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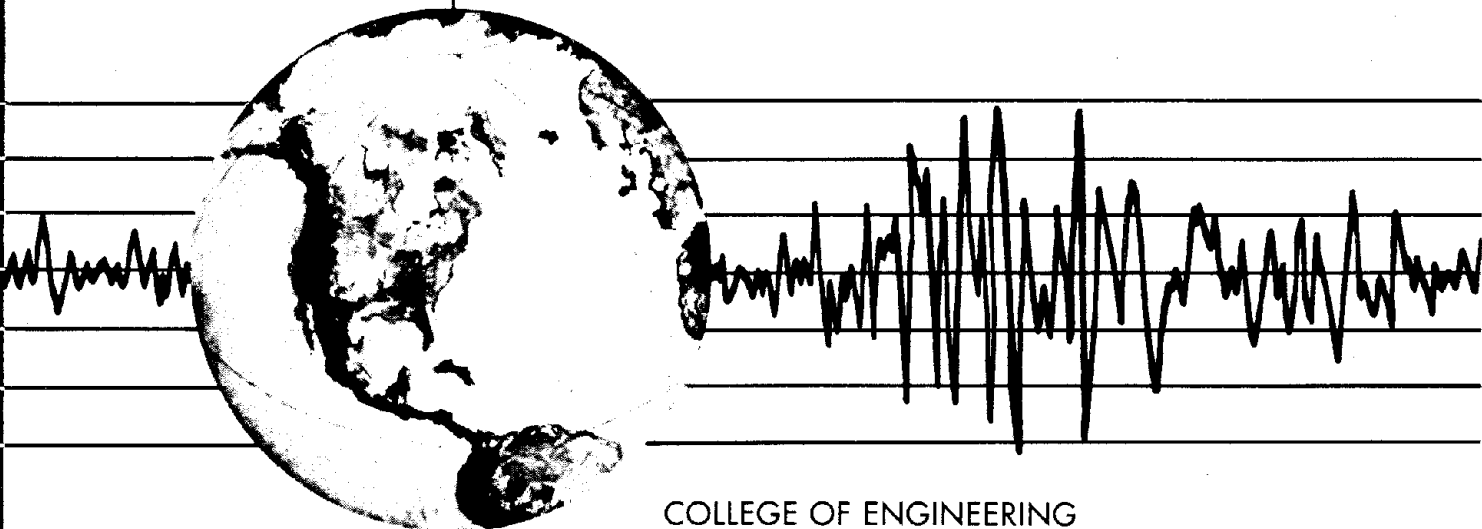
EARTHQUAKE ENGINEERING RESEARCH CENTER

# BEHAVIOR OF PRE-NORTHRIDGE MOMENT RESISTING STEEL CONNECTIONS

by

TZONG-SHUOH YANG  
EGOR P. POPOV

Report to Sponsors:  
National Science Foundation  
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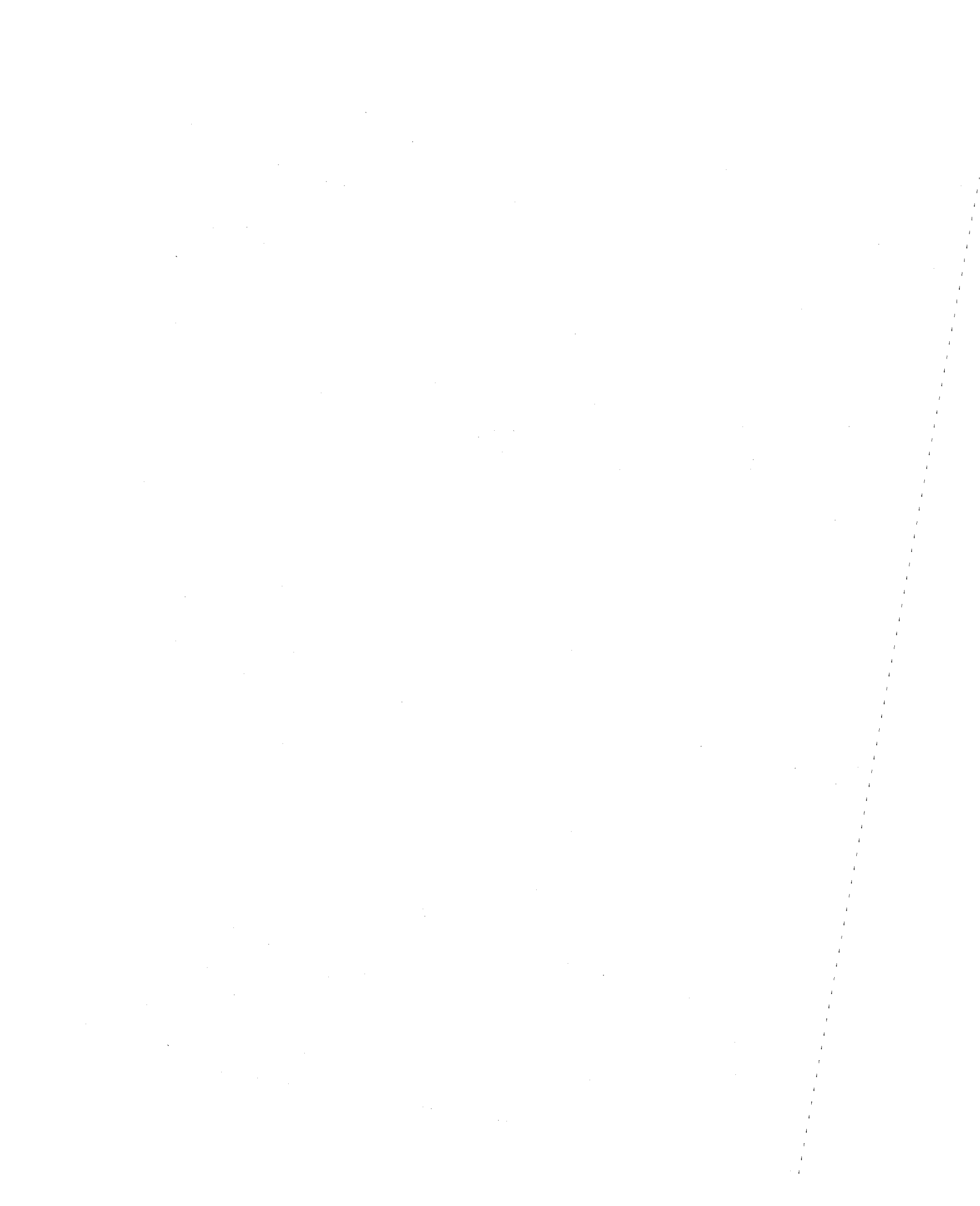


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## Abstract

The basic reasons for the fractures that occurred in steel moment-resisting connections during the 1994 Northridge earthquake are examined from a fundamental point of view. This examination begins with a discussion of material properties, and calls attention to the shortcomings of the conventional tension test. The stress-strain diagrams for specimens having a circular groove around the specimen (resembling a condition at the critical weld at a connection) are entirely different, exhibiting a brittle fracture compared with a ductile response for a bar of constant cross-section. The misleading ASTM requirement for minimum strength with no specified maximum results in a melee of actual strengths in use of which the designer is unaware. This prevailing condition makes it impossible to design rationally.

Next, the possible modes of failure are examined, showing the very limited view in the code design. Then, a simplified and more accurate analysis of the beam-column connection is examined.

Based on the above background, three SAC Pre-Northridge specimen tests subjected to cyclic loading are critically examined. Good comparisons are found using the above theory. However, the effect of the backing bars on the capacity of the connection need to be studied in more detail.

Recognizing that the unfused material between a column face and a backing bar forms an "artificial" edge crack, the methods of nonlinear finite element analysis combined with fracture mechanics were brought to bear. In the finite element analysis, the backing bar became one of the parts in a three-dimensional model of the connection. Using these procedures, it was possible to predict the instant of fracture, and to construct analytically complete hysteretic loops for the specimens. Remarkable agreement between these loops and experimental ones was achieved.

It is interesting that one of the two identical specimens fractured at a smaller tip load applied to the cantilever on a cold, murky day. This was predicted by the fracture mechanics theory, as the ambient temperature on that day was about 10°F lower than during the test with the other specimen.

The report concludes by clearly showing that, at higher applied loads, the bottom backing bar develops decidedly higher stresses at the column face than does the upper backing bar. These studies also indicate that, instead of removing the backing bars and applying a closure weld, a less expensive method of sealing the vertical "artificial" crack with a small weld may be almost equally effective.

## **Introductory Remarks**

Because of the urgency created by the January 17, 1994 Northridge earthquake, which caused numerous failures of steel-moment resisting connections in buildings, this report is released based on the as yet unpublished doctoral dissertation of Tzong-Shuoh Yang, prepared under the supervision of Egor P. Popov. It is the belief of the authors that this information should be made available to structural engineers and code formulating authorities at the earliest possible date.

## **Acknowledgements**

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The availability of data from the SAC experiments was necessary for some formulations. In this regard the cooperation of Stephen A. Mahin of SAC and UCB as well as of James O. Malley of SAC and Degenkolb Engineers is also greatly appreciated.

The kindness of Professor Albert P. Pisano of the Mechanical Engineering Department deserves special mention, as he provided access to his powerful computer facilities for the extensive analyses presented in this report.



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## Introduction

Before the 1994 Northridge earthquake, steel moment-resisting frames (MRFs) were considered ductile by engineers. The dream was broken suddenly after the earthquake. Many brittle failures were reported throughout the greater Los Angeles area [6, 30]. Most serious fractures occurred at the welded beam-to-column connections. This has called into question the strength and ductility of such connections.

This report presents the analytical studies of pre-Northridge welded beam-to-column connections used in typical steel MRFs. In the analysis, no defects in welding material, welding procedure, or workmanship are assumed. The purpose of these studies is to give explanations for both fracture locations and failure modes of the aforementioned connections in quantitative and rational ways. The stress concentration at the juncture of a welded beam flange and a column flange is analyzed by three-dimensional elastic-plastic finite elements based on the von Mises yield criterion with associated plastic flow. The results clearly explain that the weak beam flange breaks off right at the weld due to triaxial state of stress in this region. The important effect of the weak column panel zone was not fully explored before. In this report, it is shown that the column web fractures are closely related to the weak panel zone. The important effect of the backing bar in the connection failure is analyzed next by fracture mechanics methods. The unfused backing bar side next to the column flange is interpreted as an artificial crack. Flange tension due to bending of the beam opens the artificial crack between the backing bar and the column flange, and initiates the rupture. The stress-intensity factors at the artificial crack tips of both top and bottom backing bars are calculated by the J-integral method. The results clarify why the rupture generally was initiated at the bottom flange but not at the top flange. Finally, the analytical cyclic load-deflection curve and plastic energy dissipation are compared with the three SAC<sup>1</sup> Joint Venture full-size specimens tested at the University of California at Berkeley. Good agreement between the analytical results and the experimental tests conclude the report.

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<sup>1</sup>SAC is an acronym for Structural Engineers Association of California, Applied Technology Council, and California Universities for Research in Earthquake Engineering.

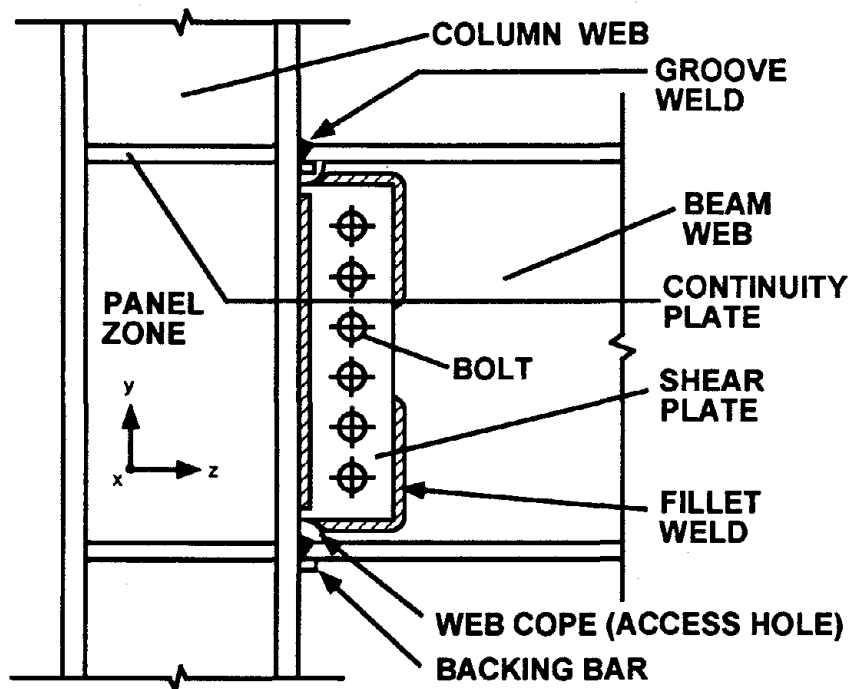
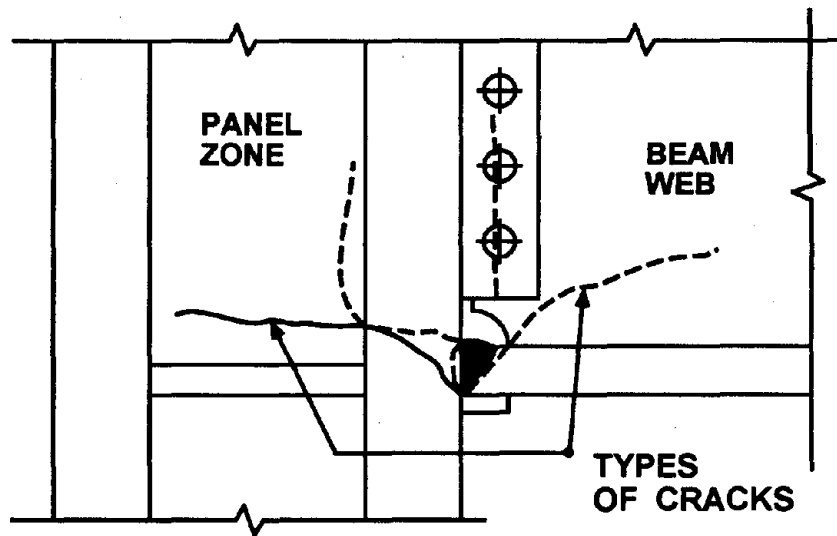


Fig. 1: A typical welded beam-to-column connection.

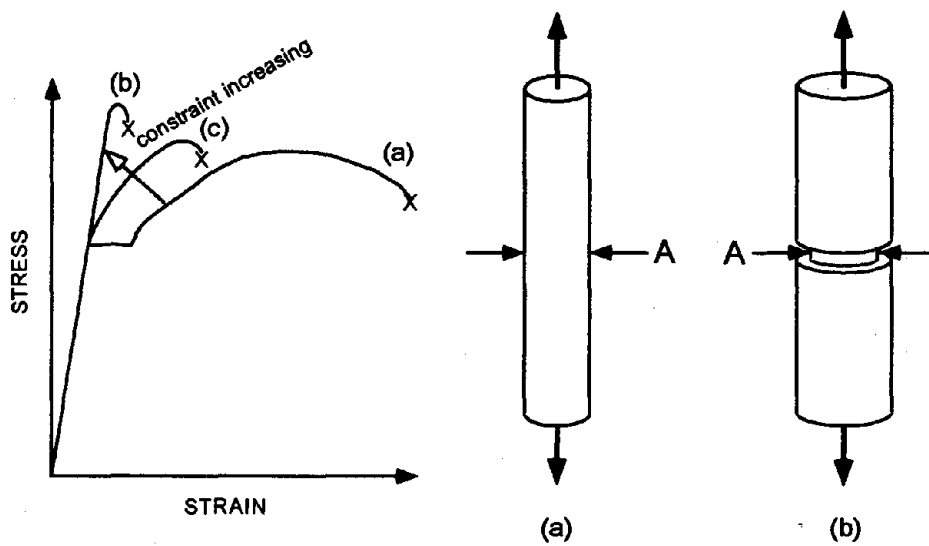
## Material Properties of Structural Steel

A typical welded beam-to-column moment resisting connection is shown in Fig. 1. The top and bottom flanges of the beam are welded directly to the column by full penetration groove welds. The beam web is bolted or welded to a shear plate, which is attached to the column by welding. The most serious rupture modes of such connections are shown in Fig. 2. The failure modes are catastrophic because they fracture at extremely high speeds without exhibiting prior ductile behavior. This violates the precept of the ductile MRF.

Before studying the non-ductile failure of the connection, some remarks on the material properties need to be made. The stress-strain curve of a small diameter uniform cylindrical steel (bar (a) in Fig. 3) loaded longitudinally to failure, will be ductile (curve (a) in Fig. 3). A small diameter bar of uniform cross-section is not restrained in the lateral direction, and allows Poisson contraction, which leads to specimen necking down and develops shear slip layers (Lueders lines) during failure. However, for a cylindrical bar with a groove or notch, such as bar (b) in Fig. 3, even though the cross sectional area at the groove is the same as bar (a), the tensile stress-strain curve is completely different. When loaded in tension, the grooved part will



**Fig. 2:** Some failure modes of the welded beam-to-column connection.



**Fig. 3:** Simple tensile test of steel specimens with the same critical cross section area: (a) cylindrical bar, and (b) grooved cylindrical bar.

have the largest stress, but due to the constraint of the larger sections outside the groove, no lateral contraction or shear flow can develop at the groove. The failure of bar (b) is caused by triaxial tension resulting in a brittle failure with no apparent yielding. The stress at breaking is near the cohesion strength of the material; its stress-strain curve is similar to curve (b) in Fig. 3. Timoshenko on page 435 of his book says [28]:

Because most of the grooved specimen remains elastic during a tensile test to failure, it will have a very small elongation, and hence only a small amount of work is required to produce fracture. A small impact force can easily supply the work required for failure. The specimen is brittle because of its shape not because of any mechanical property of the material.

