RESULTS

4.1 System Identification

4.1.1 Response of Pressure to Control Signal

The response characteristics of pressure to control signal have been identified by the shaking table experiment performed under El Centro 300 gal excitation in Series 2 (bang-bang control). The control signal was switched between 10 kgf/cm² and 45 kgf/cm². One set of the recorded time histories observed in this experiment are shown in Fig. 4.1.

A first order time delay model is assumed between the control signal and the pressure response as described in Eq. 2.6. The time constant T in the equation has been identified by minimizing the sum of the square errors between the pressure response from the experiment and that from the computer simulation. It is observed that the time delay of the pressure response to the control signal depends on whether the pressure is increased or decreased. Therefore, the time constants T_i under increasing pressure and T_d under decreasing pressure are identified separately as shown in Figs. 4.2 and 4.3. Figure 4.2 plots the sum of the square

error defined as $\sum_k (p_{sim}(t_k; T_i) - p_{exp}(t_k))^2$ where $p_{exp}(t_k)$ = pressure value observed in the experiment at $t = t_k$, and $p_{sim}(t_k; T_i)$ = simulated value at $t = t_k$ with T_i being used as time constant. The summation is over those time intervals in the entire time history where the pressure increases. Figure 4.3 plots the sum of the square errors for the time intervals where pressure decrease. The curves in these two figures indicate reasonably well defined optimal values for T_i and T_d , with $T_i = 0.029$ sec and $T_d = 0.035$ sec.

Figure 4.4 shows, in solid line, part of the time history of pressure response from the bang-bang control experiment. Also the pressure response obtained by computer simulation is shown in Fig. 4.4, in dashed line, using the first order time delay model of Eq. 2.6 with the optimal values of the time constants identified above. The simulation result is in good agreement with the experimental result, implying that the analytical model for the relationship between pressure response and control signal, Eq. 2.6, with the identified value of time constants represents the reality very well.

4.1.2 Relationship Between Friction and Pressure

In the structural model considered, the absolute value of response acceleration equals the normalized friction force f ($f = \mu g$) when the model is sliding as indicated in Eq. 2.2. Therefore, the response acceleration detected from the sensor can be used to study the behavior of the friction. From the response acceleration observed in Experimental Series 1 (passive isolation) conducted at various levels of constant pressure under El Centro excitation, the relationship between pressure p and normalized friction force f is identified.

As shown in Fig. 4.5, the relationship between the pressure and the normalized friction force follows a linear relationship. The coefficient of friction μ of about 10.2% at the pressure

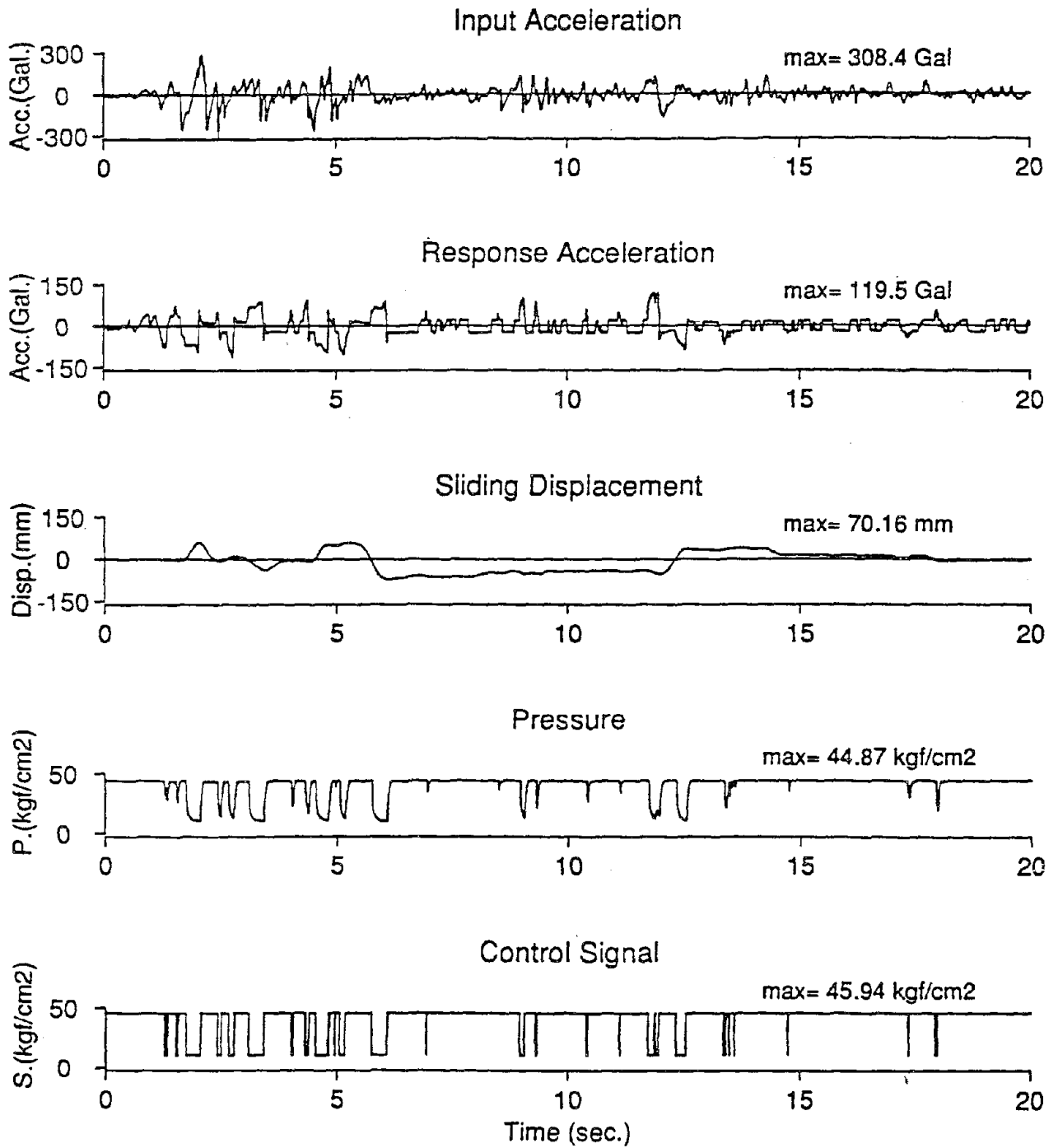


Figure 4.1: Bang-bang Control (Experiment)