There were many research studies on grooved specimens, i.e. Kirkaldy [16], Ludwik and Scheu [17], and MacGregor [18]. Stress concentration factors for a variety of grooved bars can be found, for example, in Neuber [19] and Peterson [21].

It can be seen in Fig. 1 that the welded beam flange cannot be deformed in both x and y directions because it is welded to a relatively large column flange with continuity plates. The welded beam flange also has to resist the largest bending moment caused by the loads on its span and frame drift. The stress on the beam flange outside the weld is smaller because of reduced moment and lesser lateral strain constraint. The welded beam-to-column connection has the strain constraint and the largest stress, which make it essentially like a grooved bar except of different shape. It can be expected that the tensile stress-strain curve right at the beam flange weld will be between curves (a) and (b) in Fig. 3 depending on the degree of constraint, web cope (access hole) size, and bending moment gradient.

Another important fact regarding the mechanical properties of today's steel is that the yield strength of A36 steel is no longer 36 ksi. Fig. 4 shows a series of tested stress-strain curves of coupons cut from A36 W12×26 beams. The average yield strength of A36 steel is about 48 ksi, the ultimate strength of A36 steel is about 70 ksi.

Table 1 lists *minimum* yield strength,  $F_y$ , and minimum ultimate strength,  $F_u$ , for certain ASTM steels given in the AISC specifications. The ANSI-ASTM standard B483-78 defines minimum strength as follows:

Standard mechanical property limits for the respective size ranges are based on an analysis of data from standard production material and are



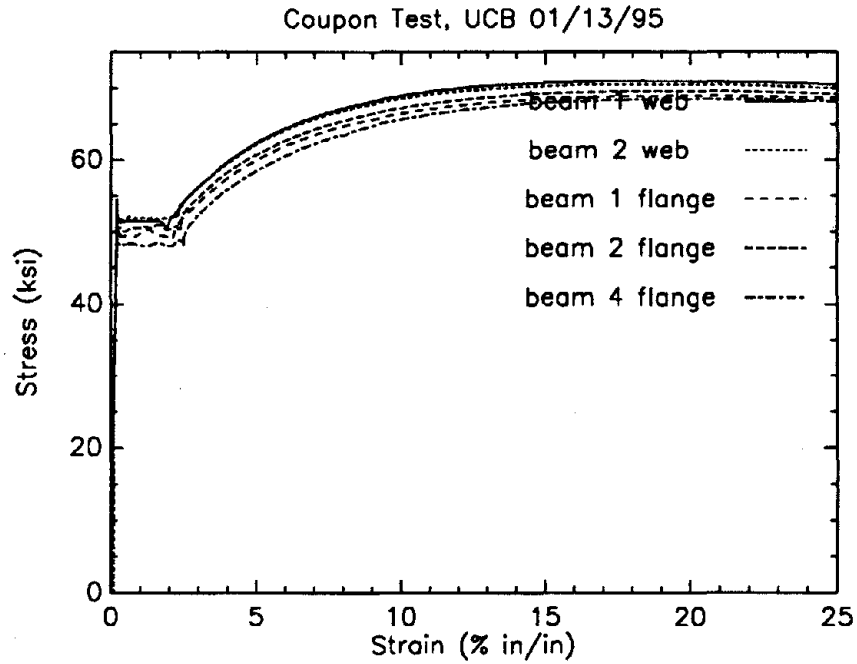


Fig. 4: Stress-strain curves of a series of tensile tests for A36 steel.

ASTM Number	A36	A592	A572 Gr50	A588	A514
Steel Type	Carbon	Carbon	Low alloy	Stainless	Alloy Q&T†
Minimum $F_y$ (ksi)	36	42	50	50	100
Minimum $F_u$ (ksi)	58	60	65	70	110

†Q&T: Quenching and Tempering

Table 1: Specified minimum strengths of certain ASTM steels.

established at a level [at] which at least 99 percent of the production of values obtained from all standard material in the size range meets the established value.

The unqualified word *minimum* may be misleading, since there is a chance that the materials involved may have a strength much higher than the minimum. In the statement, there is no specification for the upper bound of steel strength. For structural elements loaded to the allowable stress level, the statement is adequate for design. However, in seismic resisting structures, many elements can be stressed beyond the yield strength during a strong earthquake to develop the needed ductile behavior. The ASTM specifications provide no information for designers to control deformation because the material strength can be much higher than the minimum value. The A36 materials tested in the laboratory have an average yield strength 33% over the specified minimum value (Fig. 4). Such high variance on strength makes stress analysis meaningless.

The AISC specification states that:

Certified mill test reports or certified reports of tests made by the fabricator or a testing laboratory in accordance with ASTM A6 or A568, as applicable, and the governing specification shall constitute sufficient evidence of conformity with one of the ASTM standards. Additionally, the fabricator shall, if requested, provide an affidavit stating that the structural steel furnished meets the requirements of the grade specified.

The AISC statement ensures that materials having these minimum strengths are actually used in the construction, but the designers have no way to know the actual material strengths during design until the materials reach the fabrication stage.

A more appropriate material specification would be to give the average material strength with a specified small variance. Designers can use the average as the nominal design value and use the variance for reliability analysis.

## **Design Strategy**

The most essential characteristic of MRF is the requirement that plastic hinges be formed near connections during severe loading conditions. These plastic hinges provide strength and ductility to dissipate energy hysteretically. As was stated earlier, it is impossible to develop large plastic deformation right at the beam-column weld

Failure Type	Minimum Capacity	Failure Mode
1	$C_m^b$ or $C_v^b$	beam flange or shear plate rupture (crack 3 in Fig. 6)
2	$C_m^c$ or $C_v^c$	fracture through column web or divot pullout from column flange (crack 1 or 2 in Fig. 6)
3	$C_{cr}^b$	buckling of beam near connection and formation of plastic hinge (weak beam-strong column connection)
4	$C_{cr}^c$	buckling of column near connection and formation of plastic hinge (strong beam-weak column connection)

**Table 2:** Four possible failure types of a steel MRF connection.

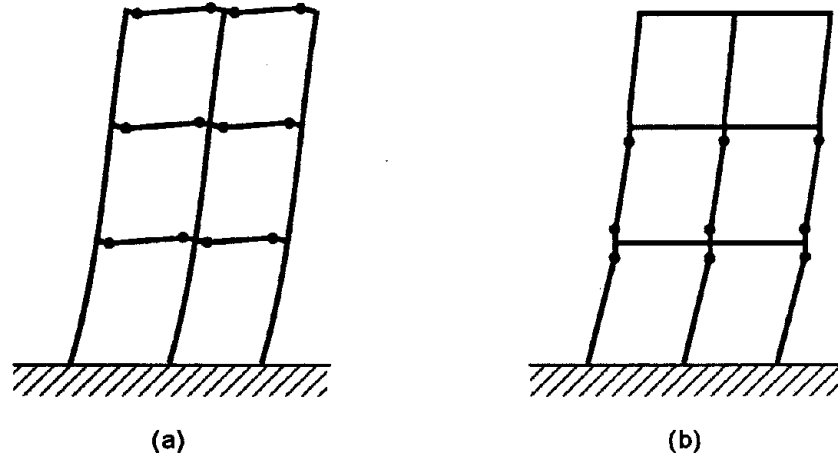
location. Thus, the plastic hinges can only be formed at the beam or column section near the connection.

In order to establish the design strategy, let the resisting capacity at the connection be defined by the following symbols:

- $C_m^b$  = Moment resisting capacity of beam at the connection
- $C_v^b$  = Shear resisting capacity of beam at the connection
- $C_m^c$  = Moment resisting capacity of column at the connection
- $C_v^c$  = Shear resisting capacity of column in the panel zone
- $C_{cr}^b$  = Local buckling strength of beam near the connection
- $C_{cr}^c$  = Local buckling strength of column near the connection

The superscripts *b* and *c* are used to designate *beam* and *column*, respectively. The actual resisting capacity of a connection is controlled by the minimum of these six values<sup>2</sup>. The minimum resisting capacity is based on the failure type. The four possible failure types are given in Table 2. In this table, Types 1 and 2 correspond to sudden fractures and should be avoided. By developing plastic hinges near the connections, Type 3 and 4 mechanisms assure good strength and ductility (Fig. 5). For beam flange connections welded directly to the column flange, the resisting capacity  $C_m^b$  is always smaller than the ultimate moment capacity of the beam. In such a situation, in order to develop a Type 3 mechanism, a non-compact beam section must be

<sup>2</sup>The resisting capacity is also affected by backing bars at the welds. This will be discussed later.



**Fig. 5:** Two alternative plastic hinge mechanisms for a typical MRF: (a) Type 3 mechanism and (b) Type 4 mechanism.

used. The derivations for  $C_m^b$  and  $C_{cr}^b$  are given in the next section. It is to be noted that the kinematic mechanisms shown in Fig. 5 are associated with early inelastic action and do not represent collapse mechanisms.

## Simplified Stress Analysis

In the linear elastic range, the stress in the  $z$  direction at the outer-most fiber point A, shown in Fig. 6 near the weld to the connection, is denoted as  $\sigma_{zz}^A$ . The nominal value of  $\sigma_{zz}^A$  can be calculated from

$$\sigma_{zz}^A = \frac{M}{S_x^b} \quad (1)$$

where  $M$  is the applied bending moment, and  $S_x^b$  is the section modulus of the beam. The corresponding strain is expressed as

$$\epsilon_{zz}^A = \sigma_{zz}^A / E \quad (2)$$

where  $E$  is the Young's modulus of the beam.

Point A is not constrained by the weldment. On the other hand, point B is also at the outer-most fiber of the beam at the center of the beam-column junction. As shown in Fig. 6, point B is restrained by the weldment, which is directly attached to a wide thick column flange and cannot displace in either the  $x$  or  $y$  direction, hence  $\epsilon_{xx}^B = \epsilon_{yy}^B = 0$ . The stress state of point B is in the transition zone from plane stress

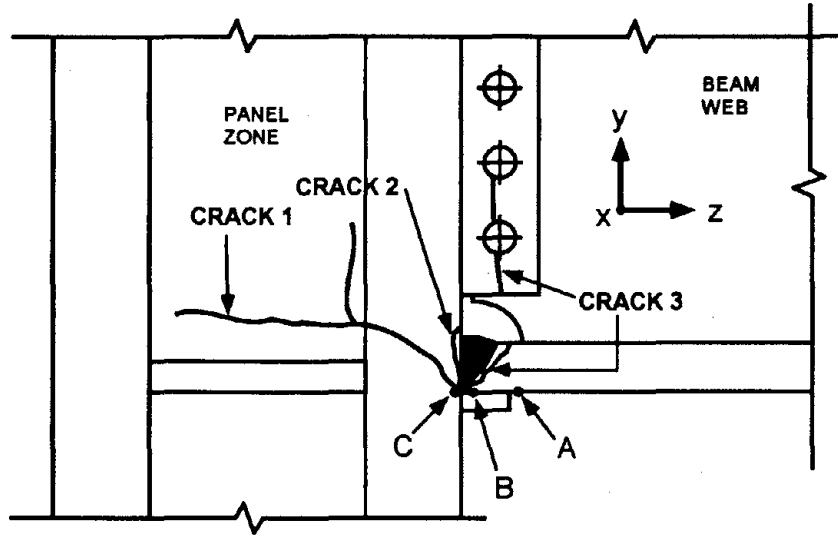


Fig. 6: Critical points in the connection - point A on beam flange, point B on beam-weldment junction, and C at column flange.

to plane strain. Because point B is so close to point A, its strain  $\epsilon_{zz}^B$  in direction z can be assumed to be equal to  $\epsilon_{zz}^A$ , and the stresses at point B can then be determined by Hooke's law:

$$\sigma_{zz}^B = \frac{(1-\nu)E}{(1+\nu)(1-2\nu)} \epsilon_{zz}^B = \frac{(1-\nu)E}{(1+\nu)(1-2\nu)} \epsilon_{zz}^A = \frac{(1-\nu)}{(1+\nu)(1-2\nu)} \sigma_{zz}^A \quad (3)$$

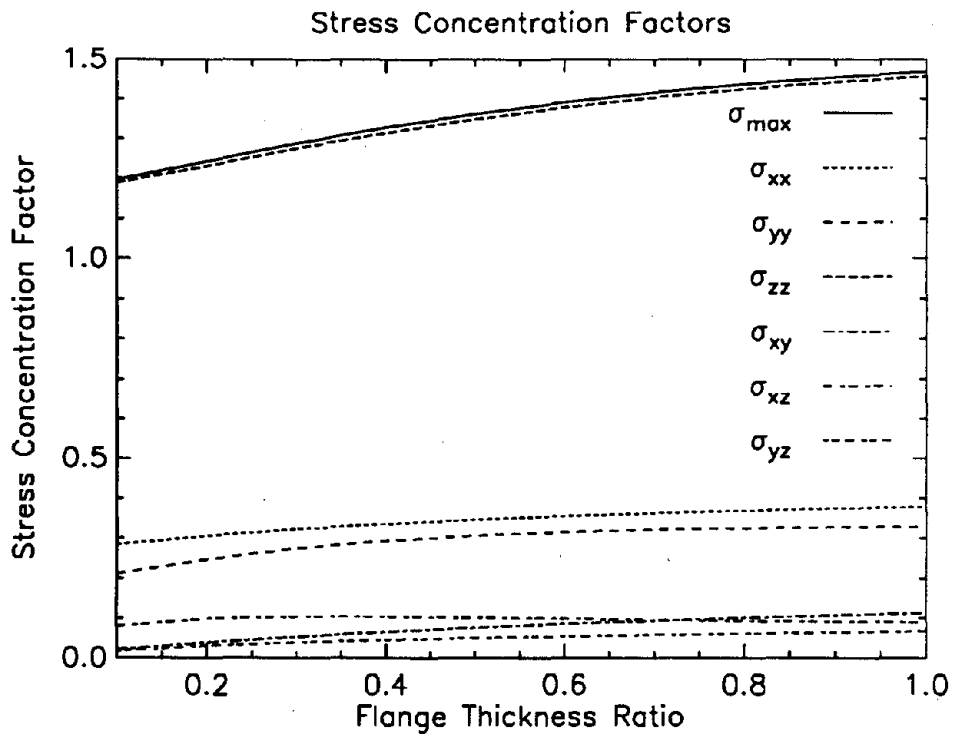
$$\sigma_{xx}^B, \sigma_{yy}^B = \frac{\nu E}{(1+\nu)(1-2\nu)} \epsilon_{zz}^B = \frac{\nu E}{(1+\nu)(1-2\nu)} \epsilon_{zz}^A = \frac{\nu}{(1+\nu)(1-2\nu)} \sigma_{zz}^A \quad (4)$$

where  $\nu$  is Poisson's ratio. Since for steel,  $\nu = 0.3$  and  $E = 29,000$  ksi, approximately, the above equations reduce to

$$\sigma_{zz}^B = 1.35 \sigma_{zz}^A \quad (5)$$

$$\sigma_{xx}^B, \sigma_{yy}^B = 0.58 \sigma_{zz}^A \quad (6)$$

To verify the adequacy of the estimated stress concentration factors derived above, a series of elastic finite element analyses was performed based on the geometry of the SAC specimen (see next section) by varying the column flange thickness while keeping the beam flange thickness constant. The results are shown in Fig. 7. The stress concentration factor of each component of stress is plotted against the beam/column flange thickness ratio. The stress concentration factors of  $\sigma_{zz}$  range from 1.2 to 1.46, which is very close to the calculated simplified value of 1.35, but the stress concentration factors for  $\sigma_{xx}$  and  $\sigma_{yy}$  are much less than the estimated value 0.58. The lower



**Fig. 7:** Stress concentration factors at juncture of beam-to-column connection calculated by elastic finite element analysis. The external load is uniformly distributed unit tensile stress  $\sigma_{zz}$  applied on the beam flange.

three curves in the figure show how small the flange shear stresses are at the juncture. Because of the low shear stresses, no shear slip can form, resulting in no ductility. For the same reason, the beam fracture is governed by the maximum-normal-stress criterion. The maximum-normal-stress criterion states that failure occurs whenever one of the three principal stresses equals the uniaxial material strength. The maximum principal tension stress  $\sigma_{max}$  of the beam flange at the juncture of the beam-column connection is plotted in Fig. 7, which is very close to the  $\sigma_{zz}$  curve.

In the inelastic range, the stress across the flange will re-distribute to become much more evenly distributed, although the greatest stress value remains at the flange center. All rolled steel members (W, M, C, etc.) possess residual stresses due to differential cooling. The flange tips and interior web parts always cool more quickly than the other parts of the flange. In this manner, the flange tips develop compressive stresses, while the residual stresses are tensile in the middle of the flange.