Identification of Time Constant T_i

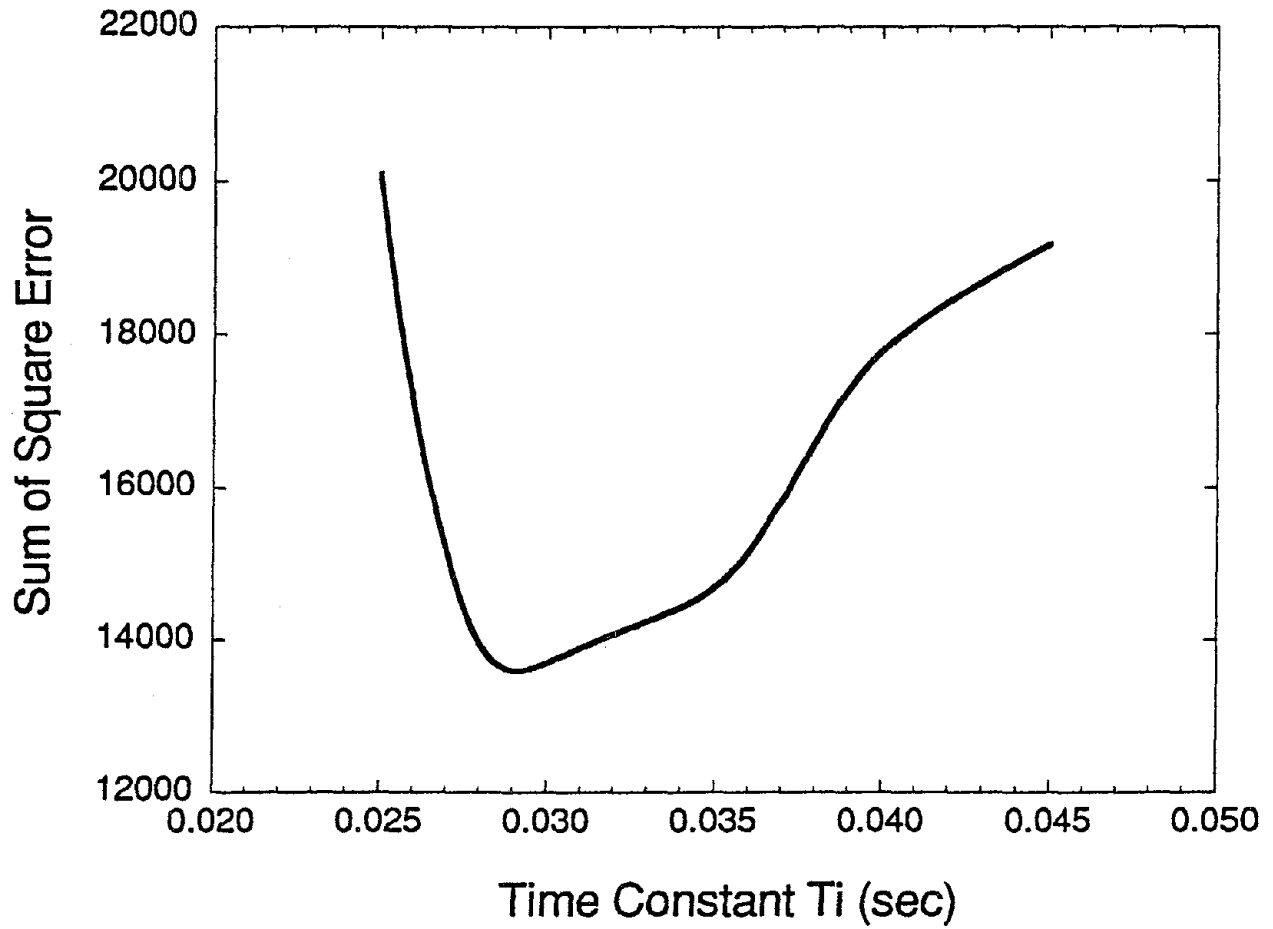


Figure 4.2: Identification of Time Constant T_i

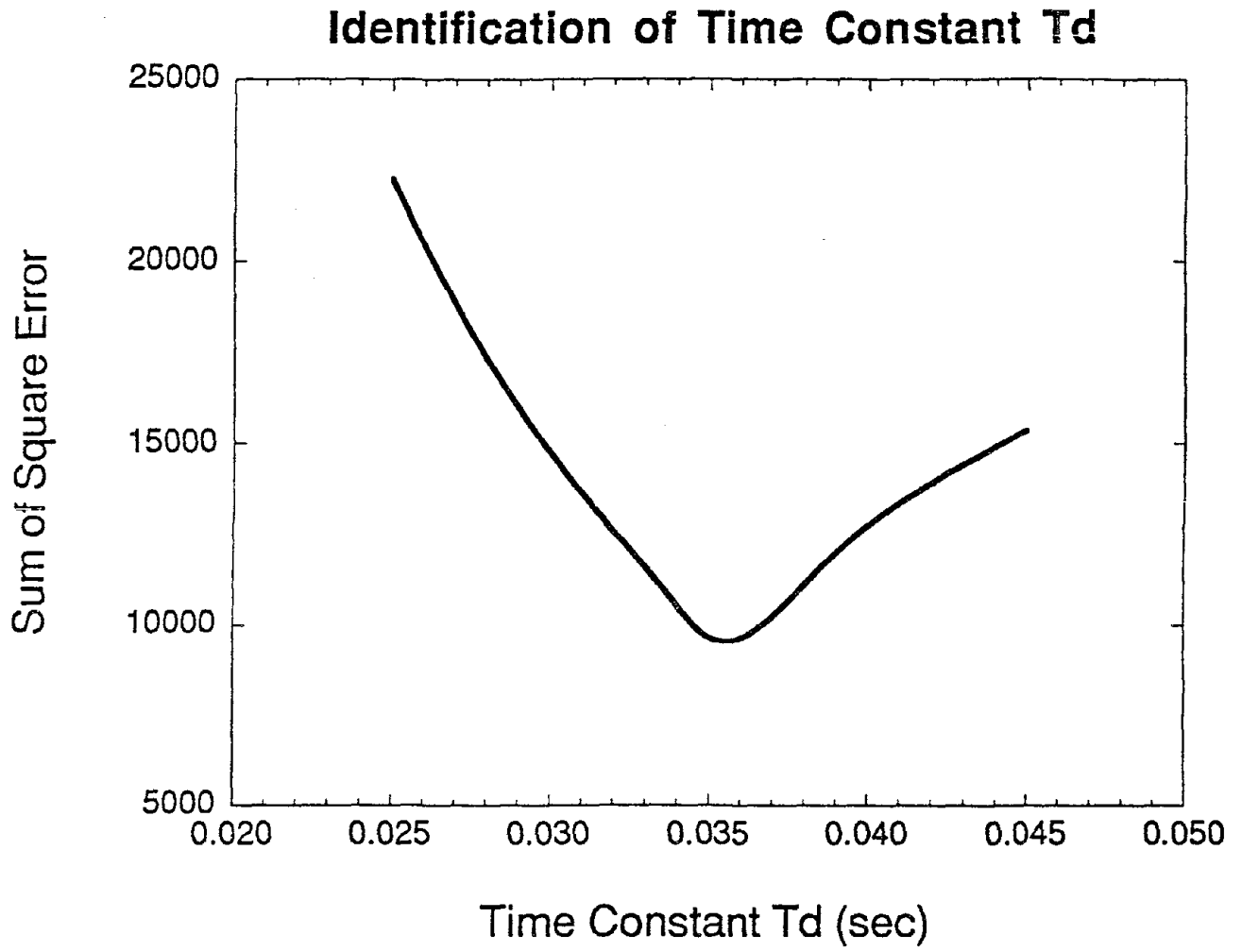


Figure 4.3: Identification of Time Constant T_d

Comparison of Experimental and Numerical Response of Pressure

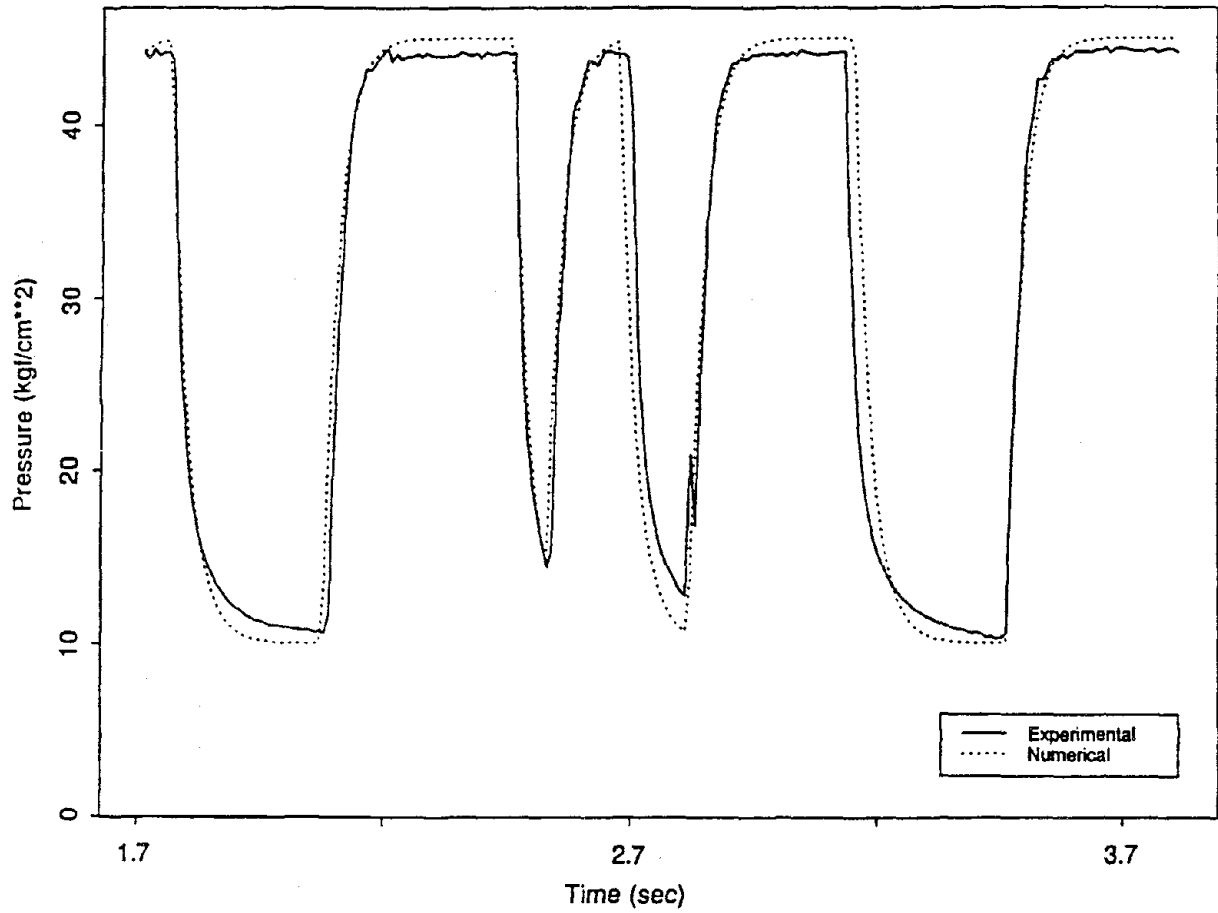


Figure 4.4: Example of Experimental and Simulated Time Histories of Pressure

of 10 kgf/cm² can, for example, be reduced to 1.6% at the pressure of 45 kgf/cm². Equation 2.7 thus can be used to describe the relationship, and the values of c_1 and c_2 are identified as $c_1 = 2.4\text{cm}^3/\text{s}^2\text{kgf}$ and $c_2 = 124.0\text{cm/s}$. As will be shown in the next subsection, however, the friction force depends not only on the pressure but also on the sliding velocity. In this respect, the value of normalized friction force used in Eq. 2.7 is valid only when the sliding velocity is zero, and therefore it is rewritten as

$$f_0 = -c_1 p + c_2 \quad (4.1)$$

where f_0 is the normalized friction force at sliding velocity of zero.

4.1.3 Dependence of Friction on Sliding Velocity

It is observed from the response acceleration measured in Experimental Series 1 (passive isolation) that the the normalized friction force f decreases with the increase of sliding velocity. In fact, these measurements are used to identify the dependence of friction on sliding velocity.

Figure 4.6 shows an example set of time histories of a passive isolation experiment, in which the pressure was kept at 30 kgf/cm², corresponding to the coefficient of friction 5.6%, and the input ground motion was El Centro record with peak acceleration of about 300 gal. From the behavior of the response acceleration, it is assumed that the relationship between the normalized friction force and sliding velocity takes the following analytical form:

$$f = f_0 \frac{k^2}{\dot{x}^2 + k^2} \quad (4.2)$$

where f_0 is the normalized friction at sliding velocity of zero as defined by Eq. 3.2, while k is a constant.

Relationship between Friction and Pressure

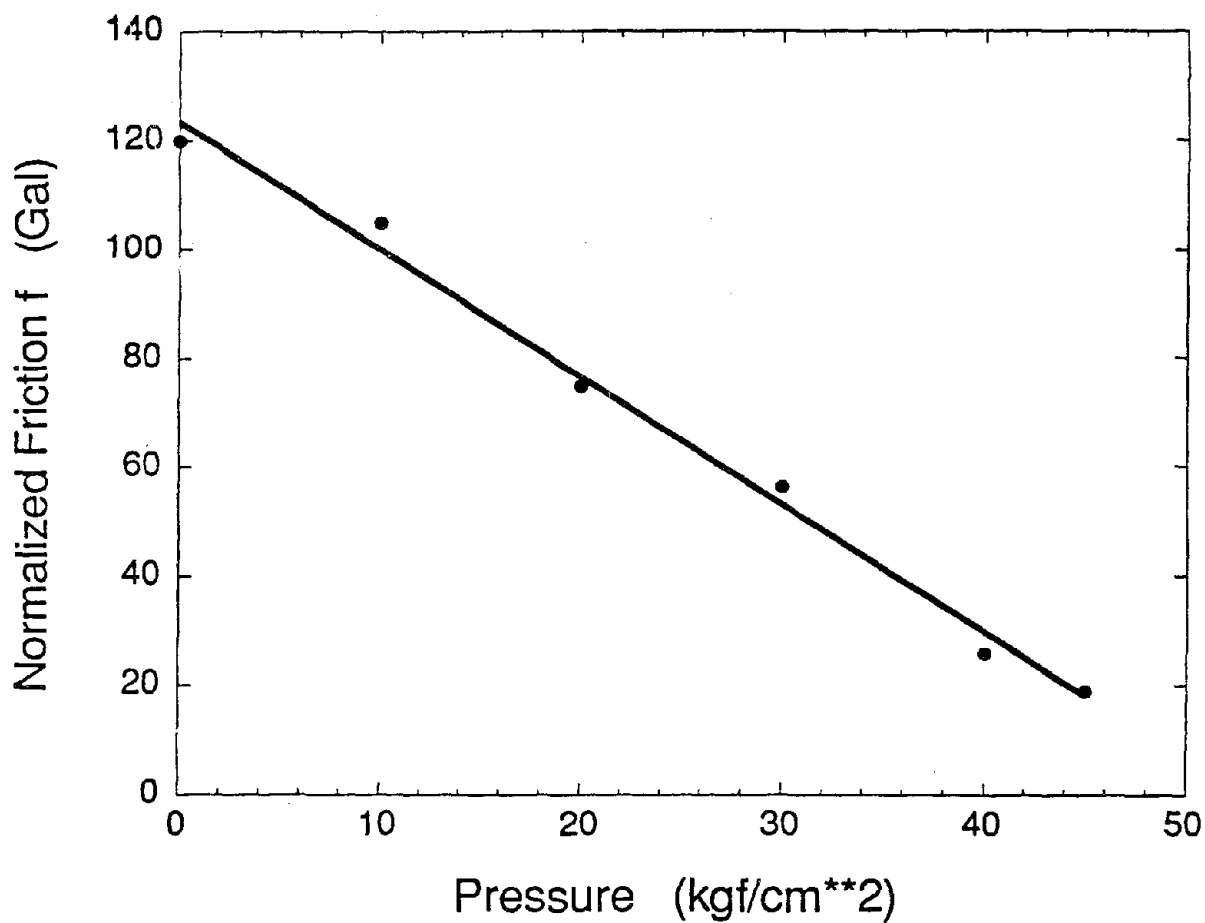


Figure 4.5: Relationship between Pressure and Normalized Friction Force

Table 4.1: Identified Parameters

T_i	29 ms	c_1	$2.4 \text{ cm}^3/(\text{s}^2\text{kgf})$
T_d	35 ms	c_2	124.1 cm/s^2
k^2	0.11 cm/s		

The value of k^2 is identified by minimizing the sum of the square errors between response acceleration shown in Fig. 4.7 and that from the numerical simulation over the entire time history ($\sum_k ((\ddot{x} + \ddot{z})_{sim}(t_k; k) - (\ddot{x} + \ddot{z})_{exp}(t_k))^2$). As shown in Fig. 4.7, the optimal value of k^2 is $0.11 \text{ m}^2/\text{s}^2$, at which the sum of square error is minimum.