The yielding moment of a member can be calculated with a sufficient degree of accuracy by the following equation

$$M_{yield}^b = (F_y - F_r)S_x^b \quad (7)$$

where  $F_r$  is the maximum compressive residual stress in either flange tip of the beam. The average compressive residual stress at the flange tips of small to medium size rolled shapes is about 13 ksi for A36 steel with 36 ksi yield strength [5, 10]. It is reasonable therefore to assume that the value of the residual stress is a fraction of the yield strength

$$F_r = \frac{13}{36}F_y = 0.36F_y \quad (8)$$

The residual stresses for large hot-rolled sections can be found in Alpsten [1]. The plastic moment capacity of a rolled section is hardly affected by the presence of residual stresses and can be calculated simply as

$$M_{plastic}^b = F_y Z_x^b \quad (9)$$

where  $Z_x^b$  is the plastic modulus of the beam. The ultimate moment capacity of a beam can be reached by bending a plastic section into the strain-hardening range. Hence the ultimate bending capacity is expressed as

$$M_{ultimate}^b = F_u Z_x^b \quad (10)$$

For a directly welded beam-to-column connection without cover plate, the bending beam moment is transferred primarily through the beam flanges into the column

regardless of the size of the shear plate. The ultimate moment capacity of the connection therefore can be calculated by

$$C_m^b = F_u Z_f^b \quad (11)$$

where  $Z_f^b$  is the plastic modulus of the beam flanges. Because  $Z_x^b$  is larger than  $Z_f^b$ , the ultimate moment capacity of the beam  $M_{ultimate}^b$  is always larger than the connection moment capacity  $C_m^b$ . If the rolled shape section is compact, no local buckling will occur before  $M_{ultimate}^b$  is reached, and the rapid failure occurs by tearing off the flange.

The local buckling stress of a beam flange can be derived from the plate buckling stress. In general, the plate buckling stress in the elastic range is given by

$$\sigma_{cr} = k \frac{\pi^2 E}{12(1 - \nu^2)} \left( \frac{t}{b} \right)^2 \quad (12)$$

where  $t$  and  $b$  are the thickness and the width of the plate, respectively.  $k$  is the plate buckling coefficient, which depends on the plate geometry and boundary conditions. For the beam flange of a wide-flange rolled shape,  $k$  is 0.7. The plate buckling stress in the inelastic range can be shown to be [8]

$$\sigma_{cr} = k \frac{\pi^2 \sqrt{E E_t}}{12(1 - \nu^2)} \left( \frac{t}{b} \right)^2 \quad (13)$$

where  $E_t$  is the tangent modulus of the material. From the buckling stress, the critical beam moment capacity  $C_{cr}^b$  can be calculated from

$$C_{cr}^b = \sigma_{cr} Z_f^b \quad (14)$$

To safely design a beam-column connection, it is desirable that

$$C_{cr}^b < C_m^b < M_{ultimate}^b \quad (15)$$

But for economic use of the material, the critical stress should obey the following relation

$$F_y \leq \sigma_{cr} < F_u \quad (16)$$

## Three SAC Pre-Northridge Specimen Tests

In order to help understand the strength and ductility of the welded moment resisting connections, three specimens have been fabricated according to the standards used



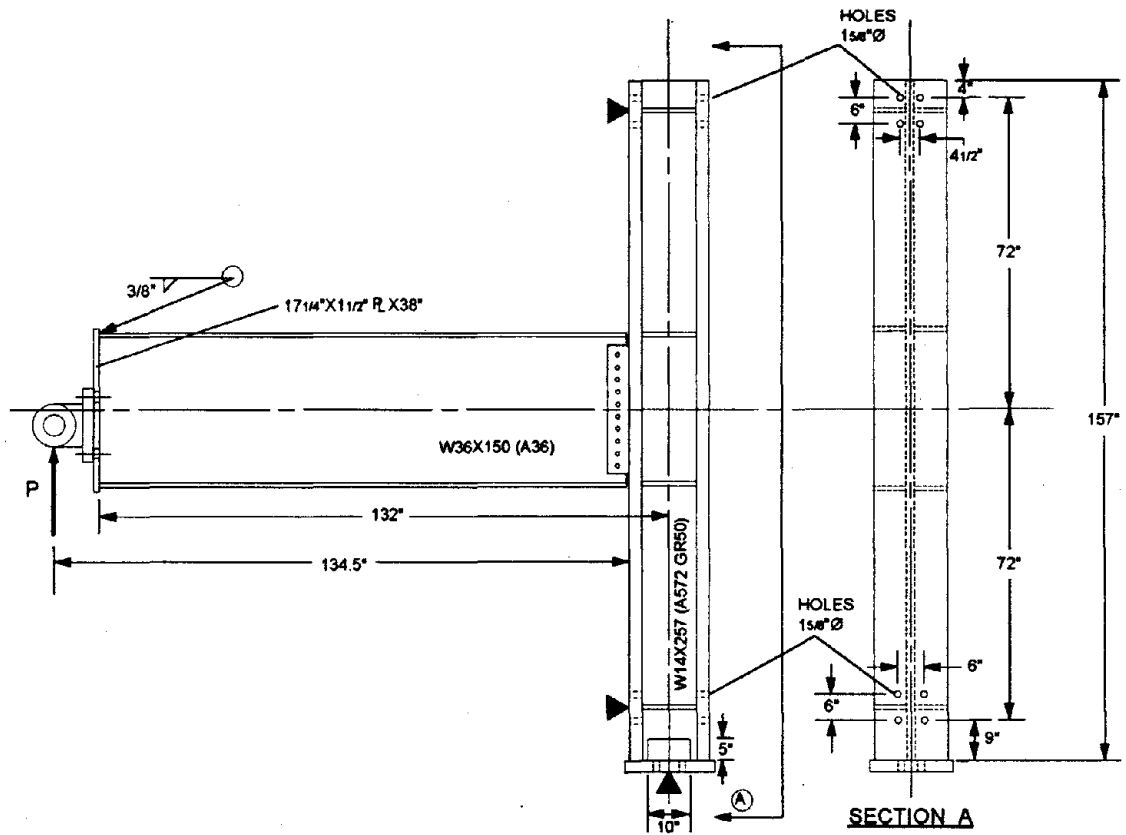


Fig. 8: Detail of the SAC PN specimens.

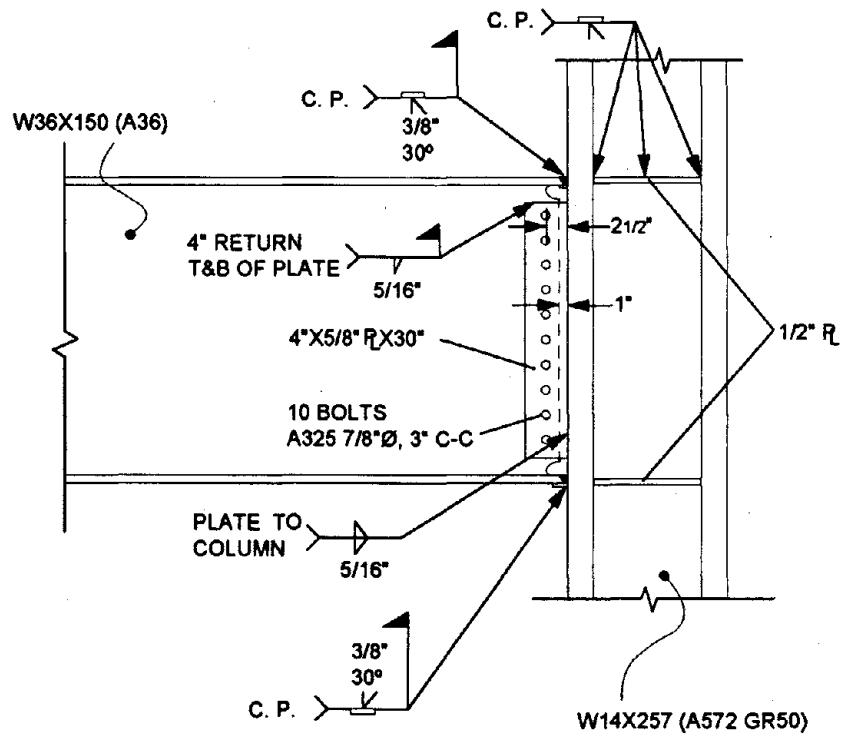


Fig. 9: Connection detail for SAC PN specimens.

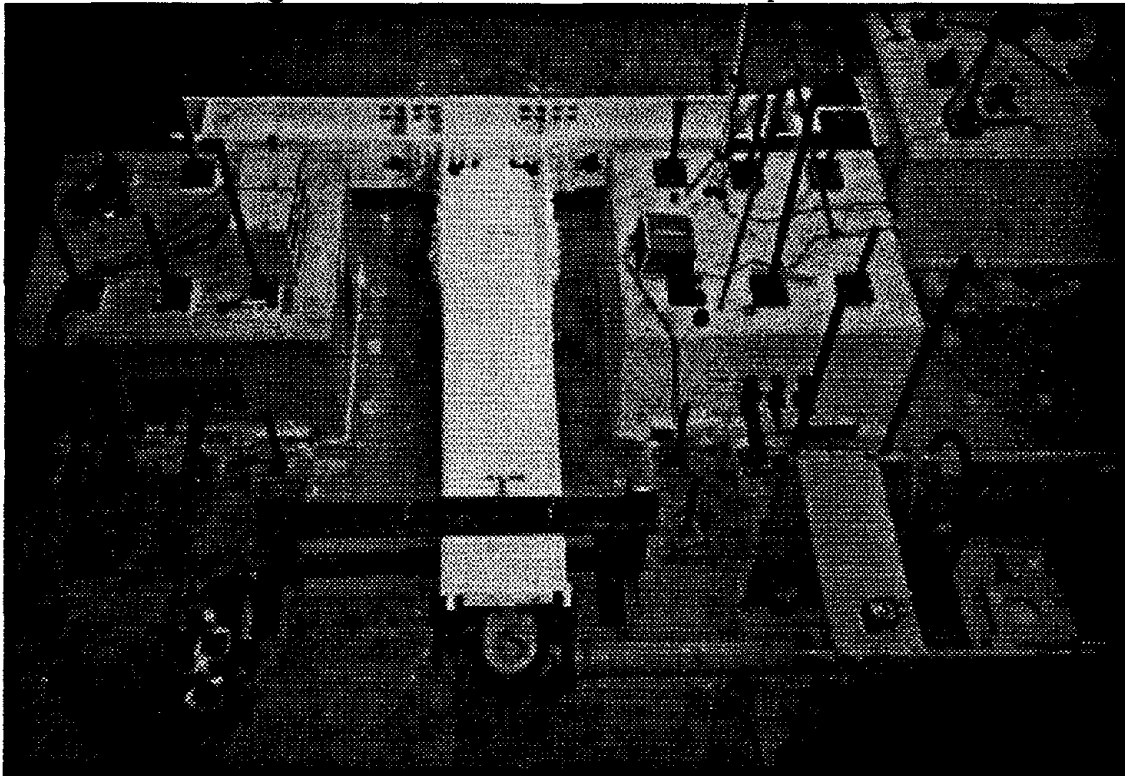
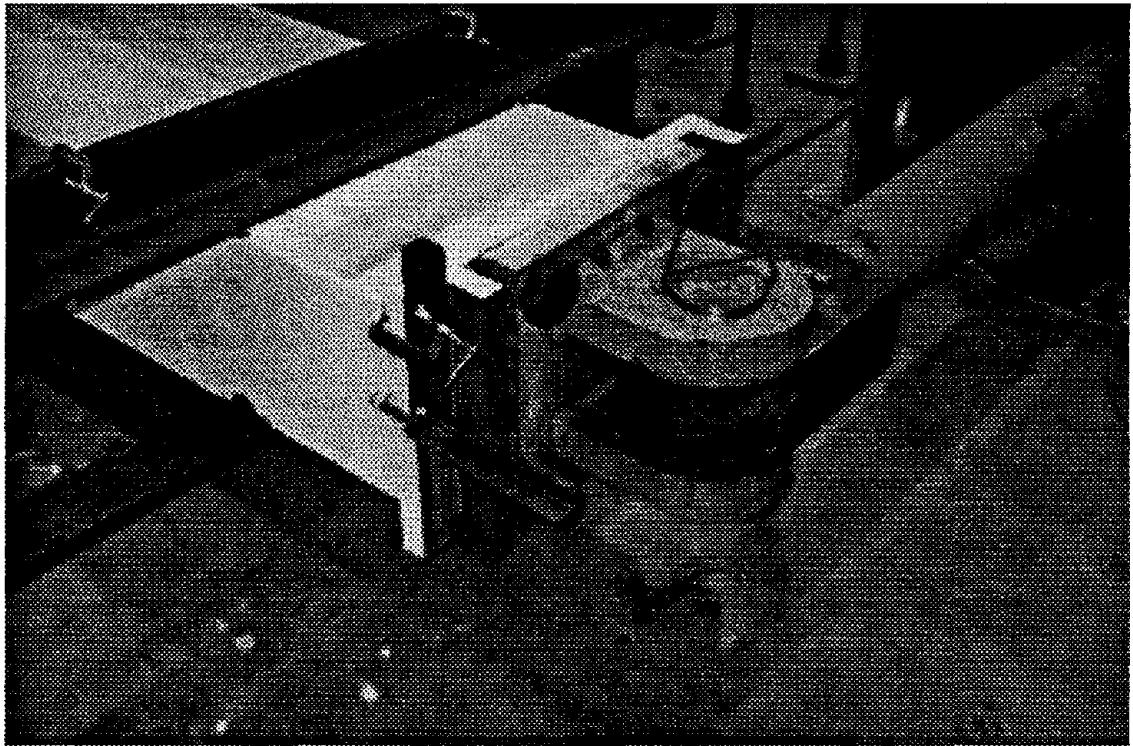
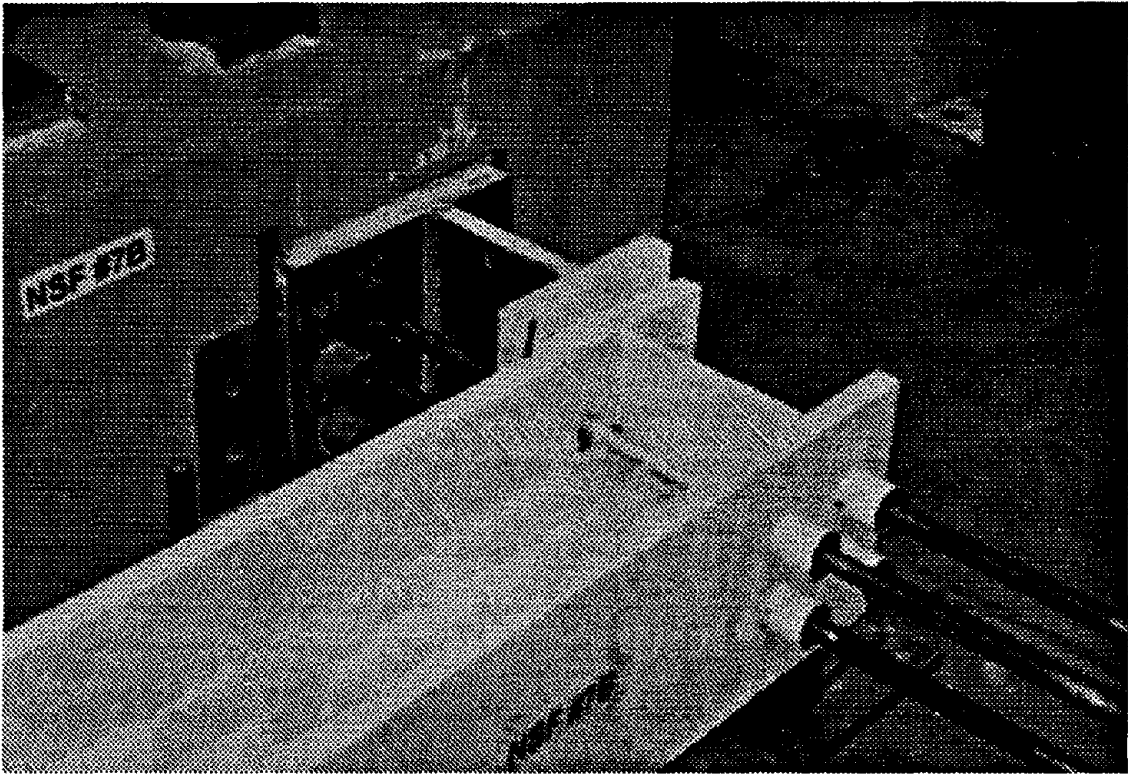


Fig. 10: Photograph showing specimen tested horizontally in laboratory. Left side of the beam is the *upper* side of the specimen.