It is found that $k^2 = 0.11 \text{ m}^2/\text{s}^2$ is also the optimal value for other passive isolation experiments where the pressure is kept at 10, 20, 40 and 45 kgf/cm^2 . Figure 4.8 shows part of the experimental and simulated time histories of response acceleration for the passive isolation performed under $p = 30 \text{ kgf}/\text{cm}^2$. Again, the simulation result matches the experimental result very well, implying that the identified model for friction closely represents the reality.

4.1.4 Response of Friction to Control Signal

Summarizing the identification results, the response of friction force to control signal can be modeled by the following equations:

$$T\dot{p} + p = u \quad (4.3)$$

$$f_0 = -c_1 p + c_2 \quad (4.4)$$

$$f = f_0 \frac{\dot{x}^2}{\dot{x}^2 + k^2} \quad (4.5)$$

The values of parameters T, c_1, c_2, k^2 identified are summarized in Table 4.1.

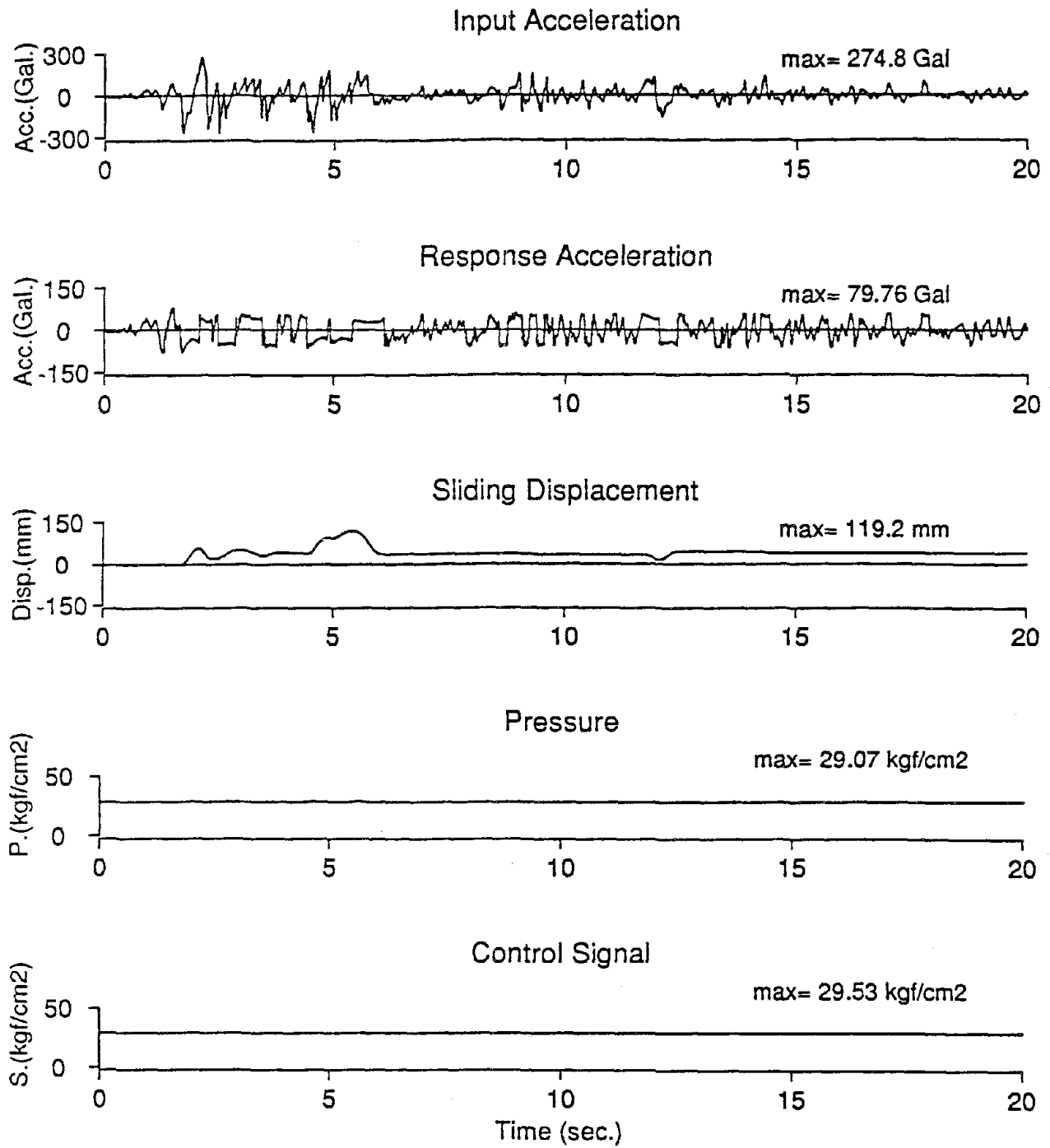


Figure 4.6: Passive (Experiment)

Identification of Velocity Dependence of Friction

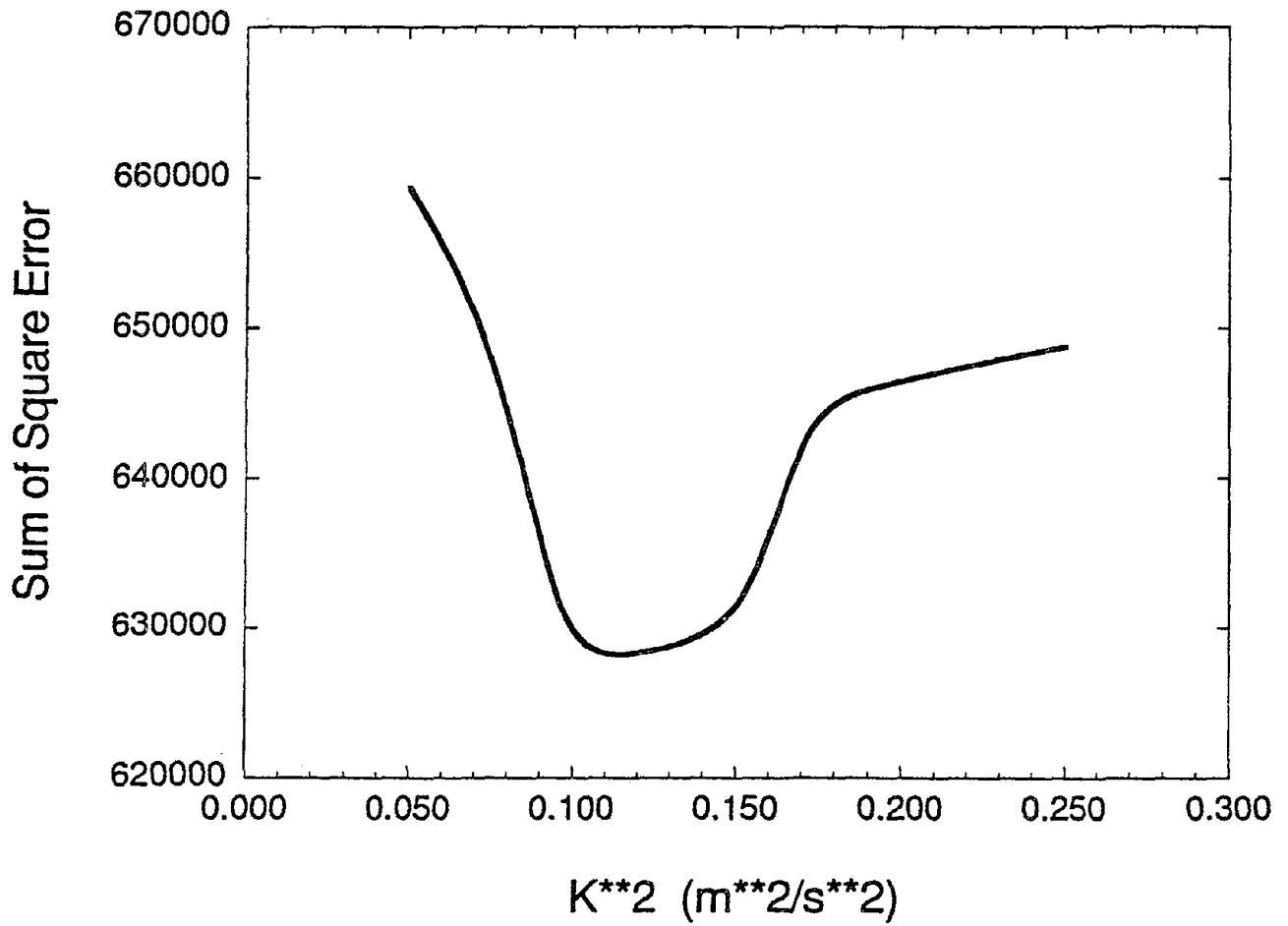


Figure 4.7: Identification of Constant k^2

Comparison of Experimental and Numerical Response Acceleration

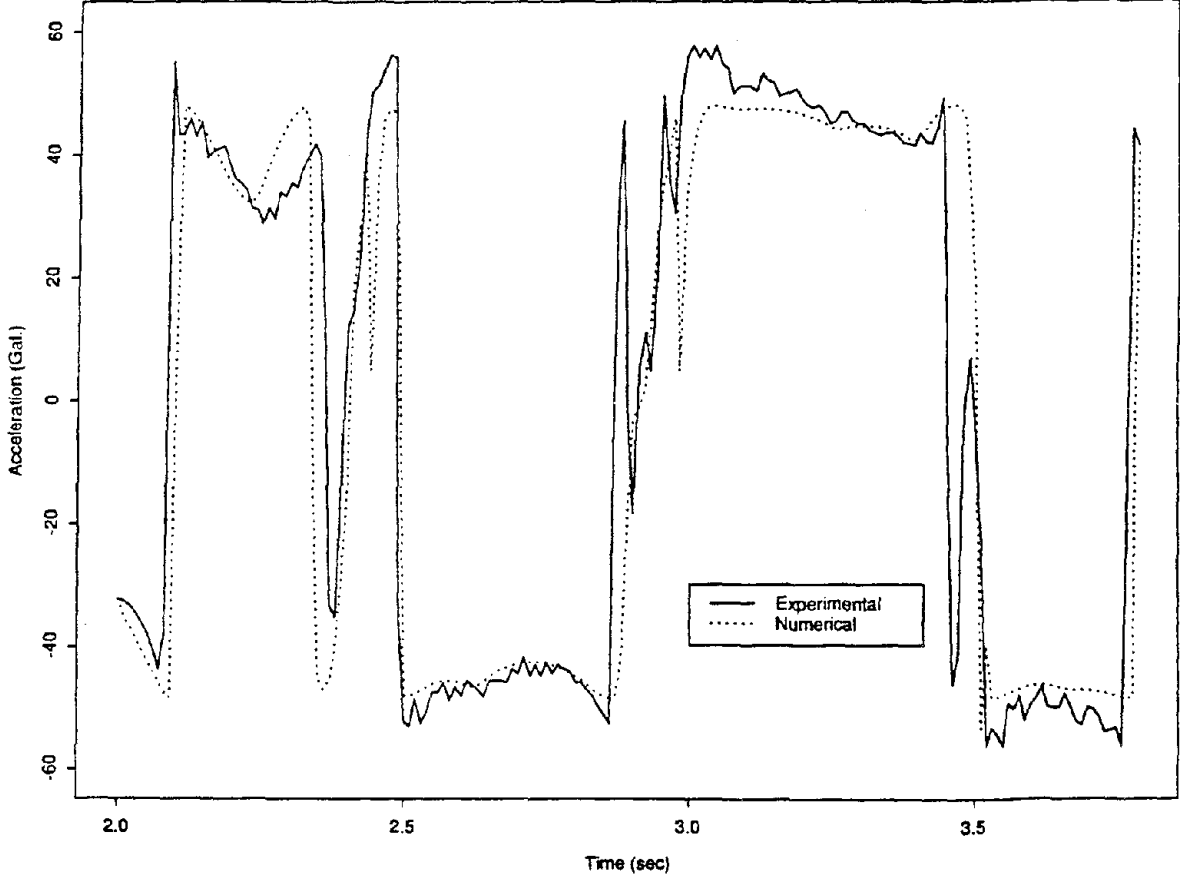


Figure 4.8: Example of Experimental and Simulated Time Histories of Acceleration

4.2 Passive and Hybrid Isolation

Typical experimental results from four series of experiments utilizing the techniques of passive isolation, bang-bang control, and instantaneous optimal control with and without time delay are compared with each other. These results are obtained under the El Centro (NS, 1940) ground motion with the peak acceleration of about 300 Gal. Figures 4.6 and 4.1 shown before give two sets of time histories for response acceleration, sliding displacement, pressure, and pressure control signal corresponding to Experimental Series 1 and 2 respectively, while Figs. 4.9 and 4.10 show two sets corresponding to Experimental Series 3 and 4.

In the passive isolation results as shown in Fig. 4.6, the pressure in the bearing chamber was kept at 30 kgf/cm^2 , corresponding to the coefficient of friction 5.6%. A large residual sliding displacement of 62 mm after the earthquake is observed.

In the bang-bang control experiment shown in Fig. 4.1, the pressure control signal is switched between 10 kgf/cm^2 and 45 kgf/cm^2 . The maximum and minimum response accelerations are consistent with the accelerations of the passive isolation cases of $p = 10 \text{ kgf/cm}^2$ and 45 kgf/cm^2 respectively. However, the residual displacement is almost zero and the maximum sliding displacement is about 20 % less than the passive isolation, showing the effectiveness of the bang-bang control in reducing the sliding displacement.

Figures 4.9 and 4.10 respectively show the results of hybrid isolation experiments using instantaneous optimal control without and with time delay, in which the feedback gains are set to: $F_f = 1.0 \text{ kgfs}^2/\text{cm}^3$ and $F_d = -45 \text{ kgf/cm}^3$, and $F_f = 16.5 \text{ kgf/cm}^2$ and $F_d = -16.4 \text{ kgf/cm}^3$ with time delay. In both cases, the pressure control signal is confined between $u_{max} = 45 \text{ kgf/cm}^2$ and $u_{min} = 10 \text{ kgf/cm}^2$. The response of instantaneous optimal control,

depicted in Figs. 4.9 and 4.10, show the results similar to that of the bang-bang control experiment. This is because the feedback gains used in these experiments force the control signal to reach the minimum value of 10kgf/cm^2 primarily for the purpose of confining the sliding displacement to a small value under the ground acceleration as large as El Centro earthquake with peak acceleration of about 300 gal. The instantaneous optimal control with time delay (Fig. 15(a)) shows the best control performance among the three control algorithms developed and tested in the experiments.

The difficulty of starting initial slide associated with the passive system is alleviated in the hybrid systems.

The experimental results of passive and hybrid isolation under the Hachinohe (NS) and Taft (EW) are shown in the Appendix A.

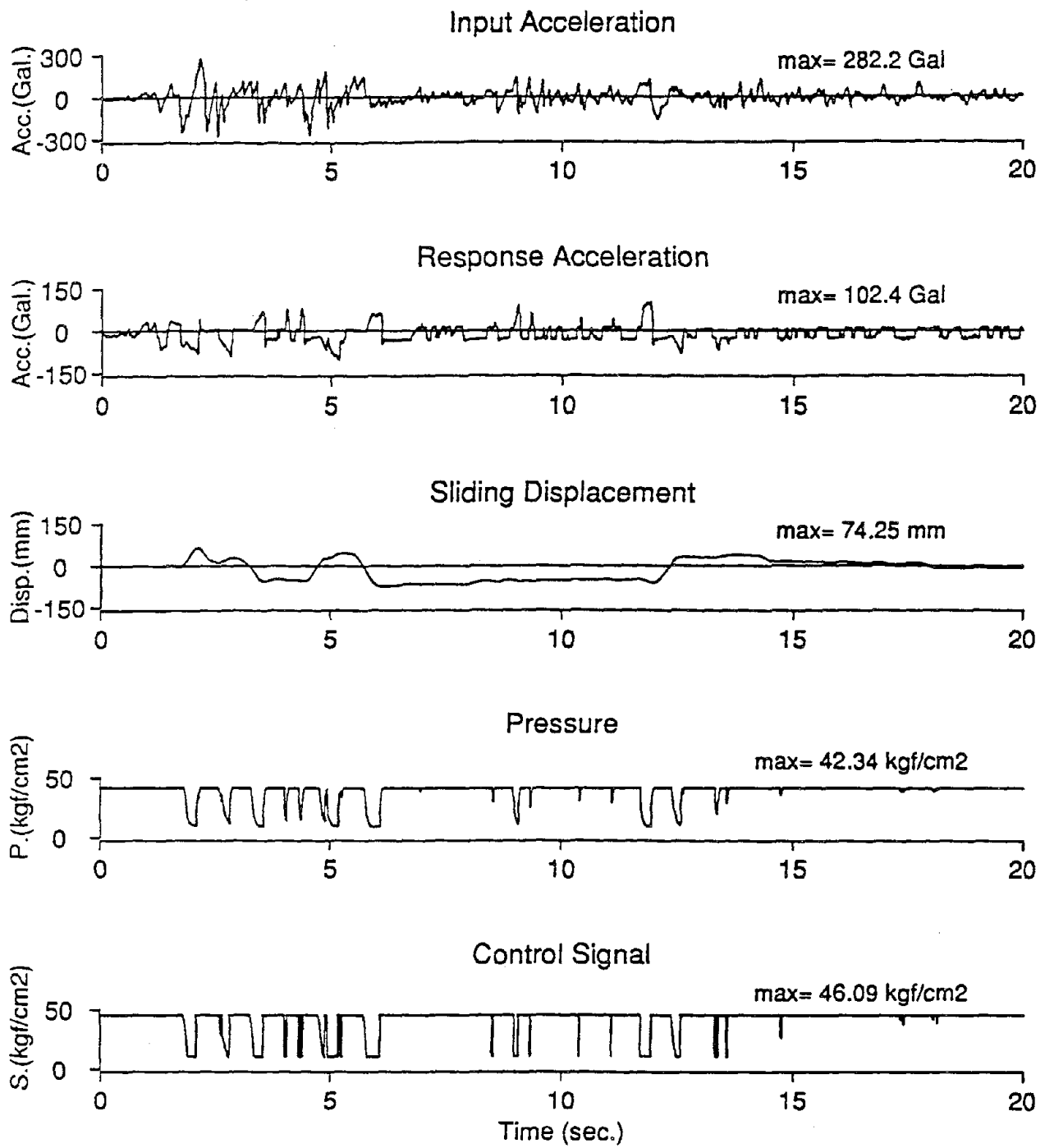


Figure 4.9: Instantaneous Optimal Control without Time Delay (Experiment)

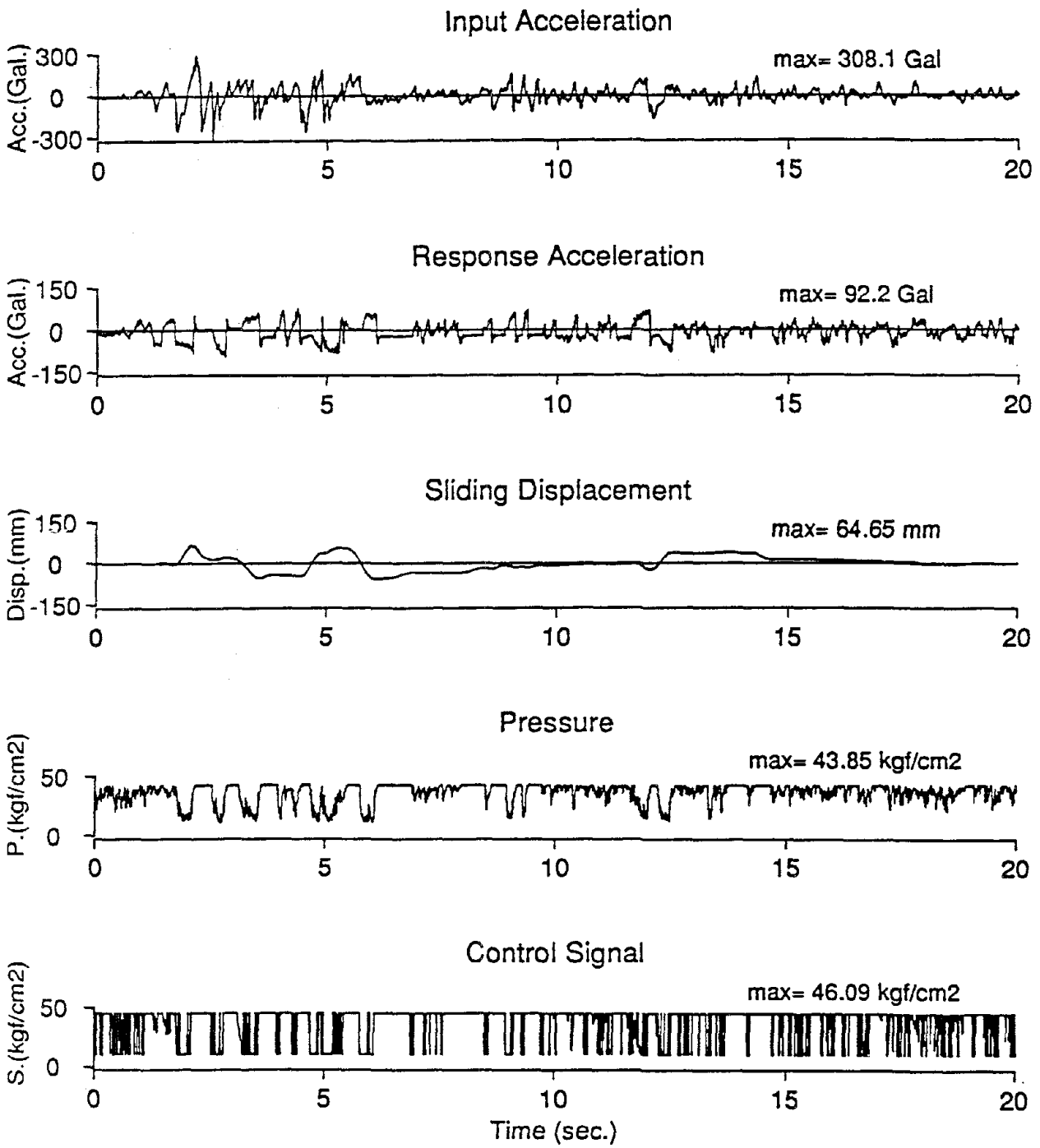


Figure 4.10: Instantaneous Optimal Control with Time Delay (Experiment)

4.3 Comparison of Hybrid and Passive Isolation

The performance of the hybrid isolation system is compared to that of the passive isolation system. Figures. 4.11 and 4.12 show the maximum response acceleration, maximum sliding displacement, and the residual displacement of the structural model with passive or hybrid isolation system under El Centro records of different peak ground acceleration. Hybrid isolation results shown in these figures are those under instantaneous optimal control algorithm without time delay.