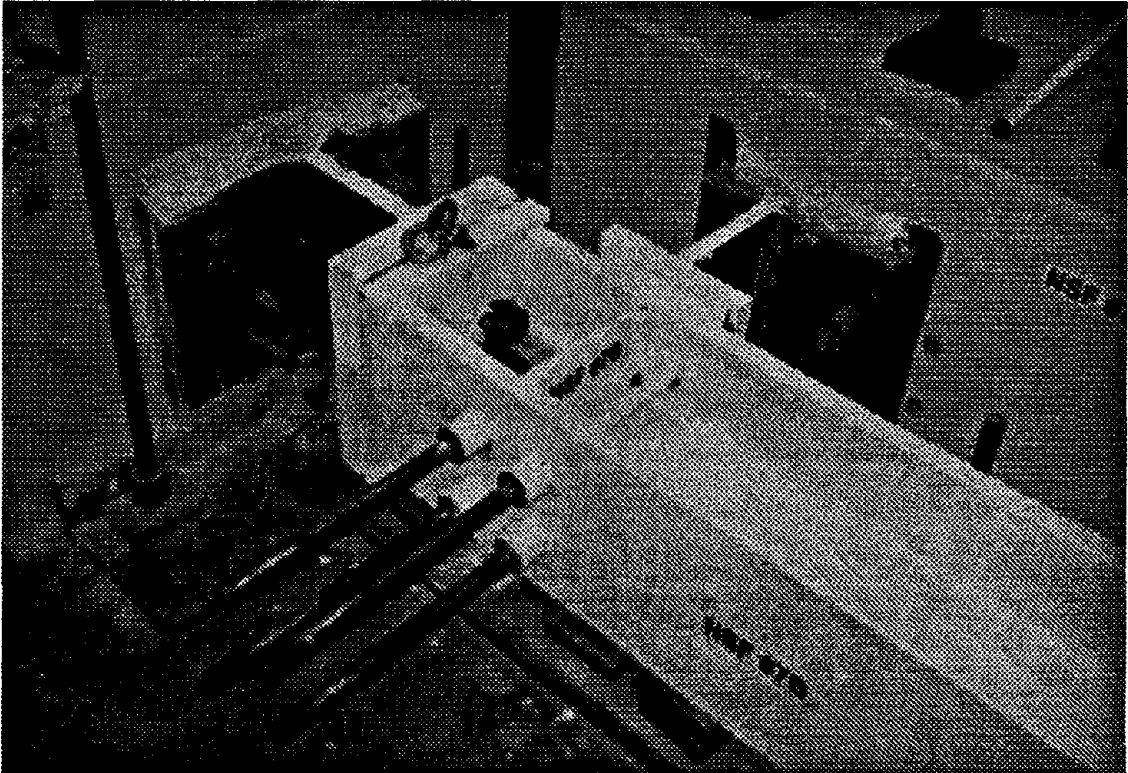


**Fig. 11:** Photograph showing the clevis connected to beam end-plate and hydraulic actuator.

before the 1994 Northridge earthquake. These were tested at UC Berkeley under the guidance of the SAC Joint Venture. The dimensions of these specimens are shown in Fig. 8. The connection detail is shown in Fig. 9. These specimens were tested in a horizontal position. The test setup is shown in Fig. 10. An end-plate is welded to the cantilever beam and bolted to a clevis, which connects to a computer-controlled 350 kips hydraulic actuator (see Fig. 11). Slowly varying cyclic loads are applied to the beam tip by the actuator. The column is simply supported at the ends by prestressed rods tightened to three reinforced concrete blocks. These reaction blocks are prestressed to the floor by high-strength rods. The reaction and loading points simulate the inflection points occurring in mid span of a frame. At the top of the column, in order to simulate a roller support at the end, the column is tightened to a reaction block by four prestressing rods (Fig. 12). At the bottom of the column, in order to simulate a hinged end, the column is tightened to the reaction blocks by prestressed rods in two directions (Fig. 13). Detailed information of the setup can be found in the thesis by Blackman [7]. During the fabrication of the specimens, two A572-Gr50 beams were mistakenly used in the first two specimens PN1 and



**Fig. 12:** Photograph showing the top of the column.



**Fig. 13:** Photograph showing the bottom of the column.

**Material Properties**  
(from Mill Certificates)

Specimen Number	Material Size & Spec	Yield Strength $F_y$ (Elongation)	Ultimate Strength $F_u$ (Elongation)
PN1, PN2 & PN3 Column	W14×257 A572-Gr50	53.5 ksi (0.18%)	72.5 ksi (N/A)
PN1 & PN2 Beam	W36×150 A572-Gr50	62.6 ksi (0.22%)	74.7 ksi (22.5%)
PN3 Beam	W36×150 A36	56.8 ksi (0.20%)	68.7 ksi (28.0%)

**Table 3:** Material properties of the SAC Joint Venture PN specimens.

PN2. Only the third specimen PN3 was made of the correct materials. The material properties of these three specimens as determined from mill certificates, and are given in Table 3. Notice that the yield strength of A572 Grade 50 PN1 and PN2 beam is 25% over the ASTM minimum, the yield strength of the A36 PN3 beam is 58% over the ASTM minimum. The high scatter in material strengths may result in inadequate structures.

Each SAC specimen has a cantilever beam with a concentrated force  $P$  applied at its free end (see Fig. 8). The arm length  $L$  of the concentrated force to the face of the column flange is 134.5 inches. The moment capacities of these specimens can be calculated by Eqs. 7-11. The corresponding concentrated forces can also be calculated by dividing the moment by the moment arm. For PN1 and PN2 specimens, the yield force can be calculated as

$$P_{yield} = M_{yield}^b / L = S_x^b (F_y - F_r) / L = 504(62.6 - 0.36 \times 62.6) / 134.5 = 150 \text{ kips} \quad (17)$$

and the peak force can be calculated as

$$P_{peak} = C_m^b / L = Z_f F_u / L = 392.96 \times 74.7 / 134.5 = 218 \text{ kips} \quad (18)$$

Similarly, the yield and peak forces of the PN3 specimen can be calculated as

$$P_{yield} = S_x^b (F_y - F_r) / L = 504(56.8 - 0.36 \times 56.8) / 134.5 = 136 \text{ kips} \quad (19)$$

$$P_{peak} = C_m^b / L = Z_f F_u / L = 392.96 \times 68.7 / 134.5 = 201 \text{ kips} \quad (20)$$

The shear force in the column below and above the connection due to tip load  $P$  can be calculated as

$$V^c = P(L + \frac{d^c}{2})/138 = P(134.5 + 16.38/2)/138 = 1.03P \quad (21)$$

where  $d^c$  is the depth of the column (refer to Fig. 8). The axial force at the beam flange can be expressed as

$$T^b = \frac{PL}{d^b - t_f^b} = \frac{134.5P}{35.85 - 0.94} = 3.85P \quad (22)$$

where  $d^b$  is the depth of the beam and  $t_f^b$  is the flange thickness of the beam. Thus the shear force in the panel zone is

$$V^{pz} = T^b - V^c = 2.82P \quad (23)$$

and the shear capacity of the column is

$$C_v^c = 0.95d^c t_w^c \frac{F_y^c}{\sqrt{3}} = 0.95 \times 16.38 \times 1.175 \times \frac{53.5}{\sqrt{3}} = 565 \text{ kips} \quad (24)$$

where 0.95 is the effective shear area coefficient,  $t_w^c$  is the web thickness of the column, and  $1/\sqrt{3}$  comes from the von Mises yield criterion. As long as the tip load is over 200 kips, the column will have shear yielding in the panel zone.

The maximum column bending moment occurs at the section outside the panel zone. Its value is

$$M^c = V^c(69 - d^b/2) = 52.8P \quad (25)$$

The moment capacity of the column is

$$C_m^c = F_u^c Z_x^c = 72.5 \times 542 = 39,300 \text{ kip-in} \quad (26)$$

The column is very safe in bending. The shear plate of the connection is very well designed, and its strength is enough to resist the applied load.

The test results of these specimens are shown in Table 4. These values agree well with the calculated values. The moment-rotation and moment ratio-plastic rotation diagrams for three SAC specimens are given in Fig. 14.

Since the beams of three SAC specimens have compact sections and strong material, the failure modes are of a rapid fracture type. The crack in the PN1 specimen initiated at the center of the bottom beam flange-column juncture. The crack rapidly propagated through the column flange and forked out into two cracks in the column

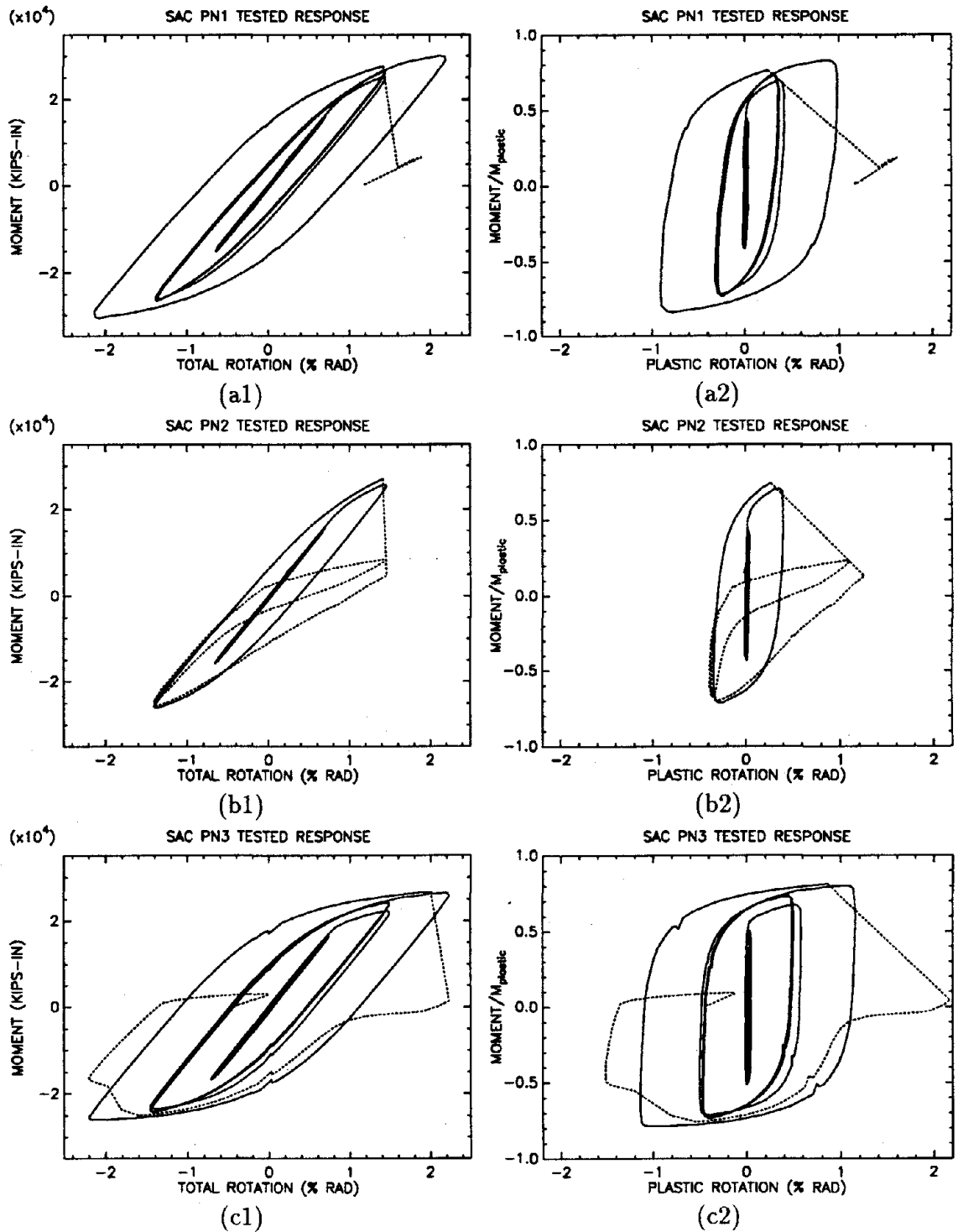


Fig. 14: Moment-rotation and moment ratio-plastic rotation diagrams for SAC PN1, PN2 and PN3 specimens.

Specimen Number	Load (kips)	Displacement Total (inches)	Displacement Beam (inches)	Post-yield cycles	Date & Temperature
PN1 - $P_{yield}$	154	1.31	1.15	$4\frac{1}{4}$	02/09/95
$P_{peak}$	225	2.91	2.63		60° F
PN2 - $P_{yield}$	153	1.34	1.11	$1\frac{1}{4}$	02/16/95
$P_{peak}$	201	1.94	1.71		50° F
PN3 - $P_{yield}$	138	1.12	1.02	$4\frac{1}{4}$	02/28/95
$P_{peak}$	199	3.02	2.88		60° F

Table 4: Test results of the SAC Joint Venture PN specimens.

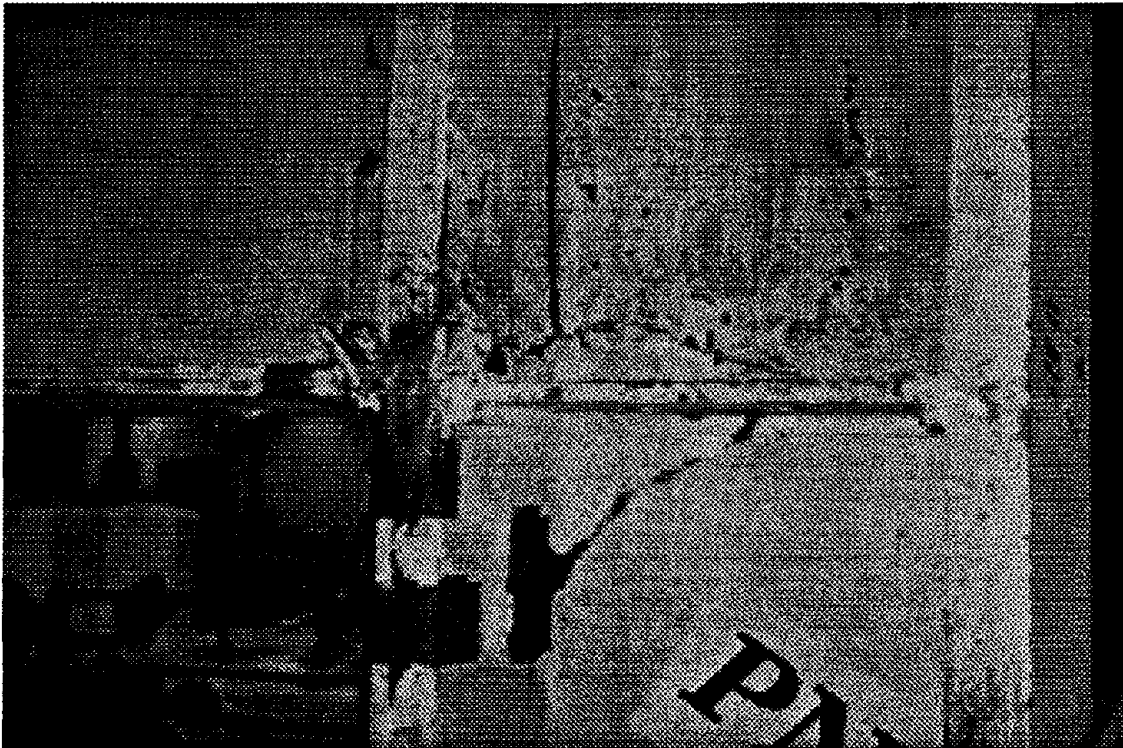


Fig. 15: Photograph of SAC specimen PN1 after test.



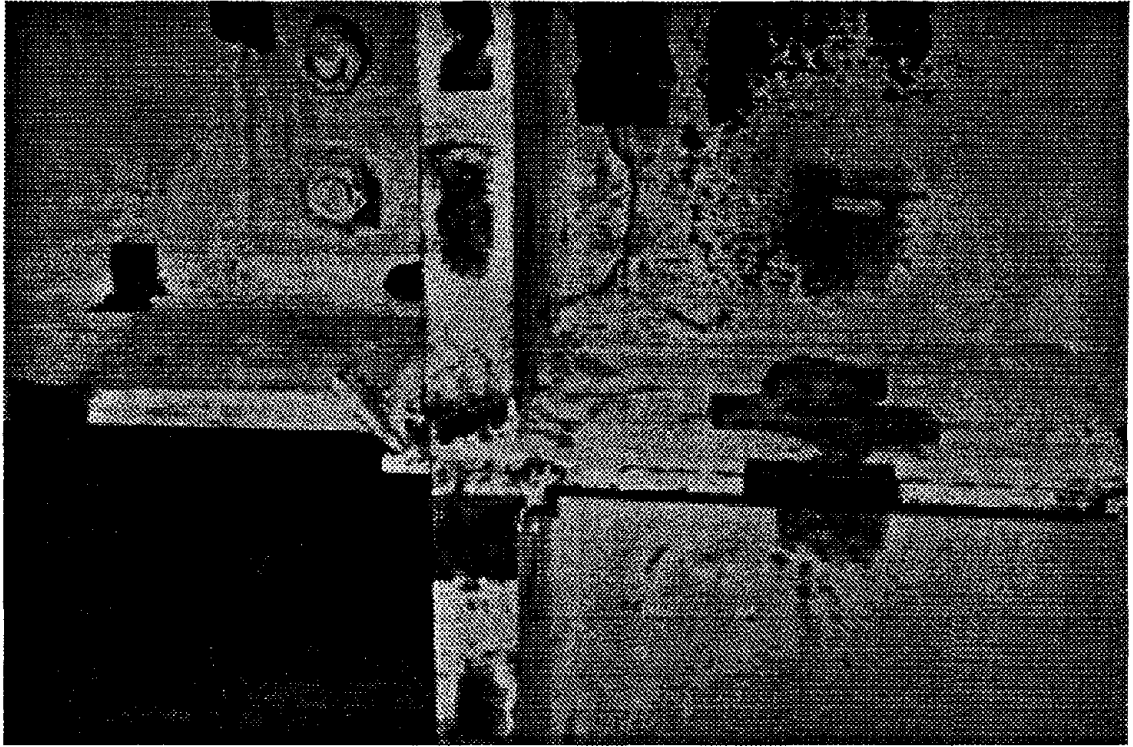
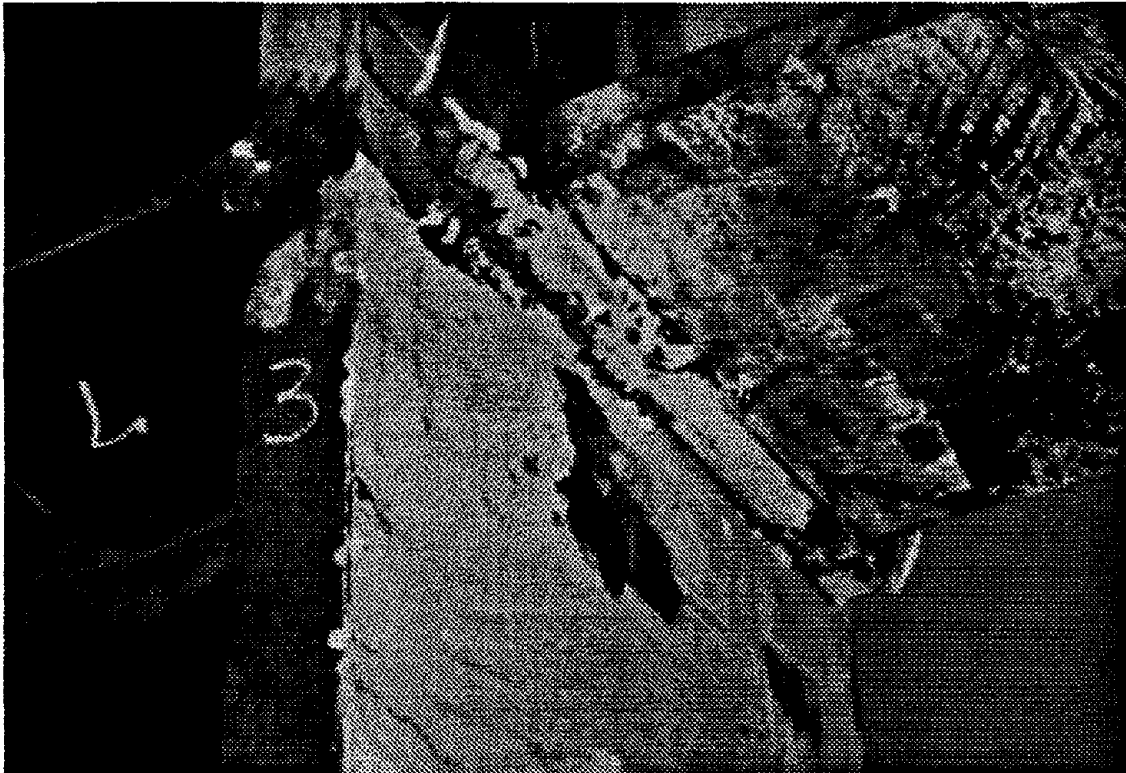


Fig. 16: Fracture pattern of SAC specimen PN2 is similar to specimen PN1.

web (see Fig. 15). At the peak load of 225 kips, the panel zone is unable to resist the applied shear force. The PN2 specimen has the same material properties as the PN1 specimen. Its fracture pattern is also similar to PN1 (see Fig. 16) except that the post-yield cycles are smaller. The peak load 201 kips is a little over the panel zone shear capacity. Specimen PN3 had a different fracture pattern compared with that of PN1 and PN2 specimens. The crack initiated at the center of the bottom beam flange-column juncture, then fractured the entire bottom beam flange (see Fig. 17). According to the classification in Table 2, PN1 and PN2 had failures of Type 2 because both had a strong beam and a relatively weak panel zone. The failure mode of PN3 is Type 1. All three specimens performed unsatisfactorily and failed in abrupt fractures. The imposed displacements and hysteresis loops for these specimens will be presented together with the analytical results in a later section.