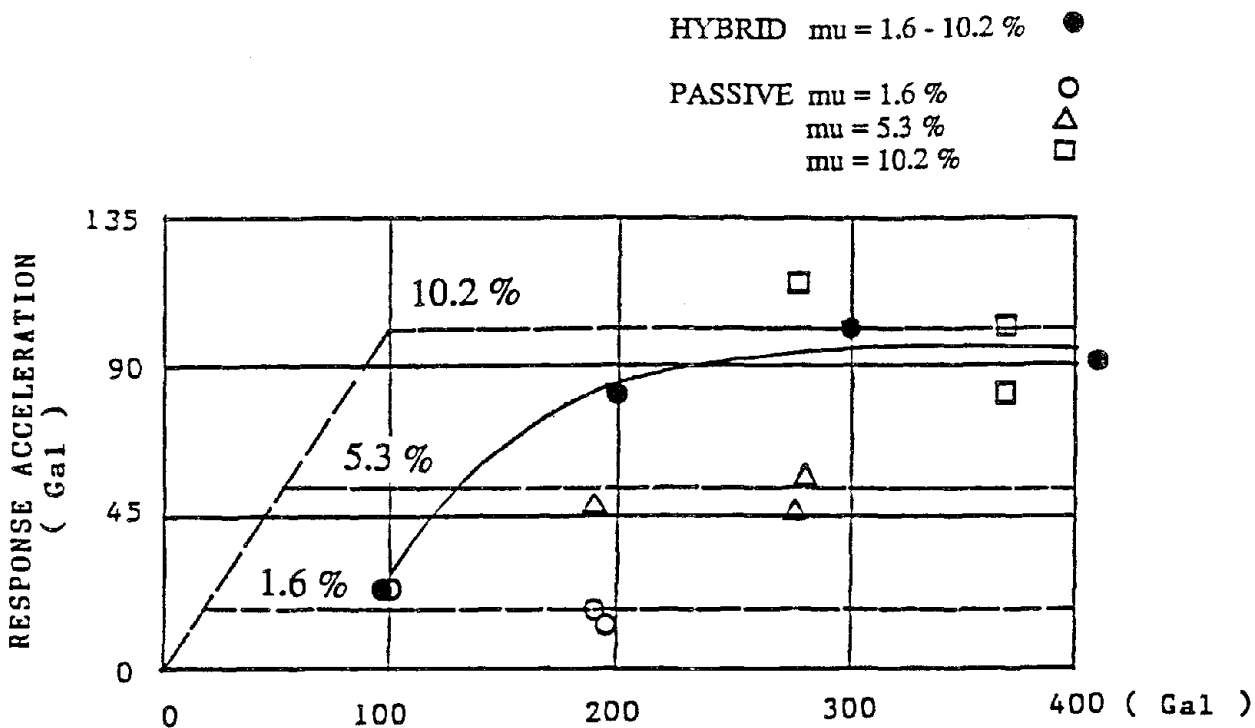
In the passive isolation, if a small friction coefficient, for example 1.6%, is used, a high level of isolation performance is expected since the response acceleration is reduced to a small level. In this case, however, the maximum displacement becomes excessive very rapidly as the input earthquake becomes more intense. On the other hand, if a large friction coefficient such as 10.2% is used, the sliding displacement can be confined within a relatively small range. However, the isolation performance in this case is limited in the sense that the acceleration can not be satisfactorily reduced. Particularly for small to medium earthquakes with peak acceleration less than 100 gal, this passive sliding isolation system does not function at all, thus the response acceleration equals the input acceleration. Such acceleration might damage the sensitive equipment inside the building.

The hybrid isolation system, however, can alleviate these problems associated with the passive isolation system. For small to medium earthquakes, the friction can be controlled to a very small level to make the structure slide easily, so that the response acceleration can be considerably reduced. For large earthquakes, the friction is controlled to prevent the excessive sliding displacement, while the response acceleration can also be kept at a small level. Another advantage of the hybrid system is clearly seen in Fig. 4.12 where the residual

sliding displacement can be maintained almost at zero level.

Experimental results verify that the proposed friction controllable sliding system does make the conventional passive sliding isolation system effective for all intensities of earthquake.

Comparison of Response Acceleration



Comparison of Sliding Displacement

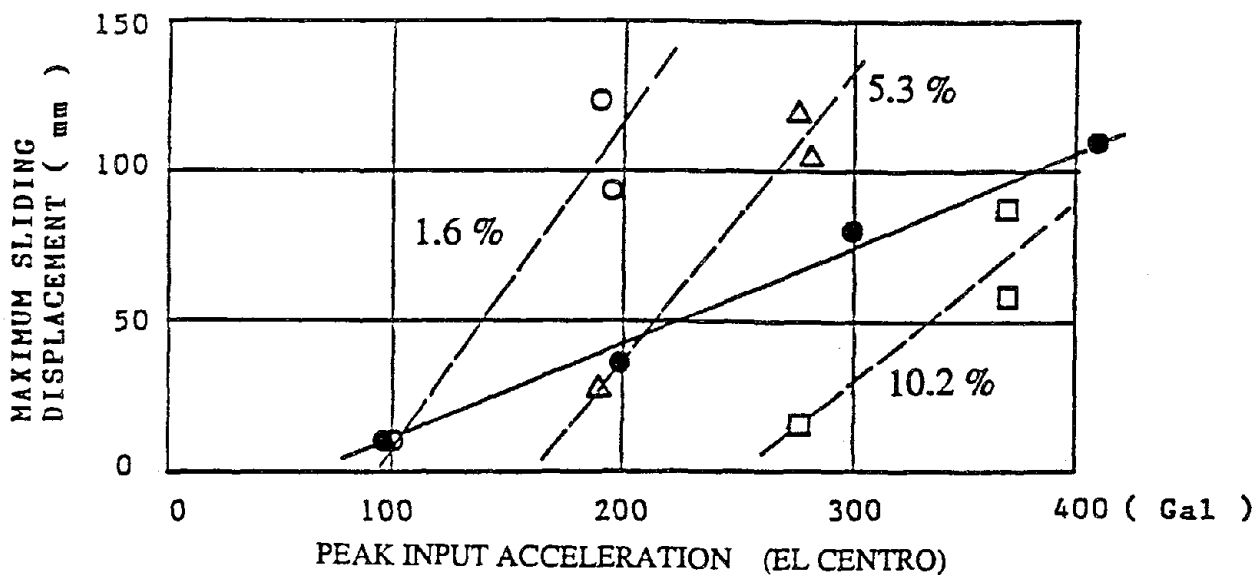


Figure 4.11: Comparison of Passive and Hybrid Isolation

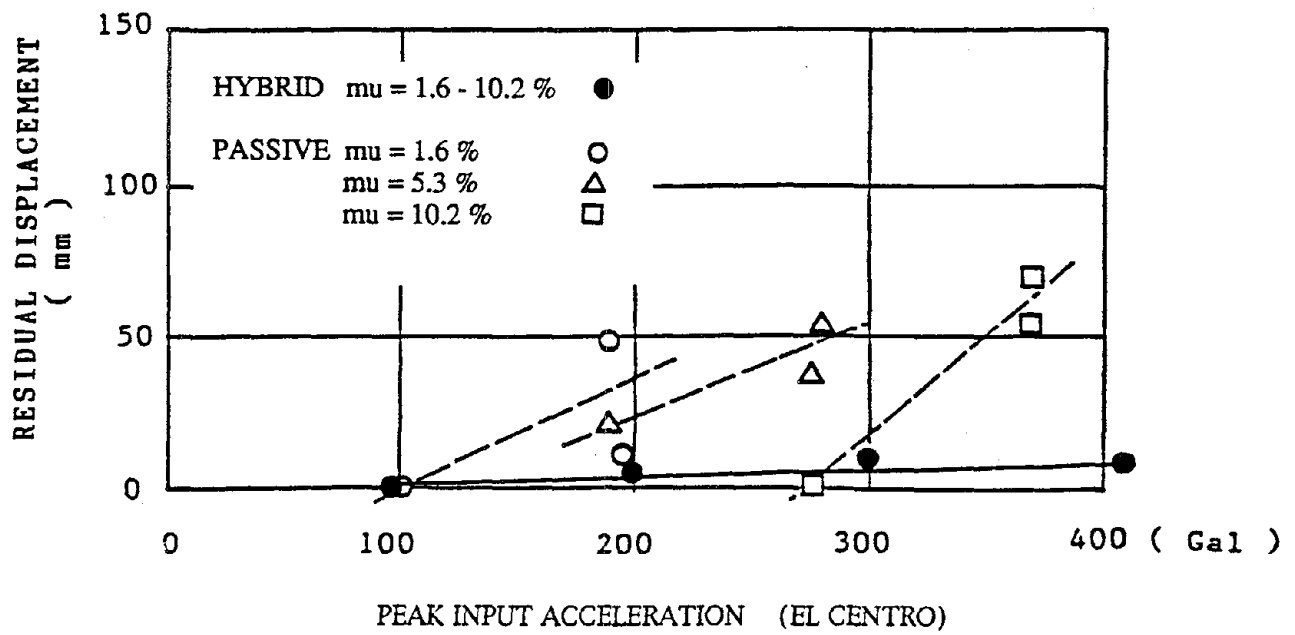


Figure 4.12: Comparison of Passive and Hybrid Isolation (Residual Displacement)

4.4 Study of Control Parameters

The effects of control parameters, such as the time interval of control Δt_c and the threshold value for the window comparator, on control performance have been studied in shaking table experiments. On the basis of these studies, the appropriate values of these parameters are determined. Although the values of feedback gains were also examined in a few cases of shaking table experiments, they were determined by computer simulation analysis.

4.4.1 Time Interval of Control

The continuous-time control algorithms developed in Section 2 can only be executed in the discrete time since a digital computer is used for on-line computation and control execution. As a consequence, response measurements are digitized as feedback signals, while control signal is sent in the form of piecewise step functions through the analog-digital converter as shown in Fig. 3.6. Hence, they are not continuous functions as called for by continuous-time control algorithms, and usually the larger the time interval for control execution, the worse the control performance becomes.

For this reason, the effect of the time interval of control on control performance has been studied in shaking table experiments performed for hybrid isolation system. The response time histories obtained in the experimental case identified as 6-3 with the time interval of control $\Delta t_c = 0.030$ sec, and experimental case identified as 6-6 with $\Delta t_c = 0.004$ sec, are shown respectively in Figs. 4.13 and 4.14. Both cases are under instantaneous optimal control without consideration of time delay, and are subjected to El Centro 300 Gal input earthquake. The time interval for the measurement of feedback signals are $\Delta t_d = 0.002$ sec in both cases.

In the response acceleration shown in Fig. 4.13 (Case 6-3), impulsive response behaviors are observed (denoted by ∇). They are, however, mostly eliminated in Case 6-6 as shown in the same figure indicating the improvement of control performance by reducing the time interval of control. The sliding displacements, however, show little difference between these two cases as seen in Fig. 4.14.

Further investigation shows that the time interval of control smaller than 0.004 sec does not significantly improve control performance. Hence, the time interval of control Δt_c has been set at 0.004 sec in all other control experiments.

It is also interesting to find that the time interval does not change the basic behavior of the structural response without introducing instability. For example, even the time interval as large as 0.03 sec does not introduce the instability of the control system as shown in Figs. 4.13 and 4.14. In this sense, this hybrid control system is quite robust.

4.4.2 Window Comparator

From the control experiments, it was found that there was some high frequency noise in the measurement of feedback signals which resulted in undesirable control signal. In fact, the use of the feedback signal including such high frequency components in experiments produced highly inefficient control results. To alleviate this difficulty, a practical method called "window comparator" is used. The method simply defines a certain threshold value, for the increment of a feedback signal over a certain time interval. If the increment is below the threshold, the variation in the feedback signal is considered as noise and the feedback signal is considered unchanged over that time interval.

The appropriate threshold values for displacement and acceleration signals were studied

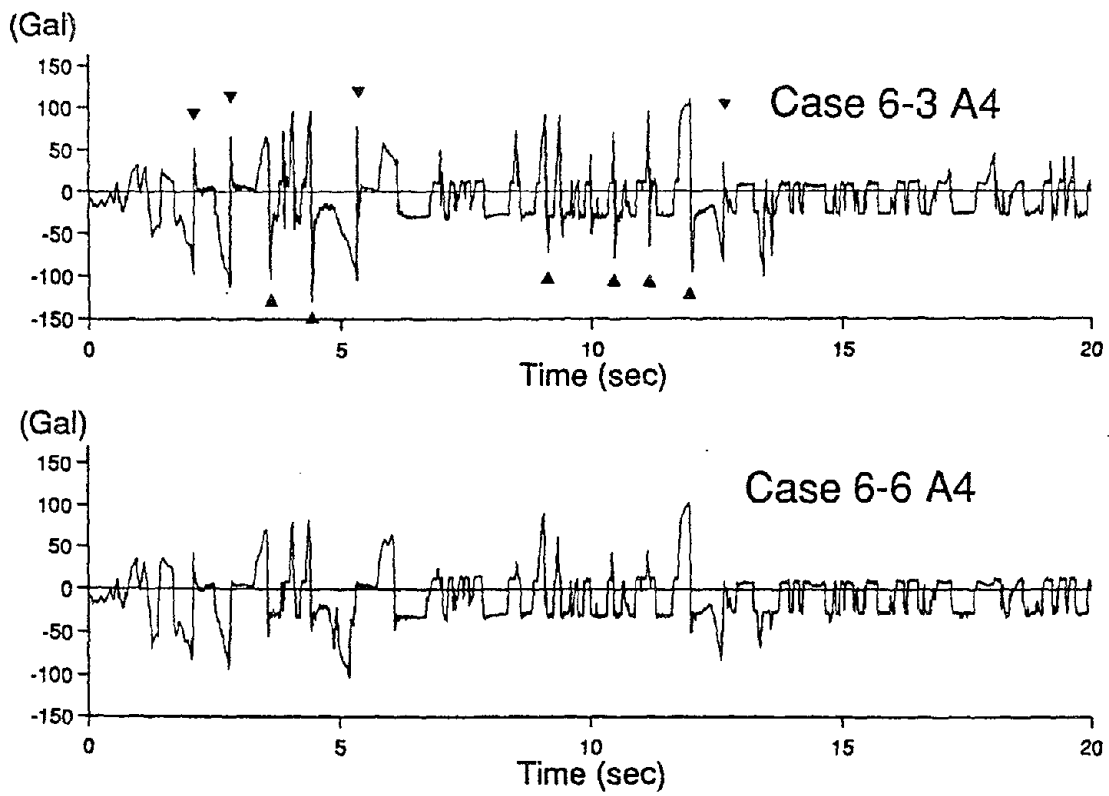


Figure 4.13: Effect of Control Time Interval on Response Acceleration

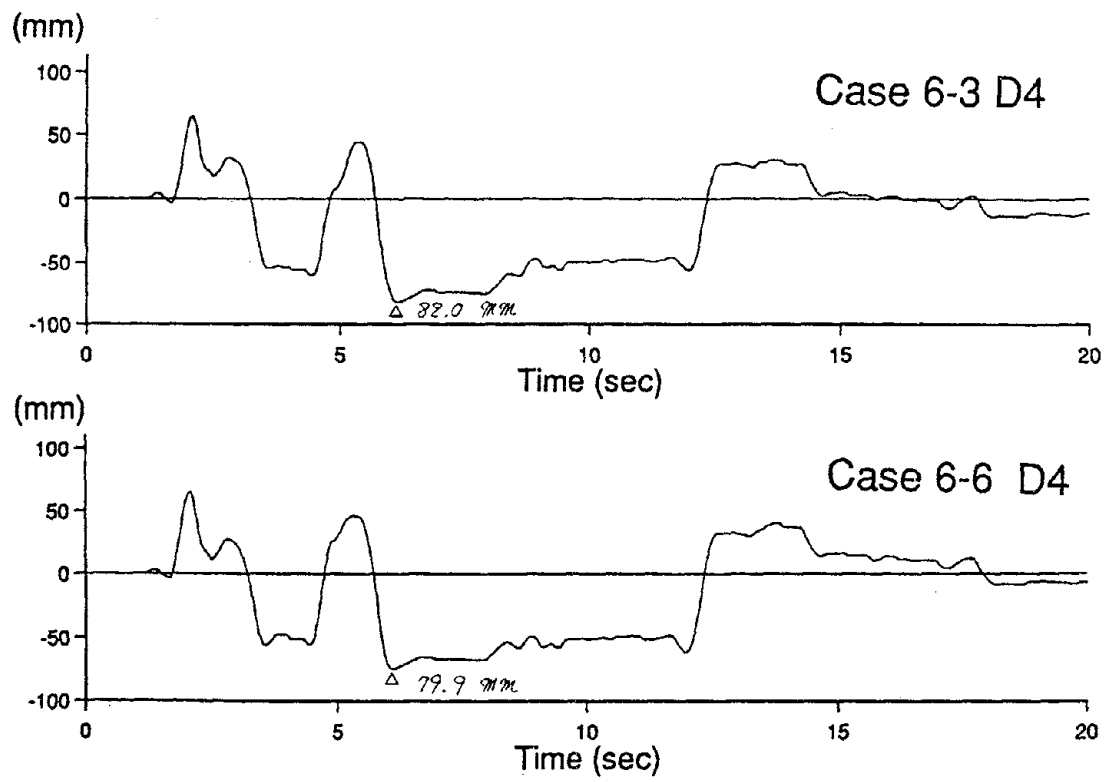


Figure 4.14: Effect of Control Time Interval on Sliding Displacement

respectively in hybrid isolation experiments using instantaneous optimal control with and without time delay. As a result, the threshold value for displacement signal was determined to be $\Delta x = 0.02$ cm for every two control time intervals of 0.008 sec, and the threshold value for acceleration signal was determined to be $\Delta(\ddot{x} + \ddot{z}) = 0.75$ gal for each control time interval of 0.004 sec. The control results are dramatically improved when these threshold values are used. To examine the significance of the threshold values, the increment Δx measured in displacement x measured for every time interval of 0.008 sec from an optimal control experiment is plotted at the end of each time interval as shown in Fig. 4.15, together with time history of displacement. This figure shows that the threshold value equal to 0.02 cm for the displacement increment consistently removes the noise-like high frequency components of the feedback signal.

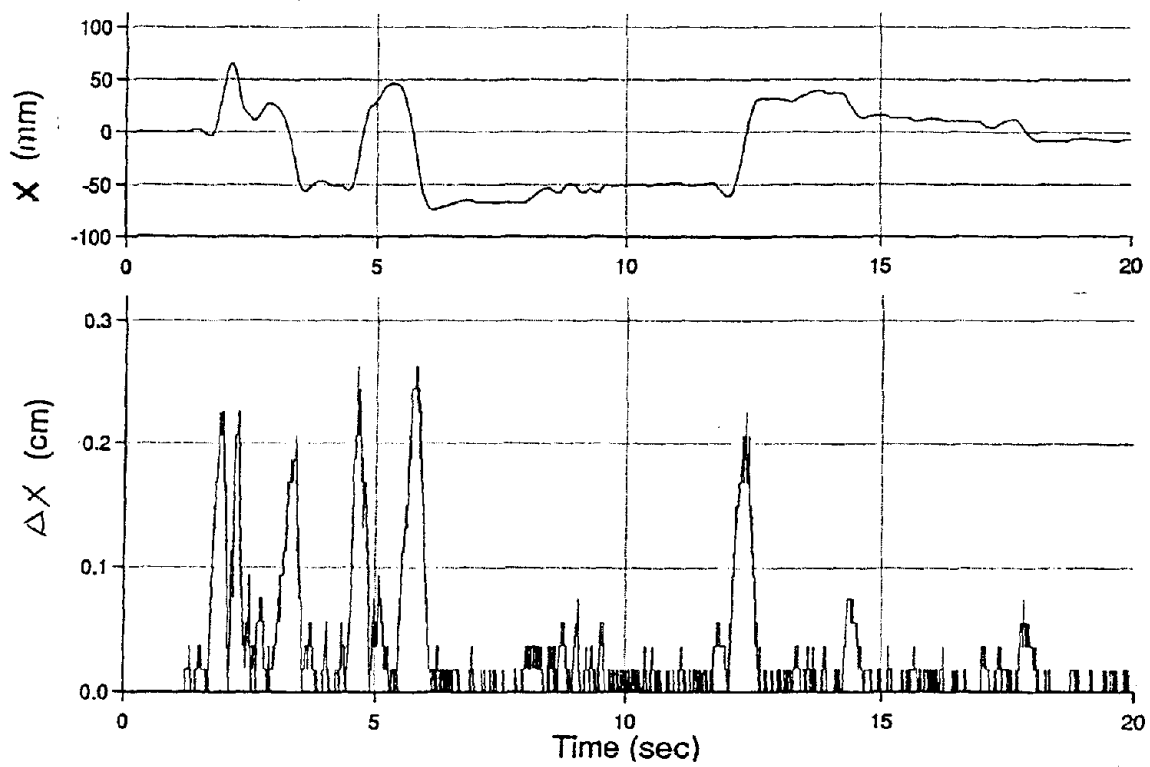


Figure 4.15: Study of Window Comparator

SECTION 5

NUMERICAL SIMULATION

A computer program for the simulation of seismic response of the structure under the hybrid or passive isolation system has been developed. Analytical models presented in the preceding sections, which are based on careful identification experiments, are utilized for computer simulation of experimental results. In this section, it will be demonstrated that these analytical models describe the real system accurately, by carefully comparing the numerical simulation results based on these models with the experimental results.

5.1 Analytical Model and Techniques for Numerical Simulation

The equations of motion of the structural model shown in Eqs. 2.1 - 2.5 and the relationships between the pressure control signal and friction described by Eqs. 4.3 - 4.5 are used for computer simulation. The parameters in these equations use the values given in Table 4.1 as identified by experiments.

Based on the Newmark's β scheme, a double precision FORTRAN routine for numerical integration of the equations of motion and equations of control system has been developed, in which β is selected to $1/6$. For the friction system such as under consideration, the precise evaluation are crucial to the accuracy of response calculation when the motion of the structure undergoes transition between the sticking and sliding phases and when the sliding velocity reverses its direction. A time step of Δt is used in the continuous phases of motion, while a much smaller time step Δt is used whenever the phase transits or the velocity reverses its direction. The following computational algorithm is proposed [19] and used.

In Phase II (sliding phase),

1. If the relative velocity of the model changes its direction, go one time interval Δt back, change the time step from Δt to Δt_t and continue the computation again.
2. When the relative velocity changes its direction again, go one time interval Δt_t back, examine whether or not the structural model goes into sticking phase, according to the criterion for transition given in Eq. 2.5.
3. If it goes into sticking phase, change equation of motion to Eq. 2.1, and alter the time step from Δt_t to Δt .
4. If it continues to slide, reverse the direction of friction force, and continue the computation in sliding phase using the time step Δt .

In Phase I (sticking phase),

1. Examine if the model goes into sliding phase according to the criterion given in Eq.

2.3. If it does, change the equation of motion to Eq. 2.2, and change the time step from Δt to Δt_t for the next time interval Δt .