It is interesting to note that if the beam of the SAC specimen had a yield strength of 36 ksi and an ultimate strength of 58 ksi, the performance of the connection would be much better and the failure mode would also be different. Compared with many successful connection tests back in the 1970s [22, 23], the design method and procedure are almost the same today. The most significant difference is that the



**Fig. 17:** Photograph showing the fractured bottom beam flange of SAC PN3 specimen after test.

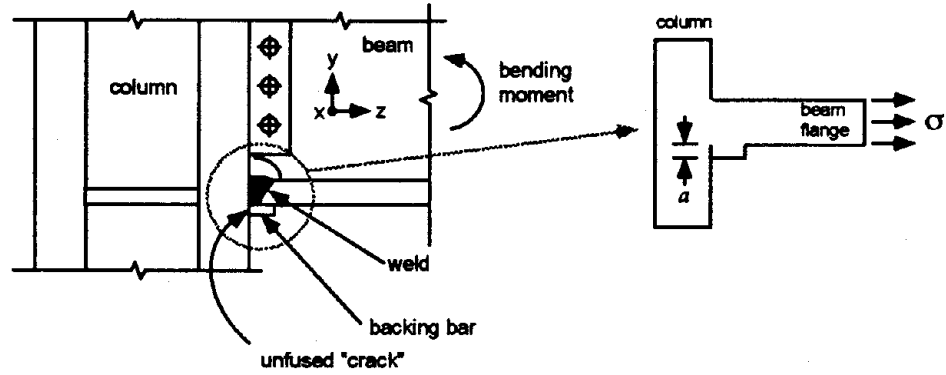


Fig. 18: The unfused backing bar surface forms an artificial edge crack.

material strength in the 1970s was much nearer to the ASTM specified minimum.

## Stress Concentration Caused by the Backing Bar

If the backing bar has not been removed after the welding, the unfused interface between the backing bar and the column flange acts as a fine crack. The length of the crack is equal to the thickness of the backing bar (Fig. 18). Theories for analyzing the stress field near cracks are now well-established and can be found in many standard texts such as Anderson [2], Broek [9], Rolfe and Barsom [25], and Suresh [27]. The stresses in the vicinity of a crack tip in an elastic material can be expressed as [13, 14, 29]

$$\begin{aligned}
 \sigma_{xx} &= \frac{K}{\sqrt{2\pi r}} \cos\left(\frac{\theta}{2}\right) \left[ 1 - \sin\left(\frac{\theta}{2}\right) \sin\left(\frac{3\theta}{2}\right) \right] \\
 \sigma_{yy} &= \frac{K}{\sqrt{2\pi r}} \cos\left(\frac{\theta}{2}\right) \left[ 1 + \sin\left(\frac{\theta}{2}\right) \sin\left(\frac{3\theta}{2}\right) \right] \\
 \sigma_{xy} &= \frac{K}{\sqrt{2\pi r}} \cos\left(\frac{\theta}{2}\right) \sin\left(\frac{\theta}{2}\right) \cos\left(\frac{3\theta}{2}\right)
 \end{aligned} \tag{27}$$

for a crack aligned in the  $x$  direction, where  $K$  is the stress intensity factor,  $r$ ,  $\theta$  are the cylindrical polar coordinates of a point with respect to the crack tip. Thus each case is characterized by the stress intensity factor having a spatial distribution of stresses. One of the underlying principles of fracture mechanics is that unstable fracture occurs when the stress-intensity factor  $K$  at the crack tip reaches a critical value  $K_c$ . To prevent a fracture failure, the computed stress-intensity factor  $K$  must be less than the critical stress-intensity factor, or fracture toughness,  $K_c$ .

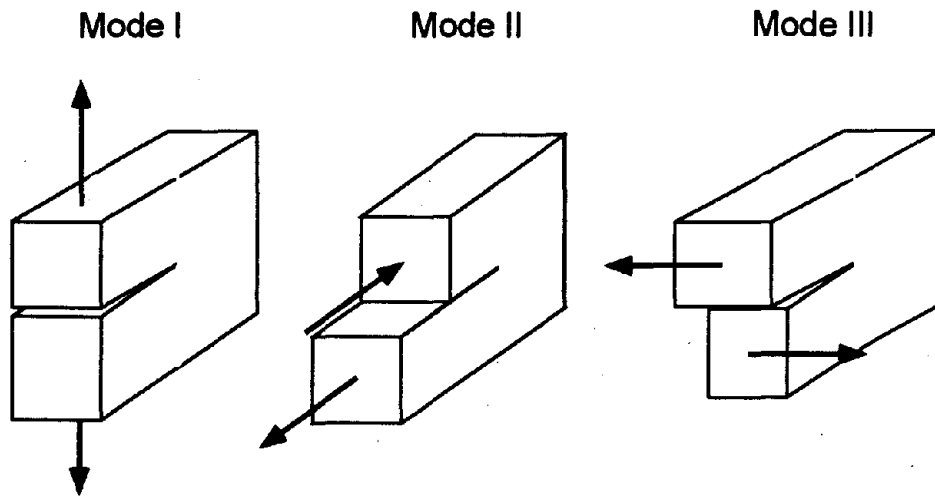


Fig. 19: The three modes of loading that can be applied to a crack: Mode-I (Opening), Mode-II (In-plane shear), and Mode-III (Out-of-plane shear).

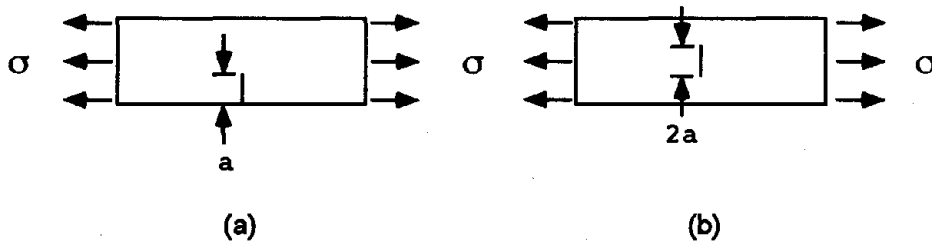


Fig. 20: A semi-infinite plate with (a) edge crack, (b) center crack subject to a remote axial stress  $\sigma$ .

The “artificial crack” between the unfused backing bar and the column flange can be characterized as an edge crack. There are three possible fracture modes of a crack (Fig. 19), namely Mode-I (Opening), Mode-II (In-plane shear), and Mode-III (Out-of-plane shear). Tension in a beam flange due to bending opens the crack in mode-I<sup>3</sup>. The Mode-I stress-intensity factor for the edge crack is shown to be

$$K_I = 1.12\sigma\sqrt{\pi a} \quad (28)$$

where  $a$  is the crack length, here the thickness of the backing bar, and  $\sigma$  is the applied nominal Mode-I stress (Fig. 20(a)). The critical stress-intensity factor  $K_{Ic}$  of Mode-I

<sup>3</sup>The shear and torsional forces in the beam can also open the crack in mode-II and mode-III, respectively, but their magnitudes are relatively small and can be ignored.

can be obtained by following the ASTM standard for determining  $K_{Ic}$  [3].  $K_{Ic}$  is temperature sensitive. For carbon steel, the transition is quite steep at temperatures above 0°F.

$K_{Ic}$  can also be obtained from the critical energy release rate  $\mathcal{G}$  (after Griffith [11]) of the material from

$$K_{Ic} = \sqrt{\mathcal{G}E'} \quad (29)$$

where  $E'$  is the modified modulus of elasticity, it can be computed as

$$E' = \begin{cases} E & \text{for plane stress} \\ E/(1 - \nu^2) & \text{for plane strain} \end{cases} \quad (30)$$

where  $E$  is the Young's modulus and  $\nu$  the Poisson's ratio. The crack between the backing bar and column flange is long enough to be considered as a plane strain problem.

The critical energy release rate  $\mathcal{G}$  is temperature sensitive. For structural carbon steel, it is known that  $\mathcal{G}$  is at least 15 lb-ft (see [4, 26]). The critical stress-intensity factor can be calculated to be

$$K_{Ic} = 75.7 \text{ ksi}\sqrt{\text{in}} \quad (31)$$

Based on this value and in the case that  $K_I = K_{Ic}$ , if the thickness of the backing bar is 3/8 inch, then the nominal stress  $\sigma_{zz} = \sigma$  in the beam flange, according to Eq. 28, cannot be more than 62 ksi.

There are two ways to reduce the deleterious effect of the backing bar. A direct method is to remove it using a carbon arc. Once the backing bar has been removed, the artificial crack no longer exists. But this method has a high probability of damaging a good weld above the backing bar. Another method to reduce the stress concentration caused by the backing bar is to apply a fillet weld under it to close the crack. A center crack of length  $2a$  occurring away from the edges, such as the one shown in Fig. 20(b), has a somewhat smaller stress-intensity factor, namely,

$$K_I = \sigma\sqrt{\pi a} \quad (32)$$

Here,  $a$  is half of the backing bar thickness. Therefore, an additional fillet weld under the backing bar will reduce the stress-intensity factor and improve the connection fracture resisting capability.

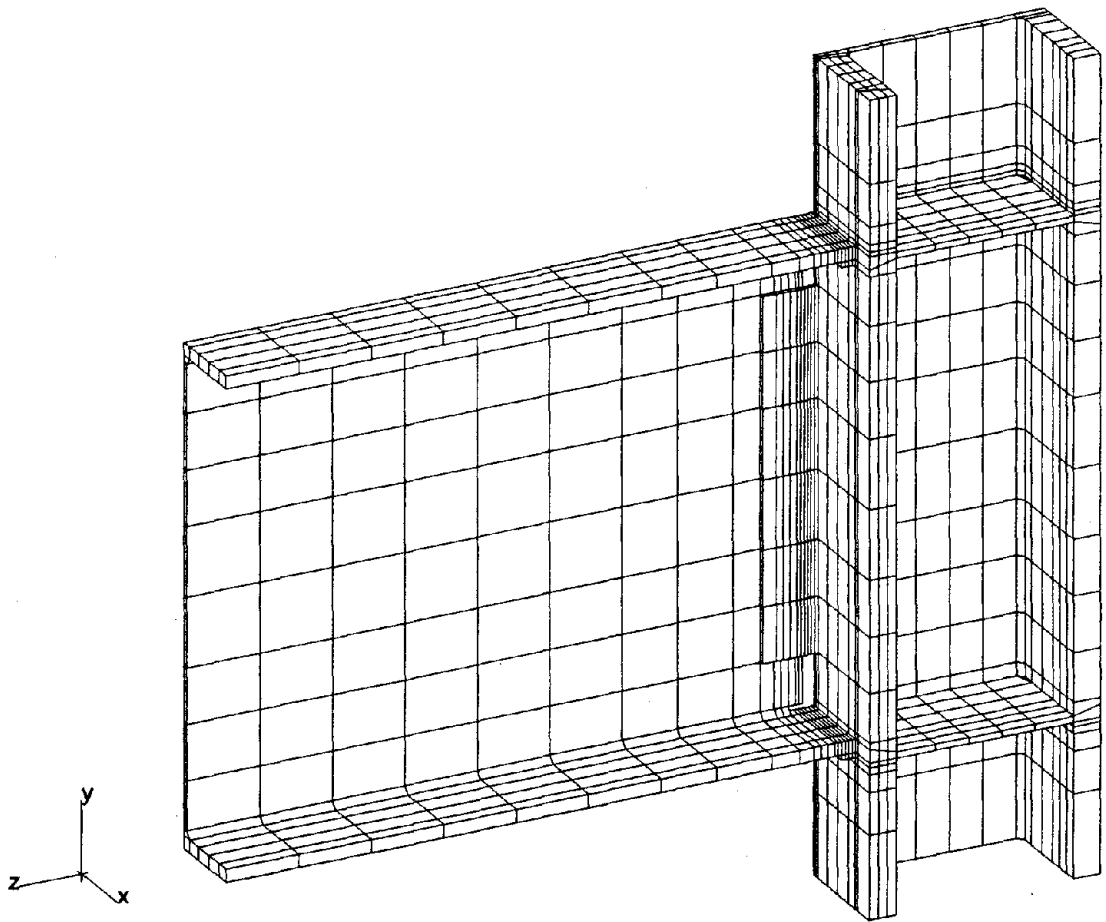
In this section, it is assumed that the stress  $\sigma$  across the beam flange is uniform with linear elastic behavior, but in reality,  $\sigma$  is not uniform. The geometry of the

backing bar crack is not similar to the edge or center crack shown in Fig. 20. Applying Eqs. (28) and (32) to the backing bar crack problem is oversimplified. To account for the effects of the geometry, non-uniform stress distribution, and elastic-plastic material behavior, the numerical  $J$  contour integration by finite element method must be used.

## Nonlinear Finite Element Analysis

In order to accurately evaluate the stress distribution and the stress-intensity factor at the connection, the SAC specimens are modeled on eight-node brick elements in the ABAQUS finite element package [12]. The element mesh of the connection part of the specimen is shown in Fig. 21. Note that the backing bars together with the artificial cracks mentioned previously are introduced in the model. As of this writing, the SAC specimens are being retrofitted for re-testing, and no tension tests for the material are available at the moment. Because the true material properties are unknown, the material properties used in the finite element model are based on the Mill Certificates as given in Table 3. The coupon test will be scheduled right after the completion of structural testing of the retrofitted connections. The material properties were modeled by von Mises yield criterion with associated plastic flow.

To fully understand the stress distribution and propagation during the structural testing, the same imposed beam tip displacements were used in the finite element computations. Because the applied loads are slowly varying in the test, the inertia loads are ignored in the finite element analysis. Figure 22 displays the imposed tip displacements for experimental and analytical studies of the PN1 specimen. The analysis steps shown in the figure are not evenly spaced because it requires more steps in the nonlinear region. To accelerate the analysis, low amplitude displacement cycles are ignored in the analysis, these including two 0.1 in. cycles, three 0.25 in. cycles, and three 0.5 in. cycles. Because the hot rolling residual stresses and the heat-affected zone residual stresses are not fully known, such effects are also ignored in analyses. Such simplification make the corners in the analytical hysteresis loops sharper than the experimental ones. The experimental and analytical hysteresis loops for the SAC PN1 Specimen are presented in Fig. 23. The experimental and analytical imposed displacements and hysteresis loops for the SAC PN3 specimen are shown in Figs. 24 and 25, respectively. The low amplitude displacement cycles are also ignored in the PN3 analysis. By integrating the hysteresis loops step-by-step, the total energy



**Fig. 21:** Finite element mesh for SAC Pre-Northridge PN connection. Only one half of the specimen is modeled.

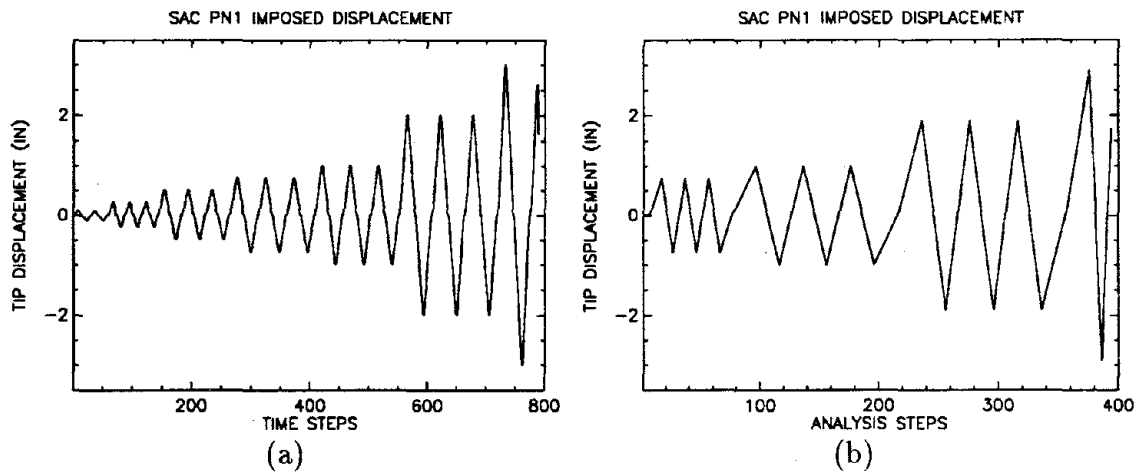


Fig. 22: (a) Imposed tip displacements used in testing SAC PN1 specimen, and (b) imposed tip displacements used in finite element analysis.