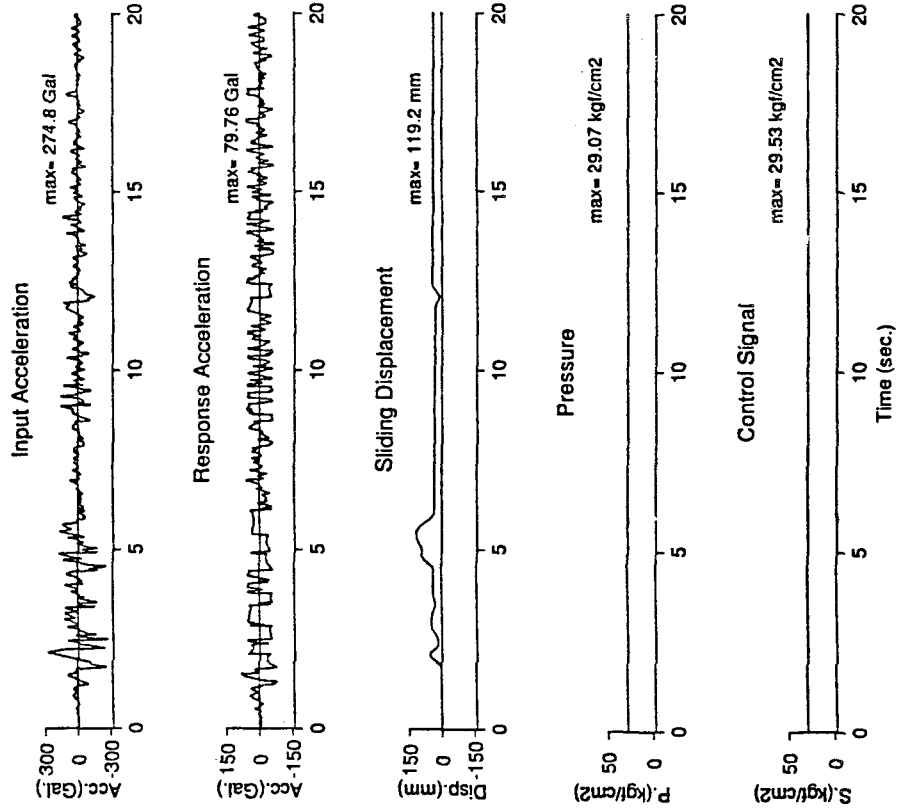
The accuracy of the simulation results is verified by comparing the results with those by finer time steps Δt_t on several simulation cases, and the time step Δt_t has been determined to be 0.00001 sec . The value of time step Δt should be at least 0.002 sec in the simulation of hybrid isolation, because this value was used as the time interval for the measurement of feedback signals in the hybrid control experiments as mentioned before. In the simulation of the response of the building which is passively isolated, however, the value of the time step $\Delta t = 0.01$ sec is used and it appears to be fine enough for the numerical integration.

5.2 Comparison of Simulation with Experimental Results

Shaking table experimental results described in the preceding sections are simulated by numerical analysis. Some examples of time histories from simulation of passive isolation and hybrid isolation using bang-bang control as well as instantaneous optimal control without and with time delay are shown in Figs. 5.1 - 5.4 respectively, together with the corresponding experimental results. In this respect, several remarks seem in order:

1. On the whole, numerical simulation results show remarkably high degree of agreement with experimental results. This demonstrates that the analytical model with the parameter values used represents the reality very well.
2. In simulation, the static coefficient of friction is assumed to be the same as the sliding coefficient of friction. Therefore, simulation could not show the maximum response acceleration, which is due to the static friction, in the passive isolation experiment.

Experiment



Simulation

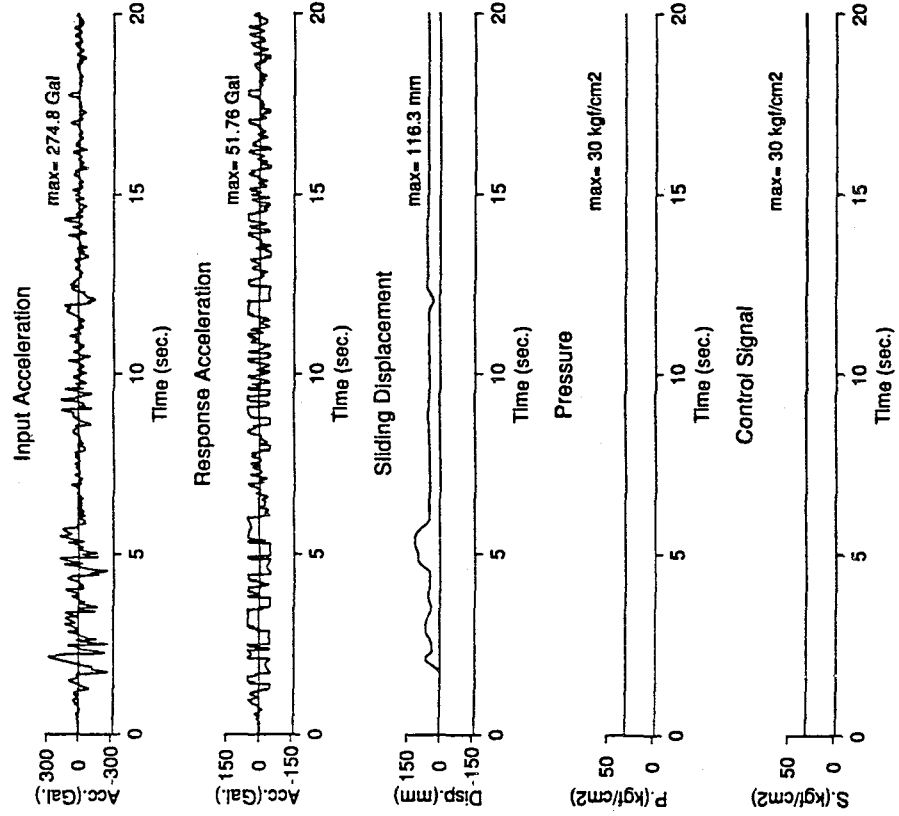


Figure 5.1: Passive Isolation

Experiment

Simulation

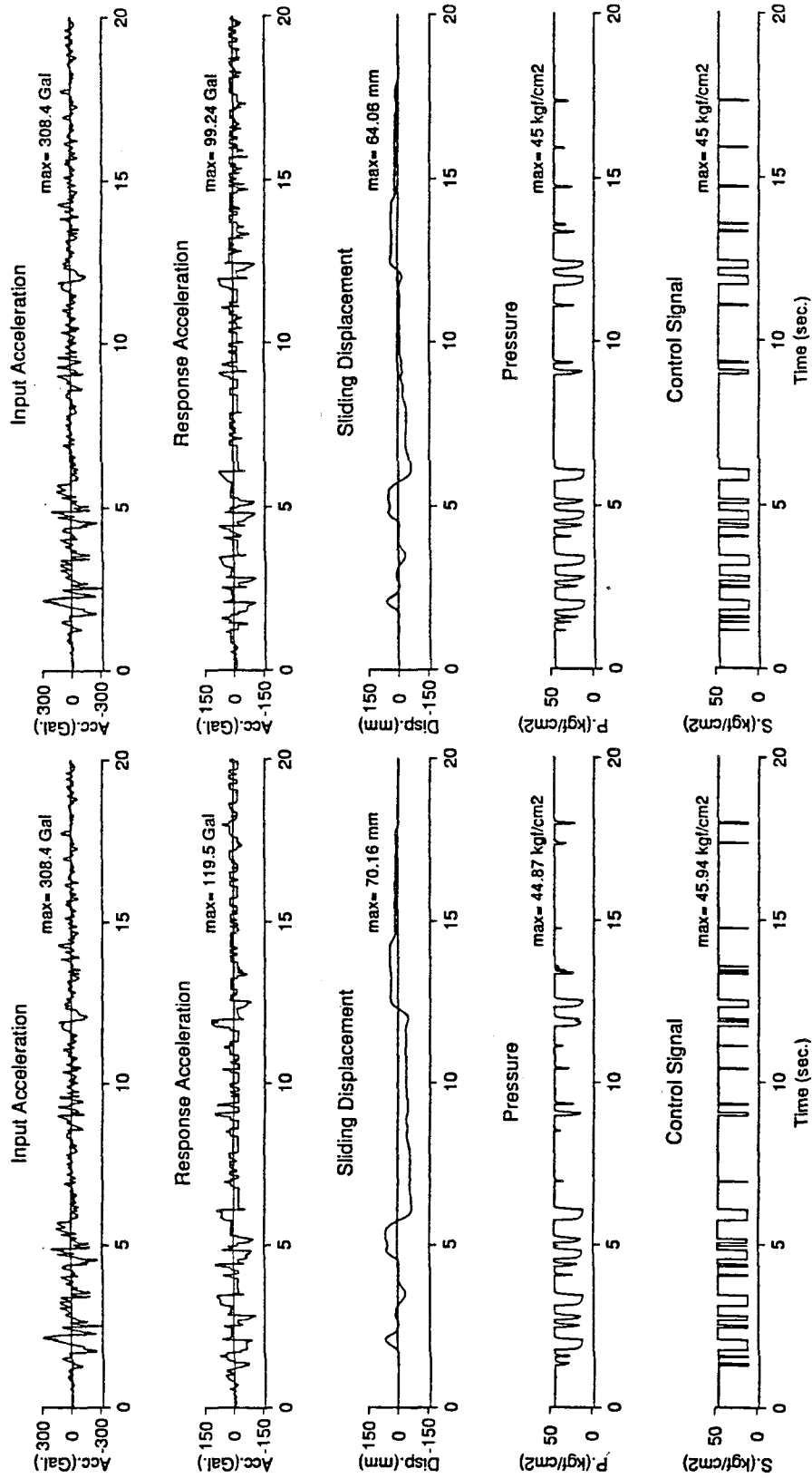
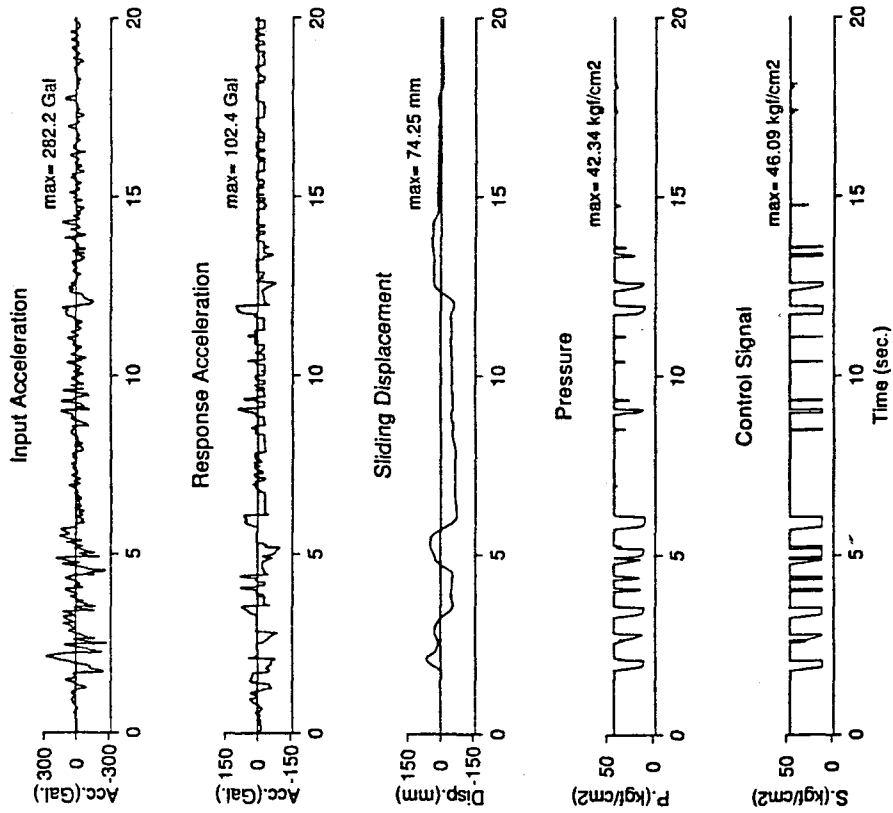