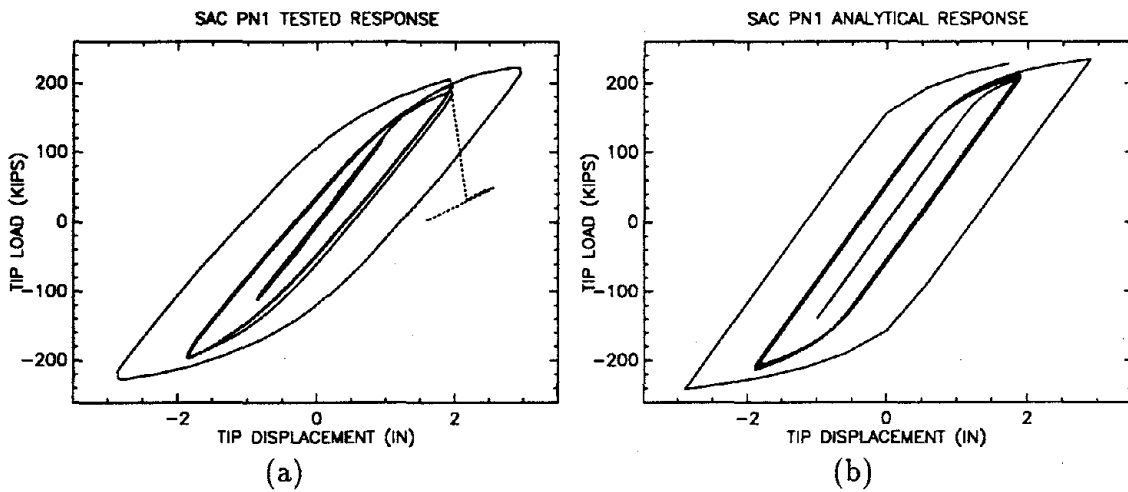
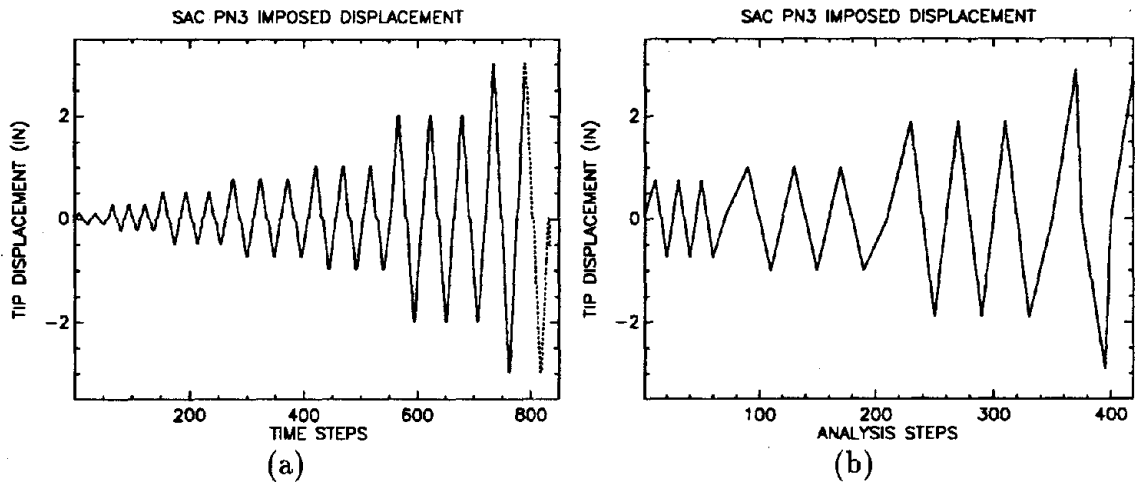
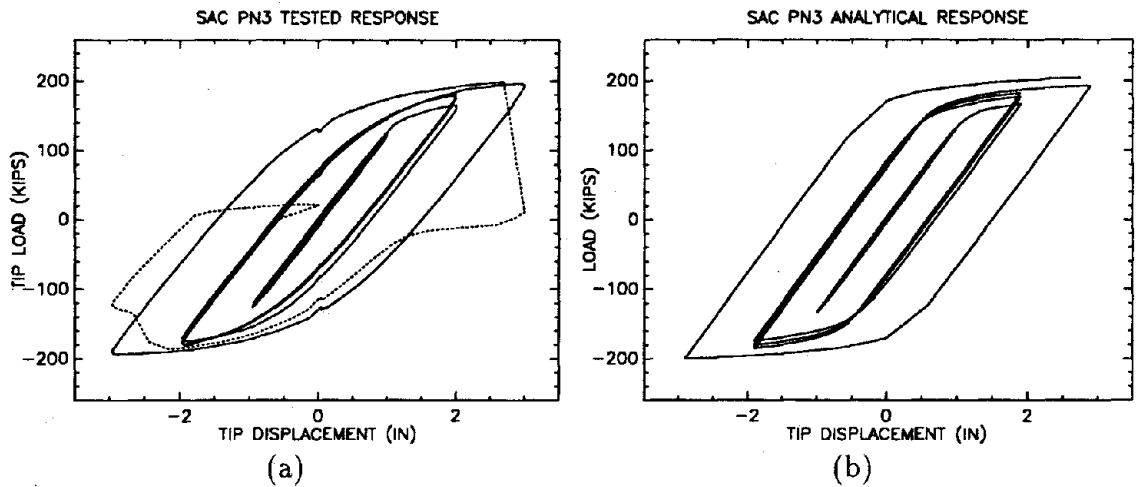


Fig. 23: (a) Experimental and (b) analytical hysteresis loops of SAC PN1 specimen (Displacements are measured in the loading direction).





**Fig. 24:** (a) Imposed tip displacements used in testing SAC PN3 specimen, and (b) imposed tip displacements used in finite element analysis. The dashed line represents the imposed displacements after the bottom beam flange was fractured.



**Fig. 25:** (a) Experimental and (b) analytical hysteresis loops of SAC PN3 specimen (Displacements are measured in the loading direction). The dashed curve represents the response after the fracture of the bottom beam flange.

diagram can be constructed. The total strain energies of PN1 and PN2 specimens are given in Fig. 26. The wavy shape curve in the total energy diagram is due to the restoration of elastic strain energy during load reversals. Those unrecoverable energies are dissipated energy. The same diagrams for the PN3 specimen are given in Fig. 27. These figures lead to the conclusion that none of the three SAC specimens possessed adequate energy dissipating capacities.

Among the three SAC specimens, PN2 has the least ductile behavior. The imposed displacements and load-deflection responses are shown in Fig. 28. Because the PN2 specimen is identical to the PN1 specimen, the analytical response of the PN1 specimen can be used for comparison. The total energy diagram for the PN2 specimen is given in Fig. 26(a).

A perspective view of the connection stress distribution is shown in Fig. 29. The highest stressed spots are at the beam flange weld and in the panel zone. The material yielding in the panel zone starts at the center and then gradually expands outward. During the test, the whitewash was continuously breaking off in the panel zone. When the tip load reached about 200 kips, the entire panel zone yielded. The analytical stress contours in Fig. 30 agree with this observation. Figures 31 and 32, respectively, show views of the stress contours of the top and bottom beam flanges, together with the continuity plates and column section in the plastic range. In the elastic range, the largest stress occurs at the center of beam flange-column juncture. The stresses become much more evenly distributed across the beam flange in the plastic range, but the center of the flange still has the largest stress. All the displacements in these figures are magnified 10 times. Even so, there is no obvious deformation observed in the axial direction of the beam flange.

The SAC PN1/PN2 stress distributions for the bottom beam flange along the line of groove weld are shown in Fig. 33. Curves shown are for beam tip loads of 21, 41, 62, 82, 103, 117, 142, 200, and 225 kips. Shearing stresses are not shown because their values are small. The maximum shearing stresses at 225 kips tip load are 3.6 ksi, -17.4 ksi, 17.5 ksi for  $\sigma_{xy}$ ,  $\sigma_{xz}$ , and  $\sigma_{yz}$ , respectively. Small shearing stresses imply small shearing deformations and small ductility of the connection.

The experimental strains at the center of the panel zone are plotted against the applied tip load for three SAC specimens (see Fig. 34). The load-shear strain diagrams show large shear strain in the PN1 panel zone, but much smaller shear strain in the PN2 and PN3 panel zones. The PN3 panel zone resists smaller shear force, its panel zone shows only slight yielding but no fracture. From the material strength point of

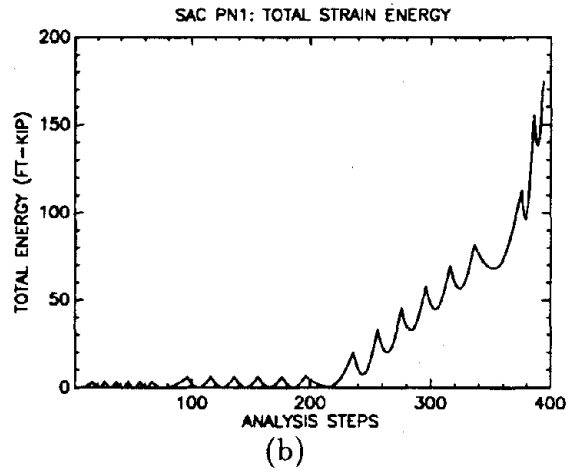
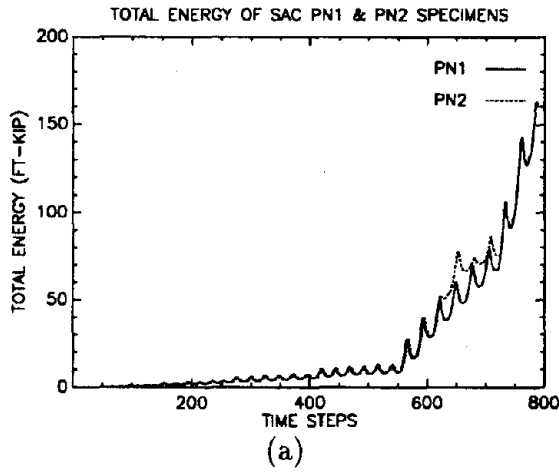


Fig. 26: (a) Experimental and (b) analytical total strain energy diagrams of SAC PN1 and PN2 specimens.

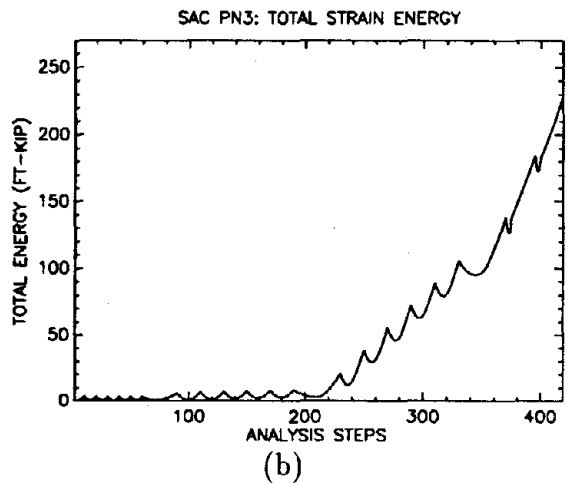
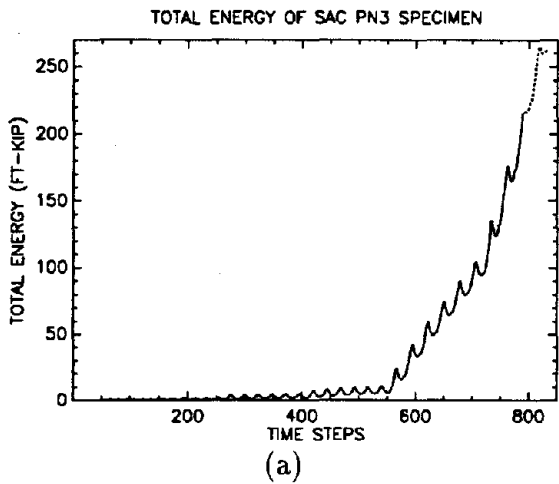
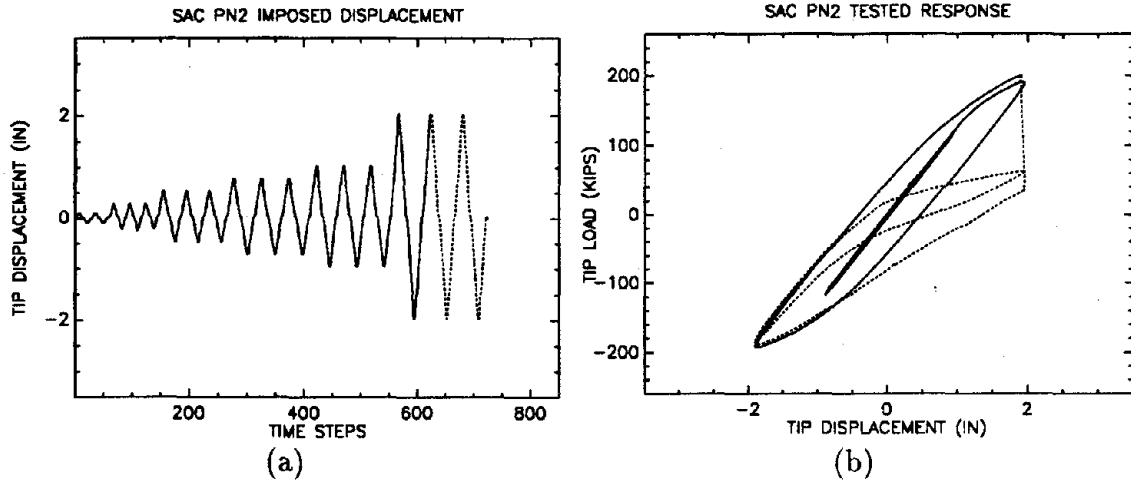


Fig. 27: (a) Experimental and (b) analytical total strain energy diagrams of SAC PN3 specimen.



**Fig. 28:** (a) Imposed displacements and (b) hysteresis loops of SAC PN2 specimen (Displacements are measured in the loading direction). The dashed curves represent the response after the fracture of the bottom beam-column juncture.

view, there is no doubt that PN1 failed in the panel zone by overstressing, but it is difficult to reach the same conclusion for PN2, although both have the same fracture pattern in the panel zone. It is much simpler to explain this by fracture mechanics.

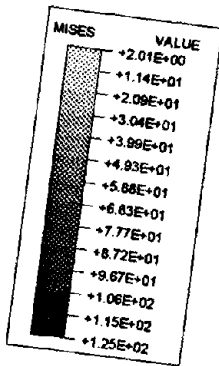
The  $J$  contour integrals along the top and the bottom backing bar crack tip are evaluated. In calculating the  $J$  contour integral using the finite element method, the nonlinear elastic-plastic material properties are considered. Rice had formulated the  $J$  contour integral defined to be [24]

$$J = \int_{\Gamma} \left( w dy - T_i \frac{du_i}{dx} ds \right) \quad (33)$$

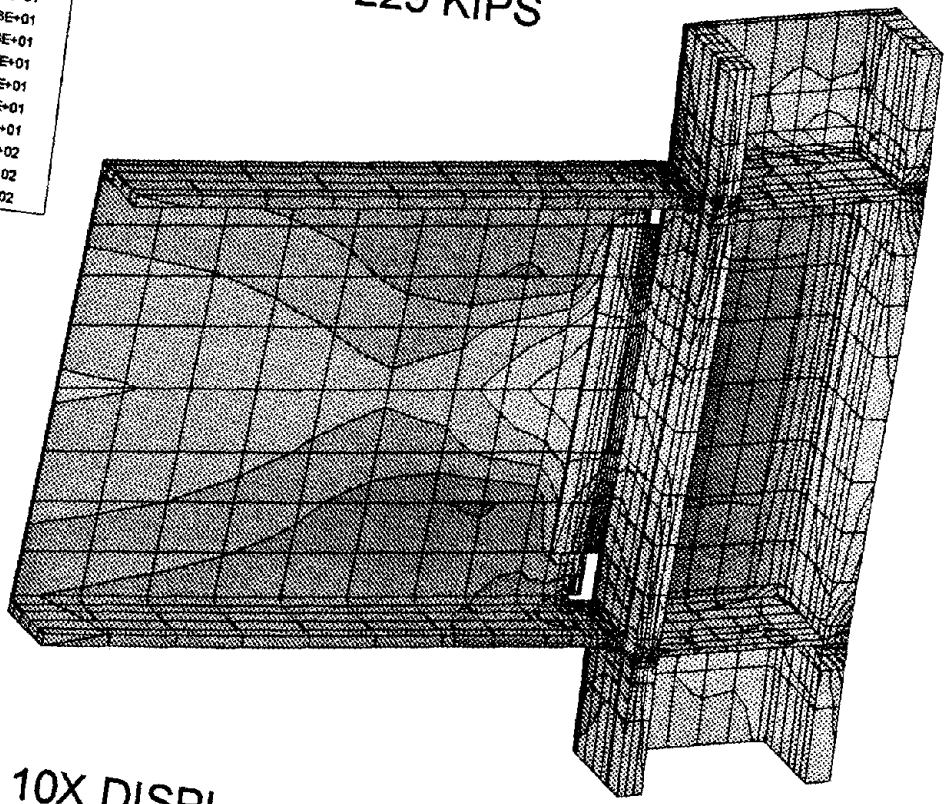
for a crack aligned in the  $x$  direction. Here,  $\Gamma$  is any contour from the lower crack face counterclockwise around the crack tip to the upper face. The path length along this contour is  $s$ ,  $w$  is the strain energy density defined as

$$w = \int_0^{\epsilon_{ij}} \sigma_{ij} d\epsilon_{ij} \quad (34)$$

where  $\sigma_{ij}$  and  $\epsilon_{ij}$  are the stress and strain tensors, respectively.  $T_i u_i$  are work terms for components of surface traction on the contour path,  $T_i$ , move through displacements,  $du_i$ . The  $J$  contour integral is equal to the energy release rate for a linear or nonlinear elastic material under quasi-static conditions. The integral was shown to be independent of choice of path for a crack with stress-free faces. Since the critical value  $J_c$  for A572 Grade 50 is unavailable, it is impossible to justify the initiation of fracture

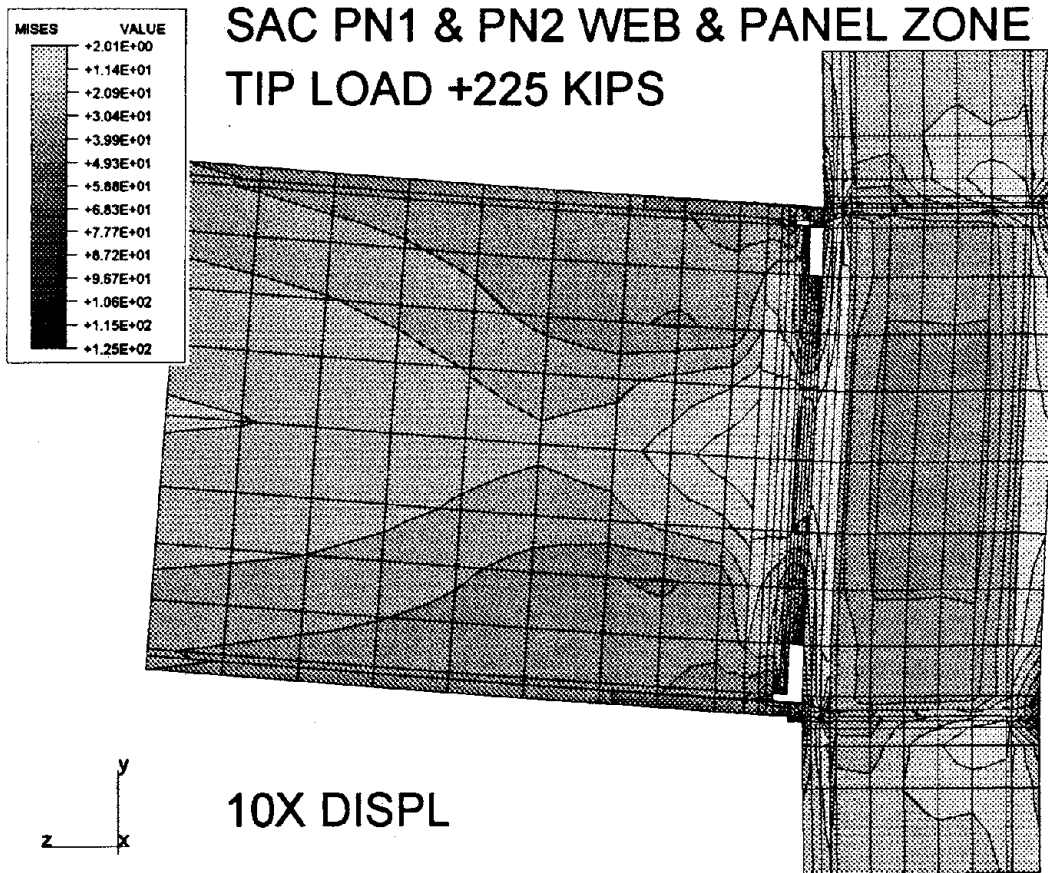


SAC PN1 & PN2  
TIP LOAD +225 KIPS

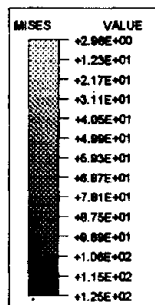


10X DISPL

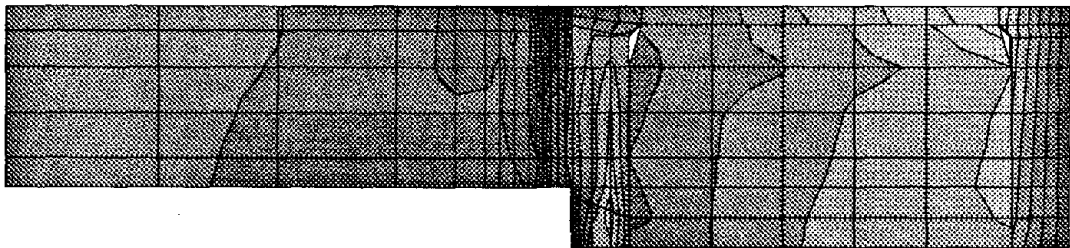
Fig. 29: Von Mises Stress contours for SAC PN1/PN2 under 225 kips tip load.



**Fig. 30:** Web and panel zone stress contours for SAC PN1/PN2 under 225 kips tip load.



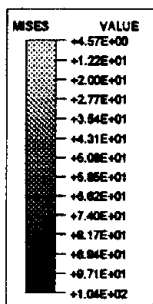
SAC PN1 & PN2 COMP. FLANGE & STIFFENER  
TIP LOAD +225 KIPS



10X DISPL



Fig. 31: Top beam flange and continuity plate contours for SAC PN1/PN2 under 225 kips tip load.



SAC PN1 & PN2 TENSION FLANGE & STIFFENER  
TIP LOAD +225 KIPS

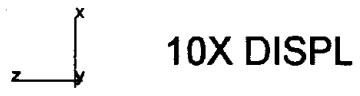
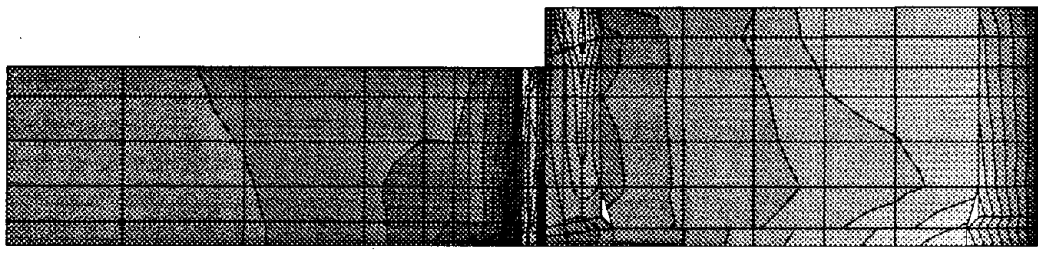
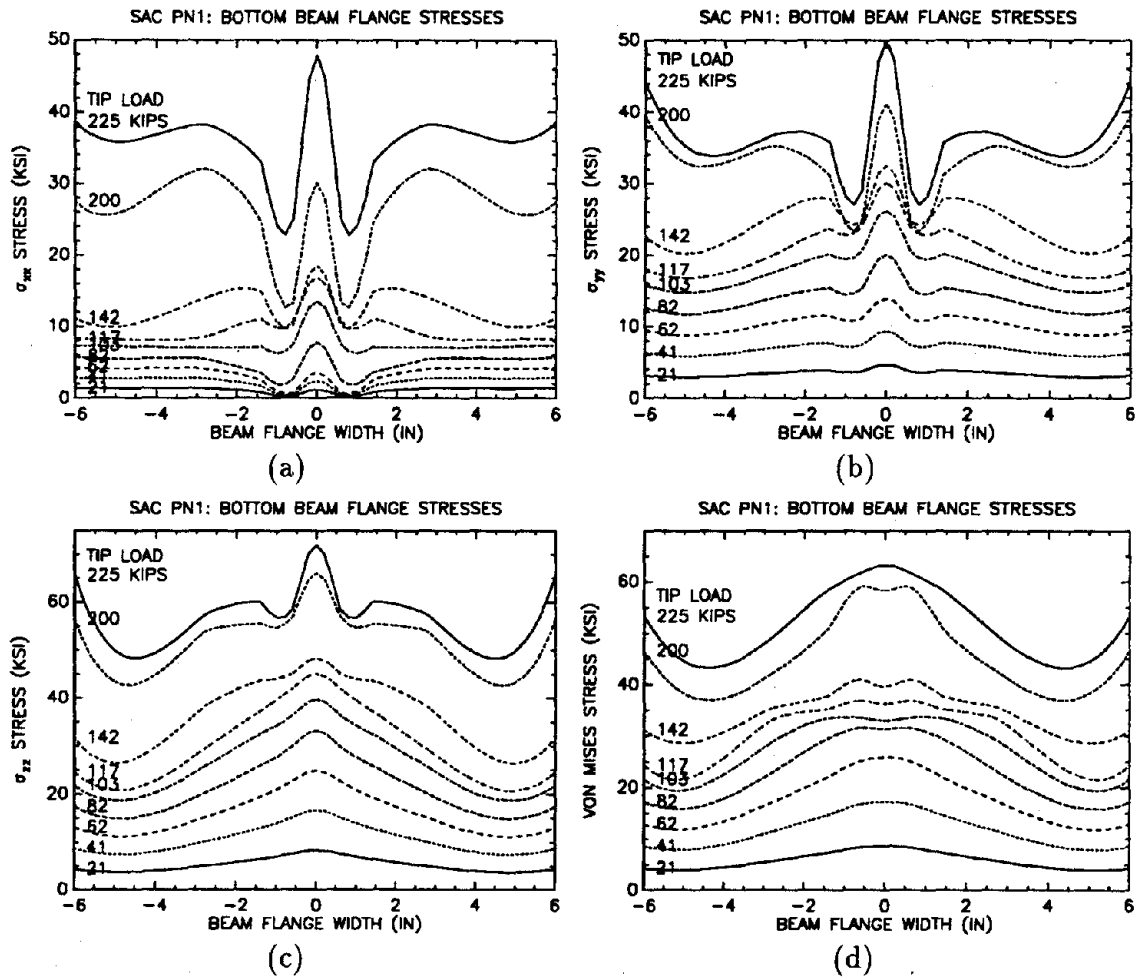


Fig. 32: Bottom beam flange and continuity plate contours for SAC PN1/PN2 under 225 kips tip load.





**Fig. 33:** Stress distribution across the bottom beam flange at weld for SAC PN1 and PN2 specimens: (a)  $\sigma_{xx}$ , (b)  $\sigma_{yy}$ , (c)  $\sigma_{zz}$ , and (d) von Mises stress.

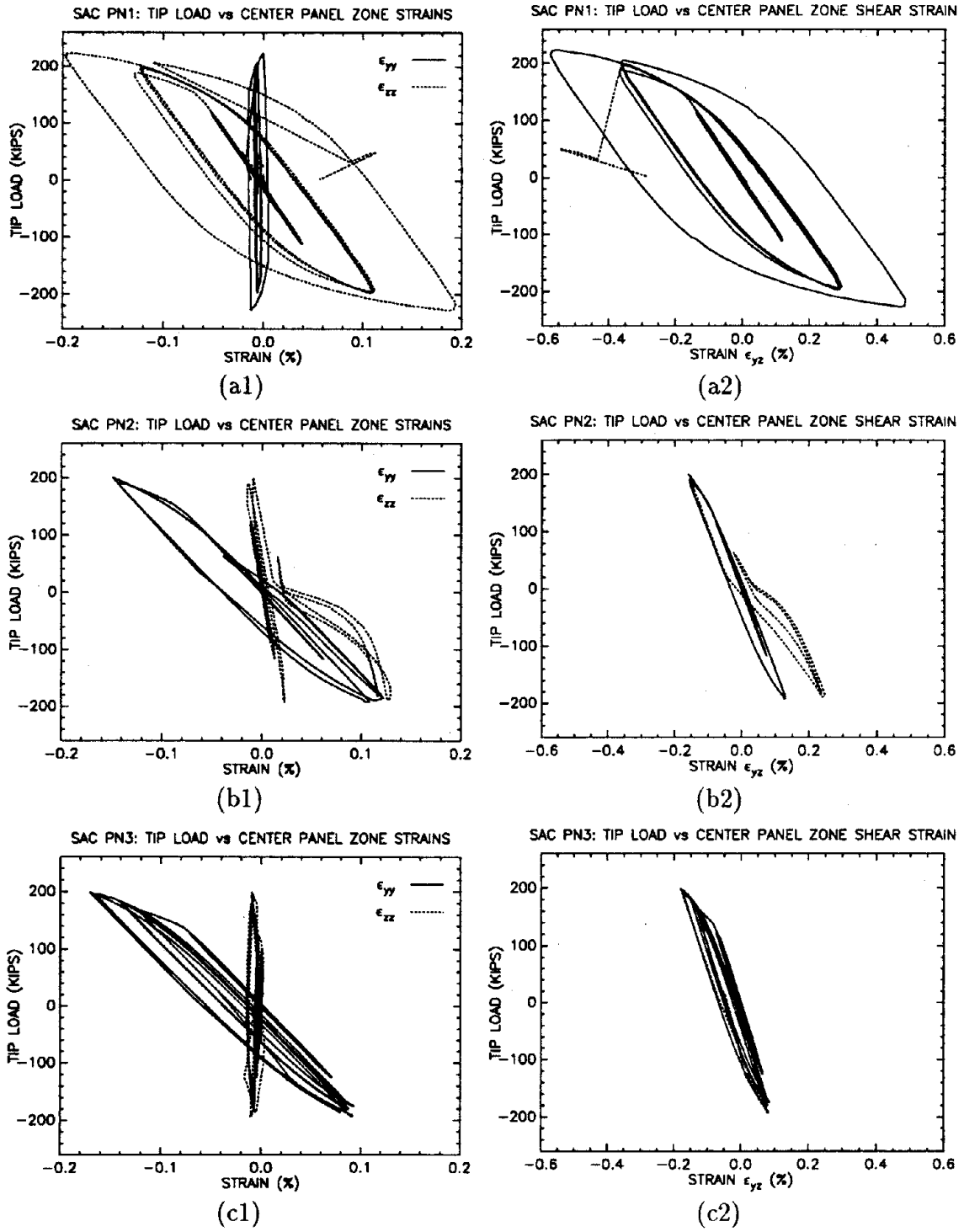


Fig. 34: Center panel zone strains vs tip load for SAC PN1, PN2 and PN3 specimens.