Figure 5.2: Hybrid Isolation under Bang-Bang Control

Experiment



Simulation

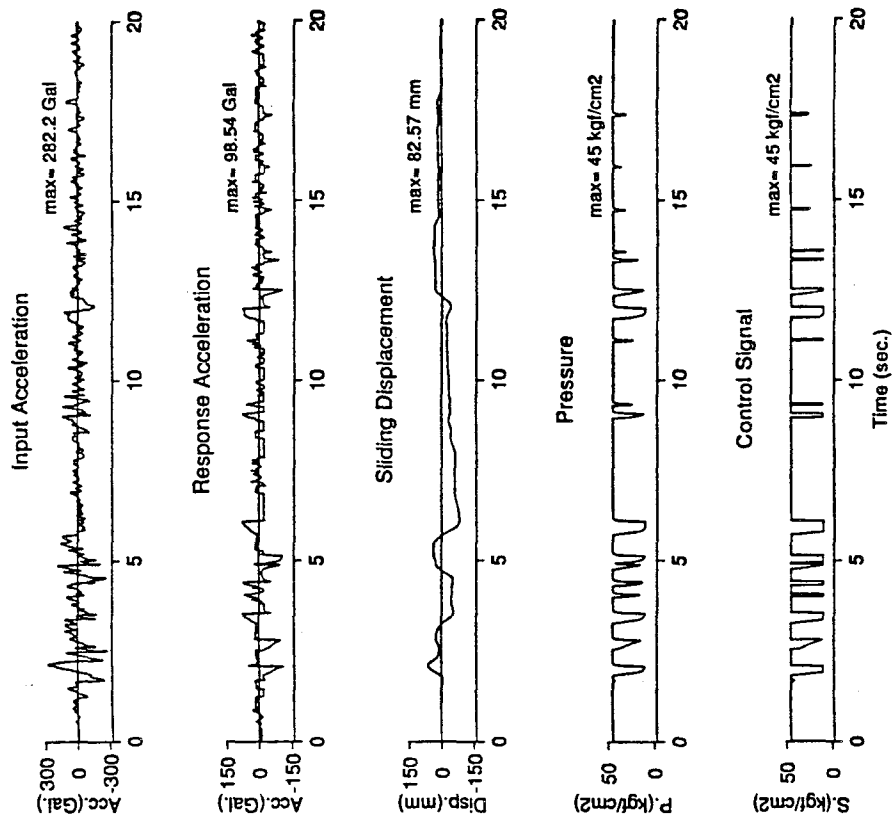
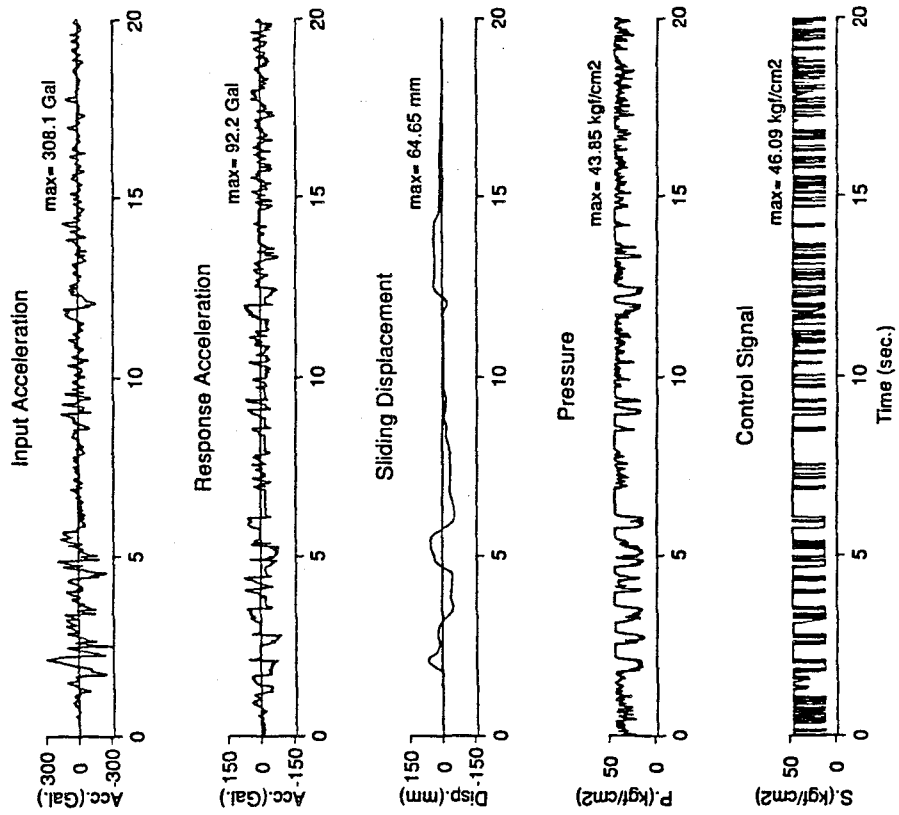


Figure 5.3: Hybrid Isolation under Optimal Control without Time Delay

Experiment



Simulation

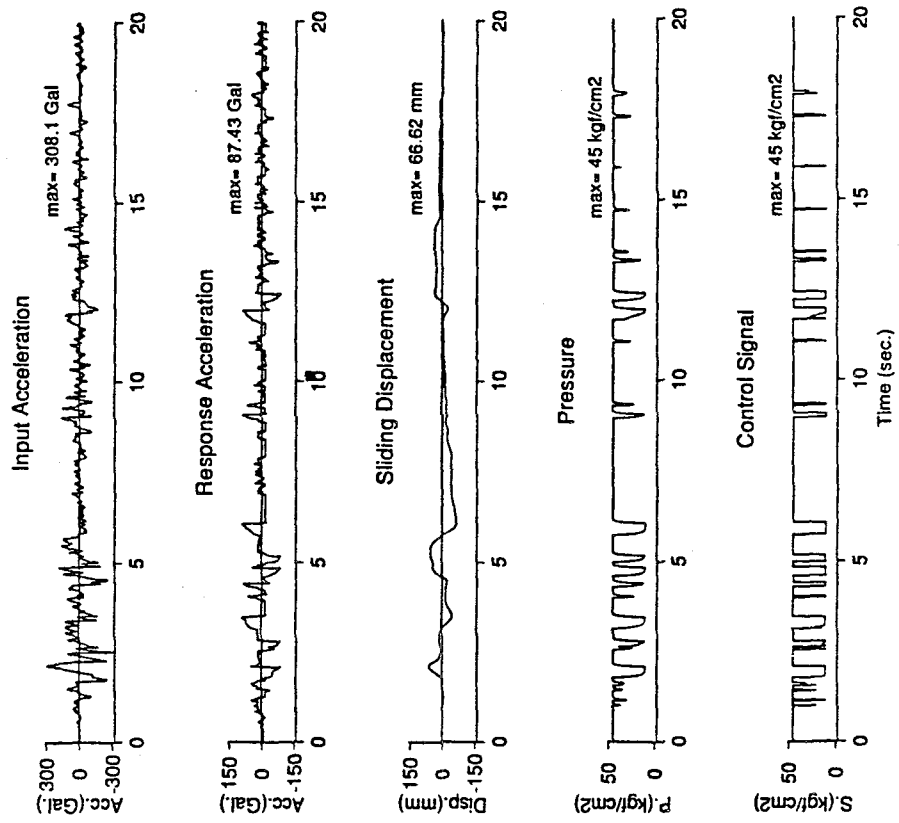


Figure 5.4: Hybrid Isolation under Optimal Control with Time delay

5.3 Robustness of System

In the experiments and computer simulation, it is found that the proposed hybrid isolation system is quite robust.

1. The instantaneous optimal control algorithms in Section 2 are developed on the reduced-order models, ignoring the following aspects in the real models: (i) sticking phase of motion (Eq. 2.1), (ii) influence of sliding velocity on the friction force (Eq. 3.3), and (iii) time delay in the pressure response to the control signal (Eq. 2.6), obviously when developing the algorithm without time delay. However, the reduced-order model did not adversely affect the control performance in a significant way. The so called “control spillover” and instability of the control system were not observed. For example, the comparison between the experimental results of the instantaneous optimal control with time delay (see Fig. 4.10) and without time delay (see Fig. 4.9) showed little difference in the response behavior. This suggests that neglecting the time delay does not degrade the control effectiveness considerably.
2. The time interval of control implementation can be relatively long, as mentioned before.
3. The feedback control signal obtained from the instantaneous optimal control algorithm is not a function of the system parameters, as shown in Eqs. 2.23 and 2.30. Hence, the control efficiency in this case is not affected by parameter variations.

SECTION 6

CONCLUSIONS

A systematic study on a hybrid sliding seismic isolation system using friction controllable bearings has been presented, including the following aspects:

1. A hybrid isolation system using friction controllable sliding bearings has been proposed for controlling response of a structure subjected earthquakes ranging from small to large intensities.

This hybrid isolation system also has the following general advantage: the system requires smaller amounts of energy and power than the corresponding actuator-controlled system, and as a consequence, the use of accumulators for the source of energy is possible, thus eliminating the necessity of an emergency energy supply system.

2. Control algorithms, instantaneous optimal control and bang-bang control have been developed for controlling the friction force in the hybrid sliding isolation system. Standard control theory is difficult to apply in a straightforward fashion, in this case where the control force has a nonlinear feature.

3. A prototype hybrid sliding isolation system using friction controllable bearings has been physically developed, and shaking table experiments were performed on a rigid structural model equipped with such a hybrid system. A computer code has been developed for real-time on-line control implementation. The dynamic characteristics of the control system between bearing pressure and sliding friction have been identified. The results of hybrid sliding isolation experiments were compared with those of passive isolation.
4. Computer codes for the simulation of structural response under passive or hybrid control have been developed, and simulation analysis has been performed. The numerical results have been compared with experimental results.

Through the experimental and analytical study, the following conclusions have been obtained:

1. Significant advantage of the proposed hybrid sliding isolation system has been demonstrated: (1) for the small to medium earthquakes, the friction is controlled to make the structure slide easily to reduce the transfer of the seismic force to the structure to a minimum; (2) As the input earthquake becomes more intense, the friction is controlled to confine the sliding displacement of the structure to an acceptable range, while at the same time to keep the transfer of seismic force as small as possible. Such intelligent features of the friction controllable hybrid system does make the conventional passive sliding isolation system effective for all intensities of earthquakes.
2. Control algorithms developed for control of nonlinear friction force proved to be effective in achieving the desired control performances. In addition, they are practical and

easy for real-time on-line control operations.

3. The analytical model of dynamic characteristics between the bearing pressure control signal and the friction on the sliding interface has been identified. Computer simulation results excellently match the experiments. This implies that the analytical model represents the actual system very well, showing the possibility of utilizing the model to perform analytical study on other types of real structures equipped with the hybrid isolation system under different earthquake conditions.
4. The hybrid sliding isolation system appears to be quite robust, demonstrating the high potential for the application of the system to actual structures.

In the immediate future, possible implementation of the hybrid sliding isolation system to existing full-size structures such as buildings and girder bridges will be explored.

SECTION 7

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