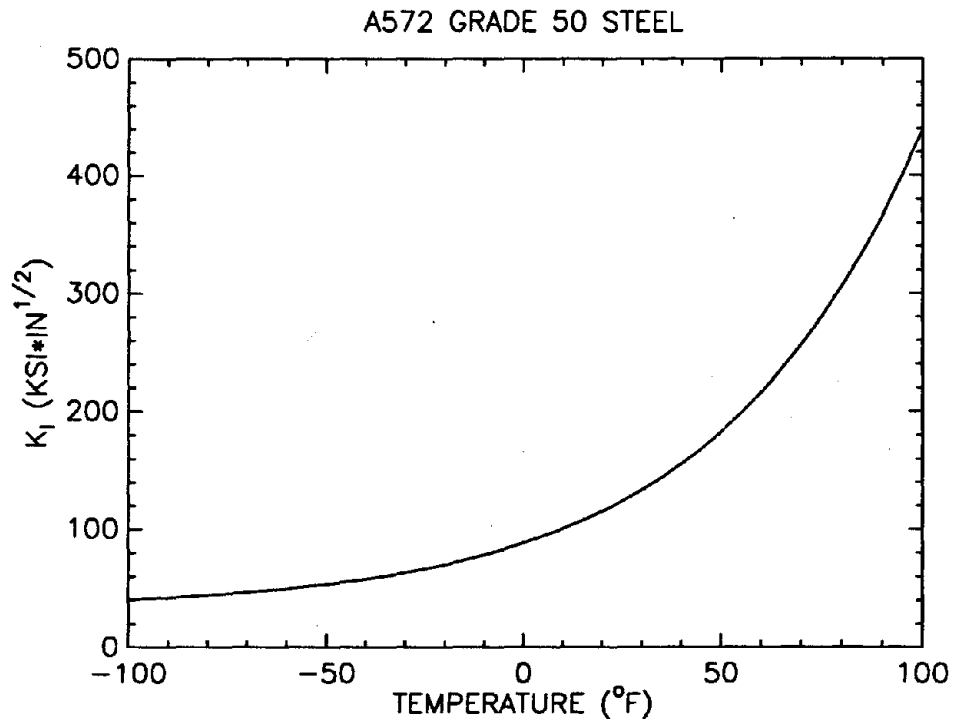


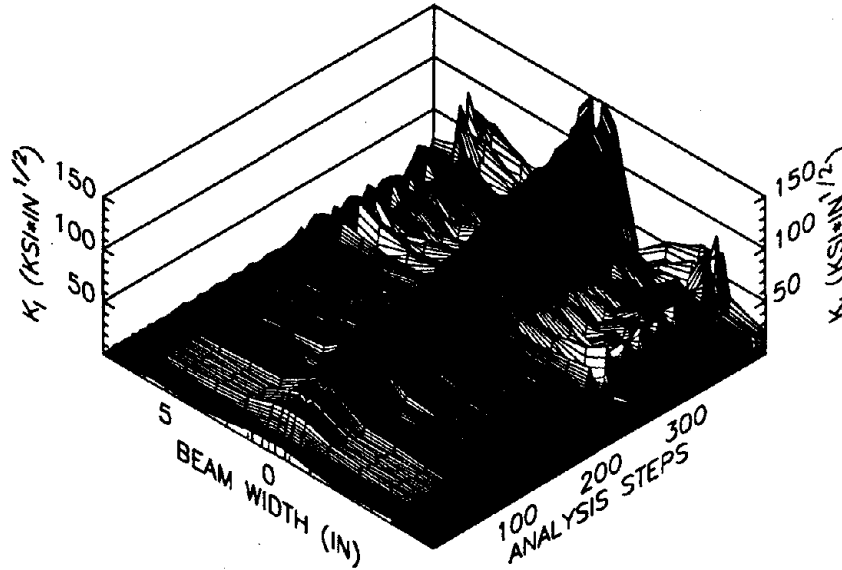
Fig. 35: Stress intensity factors plotted against temperatures obtained from 1.5-in.-thick plates of A572 steel.

based on the computed  $J$  values. Fortunately, the specific temperature induced variation in stress intensity factors for the 1.5-in.-thick A572 Grade 50 specimens tested is available and presented in Fig. 35 [20]. In order to compare the calculated  $J$  values with the known  $K_{Ic}$ , an equivalent  $K_I$  is calculated [15]:

$$\frac{K_I^2}{E'} = J \quad (35)$$

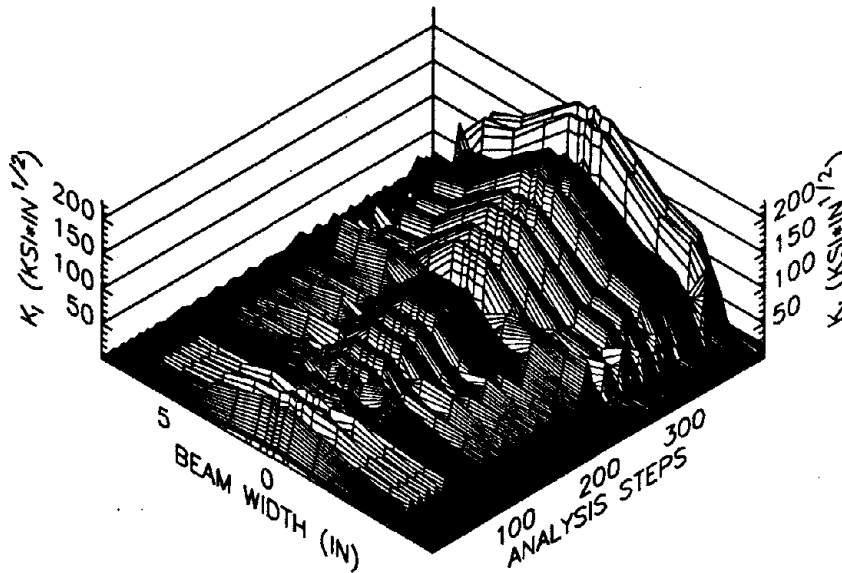
where  $E'$  is defined in Eq. (30). It is thus conceptually equal to the Griffith term  $\mathcal{G}$ . Strictly speaking, the above equation only holds under elastic conditions. But if the plastic zone at the crack front is small, this equation is valid. The equivalent  $K_I$  values from the computed  $J$  values for the top and the bottom backing bar for PN1 and PN2 are shown in Fig. 36. In the figures,  $K_I$  values are plotted against beam width and analysis steps. It can be seen that the largest  $K_I$  occurs at the center of the beam flange-backing bar juncture. If the connection fractures, it will start at the point with the largest stress intensity factor. It is interesting to re-plot the largest stress intensity factor vs. the applied tip load (see Fig. 37). The growth of  $K_I$  due to cyclic load can be seen. The critical stress intensity factors at 0°F, 50°F and 60°F taken from Fig. 35 are also plotted in the diagram. The most interesting finding in

SAC PN1:  $K_I$  FOR THE TOP BACKING BAR



(a)

SAC PN1:  $K_I$  FOR THE BOTTOM BACKING BAR

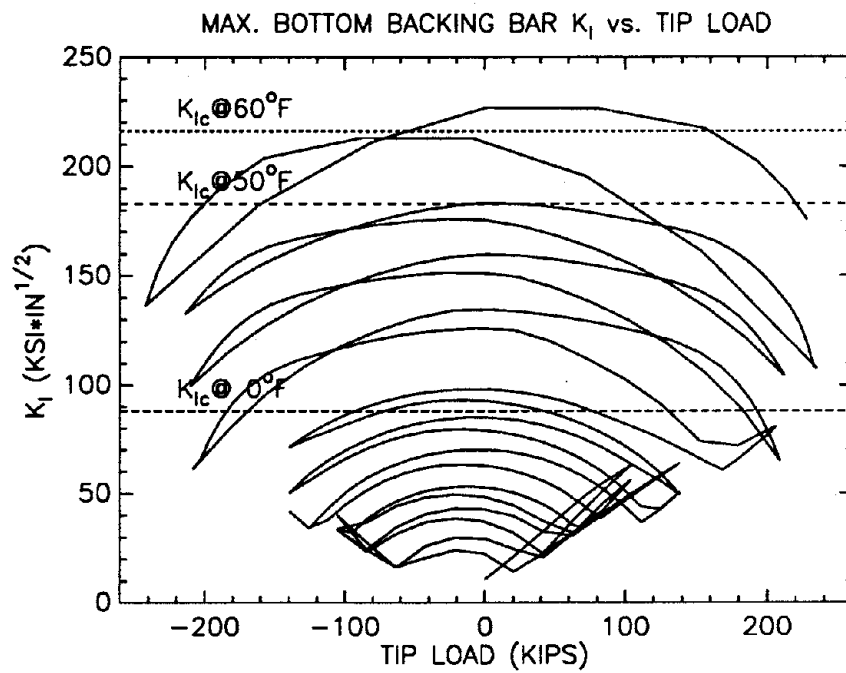
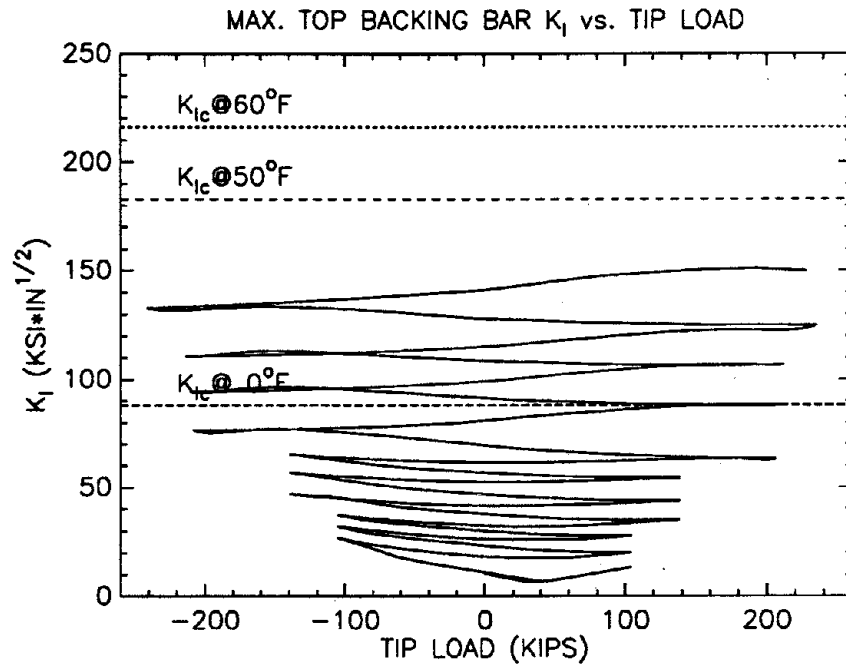


(b)

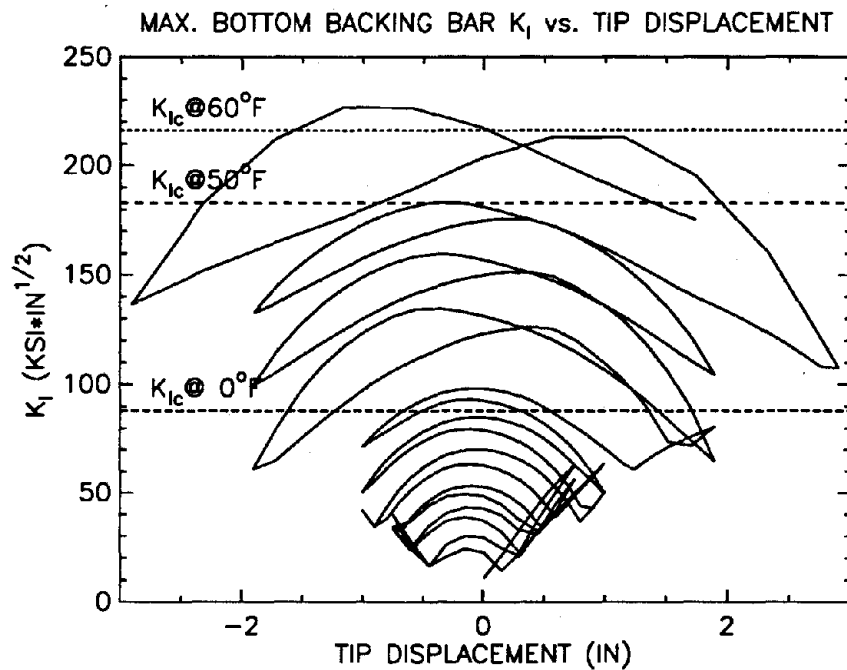
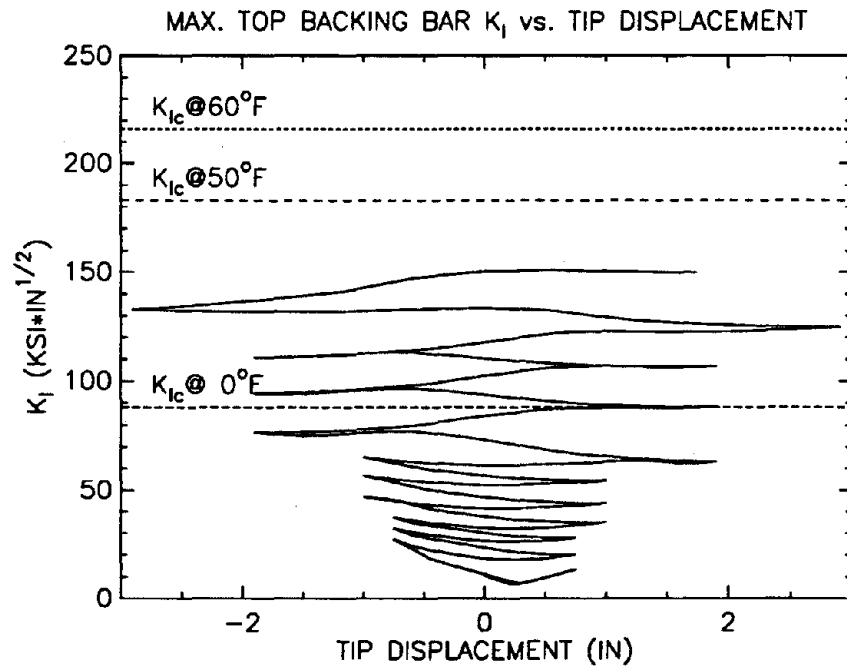
**Fig. 36:** Stress-intensity factors plotted across beam width and number of analysis steps at (a) top and (b) bottom backing bars for SAC PN1 and PN2 specimens.

the figure is that the bottom backing bar  $K_I$  at a given time is larger than that of the top backing bar. This clearly explains why most of the connection fractures during the Northridge earthquake initiated at the bottom backing bar.

It is also interesting to re-plot the largest  $K_I$  vs. tip displacement (see Fig. 38). It can be seen that the stress intensity factor keeps on growing under the same-amplitude cycles, elastic or inelastic; the larger the displacement amplitude, the larger the  $K_I$  growth rate. When  $K_I$  grows over the  $K_{Ic}$ , the connection will fracture. Based on the  $K_I$  growth rate, it is easy to predict the low-cycle fatigue fracture of the connection. The effect of temperature can be seen in these plots; the higher the temperature, the larger the  $K_{Ic}$ , and the less likely is the connection to fracture. The SAC PN1 and PN2 specimens are theoretically identical, but the PN1 specimen sustains more cycles of loading than PN2. One of the reasons to explain this is the different temperature. PN2 was tested on a cooler day. The low  $K_{Ic}$  value at the lower temperature meant PN2 fractured earlier.



**Fig. 37:** Maximum stress-intensity factors vs. tip load at (a) top and (b) bottom backing bars for SAC PN1 and PN2 specimens.



**Fig. 38:** Maximum stress-intensity factor vs. tip displacement at (a) top and (b) bottom backing bars for SAC PN1 and PN2 specimens.

## Conclusion

On the basis of these limited analytical and experimental studies, the following conclusions can be drawn:

1. Elementary mechanics calculations can predict the connection capacity very well as long as the material properties are known in advance.
2. The connection made by directly welding a compact beam flange to a column cannot attain the plastic moment of the beam. To protect the connection from failure, a weaker beam should be used or reinforcement of the connection should be made. A weaker beam means a beam with a non-compact section such that the local buckling moment is smaller than the connection moment capacity. If a compact beam must be used, the beam can be made weaker by drilling holes in the flange near the connection so that the moment transferred to the connection is smaller (Fig. 39). There are several ways to reinforce the connection; one of the methods is by using flange cover plates with thickness greater than the beam flange thickness. But such connection requires two welds, a weld to the beam and a weld to the column, which increases the cost substantially.
3. The elastic stress concentration factor at the beam flange of the welded beam-to-column connection can range between 1.2 to 1.46. The stresses will redistribute much more evenly across the flange when loaded into the plastic range. The largest stress is located at the center of the welded flange.
4. Triaxial loading makes steel at a connection fail without exhibiting yielding ductile behavior. This is due to the state of stress and not because of the material property. The demand for ductility should be dependent on the material yielding near the connection area.

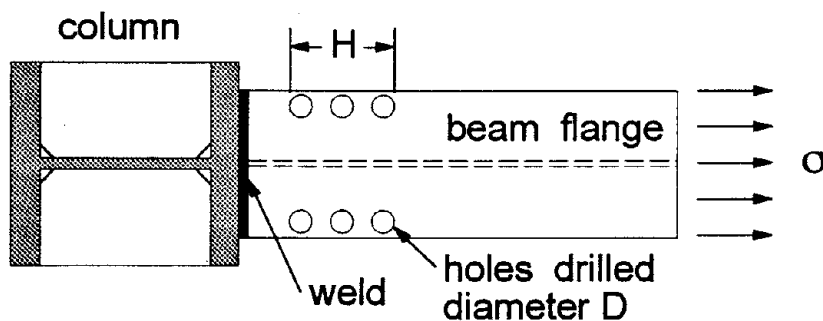


Fig. 39: Connection protection by beam flange perforation.



5. Material properties of steel, such as yield strength and ultimate strength, should be regulated to have a narrow range instead of prescribing only the minimum strength. Otherwise, an engineer cannot design a structure with tolerable bounds on response. Today's A36 steel has an average yield strength 33% over the minimum. This fact is not reflected in present design codes nor in education. This high strength does not enable the connection to develop a plastic hinge during a strong earthquake and causes its failure in brittle fracture.
6. The column web fracture is due to a weak panel zone. Using doubler plate in the panel zone may solve the problem, but will increase the construction cost. The best solution is to avoid connecting a strong beam to a column with weak web. Connecting a beam to the minor axis of a column allows for the use of column flanges to resist the shear.
7. The dimensions of many rolled shapes should be made to better proportions. For example, the W14×257 section used in the SAC specimens has a large moment capacity but a very small capacity in shear resistance.
8. The unfused surface between the backing bar and the column can be characterized as an edge crack. If the backing bar cannot be removed, an extra fillet weld under the backing bar can close the crack and makes the stress-intensity factor smaller, and thus safer. During load reversals, the stress intensity factor at the bottom backing bar crack is higher than that at the top backing bar, resulting in greater probability of initial fracture at the bottom weld during an earthquake.
9. Welded connections exposed to outside temperatures should be designed very carefully because steel has a lower critical stress-intensity factor at low temperatures. This is especially true for connections with backing bars and welding flaws.
10. Energy dissipation at a connection by the means of material yielding is notoriously unreliable. A small variation from the design value in beam or column strength will easily result in a totally different energy dissipating mechanism and failure mode.

The following important issues were not considered in this limited study, and require further research:

1. Finite element analyses need to be re-done using the true material properties after the completion of the coupon test of the SAC PN specimens. High-order singularity elements should be used in modeling the crack for higher accuracy.
2. The residual stress distribution at the heat affected zone of a weld requires further investigation.
3. The internal flaw sizes between multiple layers of weld must be investigated, especially, since the crack growth due to cyclic yield loads induces low cycles fatigue fracture. Such a problem will stand out in a long duration earthquake.
4. The databases of fracture toughness of structural steel and welding materials in the plastic range need to be established.
5.  $J$  contour integration analysis for the backing bar with additional under bar fillet weld is useful in understanding its merit quantitatively.
6. The welded connection, unlike the bolted connection, lacks crack resisting redundancy. Further investigation of the crack arresting design is required.

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